

TRANSIENT THERMAL STRESS ANALYSIS
OF COMPOSITE STRUCTURES INCLUDING
CONTINUOUSLY VARYING PROPERTIES

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NAVAL POSTGRADUATE SCHOOL

Monterey, California



THESIS

TRANSIENT THERMAL STRESS ANALYSIS
OF COMPOSITE STRUCTURES INCLUDING
CONTINUOUSLY VARYING PROPERTIES

by

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September 1975

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(20. ABSTRACT Continued)

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Transient Thermal Stress Analysis
of Composite Structures Including
Continuously Varying Properties

by

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Lieutenant, United States Navy
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ABSTRACT

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Solutions were obtained by approximate techniques, and a computer program for transient three dimensional thermal stress analysis was developed. The finite element technique was employed for the determination of the temperature distribution and the elastic stress analysis. Fortran IV G Level was used as the programming language.

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PART I. THERMAL STRESS ANALYSIS OF
CYLINDERS WITH COMPOSITE COATINGS

The efficiency of a gas turbine engine can be increased by raising the turbine inlet temperature. However, protection against high temperature oxidation or sulfidation of the turbine blades must be provided. One method, which has been suggested, [1]*, is the application of ceramic coatings which resist high temperature corrosion. Since the material properties vary through such a composite blade, thermal stresses become an important consideration particularly during transients with non-uniform temperature distributions.

The application of these ceramic coatings has been accomplished via a process of high-rate sputtering [1]. This process requires high temperatures to insure proper adhesion of the coatings. Since the thermal life cycle of the coatings differs from the thermal conditions when the coatings are applied, the deposition temperature must be considered in the design criteria.

In order to determine the thermal stress levels within possible coating arrangements, hollow cylinders were analyzed with various material compositions. A thermal cycle consisting of a rapid heating phase followed by a cooling phase was used to generate temperature fluctuations and the thermal

* Numbers in brackets refer to similarly numbered references in the bibliography.

stress distributions. The finite element technique was used to perform the elastic stress analysis and to determine the temperature distributions.

I. SIMULATION OF THE EXPERIMENTAL MODEL

An experimental program [1] to develop the coating technique and to observe the coating during thermal transients was begun using thin walled cylinders, a simple geometry instead of the more complex turbine blade. Such cylinders were prepared by the high-rate sputter deposition of nickel and stabilized zirconia on a nickel superalloy substratum. In addition to the pure coatings, a graded composition layer, composed of nickel and zirconia, was used between the substratum and the outer layer. The cylinders were exposed to a thermal cycle consisting of a rapid heating phase followed by a cooling phase. The process was accomplished by the cross-flow of high temperature air past the hollow cylinder, followed by the use of low temperature air. The analysis reported herein attempts to analytically simulate this experiment. In doing so, several simplifying assumptions were made.

The gas flow around the cylinder was considered symmetric, therefore, only one half of the hollow cylinder was modelled. A plane, along the cylinder axis, passing through the forward stagnation point and the after stagnation point was considered an adiabatic surface. No axial variation of the heat transfer conditions was allowed, however, the local heat transfer conditions were varied in the circumferential directions.

Although during an actual experiment the cylinder was supported by some apparatus, the modelled cylinder was unsupported. Therefore, all transfer of heat occurred only in the hollow cylinder, and it was free to expand or contract in all directions.

The simulation analysis was used to determine the temperature distribution and the stress distribution present in the cylinder at various times throughout the thermal cycle. Excessive stress levels were then used to predict when and what type of failure would occur.

The classical thermal stress equations are coupled to the heat equation, however, during this simulation analysis, the heat equation was solved separately. The temperature distribution was then used to solve the thermal stress problem.

II. CYLINDER CONFIGURATIONS

Four configurations were analyzed to determine the stress levels at various times in the thermal cycle. All geometries consisted of a hollow cylinder three inches long, an inside diameter of 0.5 inches, and an outside diameter dependent on the particular coating used. The thickness of the nickel superalloy substratum was 0.05 inches, and the thickness of the coatings varied from 0.005 to 0.009 inches. Drawings of all configurations are shown in Figure 1.

A. CONFIGURATION NUMBER ONE

The first coating investigated was a 0.005 inch layer of zirconia on the nickel superalloy. Material properties changed from those of the nickel superalloy to those of the zirconia coating at 0.3 inches radius.

The finite element mesh used in this arrangement consisted of 21 isoparametric quadratic elements totaling 200 nodes. The element discretization was specified as one element in the axial direction, three elements in the circumferential direction, and seven elements in the radial direction. The seven radial elements were proportioned into four elements through the substratum and three elements through the layer.

B. CONFIGURATION NUMBER TWO

The second arrangement analyzed consisted of 0.002 inches of pure nickel on the substratum followed by 0.005 inches of

stabilized zirconia. The material properties abruptly changed at two radial positions.

The finite element mesh for this configuration consisted of 21 isoparametric quadratic elements using 200 nodes. The subdivision in the axial and circumferential directions remained as in configuration number one. The seven radial elements were arranged with three elements in the substratum and two elements in each layer.

C. CONFIGURATION NUMBER THREE

The third arrangement analyzed consisted of 0.004 inches of graded layer on the substratum. This graded layer allowed a gradual change of the material properties from those of nickel to those of zirconia. This arrangement was used to match the material properties of the substratum to those of the coatings. A 0.003 inch layer of stabilized zirconia was placed on top of the graded layer. Properties of the graded composition varied in a linear fashion as described in Section III.

This configuration was discretized into 24 finite elements totaling 226 nodes. The axial and circumferential directions were divided into one element and three elements respectively. The eight elements in the radial direction were arranged with four elements in the substratum, two elements in the graded layer, and two elements in the outer layer of zirconia.

D. CONFIGURATION NUMBER FOUR

The fourth configuration was composed of the substratum followed by a 0.002 inch layer of pure nickel. A 0.004 inch coating of graded composition was placed on the pure nickel. A 0.003 inch coating of pure zirconia was used as the outer layer.

The finite element mesh for this configuration consisted of 27 isoparametric quadratic elements using 252 nodes. The axial and circumferential directions were subdivided as in the other configurations. Nine elements were used in the radial direction, of which three were placed in the substratum, followed by two in each of the three layers.

E. COMMENTS ON THE FINITE ELEMENT MESH

Emphasis has been placed on the radial and tangential variations, therefore, the mesh was made finer for those directions. Since a limitation on the computer core requirement and computing time dictates the number of elements and nodes, the axial direction was modelled using only one element. The problem solution is a three dimensional method, therefore, ideally it would be more appropriate to have additional elements. Since the main interest was focused on the radial variation in material composition and the circumferential variation of heat transfer coefficients, the mesh used was considered appropriate. There was no axial variation in material properties or boundary conditions, therefore, no axial variation in the temperature distribution resulted.

A slight variation in the stress distribution resulted due to the three dimensional nature of the problem.

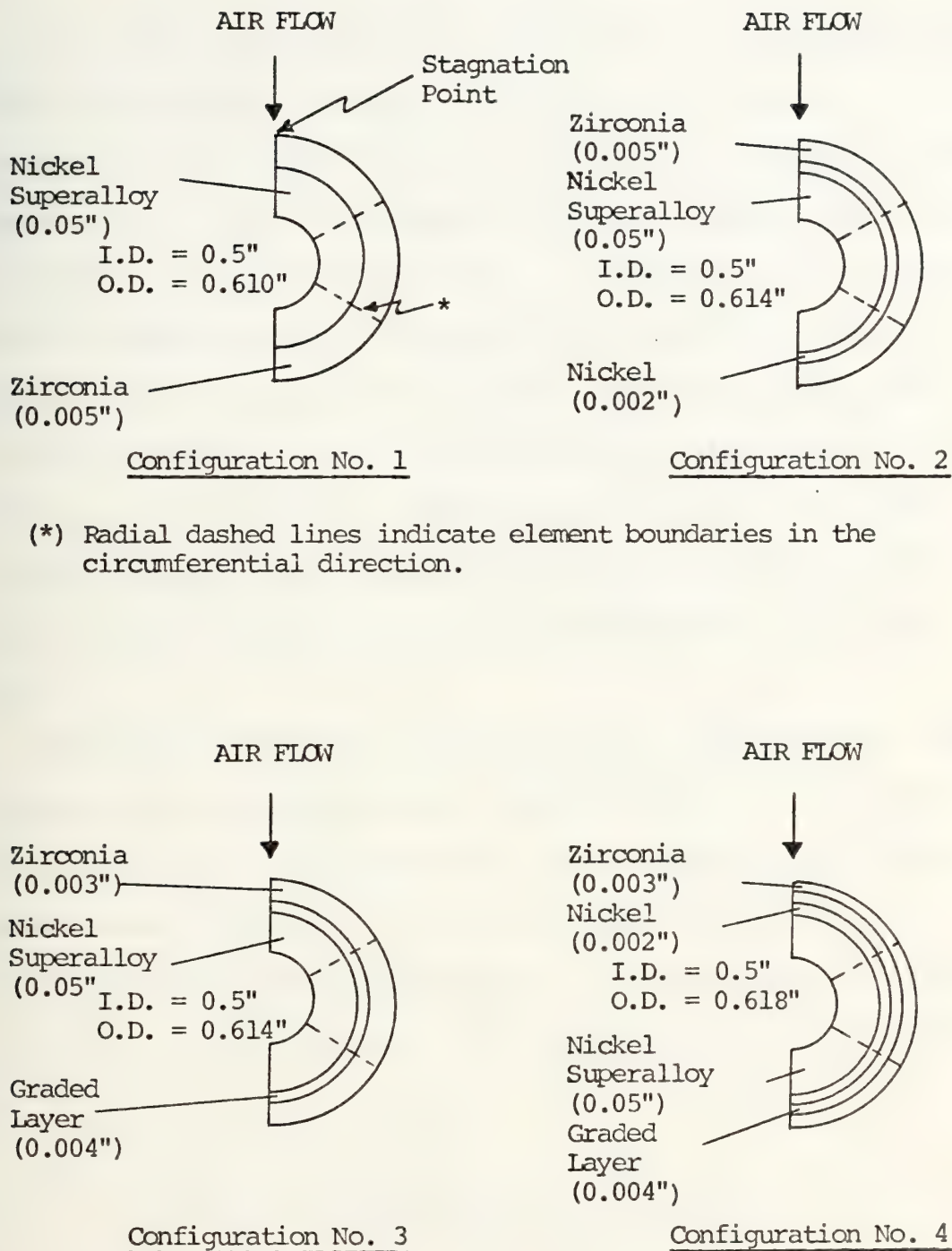


FIGURE 1. CYLINDER CONFIGURATIONS

III. MATERIAL PROPERTIES

The materials used in the analysis were a nickel-superalloy (IN-738), pure nickel, and stabilized zirconia (Zr O_2 plus Ca O). The substratum of the hollow cylinder was composed of the nickel superalloy, and the coatings were combinations of nickel and zirconia. Material property values were assumed constant throughout the temperature range investigated. The values used were the average of those available in the literature [2,3]. The numerical values for elastic modulus, Poisson's ratio, coefficient of linear expansion, thermal conductivity, and volumetric heat capacity (density times specific heat) of the various materials are given in Table 1.

Matching of material properties was attempted by using a graded composition between layers in some of the coating configurations. Approximation to the properties of these compositions was made by using a function based on the radial position. If two materials A and B have properties P_A and P_B , the property of the composition P_C was determined by

$$P_C = P_A \left(\frac{r_B - r}{r_B - r_A} \right) + P_B \left(\frac{r - r_A}{r_B - r_A} \right) \quad (1)$$

where r_A is the radius of material A and r_B , that of material B.

TABLE 1
MATERIAL PROPERTIES

Nickel Superalloy - IN-738

Modulus of Elasticity	- 31.75×10^6 PSI
Poisson's Ratio	- 0.3
Coefficient of Linear Expansion	- 8.6×10^{-6} in/in-deg.F
Thermal Conductivity	- 1.34×10^{-2} BTU/min-in-deg.F
Volumetric Heat Capacity	- 0.029 BTU/cu.in.-deg.F

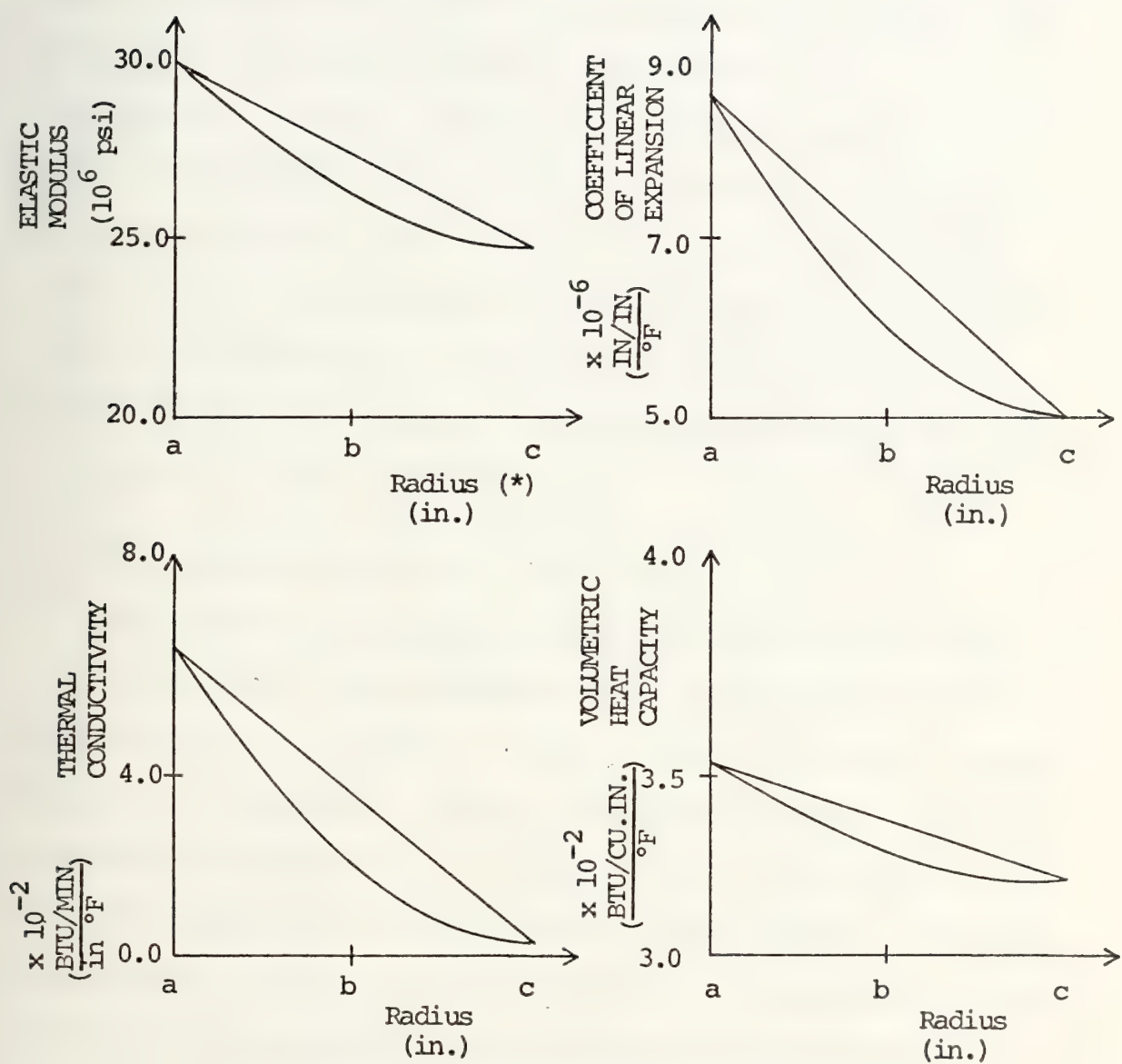
Pure Nickel

Modulus of Elasticity	- 30.00×10^6 PSI
Poisson's Ratio	- 0.3
Coefficient of Linear Expansion	- 8.6×10^{-6} in/in-deg.F
Thermal Conductivity	- 6.89×10^{-2} BTU/min-in-deg.F
Volumetric Heat Capacity	- 0.035 BTU/cu.in.-deg.F

Stabilized Zirconia

Modulus of Elasticity	- 24.8×10^6 PSI
Poisson's Ratio	- 0.3
Coefficient of Linear Expansion	- 5.0×10^{-6} in/in-deg.F
Thermal Conductivity	- 3.36×10^{-3} BTU/min-in-deg.F
Volumetric Heat Capacity	- 0.032 BTU/cu.in.-deg.F

The materials used in the graded layer were pure nickel and stabilized zirconia. The values for the various properties of the graded compositions are shown in the material property graphs, Figure 2. The property values used during the quadratic variations were determined by specifying property values for three points in the layer and then forming the quadratic function. Two of the points were specified to have properties of the inside and outside of the graded layer, i.e., the properties of nickel and zirconia. The midpoint was specified as the arithmetic mean of the properties at the midpoint of the linear variation and the properties of zirconia. The resultant quadratic variation is shown on the same plots as the linear variation.



(*) Configuration Three: $a=0.3, b=0.302, c=0.304$
 Configuration Four : $a=0.302, b=0.304, c=0.306$

FIGURE 2. MATERIAL PROPERTY GRAPHS

IV. THERMAL CYCLE CONDITIONS

The conditions which specify the thermal stress problem are classified as boundary conditions and initial conditions. With respect to the temperature distribution problem, the boundary conditions consist of convection heat transfer coefficients and the external fluid temperatures, and the initial condition is the initial temperature distribution. The stress problem has boundary conditions which specify zero or given displacement at various positions. In addition, the zero-stress temperature distribution must be supplied since all temperature distributions are compared to the zero stress condition.

A. HEAT TRANSFER BOUNDARY CONDITIONS

The thermal cycle consisted of a rapid heating phase and a cooling phase. The cylinder wall was heated from room temperature (77 deg. F) to approximately 2000 deg. F in a period of 45 seconds. During the cooling phase, the heat transfer conditions were changed to cool the cylinder back to room temperature in approximately two minutes. During the heating phase, a convection boundary condition was maintained on the external surface and an adiabatic condition on the internal surface. The cooling phase was characterized by convection boundary conditions on both internal and external surfaces. A circumferential variation of the

external heat transfer coefficients was used during both phases. The internal heat transfer coefficient was uniform during the cooling phase. No axial variation in the heat transfer coefficients was used.

In order to determine an average heat transfer coefficient for both the heating and cooling phases of the transient cycle, the cylinder was treated thermally as a lumped capacitor. Using this average coefficient, an average Nusselt number was calculated using air as the external heating fluid. The temperature of the external fluid was assumed to be 2500 °F. A Reynolds number for flow past a cylinder was then computed using the Hilpert correlation. This Reynolds number was used to determine the local Nusselt number and therefore heat transfer coefficient for air flow past a horizontal cylinder. These values of Nusselt number were obtained from Kreith [5] and Krall and Eckert [6]. For the cooling phase, the average heat transfer coefficient was used for the internal surface. The values used during the computer analysis are shown in Table 2. These values were used for all configurations.

The resultant temperatures of the cylinder wall were considered adequate for the analysis. The temperature variation at 45 seconds was between 1850 deg.F and 2050 deg F. Room temperature (77 deg. F) was reached after 2.25 minutes into the cooling phase.

TABLE 2

Heating Phase

Fluid Temperature: 2500 deg. F
Average Heat Transfer Coefficient: 22.6 BTU/hr-ft²-deg. F
Average Nusselt Number: 22.5
Reynolds Number: 2262
Equivalent Fluid Velocity: 124.6 ft/sec

Local Heat Transfer Coefficients

Angle Position from Stagnation (degrees)	Coefficient (BTU/hr-ft ² -deg. F)
0.0	40.0
30.0	41.0
60.0	38.0
90.0	26.0
120.0	11.5
150.0	16.0
180.0	12.0

Cooling Phase

Fluid Temperature: 60 deg. F
Average Heat Transfer Coefficient: 13.5 BTU/hr-ft²-deg. F
Average Nusselt Number: 47.1
Reynolds Number: 7300
Equivalent Fluid Velocity: 22.9 ft/sec

Local Heat Transfer Coefficients

Angle Position from Stagnation (degrees)	Coefficient (BTU/hr-ft ² -deg. F)
0.0	22.0
30.0	20.5
60.0	15.0
90.0	9.0
120.0	6.0
150.0	12.0
180.0	12.0

B. STRESS ANALYSIS BOUNDARY CONDITIONS

The hollow cylinders analyzed were free to expand or contract in all directions.

C. INITIAL CONDITIONS

The cylinder was specified to be at an initial uniform temperature of 77 deg. F. The zero-stress temperature distribution was assumed to be the temperature when the coatings were applied. This was specified to be 932.0 deg. F.

V. RESULTS OF THE ANALYSIS

The results of the analysis showed that the stresses within the surface coatings varied from compression at room temperature to tension at the maximum temperature. Due to the variation of the heat transfer boundary conditions, a uniform temperature equal to the zero-stress condition was not present during the thermal cycle. The complex nature of the circumferential variation in heat transfer coefficients causes temperature and stress distributions vastly different from those common to radial heat flow.

All configurations follow the same general stress history. During the period when the temperatures are below the zero-stress condition, the coatings remain in compression. At the same time, the sign of the substratum stresses varies according to the position. At approximately 12 seconds, the temperature distributions reached the zero-stress condition and the stresses in the coatings became tensile. These tensile stresses increased as the temperature increased reaching a maximum at 0.75 minutes. All configurations followed this general scheme. The maximum tensile stress is dependent on the zero-stress condition, i.e., the higher the zero-stress temperature, the lower the maximum tensile stress. Configuration Four had somewhat lower stress levels than the other configurations; however, the difference is not considered significant.

The use of a graded layer was investigated further by specifying a quadratic composition instead of a linear composition. An adjustment of the thermal conductivity and the coefficient of linear expansion to follow a quadratic function produced only minor changes in the stress levels. When all properties were adjusted to the quadratic composition variation, no appreciable change occurred in the stress levels.

Although the maximum stress levels are excessive, the maximum strains (9.6×10^{-3}) are considered small enough to justify the use of a linear elastic analysis. In addition, the brittle nature of zirconia justifies the use of a brittle fracture failure criteria. Therefore, the elastic analysis provides a reliable method for determining whether the structure will fail or not. Using the maximum principal stress theory of failure, some prediction of the time and temperature when failure occurs can be made. If the ultimate strength of the ceramic material is set at 18.0 KSI and that of the nickel superalloy at 90.0 KSI, observation of the average tangential and axial stress levels for all configurations indicates that failure would occur at about 0.2 minutes and an average temperature of 1000.0 °F. At most, this is only a rough calculation. Observations of an actual specimen could confirm this estimate by raising and lowering the specimen to successively higher temperatures, followed by appropriate microscopic examination.

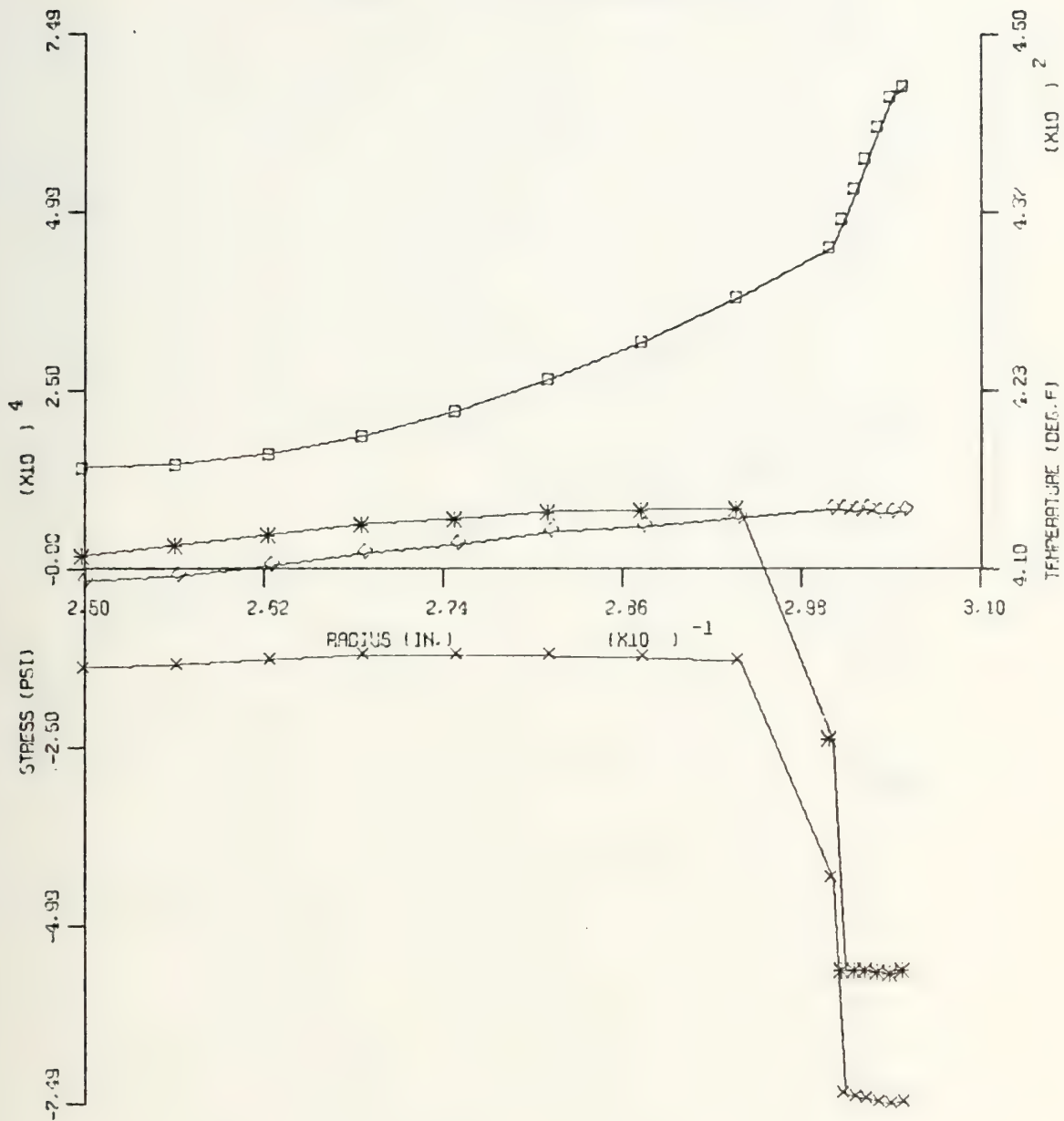
The graphical representations of the thermal stress and temperature distributions were made using the numerical values obtained for the nodes at the cylinder ends. No attempt was made to smooth out the curves and no interpolations were performed. Each plot is identified by a time and a configuration designation. For example, the first plot is identified by TIME = 0.05 CONF.1-0. This plot represents the conditions at 0.05 minutes after start of the thermal cycle for configuration number one at position zero degrees relative to the stagnation point. Each axis has been labelled and the units identified. Since the range of values for stress and temperature varies from one graph to another, the scale factors used in plotting are provided on each plot. In order to provide an overall view of the stress levels throughout the thermal cycle, a plot of extreme principal stresses versus time has been prepared for each configuration, Figures 7, 14, 21, 26.

A. CONFIGURATION NUMBER ONE

This configuration contained a single layer of stabilized zirconia on the nickel superalloy substratum. Comparison of the maximum principal stresses in all four configurations shows that configuration number one has the highest stress level. This maximum stress level (140.0 KSI) occurred at the maximum temperature, i.e., at the end of the heating phase, and was located at a radius of 0.3017 inches, the

zero angle position, and at the midpoint of the cylinder length. The largest tensile stresses present in configuration number one were all tangential and axial components of the stress state. The following figures show the radial variation of temperature and stresses at a given angular position for selected times during the heat transfer cycle: Figure 3 (a through j) at zero degrees, Figure 4 (a through j) at 60 degrees, Figure 5 (a through j) at 120 degrees, Figure 6 (a through j) at 180 degrees. (Similar sets of figures are provided for the remaining three configurations.)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.1-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

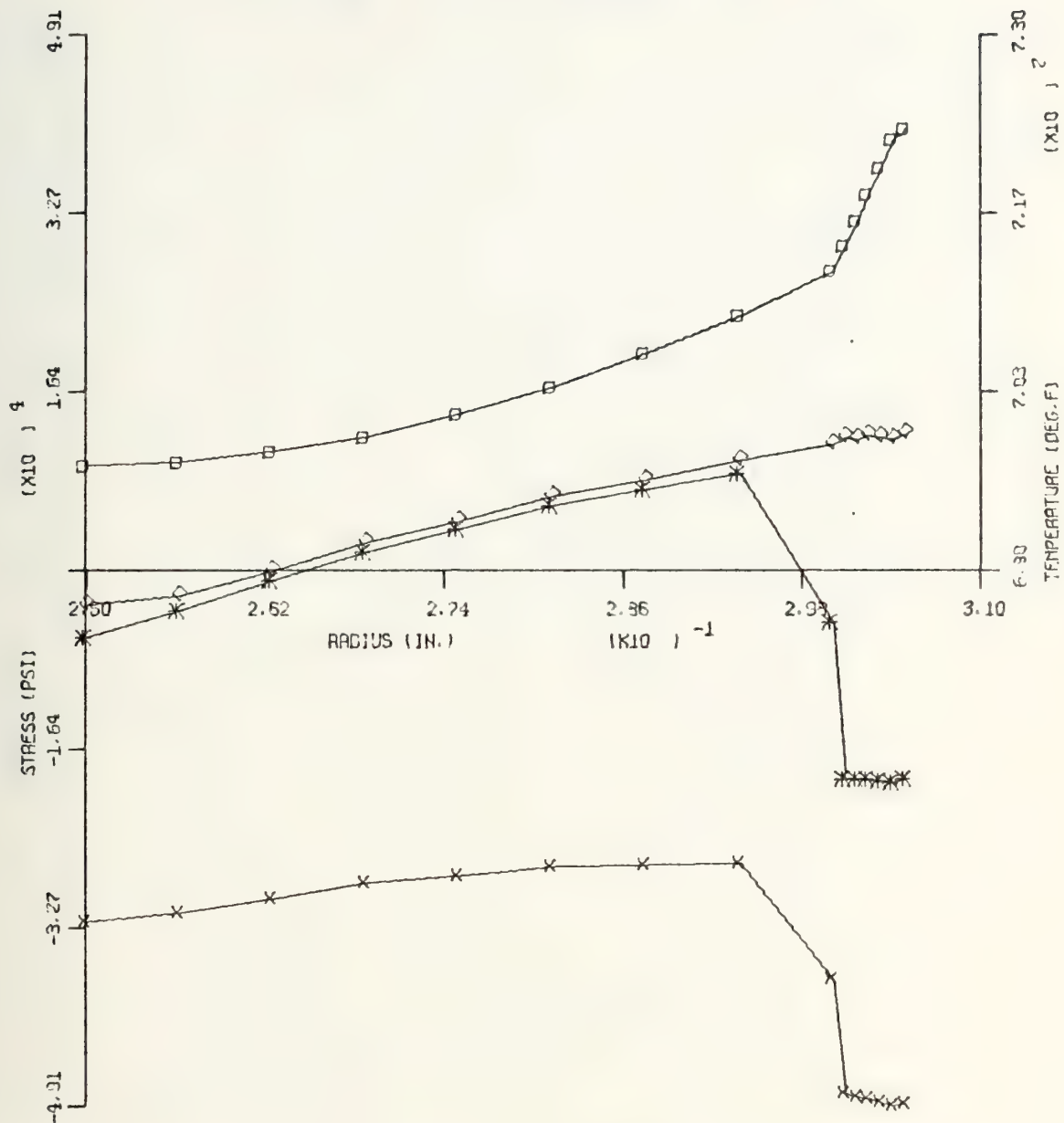
STRESS SCALE = 2.436×10^9 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 3(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.1-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

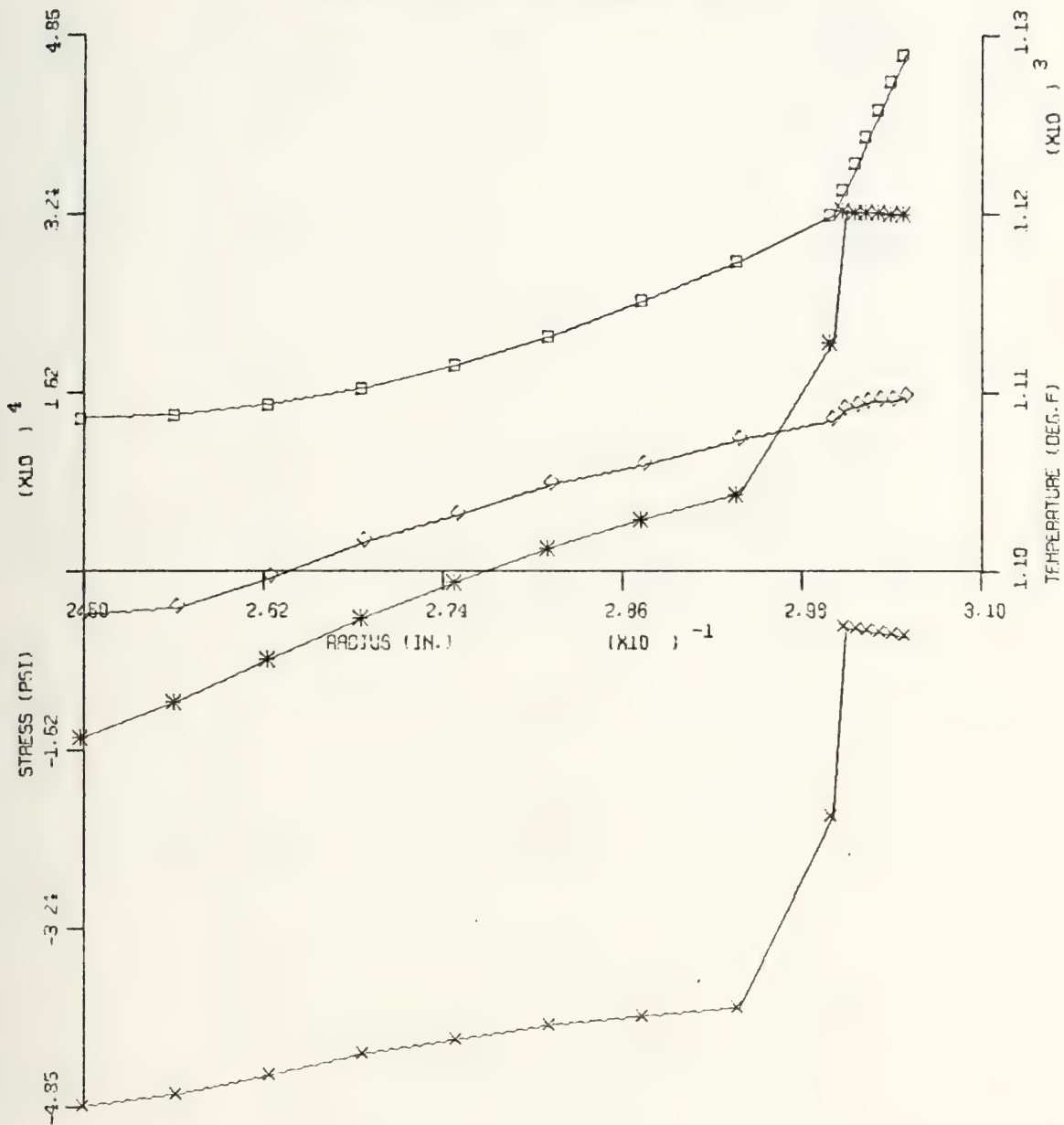
STRESS SCALE = 1.535×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 3(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.1-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

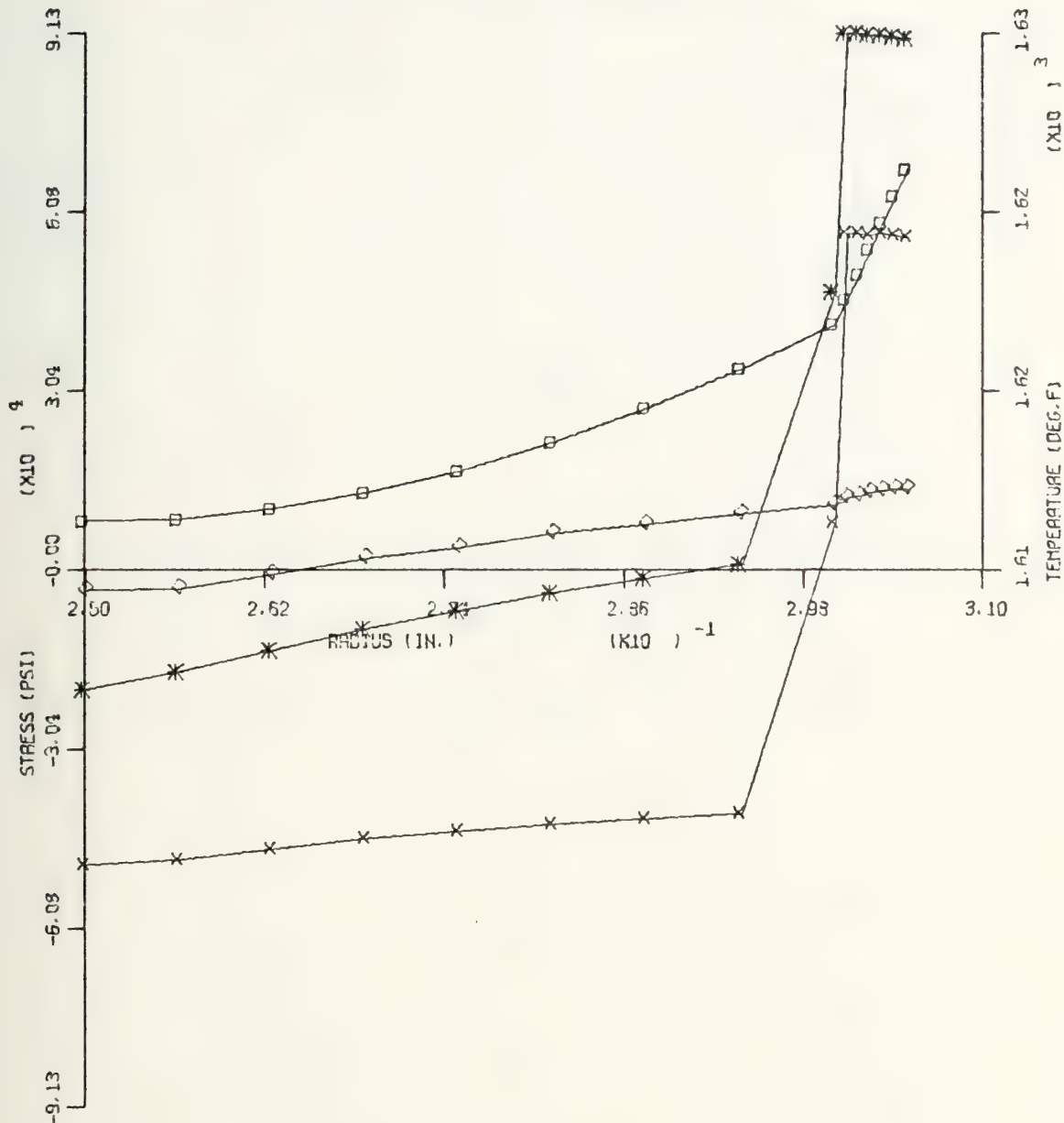
STRESS SCALE = 1.518×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 3(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.1-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

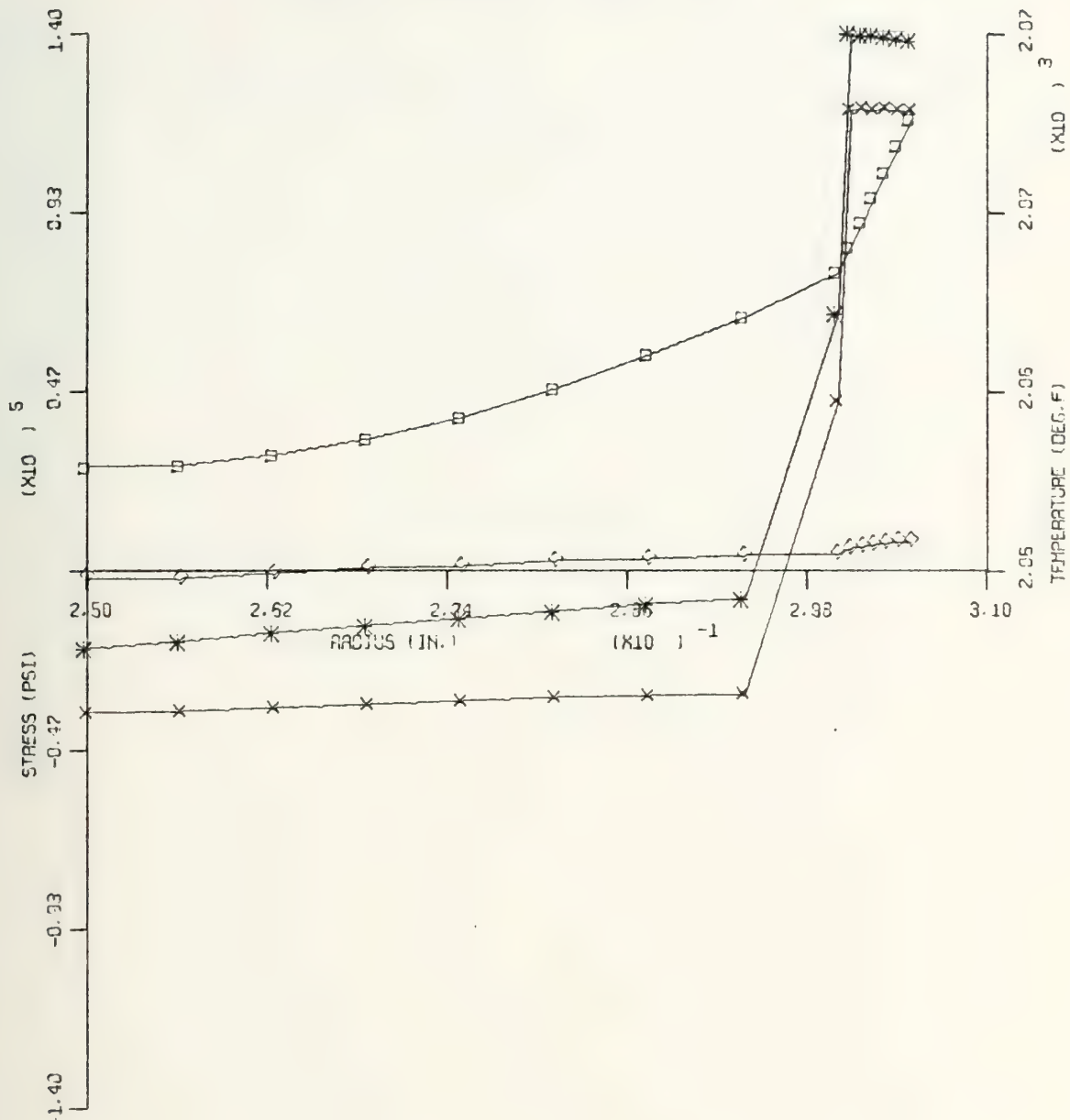
STRESS SCALE = 3.042×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 3(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.1-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

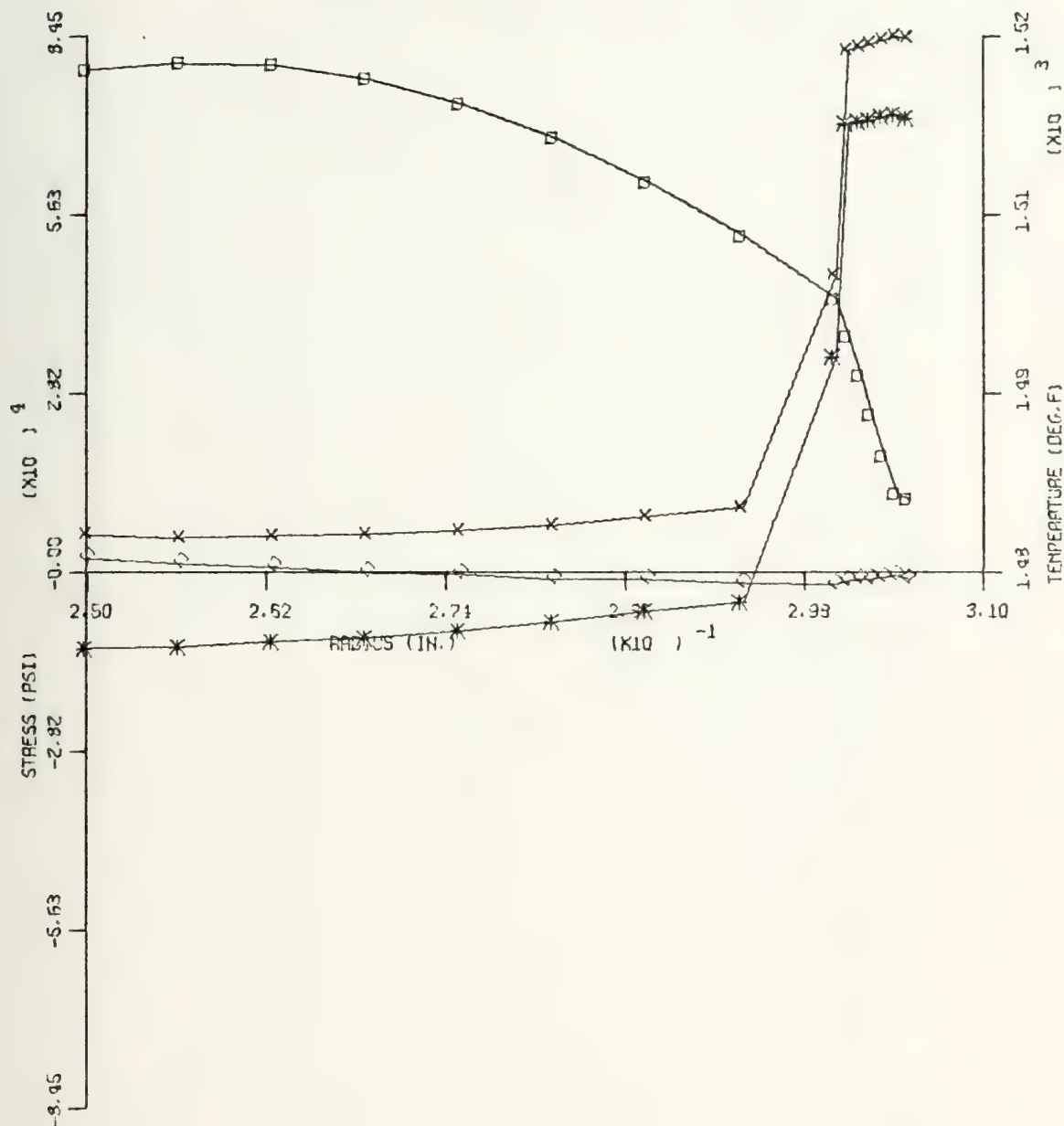
STRESS SCALE = 4.670×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 3(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.1-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

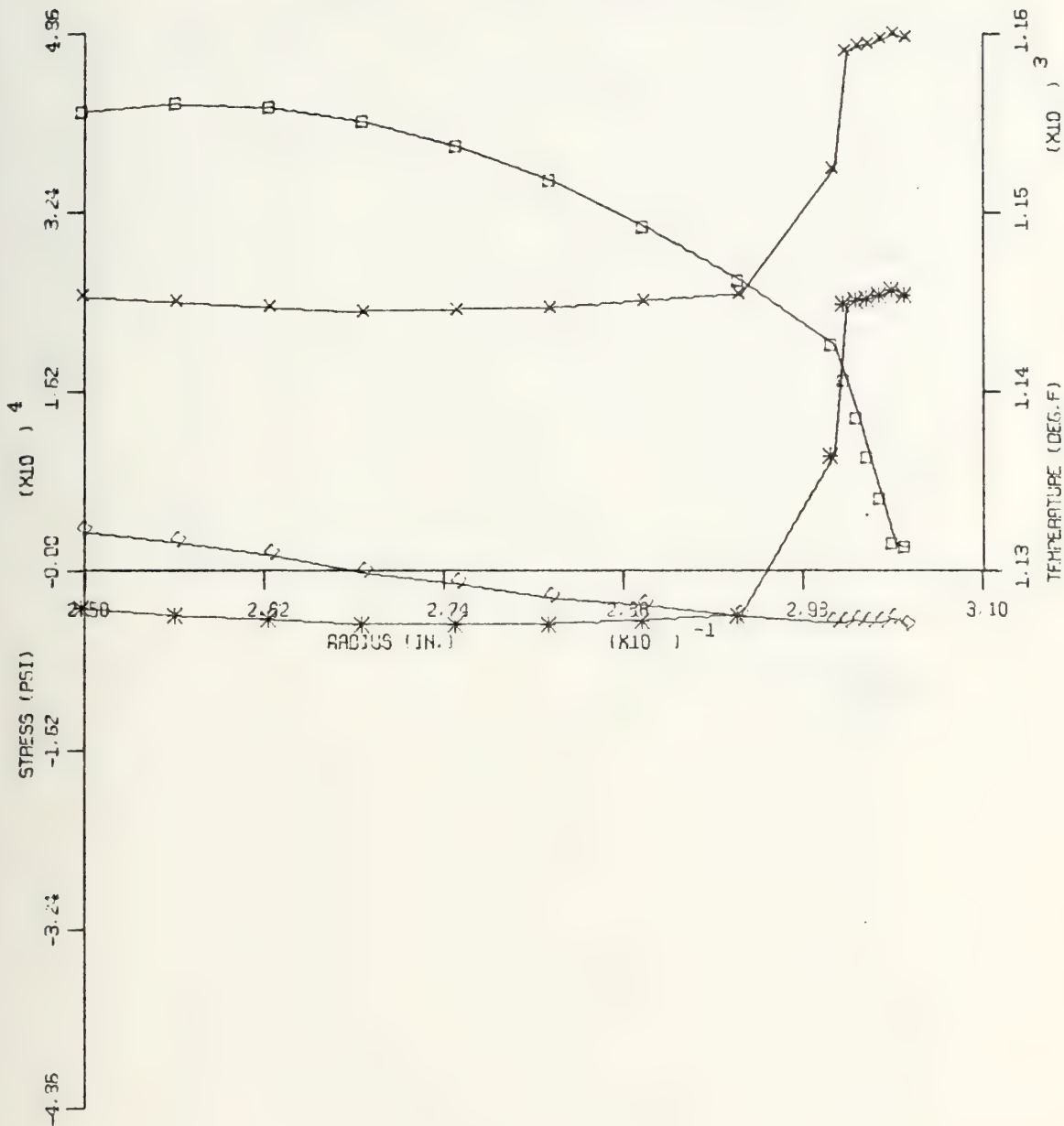
STRESS SCALE = 2.317×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 3(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.1-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

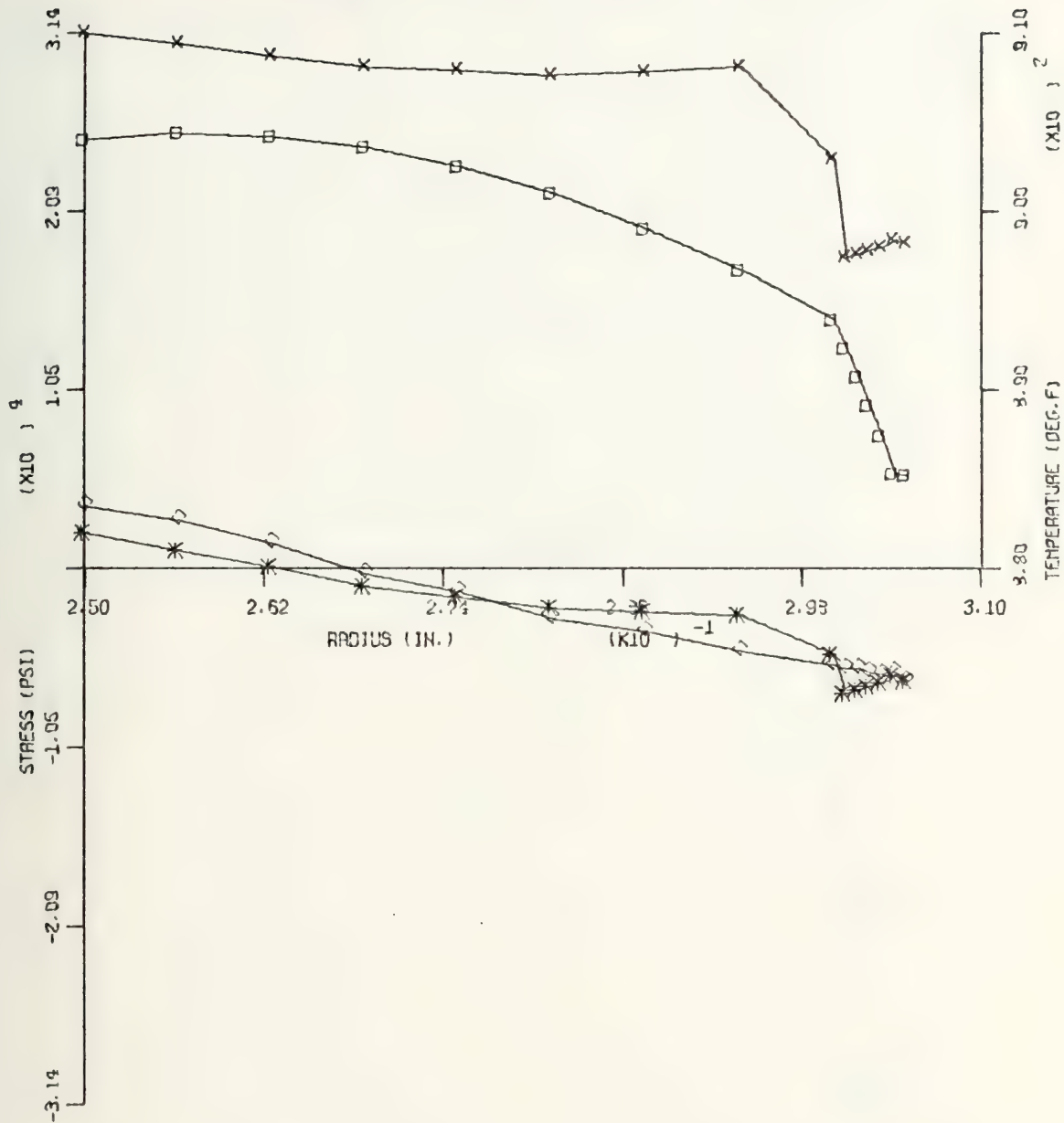
STRESS SCALE = 1.519×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 3(g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.1-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

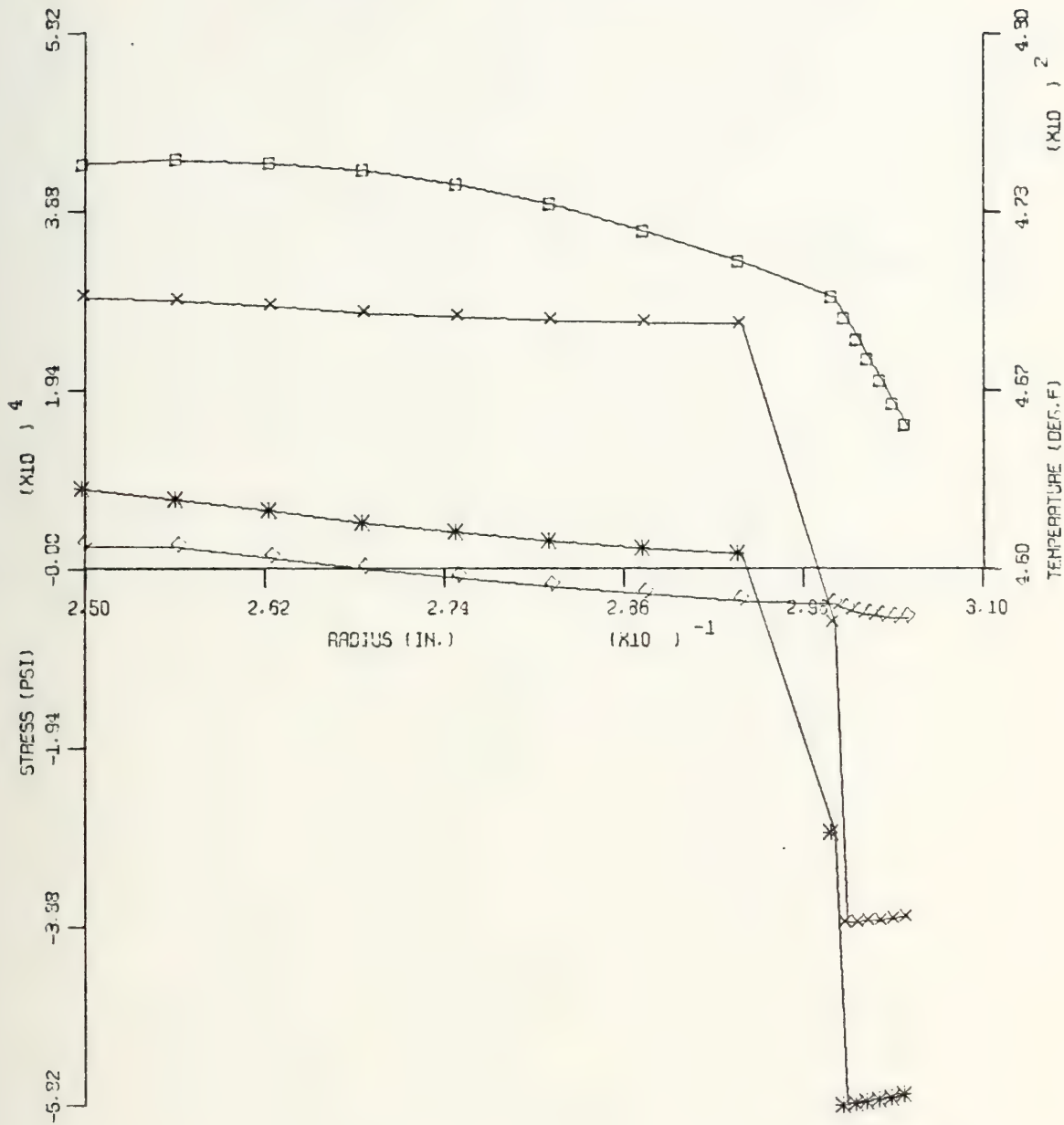
STRESS SCALE = 1.045×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 3(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.1-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

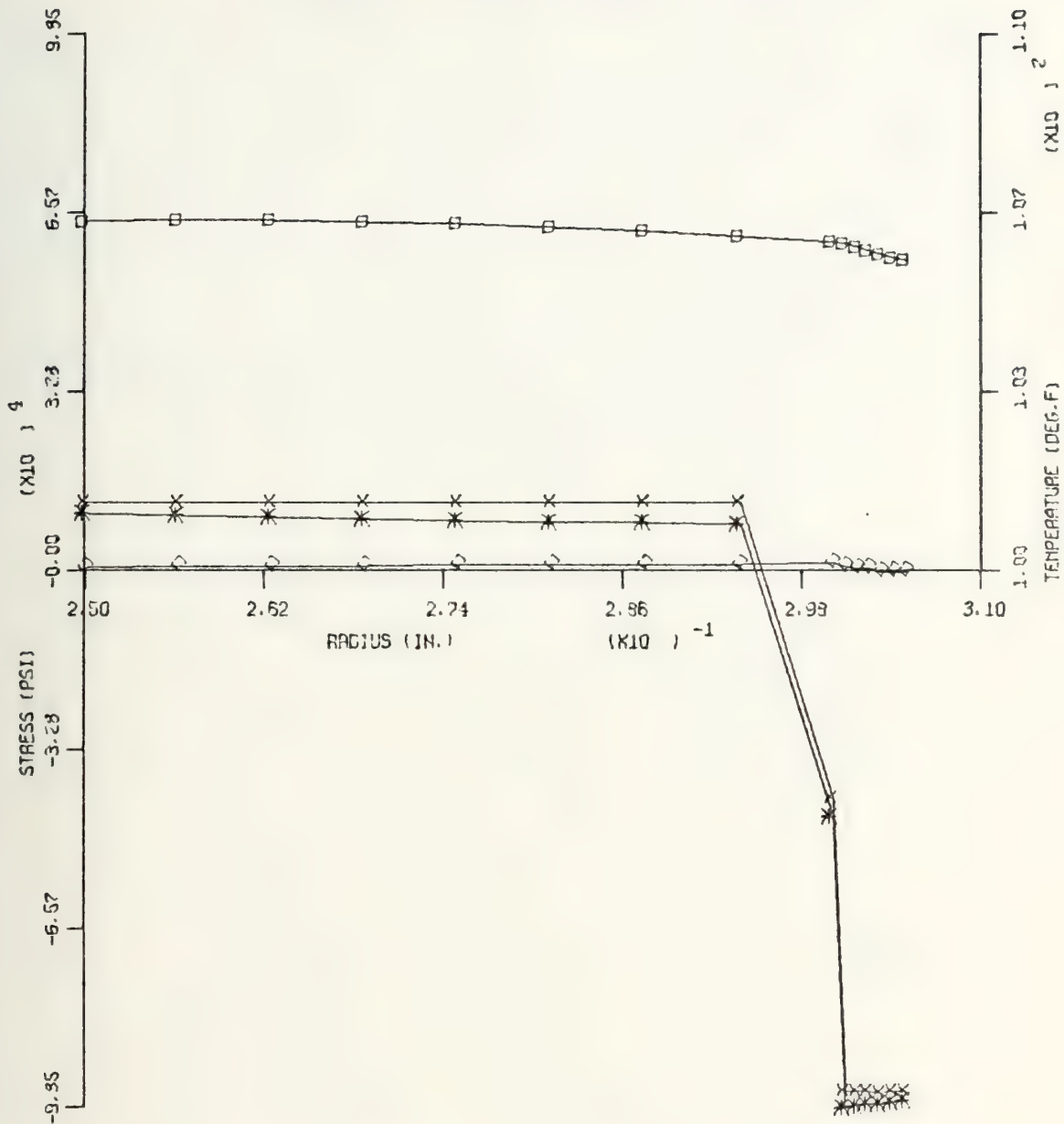
STRESS SCALE = 1.911×10^4 PSI/INCH

TEMPERATURE SCALE = 6.2 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 3(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.1-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

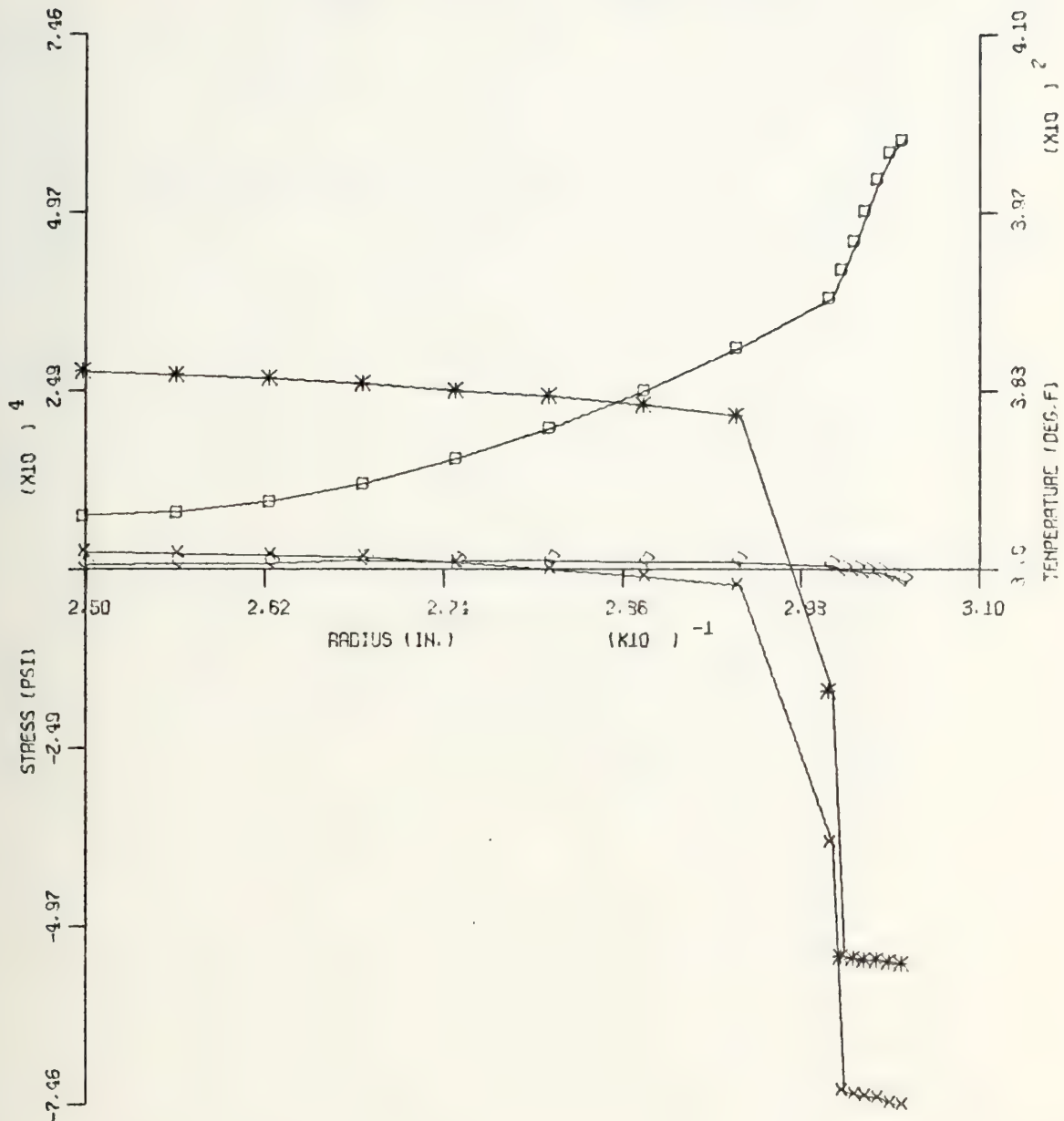
STRESS SCALE = 3.233×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 3(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF. 1-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

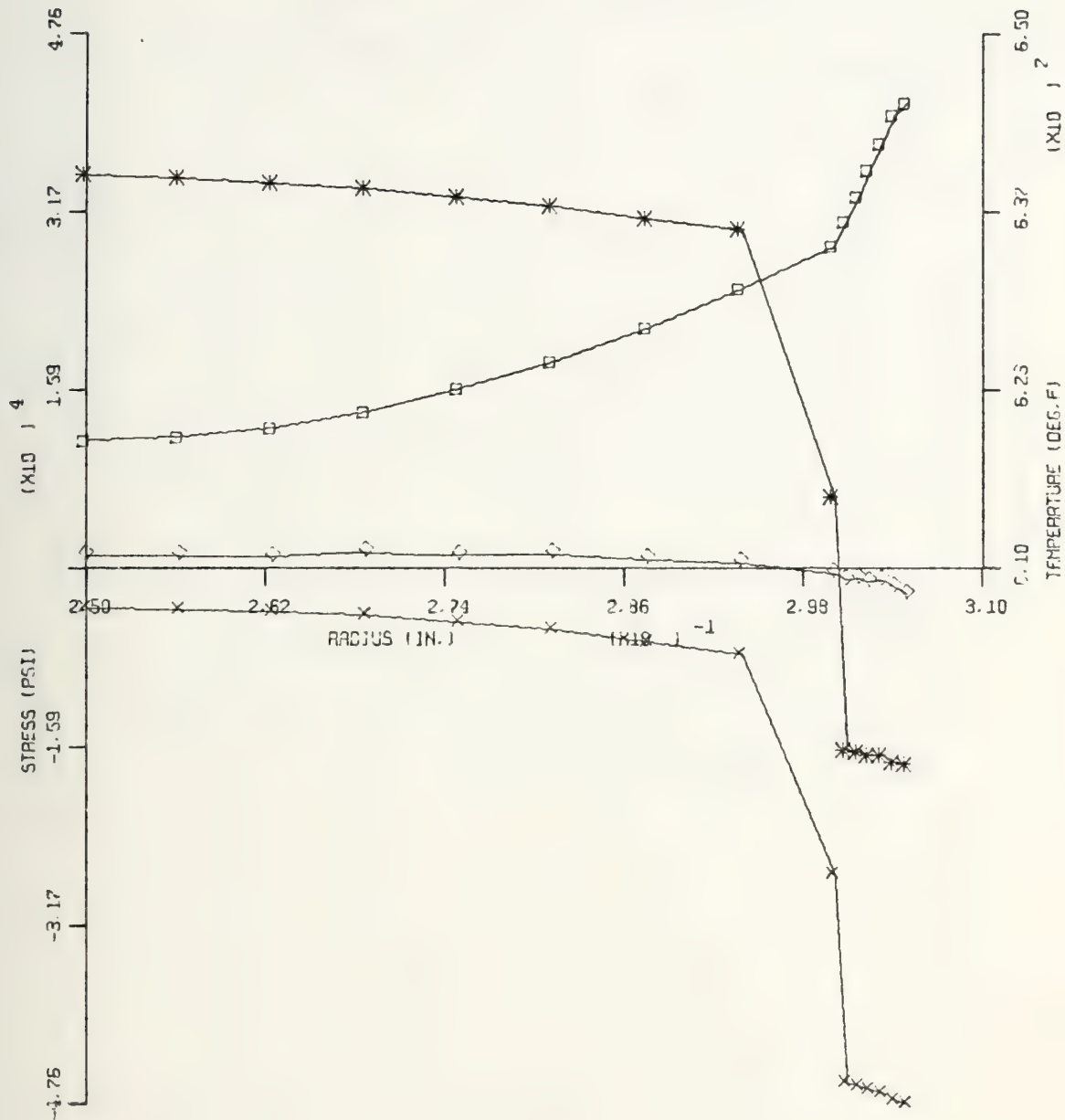
STRESS SCALE = 2.435×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 4(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.1-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

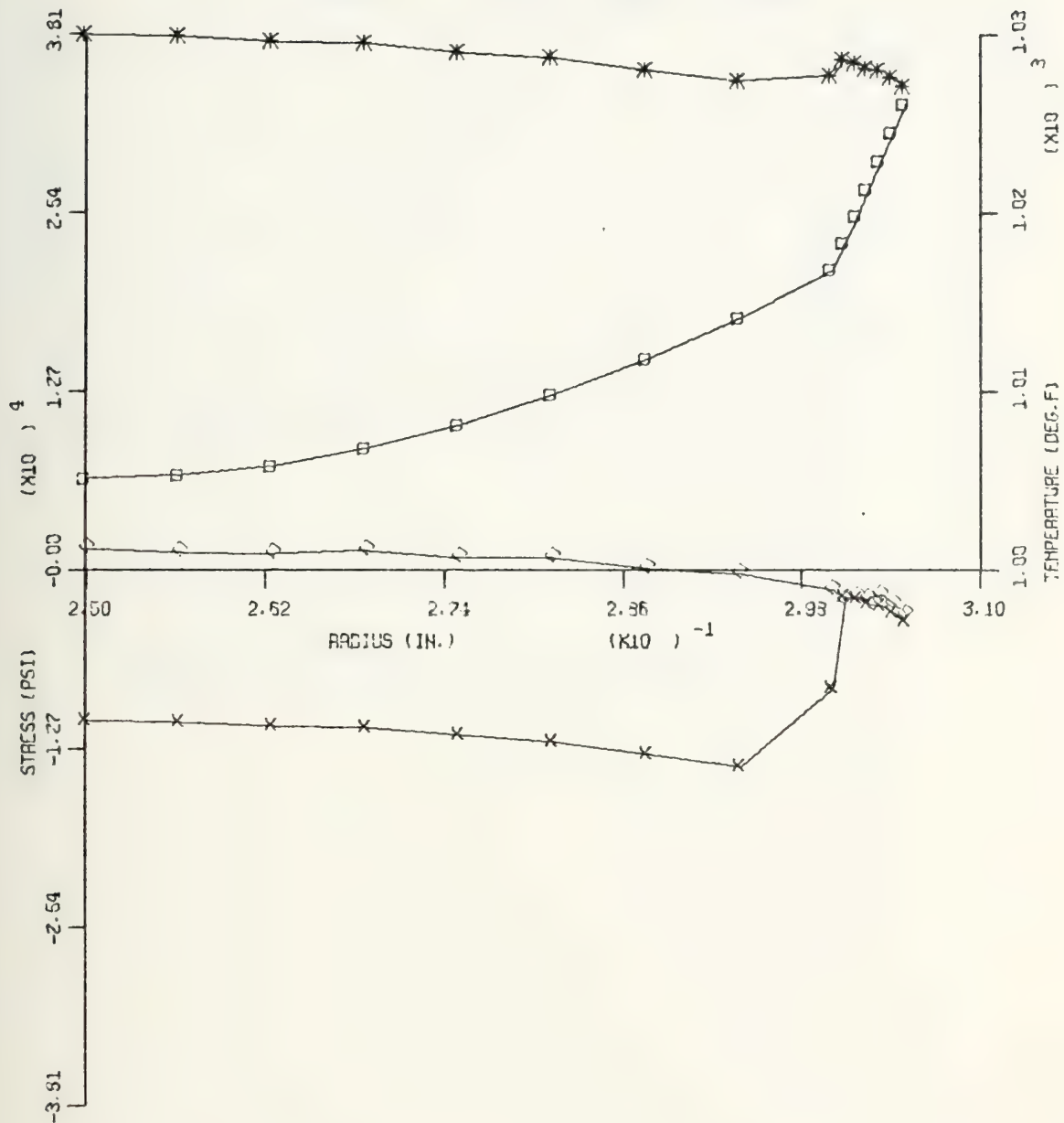
STRESS SCALE = 1.587×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 4(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.1-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

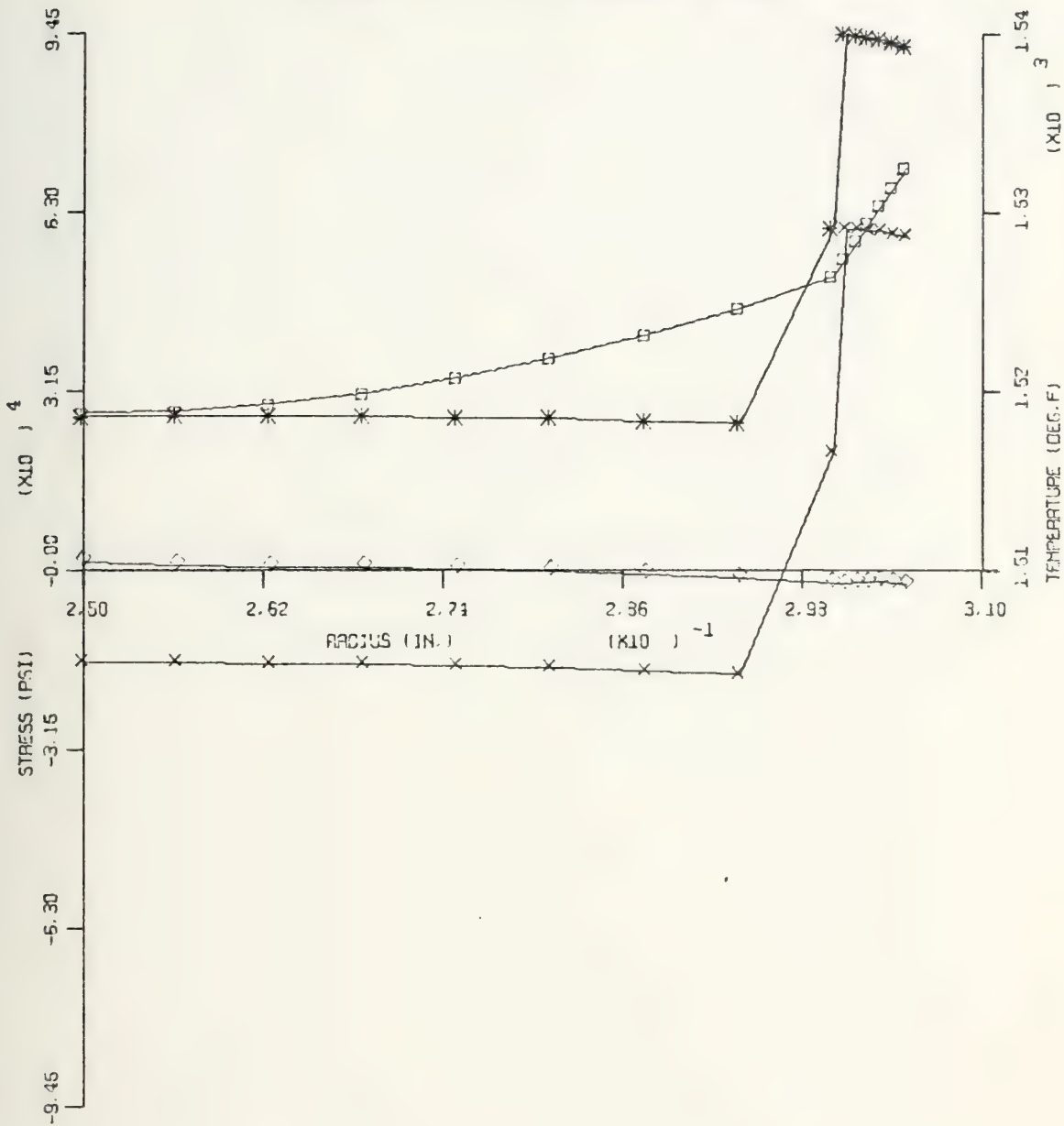
STRESS SCALE = 1.269×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 4(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.1-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

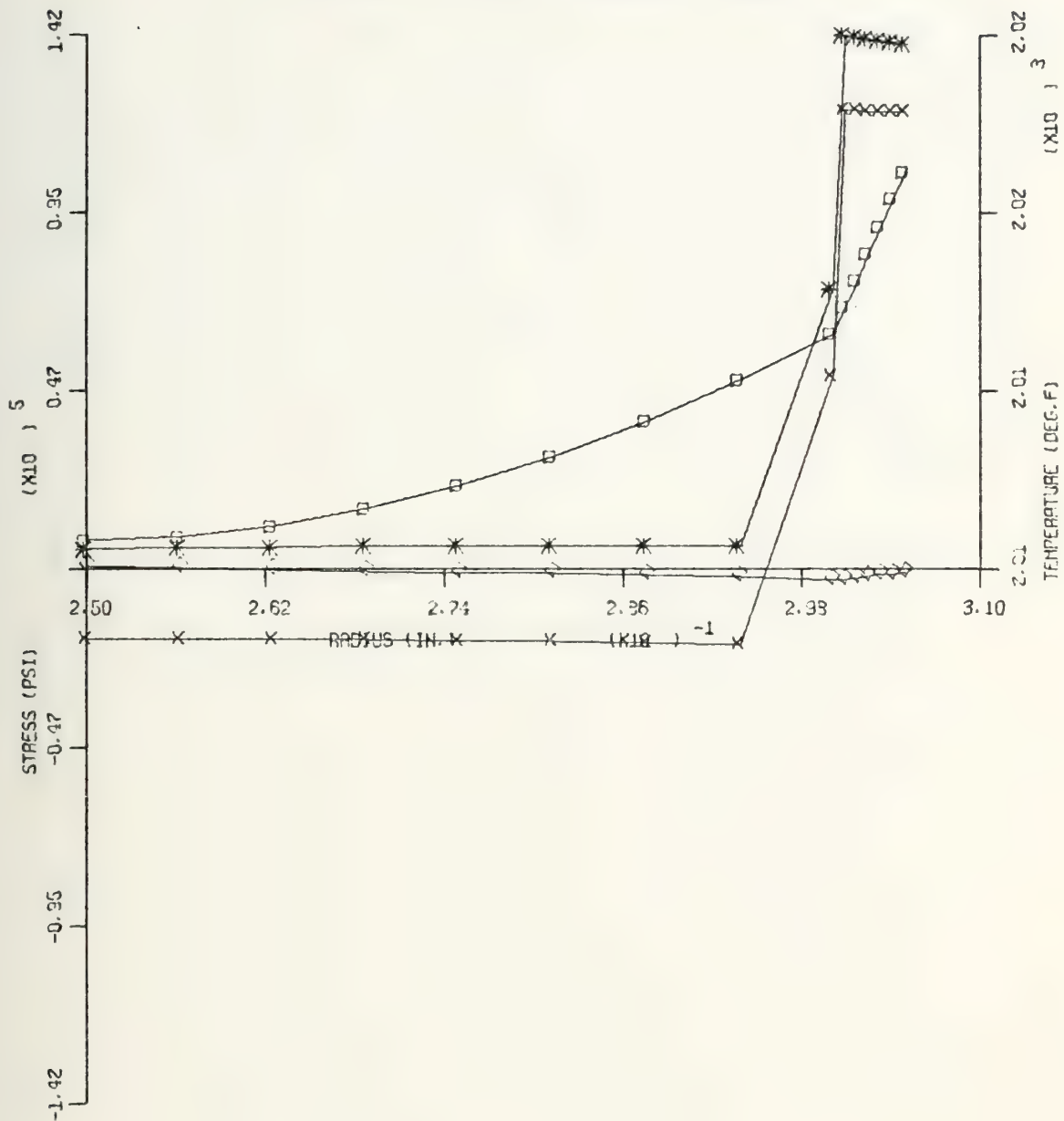
STRESS SCALE = 3.15×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 4(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.1-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

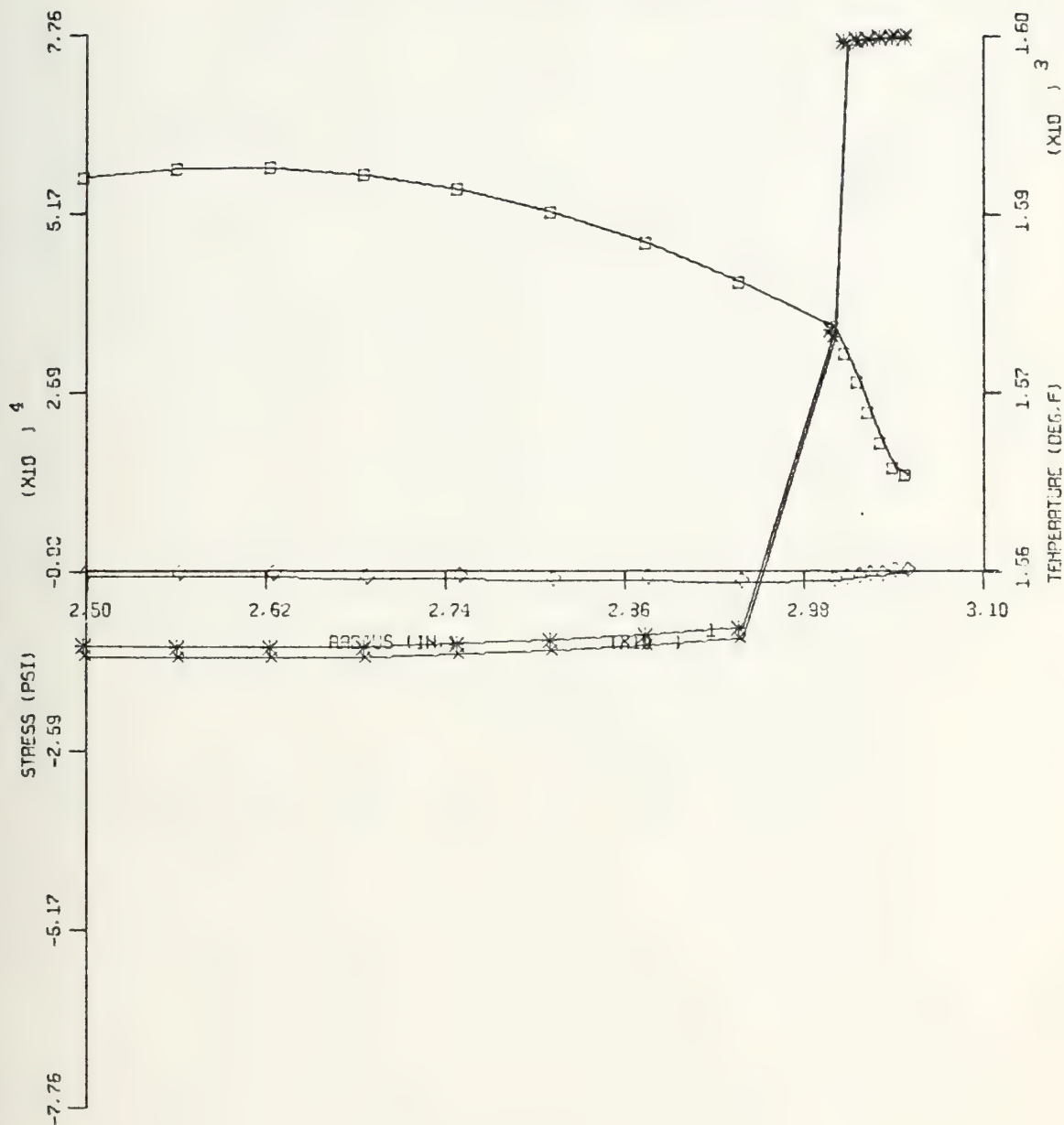
STRESS SCALE = 4.730×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 4(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.1-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

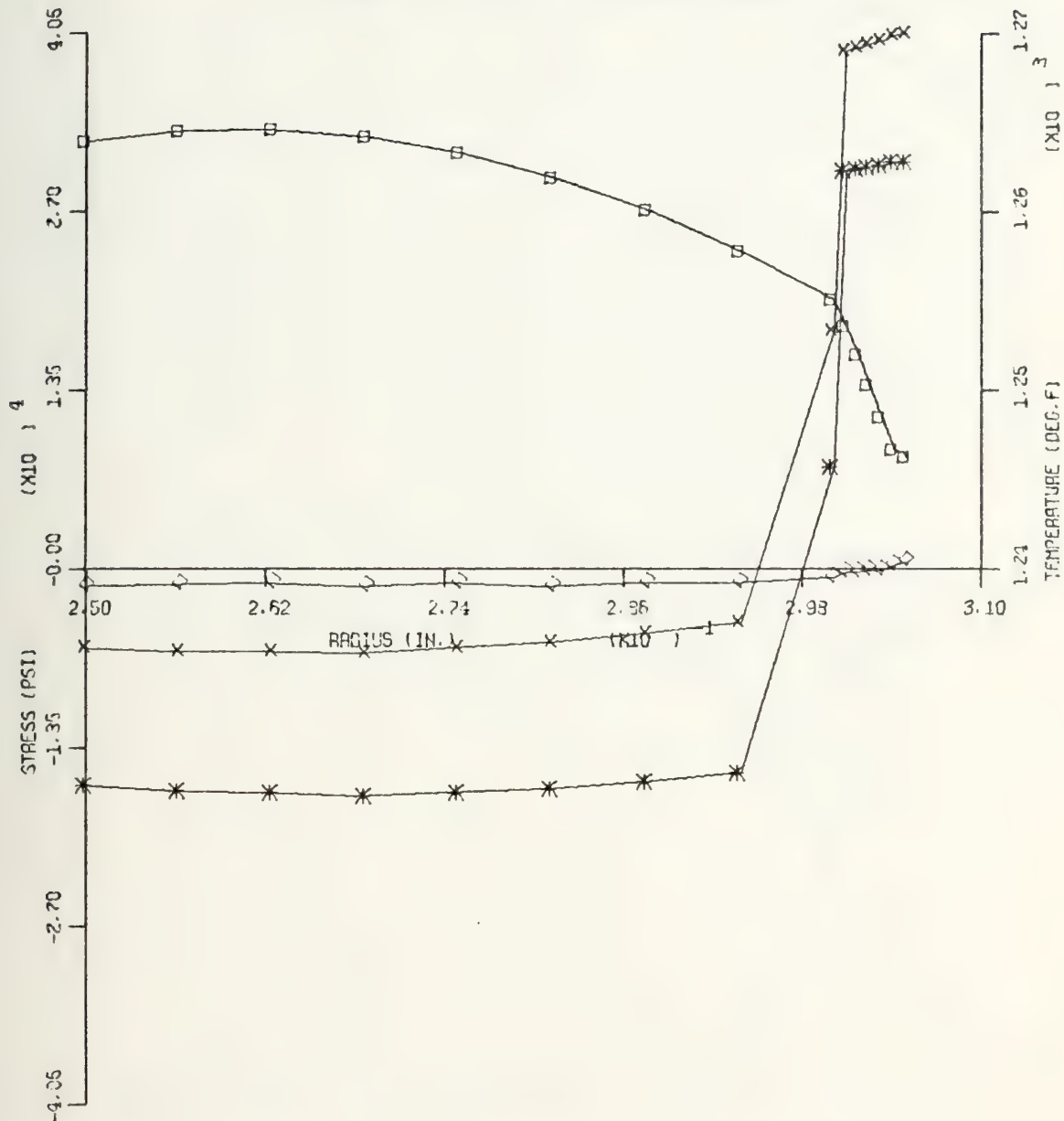
STRESS SCALE = 2.556×10^9 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 4(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.1-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

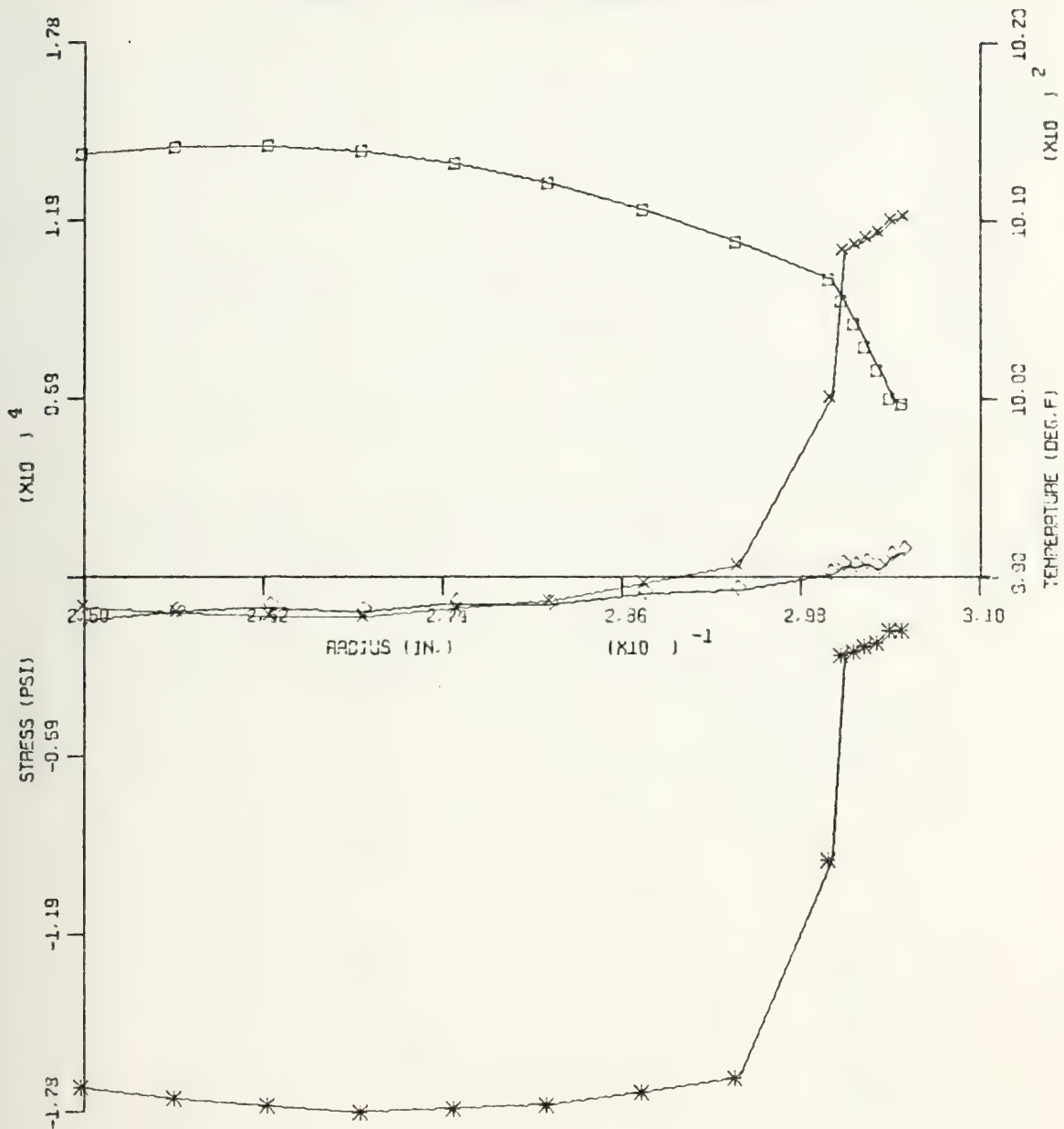
STRESS SCALE = 1.351×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 4(g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.1-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

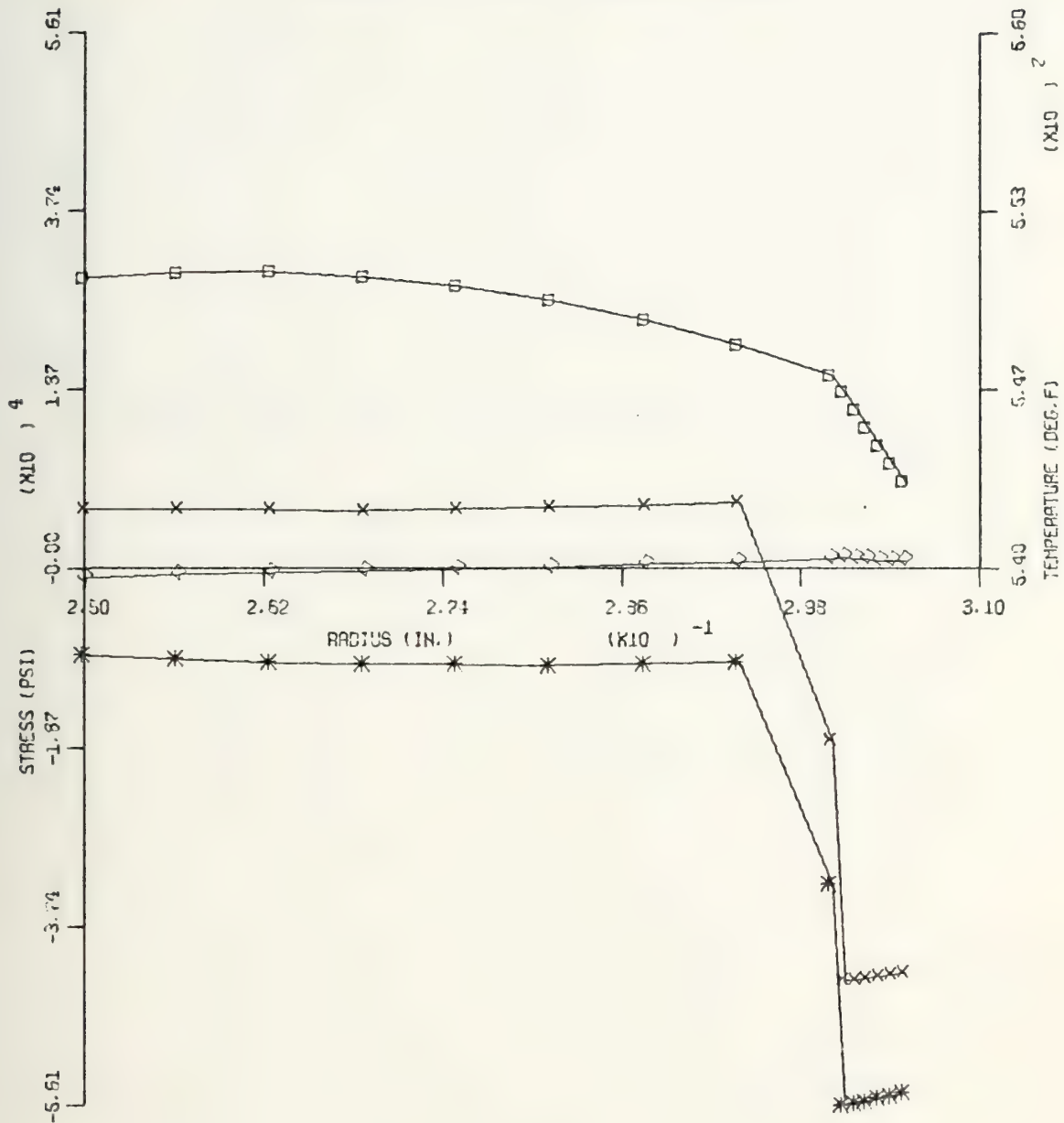
STRESS SCALE = 5.927×10^3 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 4(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.1-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

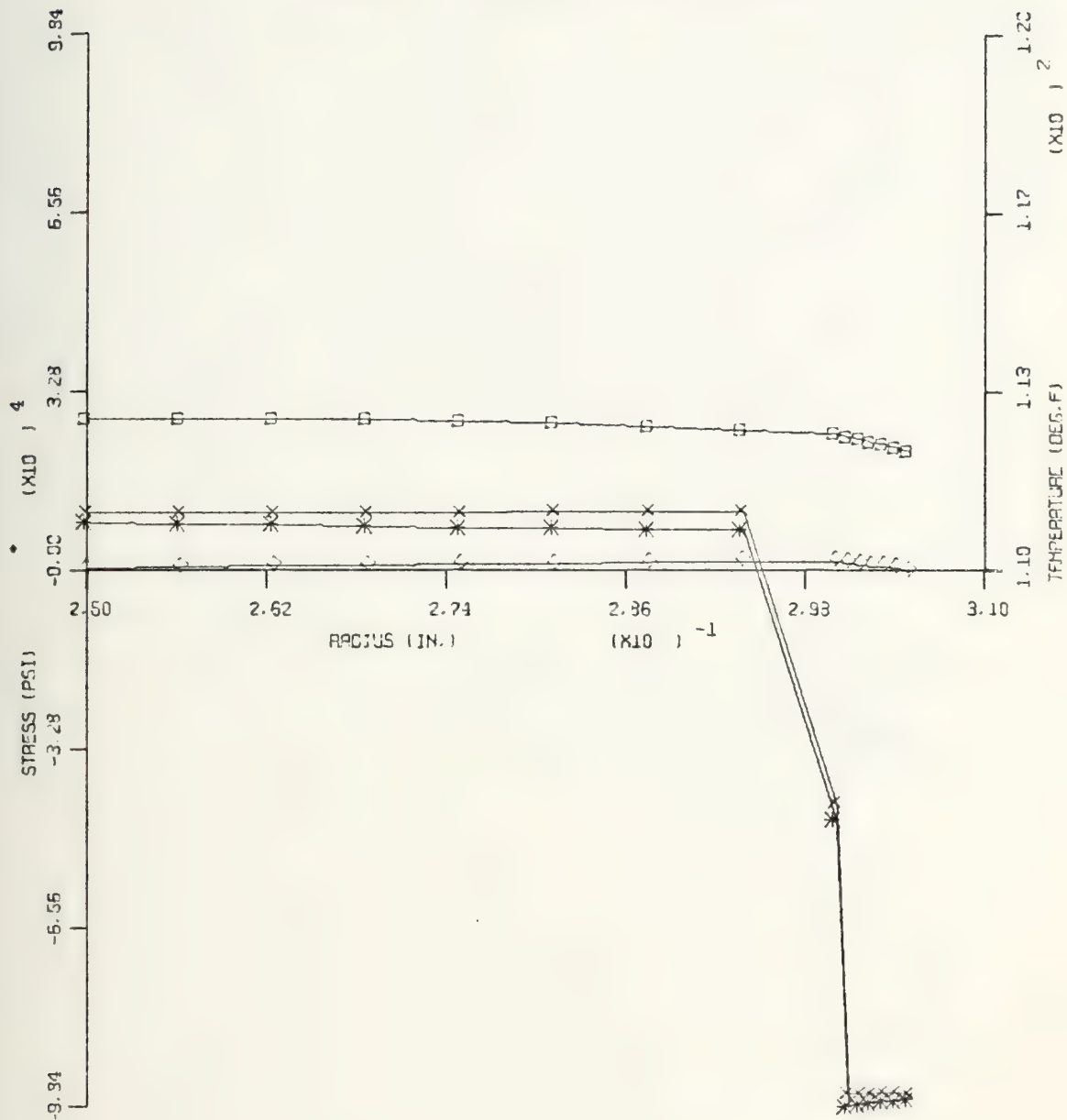
STRESS SCALE = 1.658×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 4(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.1-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

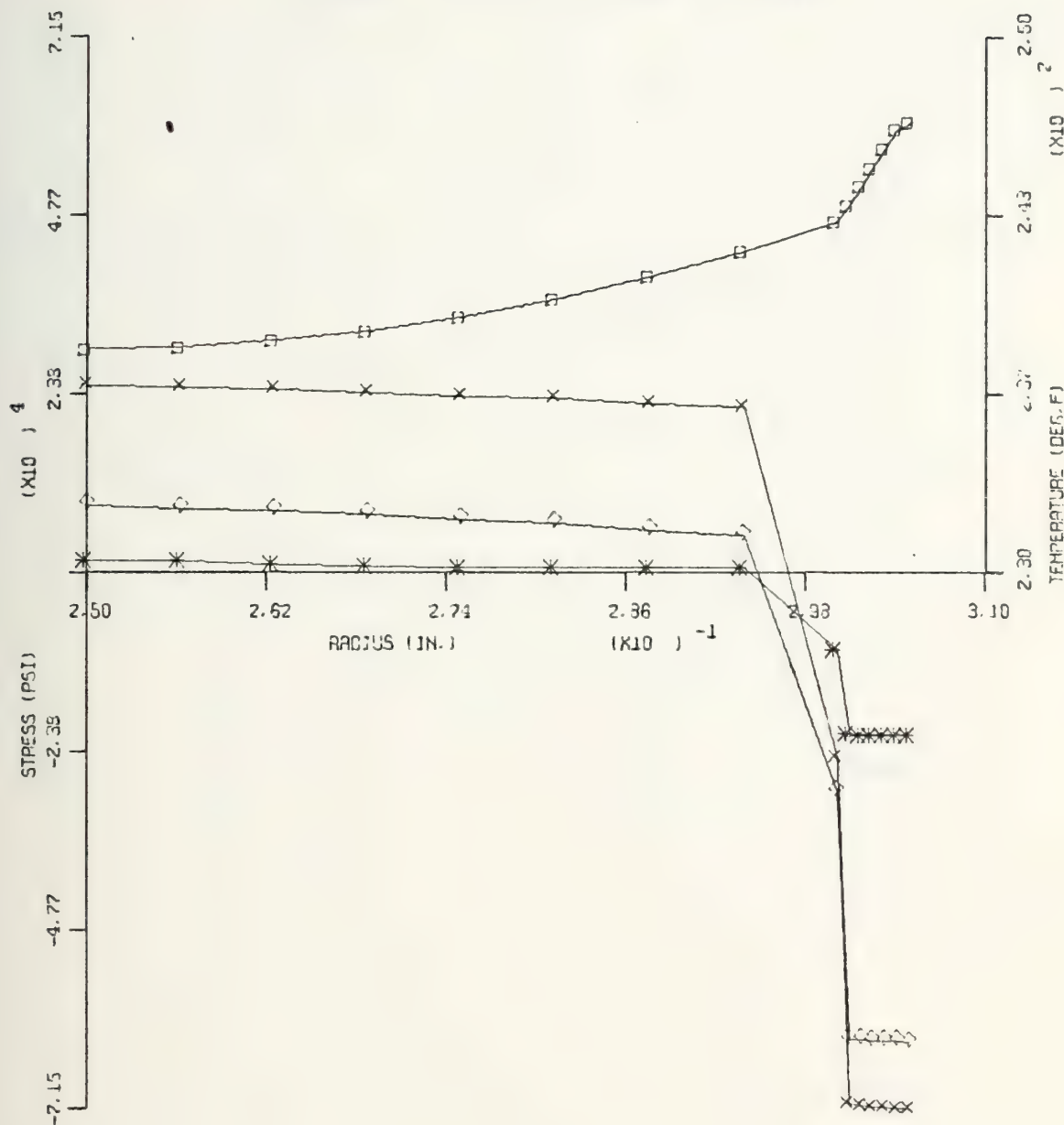
STRESS SCALE = 3.273×10^3 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 4(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.1-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

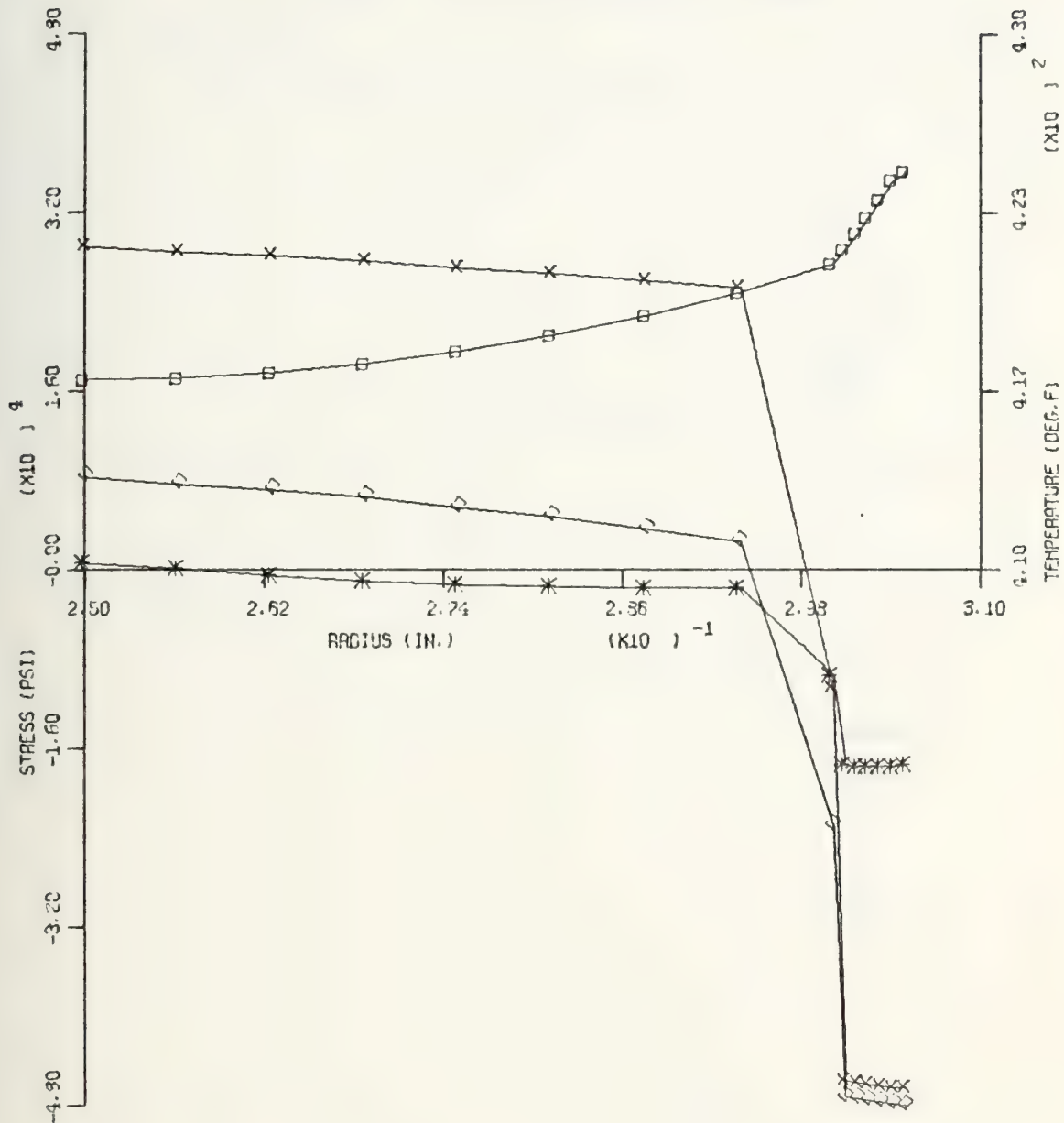
STRESS SCALE = 2.354×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 5(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.1-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

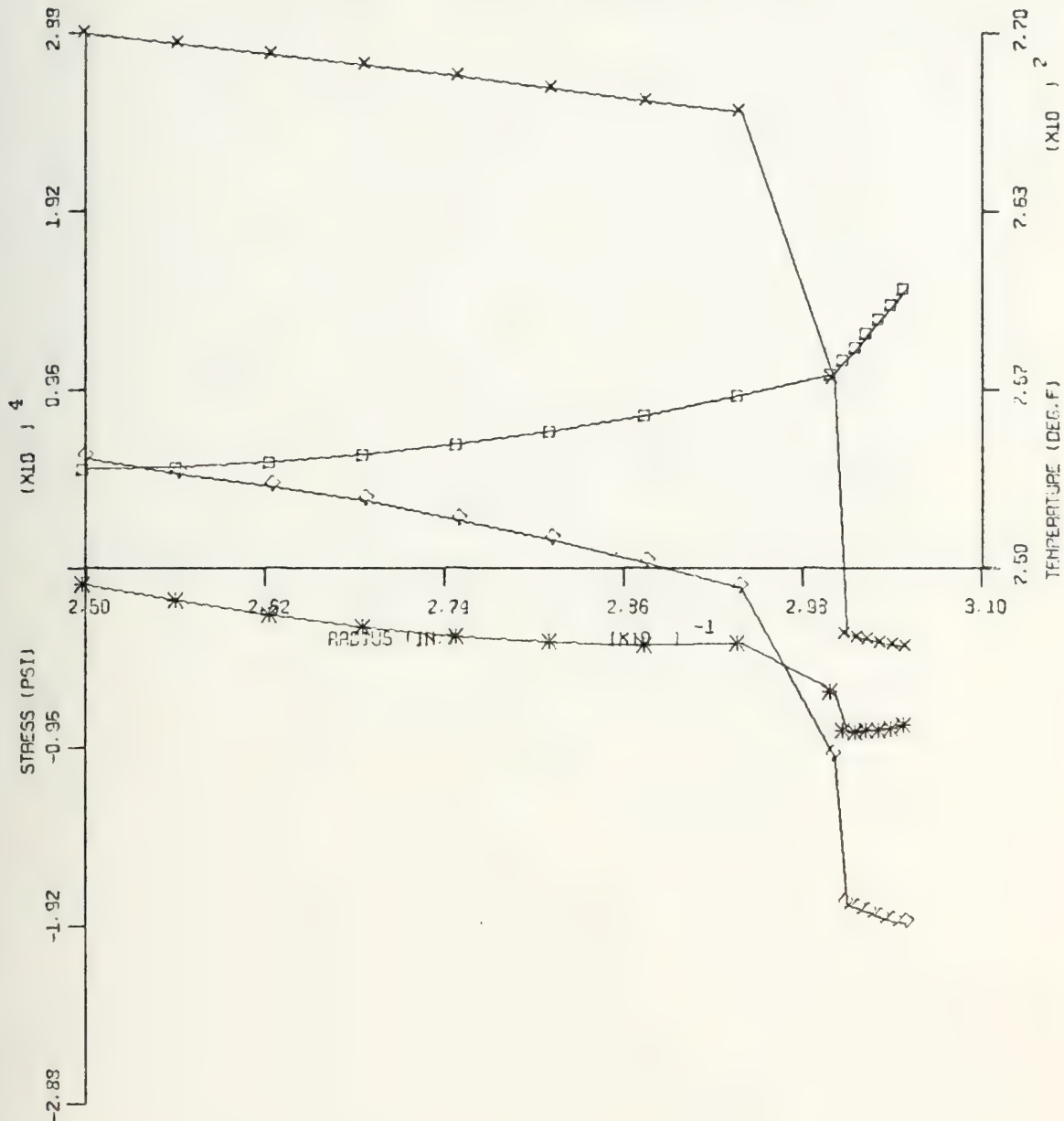
STRESS SCALE = 1.600×10^4 PSI/INCH

TEMPERATURE SCALE = 6.2 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 5(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.1-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

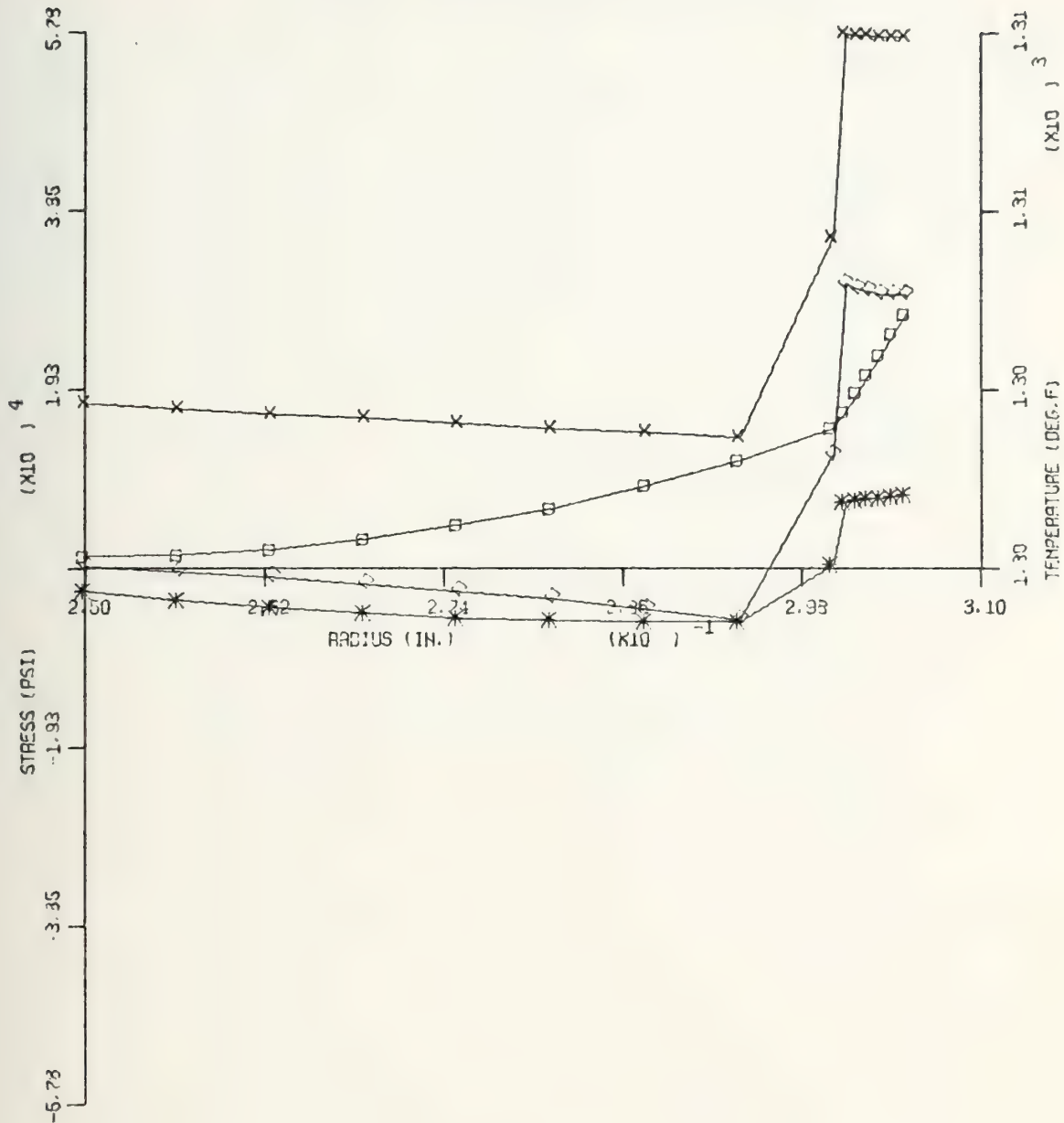
STRESS SCALE = 3.592×10^3 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 5(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.1-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

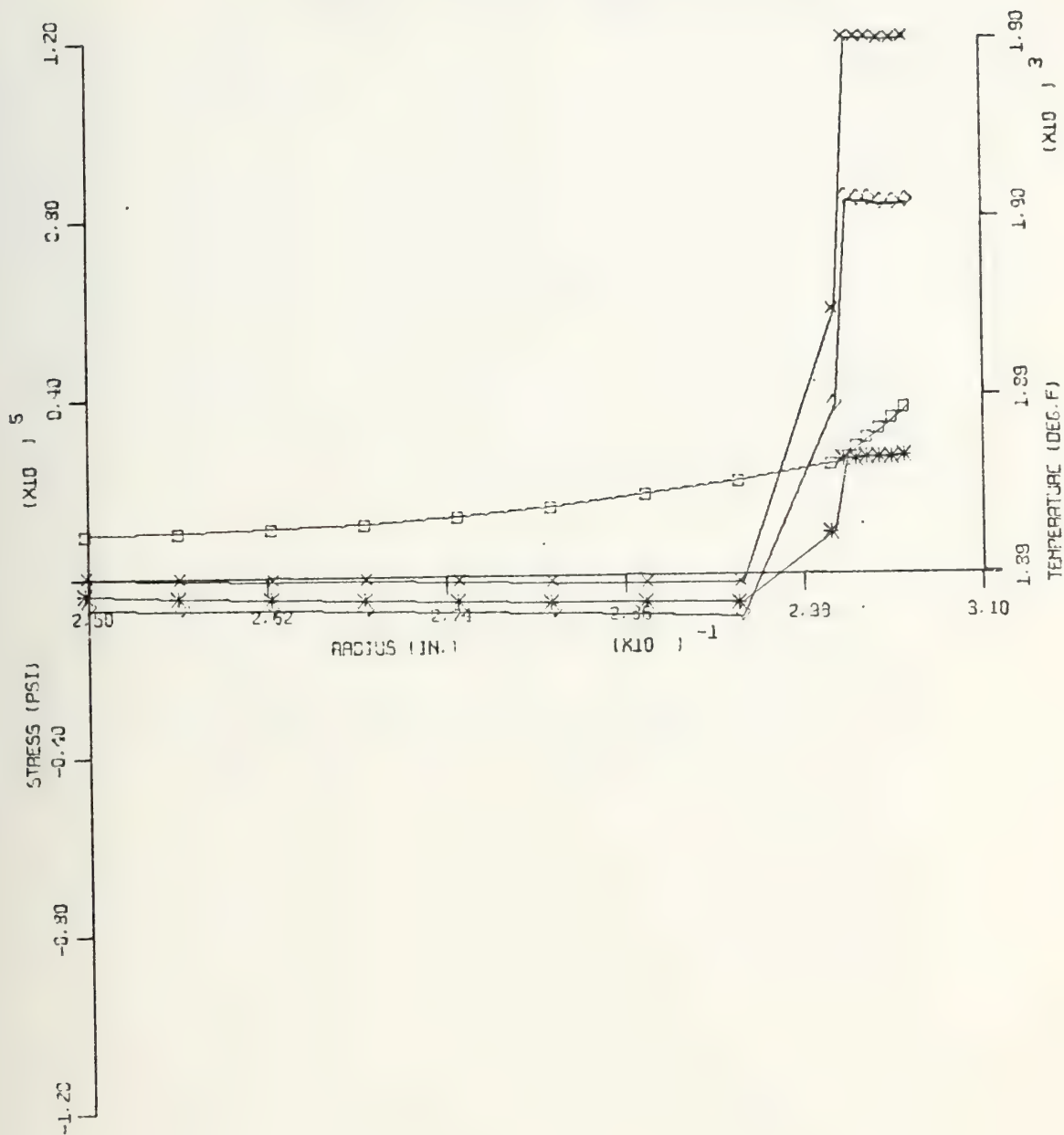
STRESS SCALE = 1.927×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 5(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.1-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

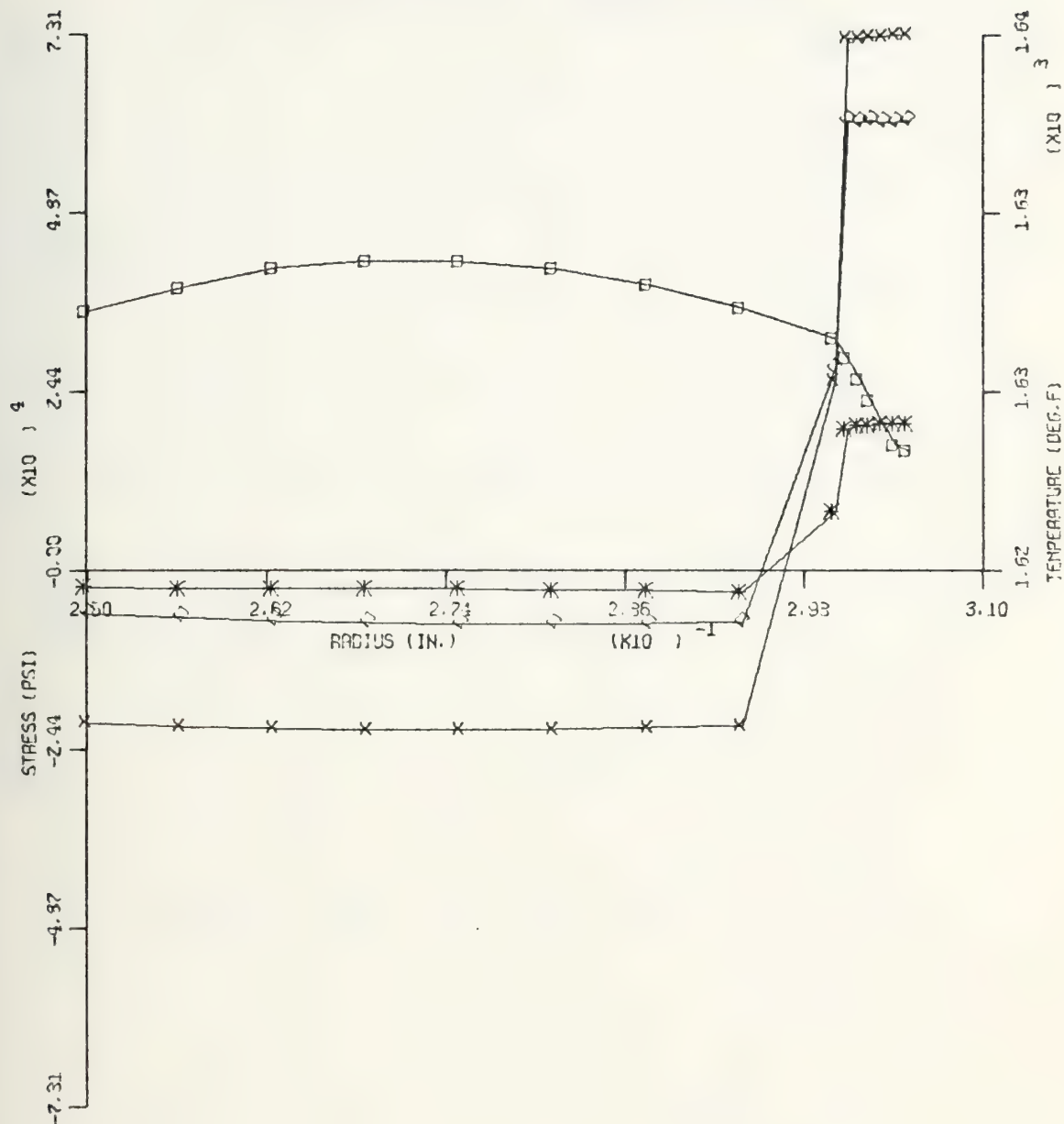
STRESS SCALE = 4.016×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 5(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.1-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

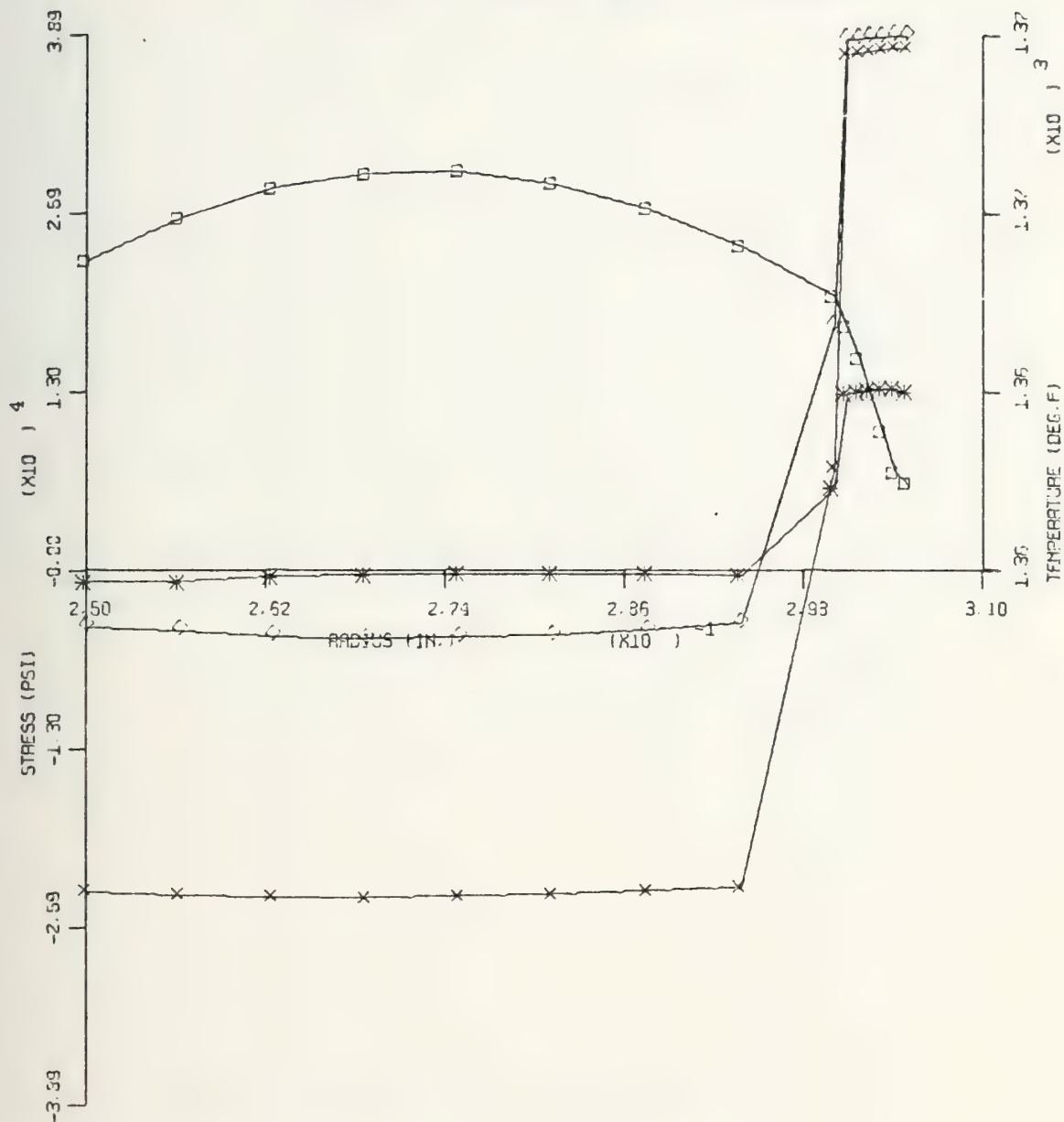
STRESS SCALE = 2.437×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 5(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.1-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

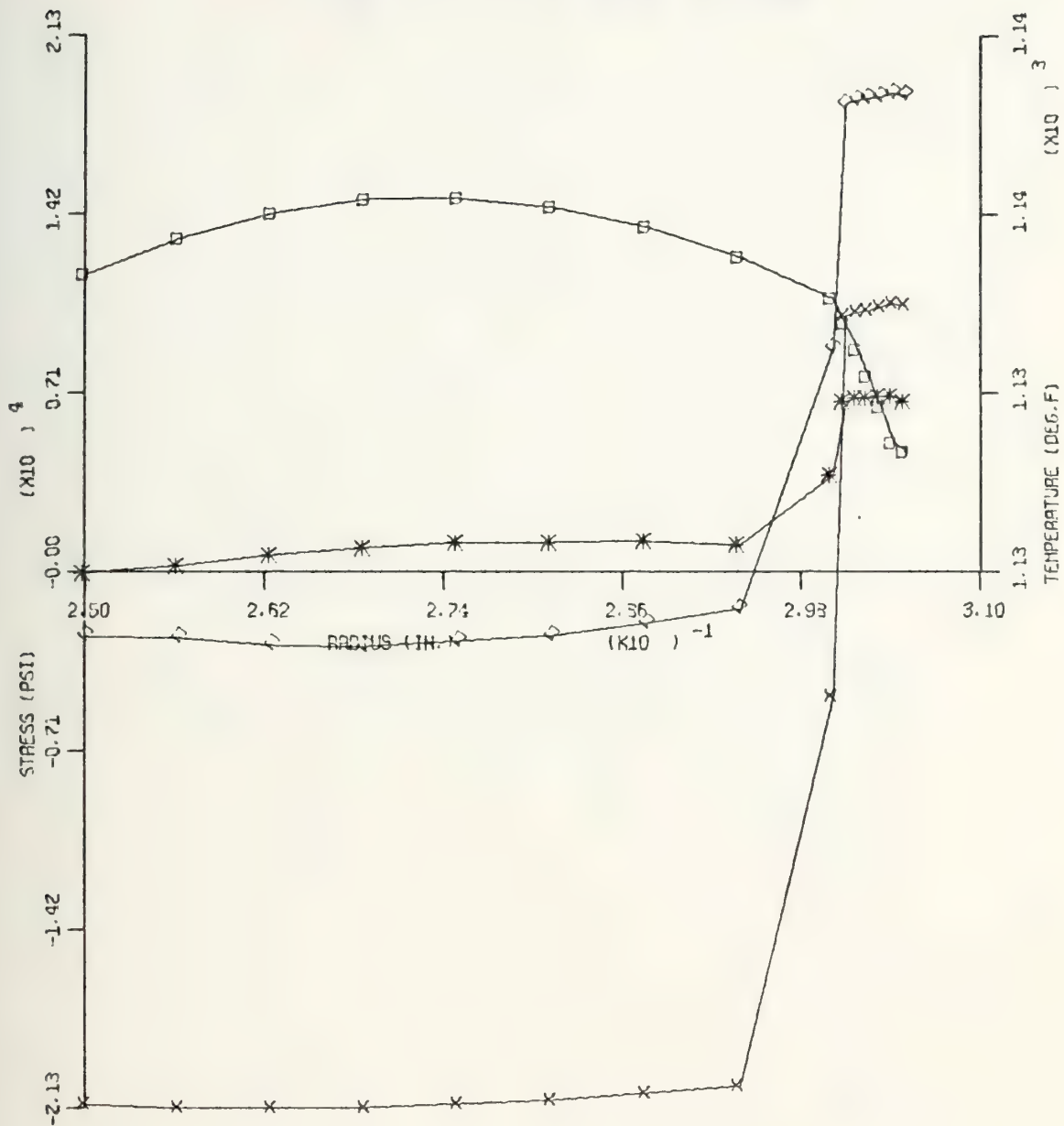
STRESS SCALE = 1.235×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 5 (g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.1-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

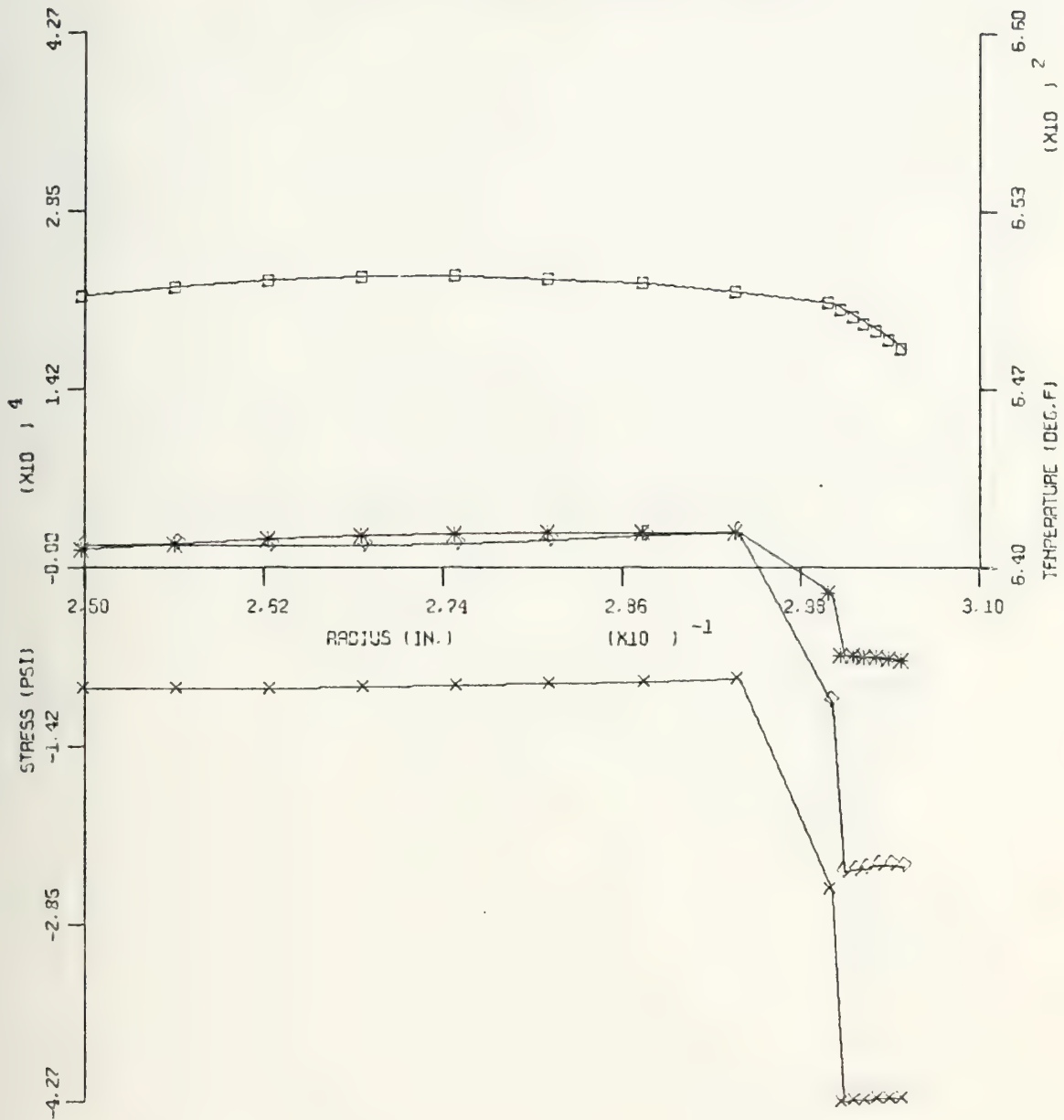
STRESS SCALE = 7.093×10^3 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 5(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.1-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

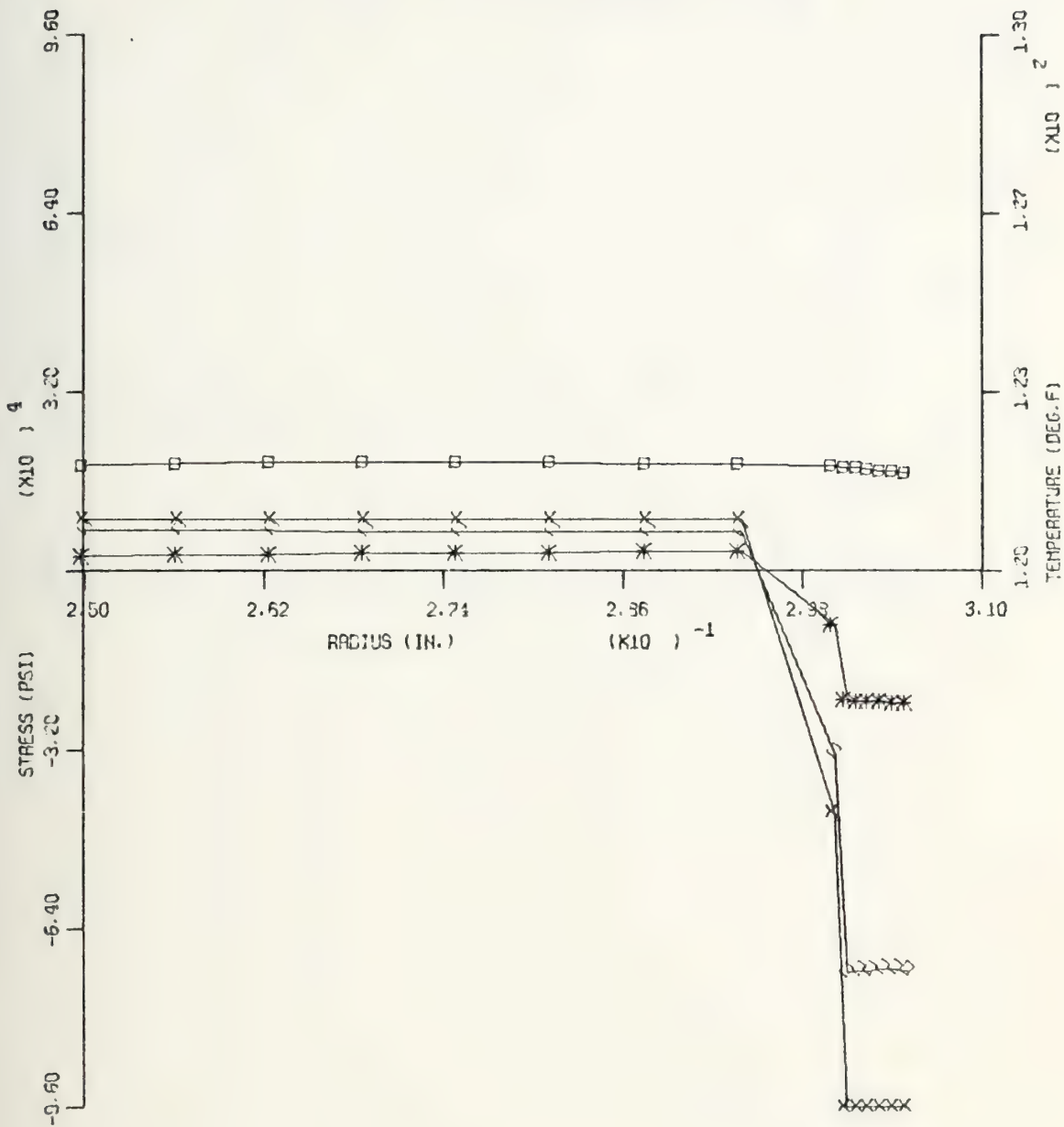
STRESS SCALE = 1.423×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 5(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.1-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

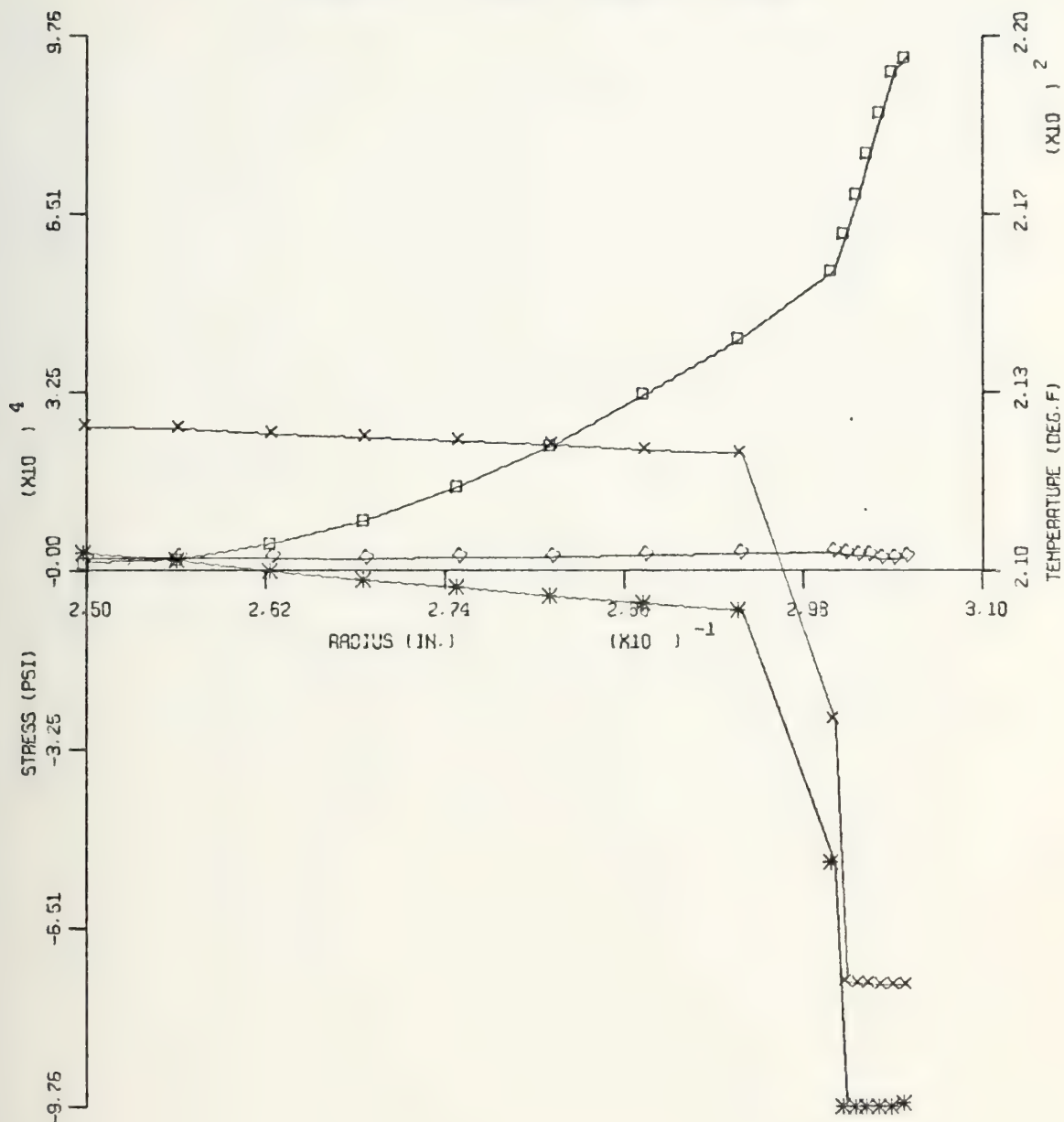
STRESS SCALE = 3.193×10^3 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 5(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.1-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

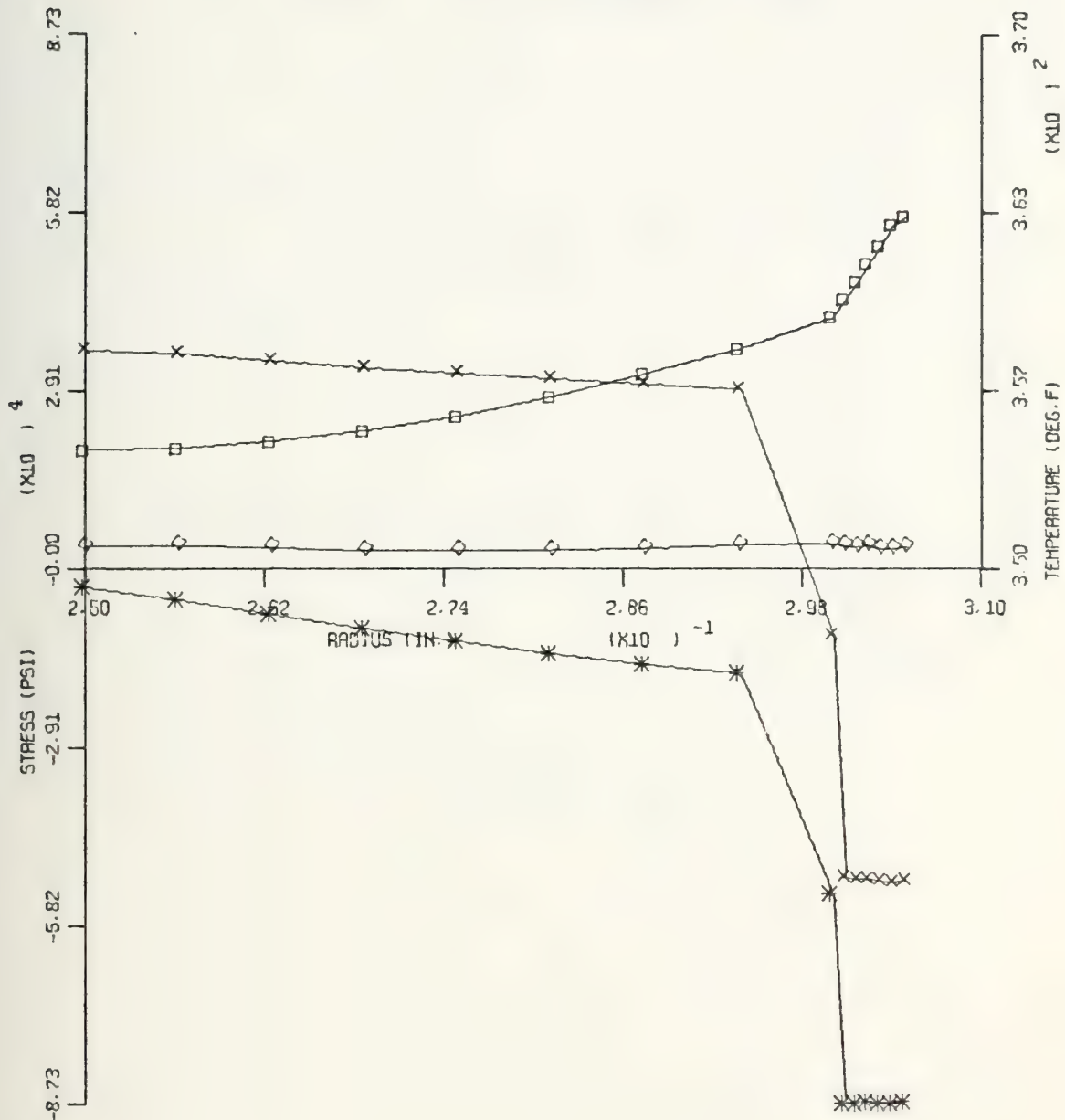
STRESS SCALE = 3.254×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 6(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.1-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

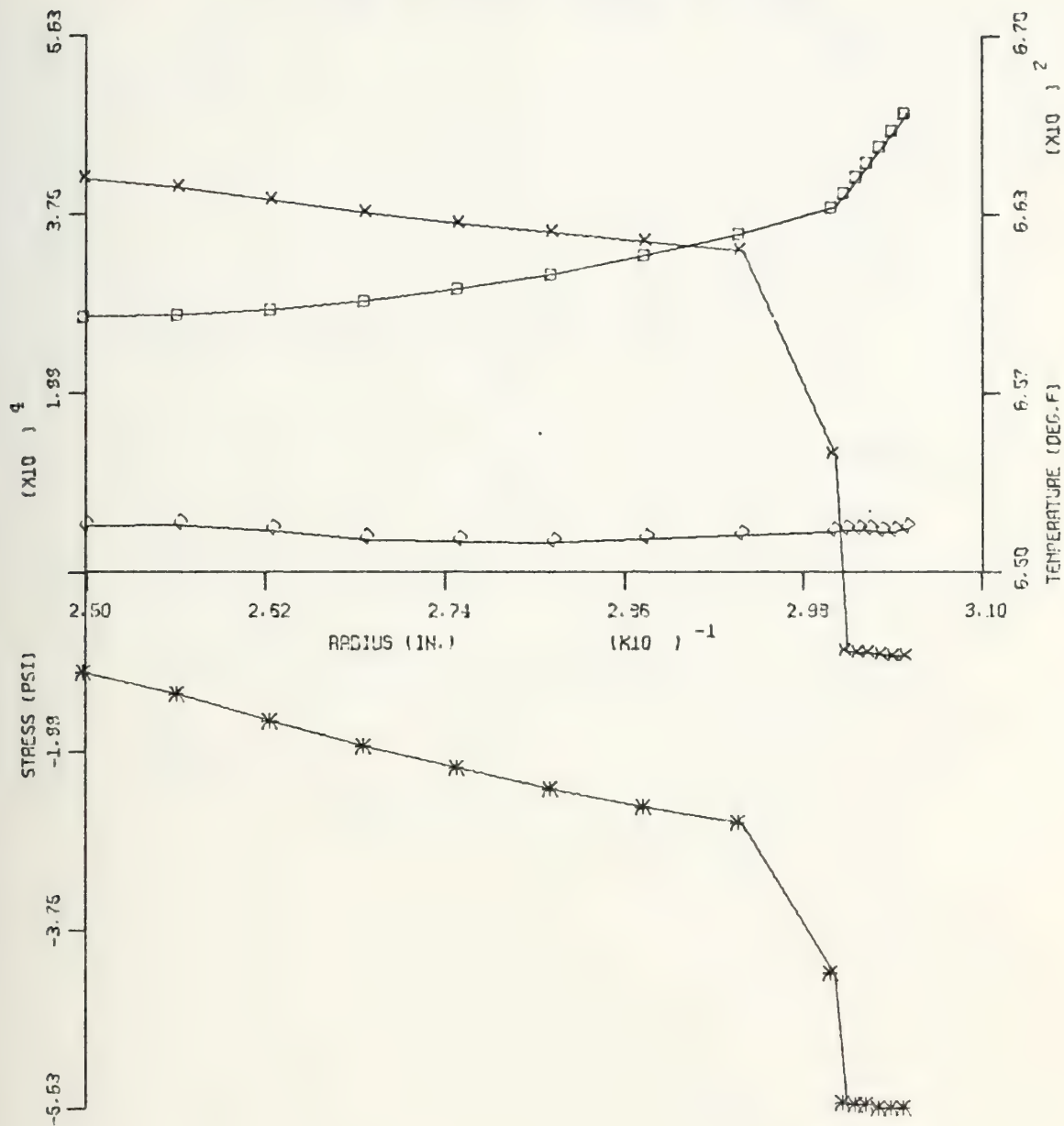
STRESS SCALE = 2.909×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 6(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.1-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

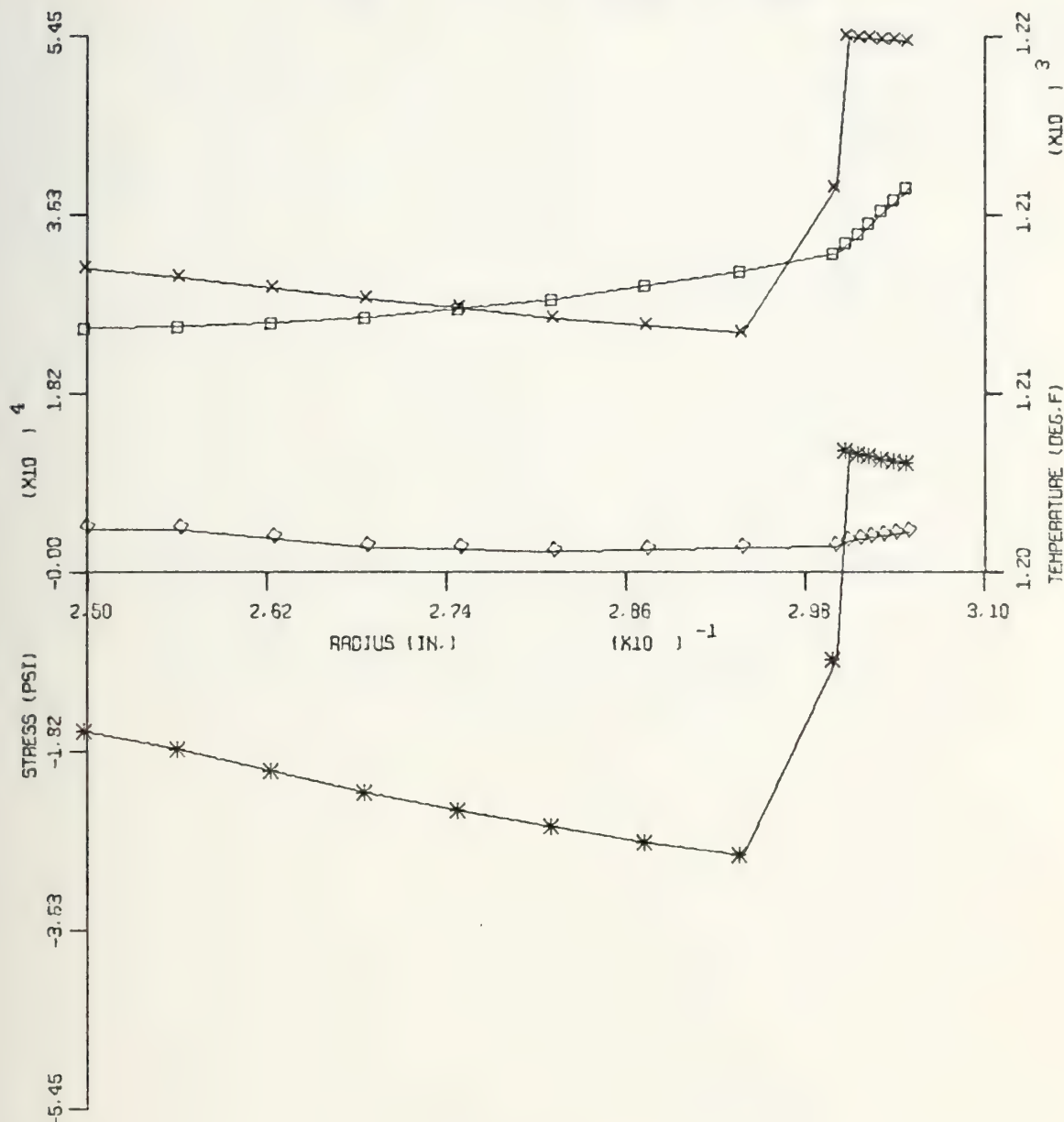
STRESS SCALE = 1.977 X 10⁴ PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 6(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.1-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

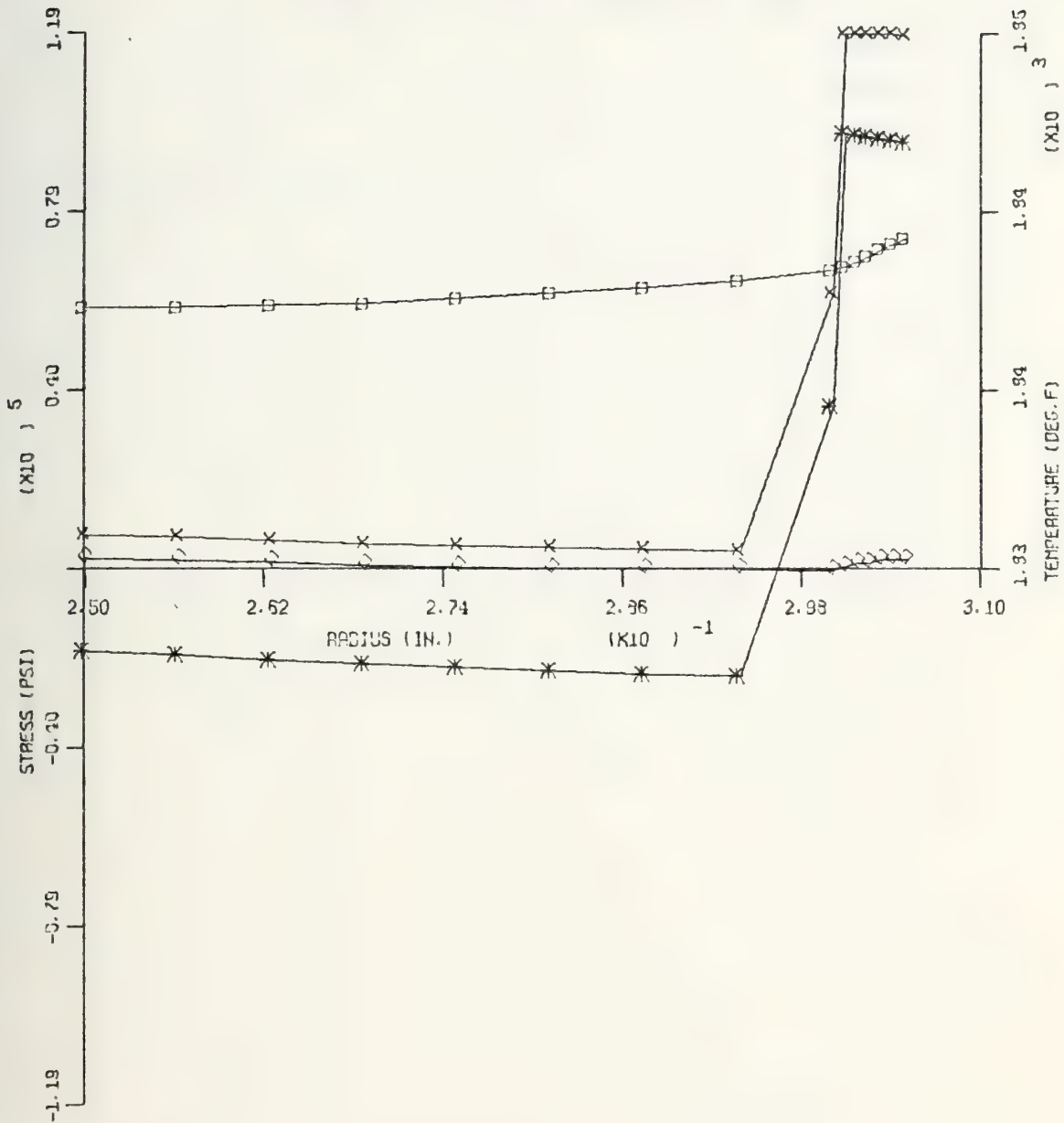
STRESS SCALE = 1.815×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 6(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.1-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

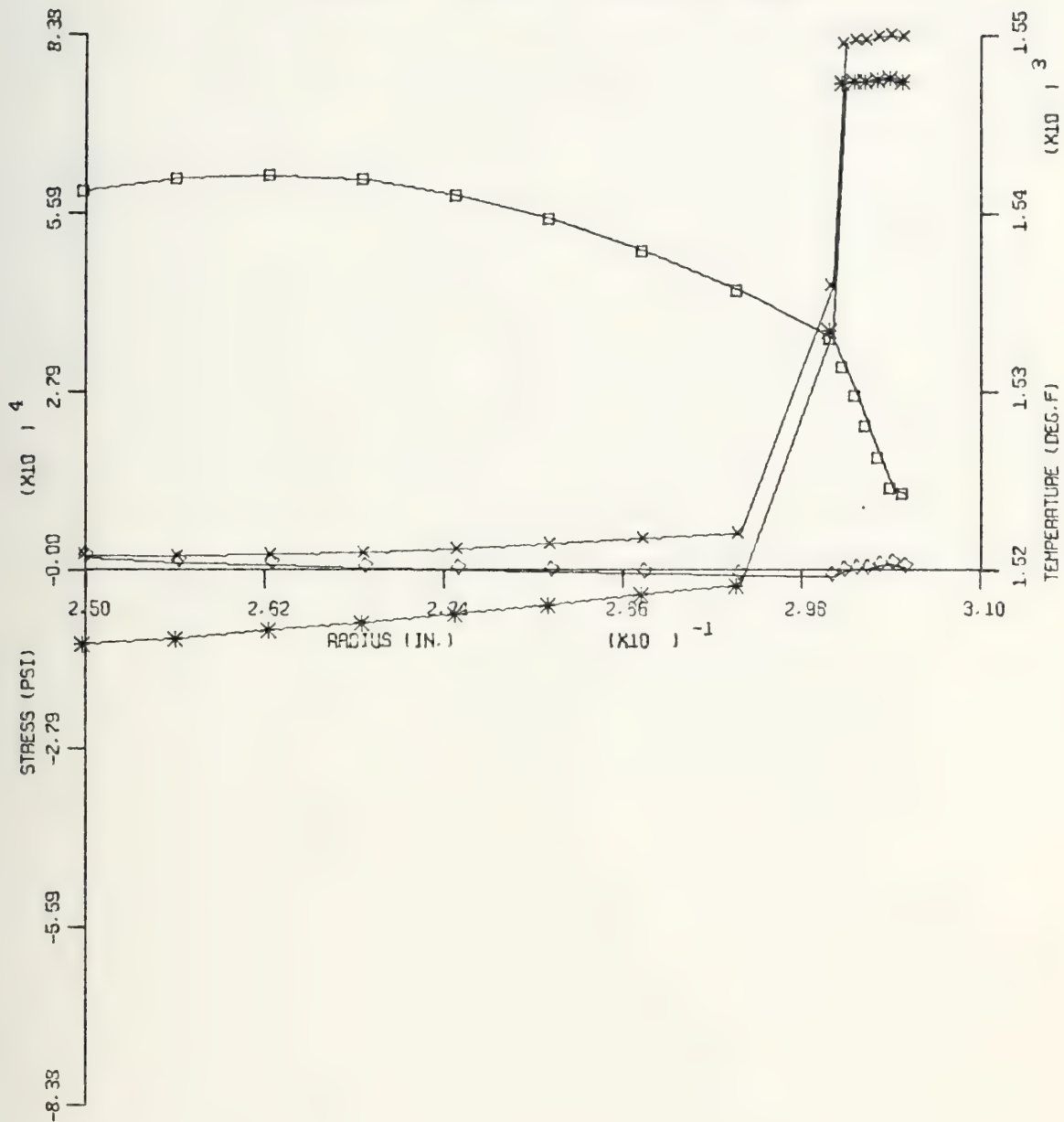
STRESS SCALE = 3.957×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 6(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.1-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

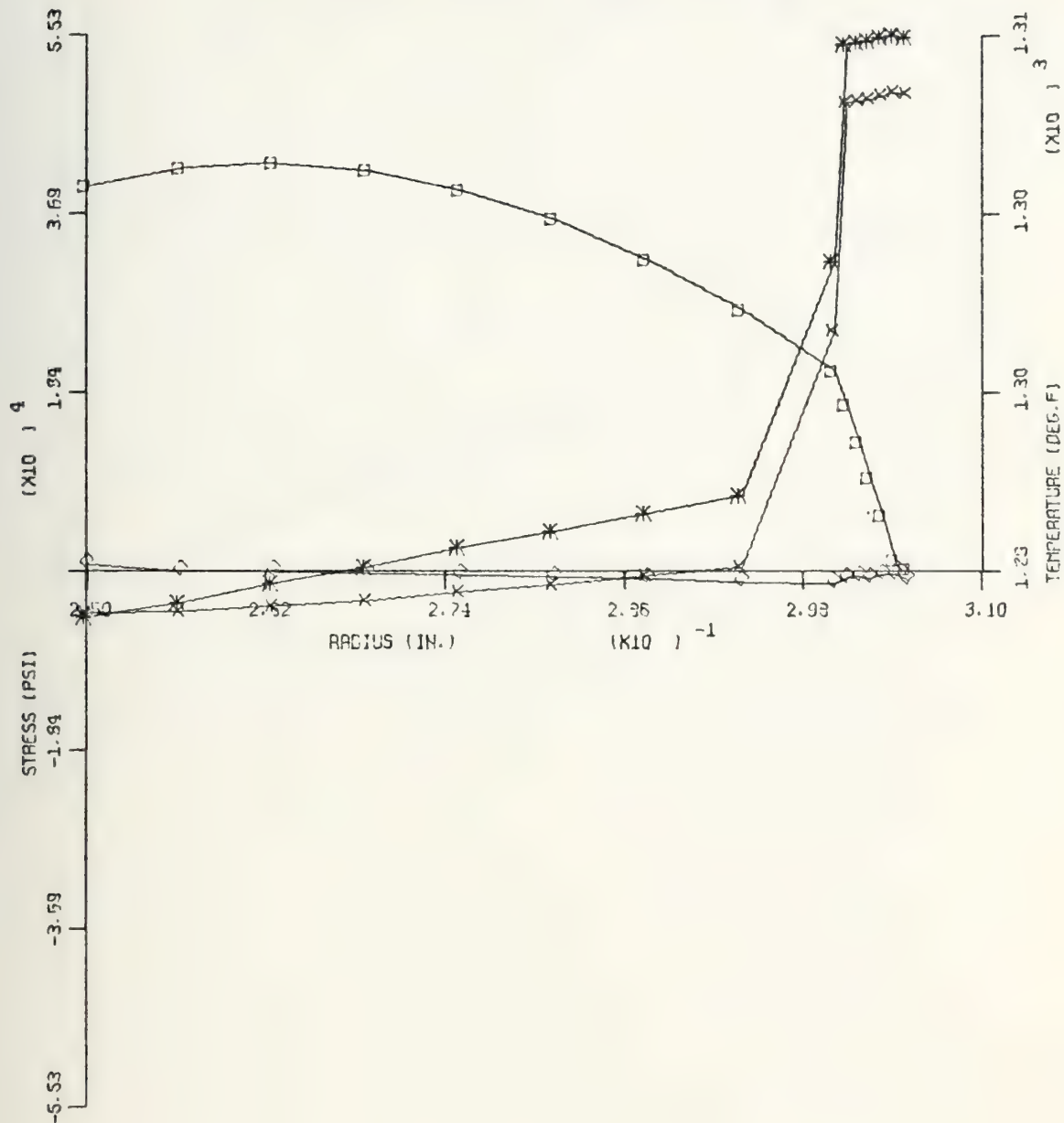
STRESS SCALE = 2.733×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 6(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.1-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

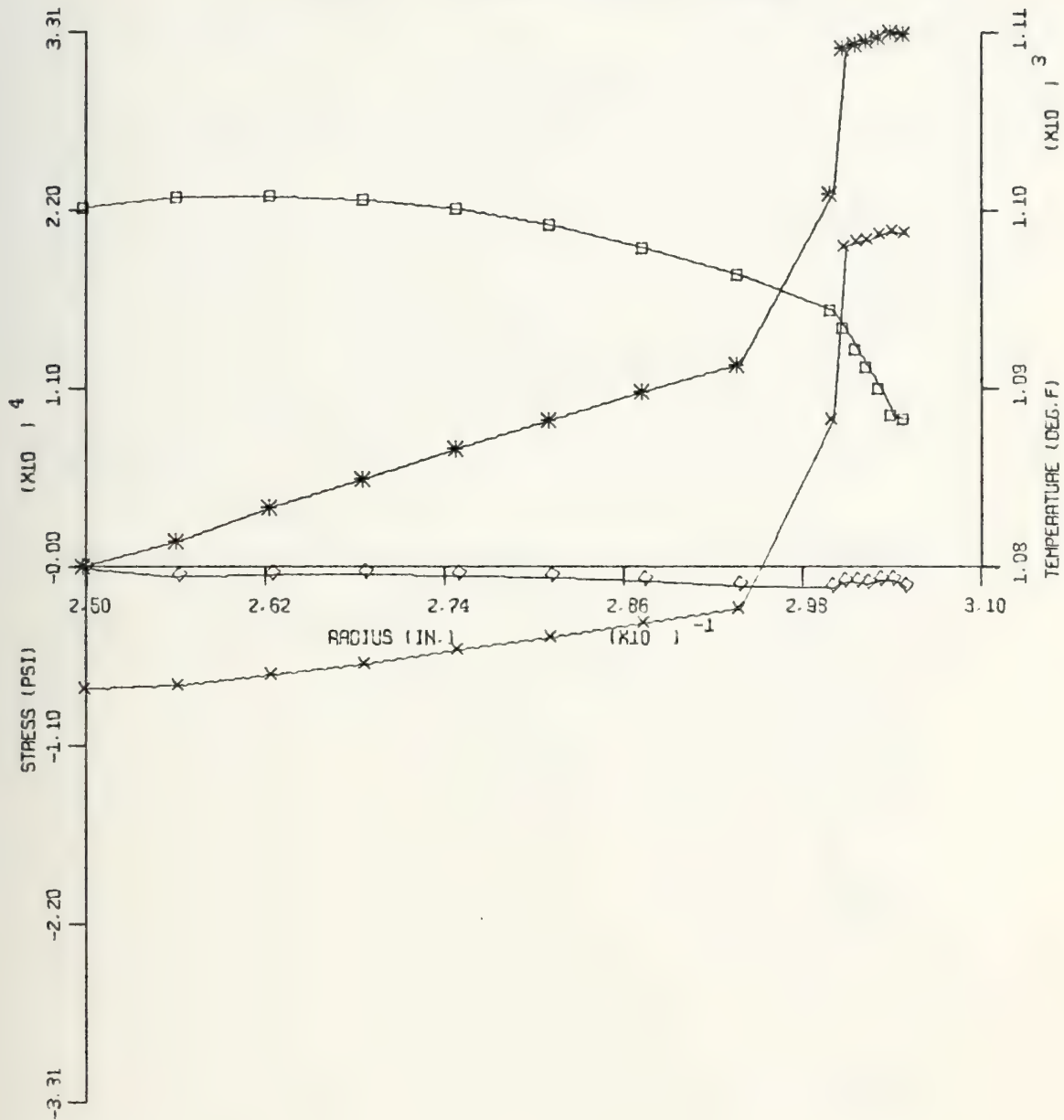
STRESS SCALE = 1.843×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 6 (g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.1-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

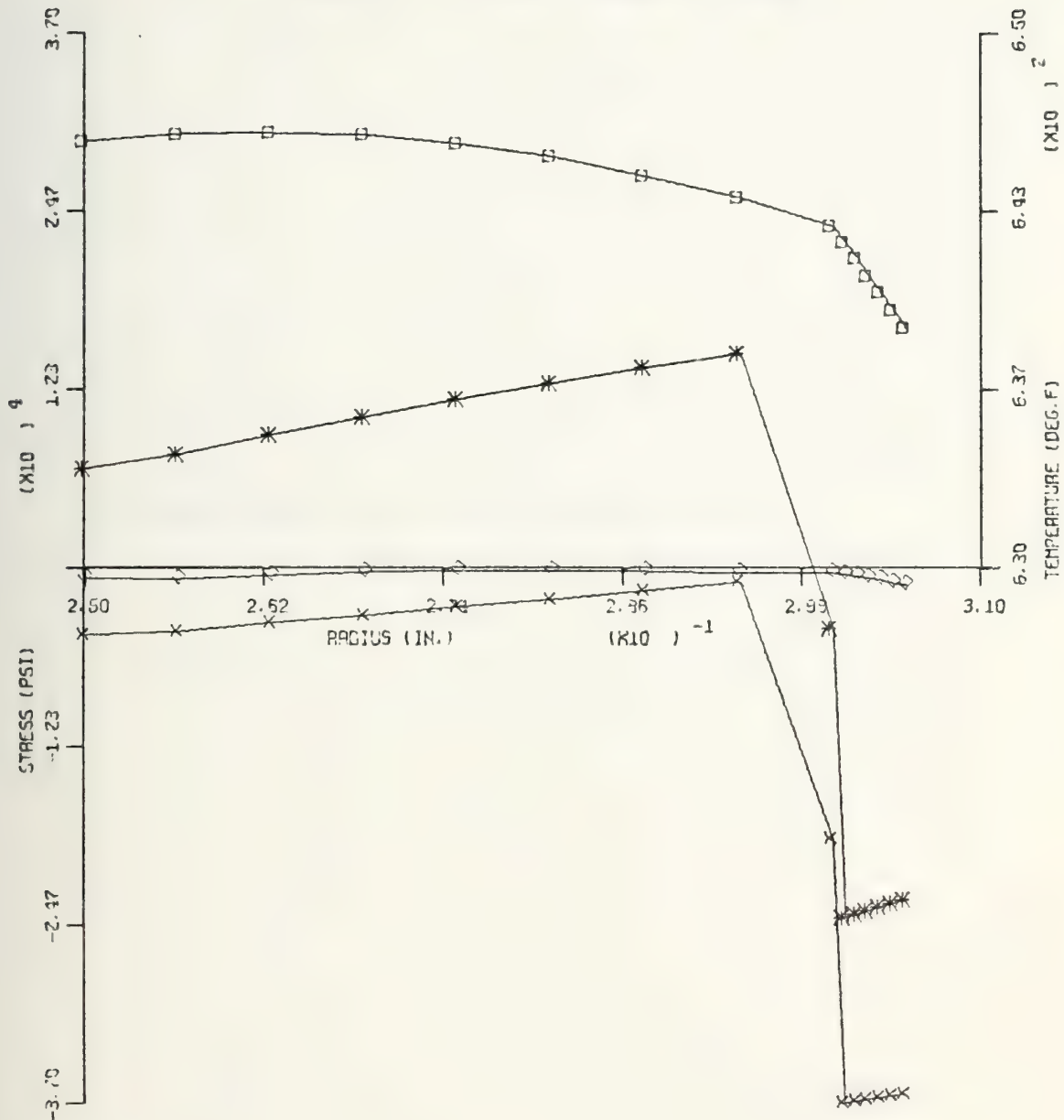
STRESS SCALE = 1.102×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 6(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.1-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

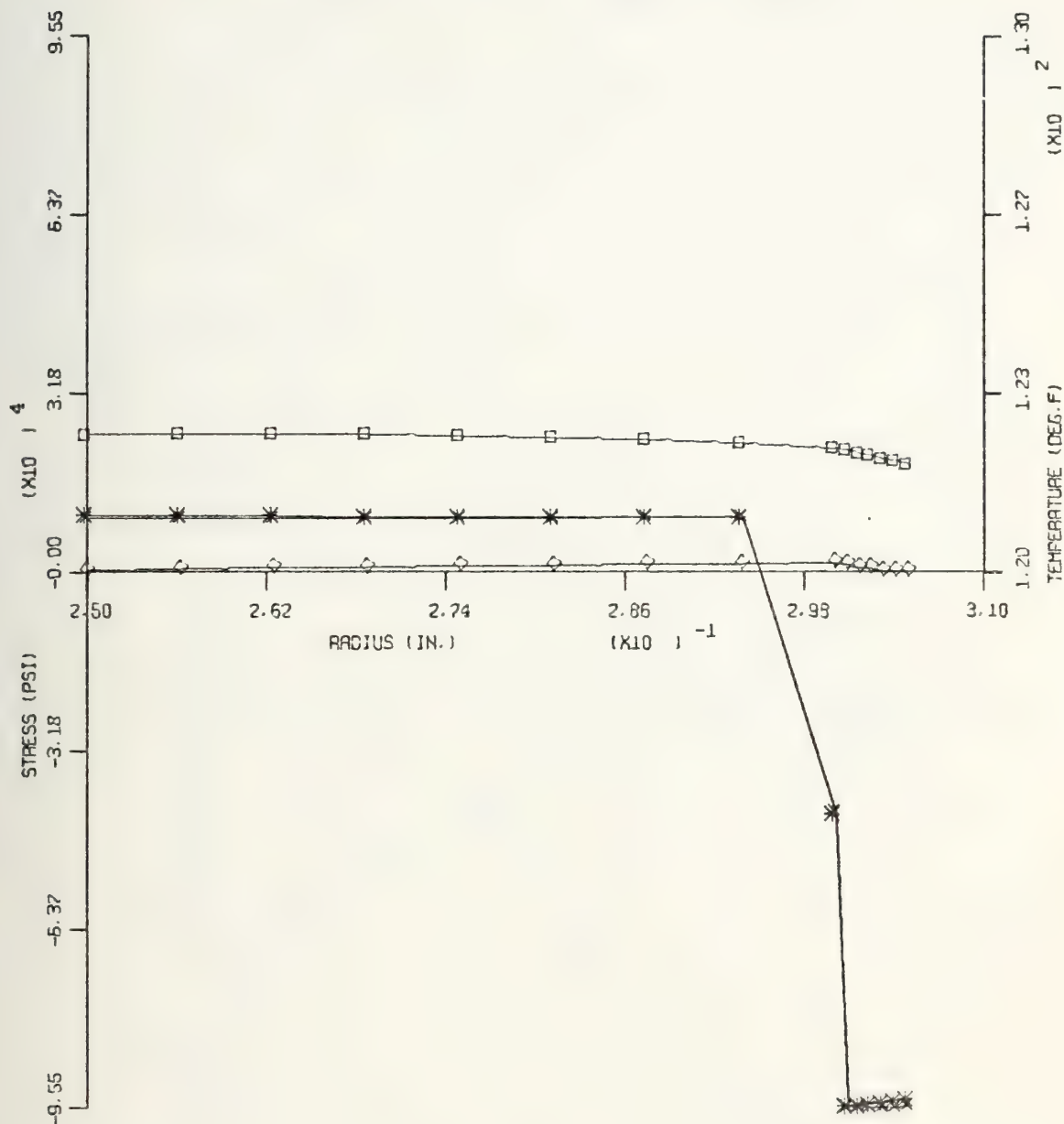
STRESS SCALE = 1.234×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 6(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.1-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 3.184×10^9 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 6(j)

MAXIMUM AND MINIMUM PRINCIPAL STRESSES
VERSUS TIME
CONFIGURATION NUMBER ONE

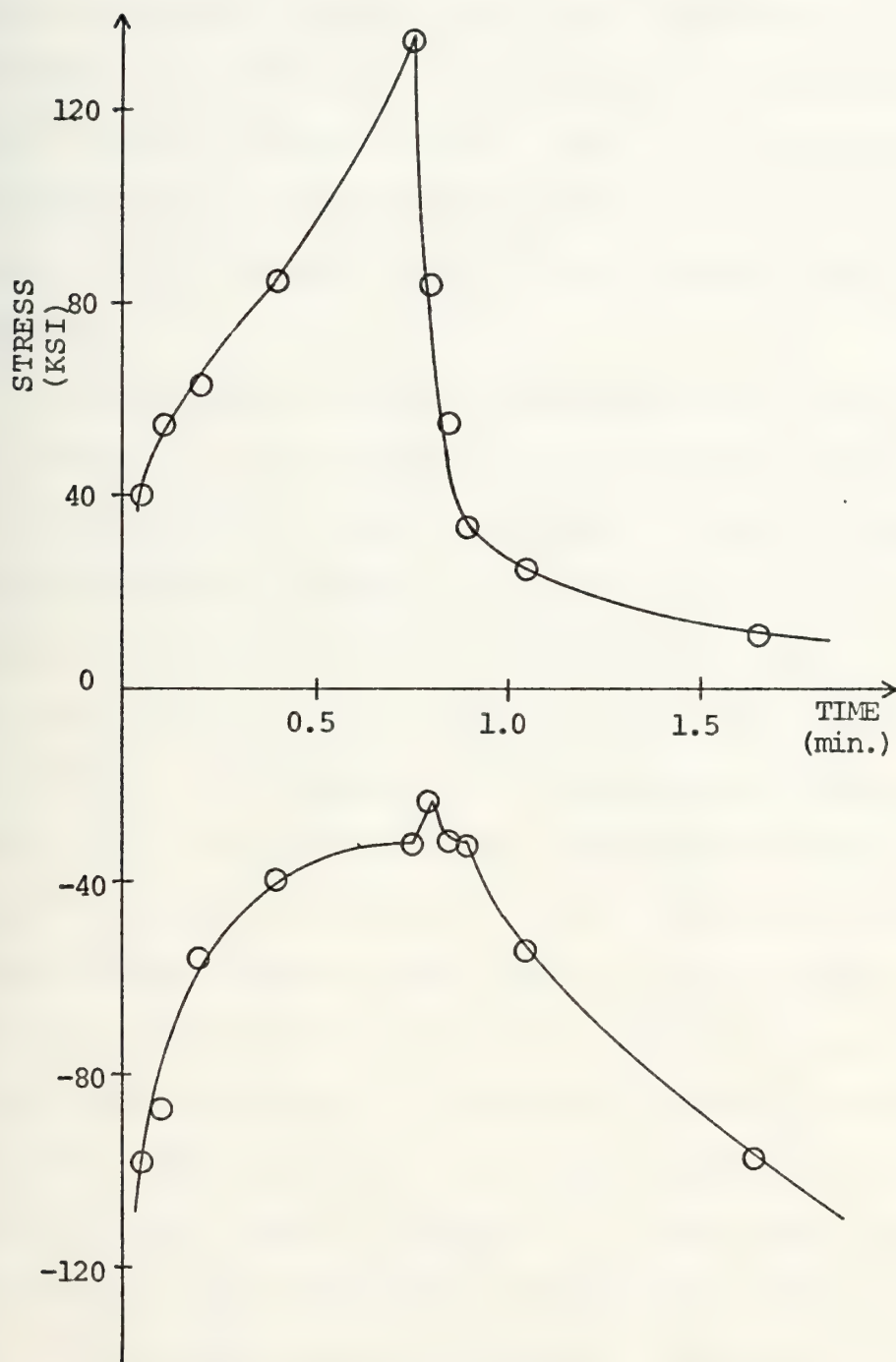


FIGURE 7

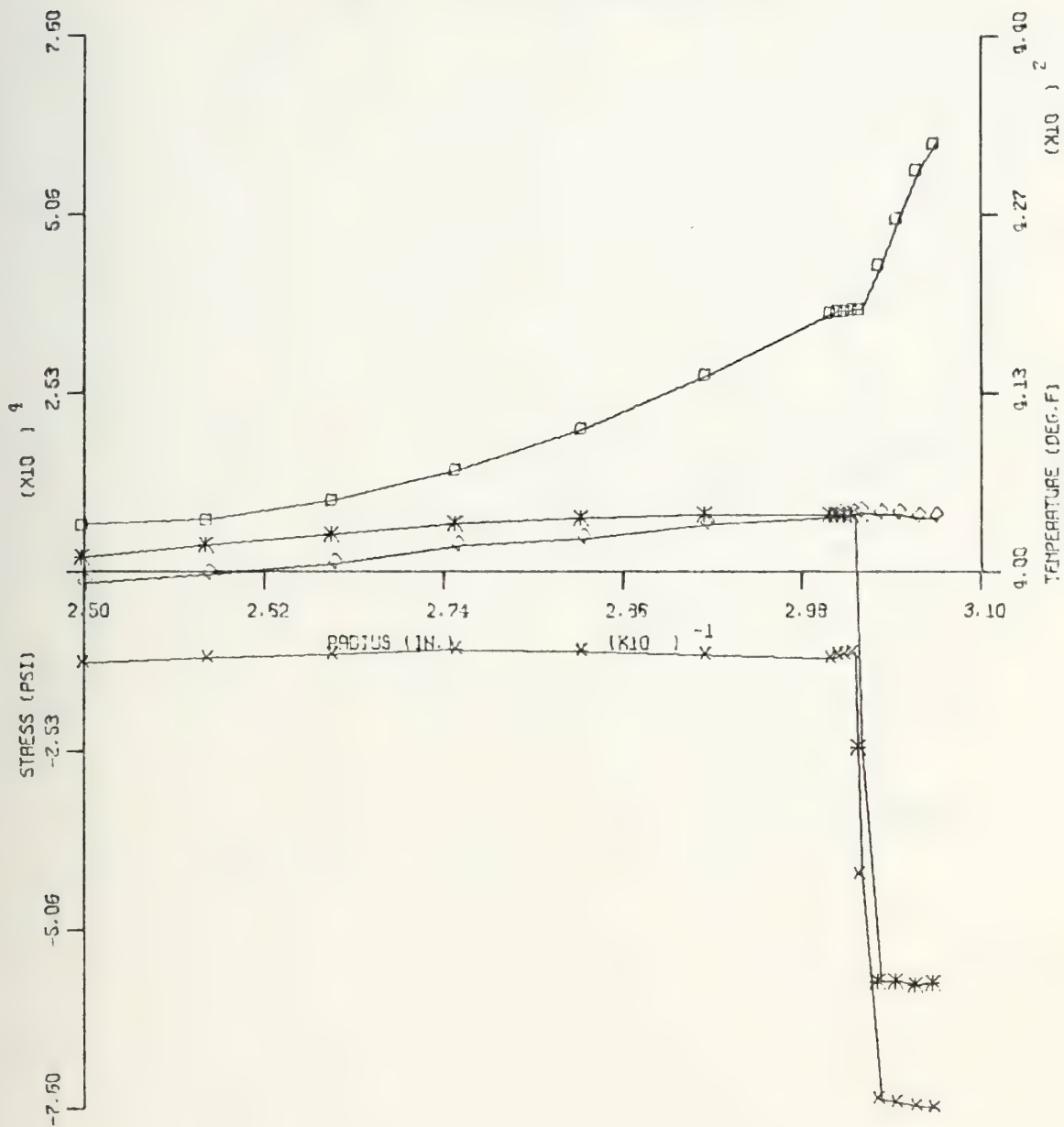
B. CONFIGURATION NUMBER TWO

This configuration contained a layer of pure nickel between the substratum and the layer of zirconia. The maximum principal stress (134.9 KSI) occurred at the maximum temperature condition and was located at a radius of 0.3045 inches, the zero angle position, and at the midpoint of the cylinder length. As with the other configurations, the larger tensile stresses were in the tangential and axial directions.

The comparison of stress and temperature distribution as a function of angle position is shown in Figure 12. The axial stress level is nearly constant as the circumferential position changes, however, the radial and circumferential stresses change at about 90 - 120 degrees. This change in the stress level coincides with a change in the temperature distribution. The change in the temperature distribution results from the changes in the temperature boundary conditions in the down stream portion of the air flow past the cylinder. Figure 12 was prepared using data for the cylinder ends at a radius of 0.3045 inches and a time of 0.75 minutes.

The time dependence of the stress levels is shown in Figure 13. The tangential and axial stresses for a position on the cylinder end at a radius of 0.3045 inches and an angle of 60° vary in the same manner as the temperature at that point. The radial stress is minimal throughout the time period.

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.2-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

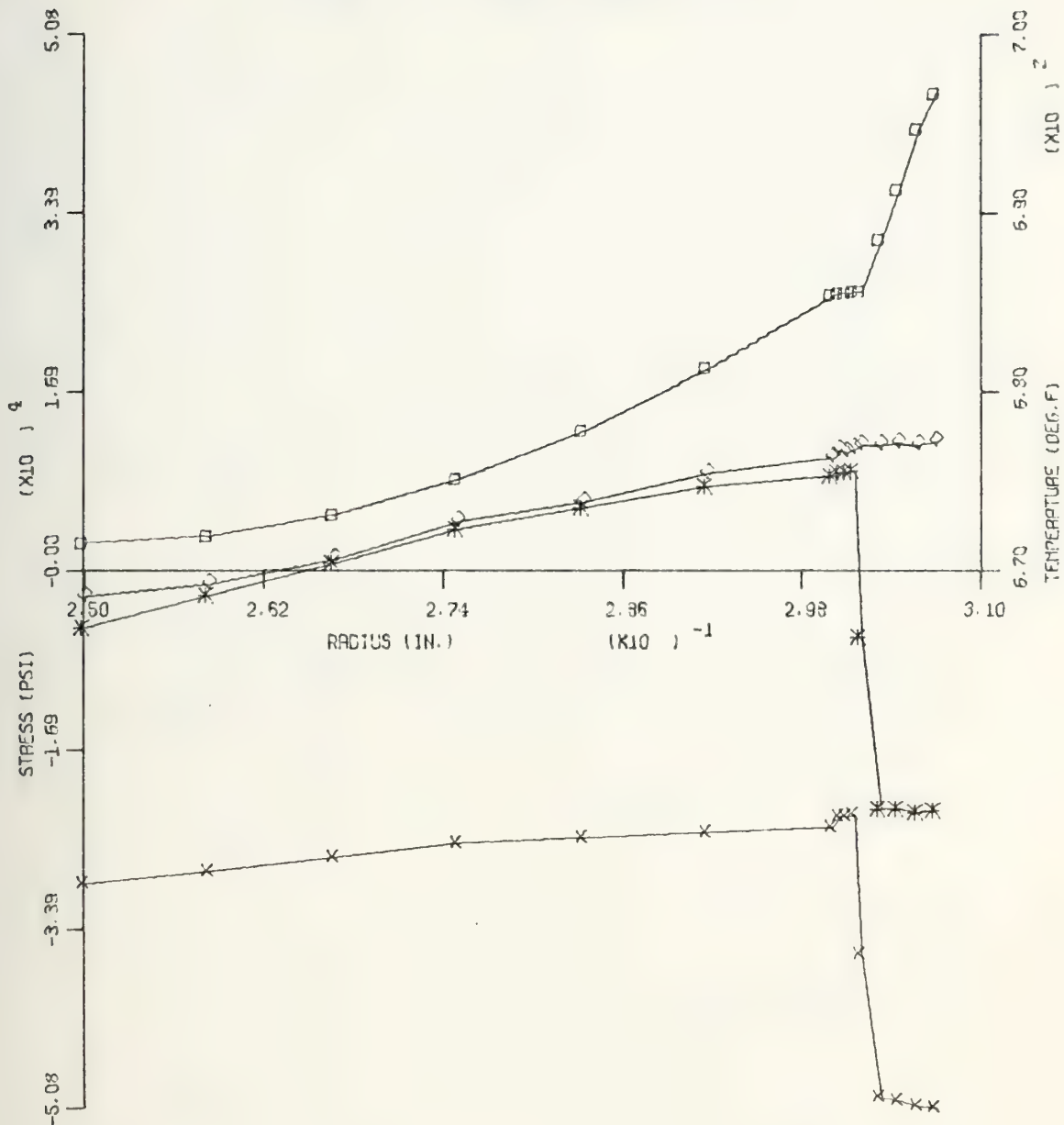
STRESS SCALE = 2.532×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 8(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.2-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

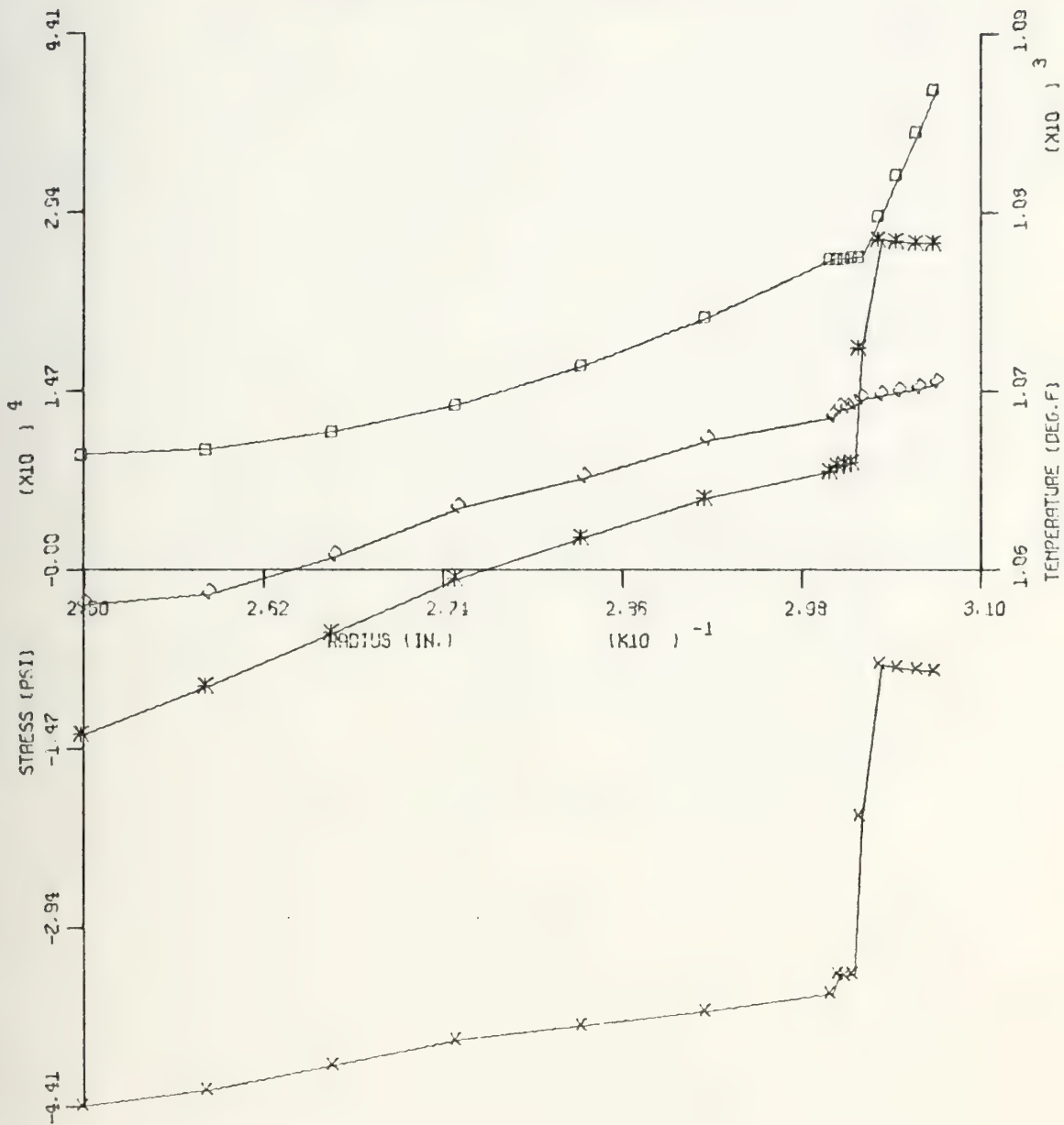
STRESS SCALE = 1.593×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 8(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.2-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

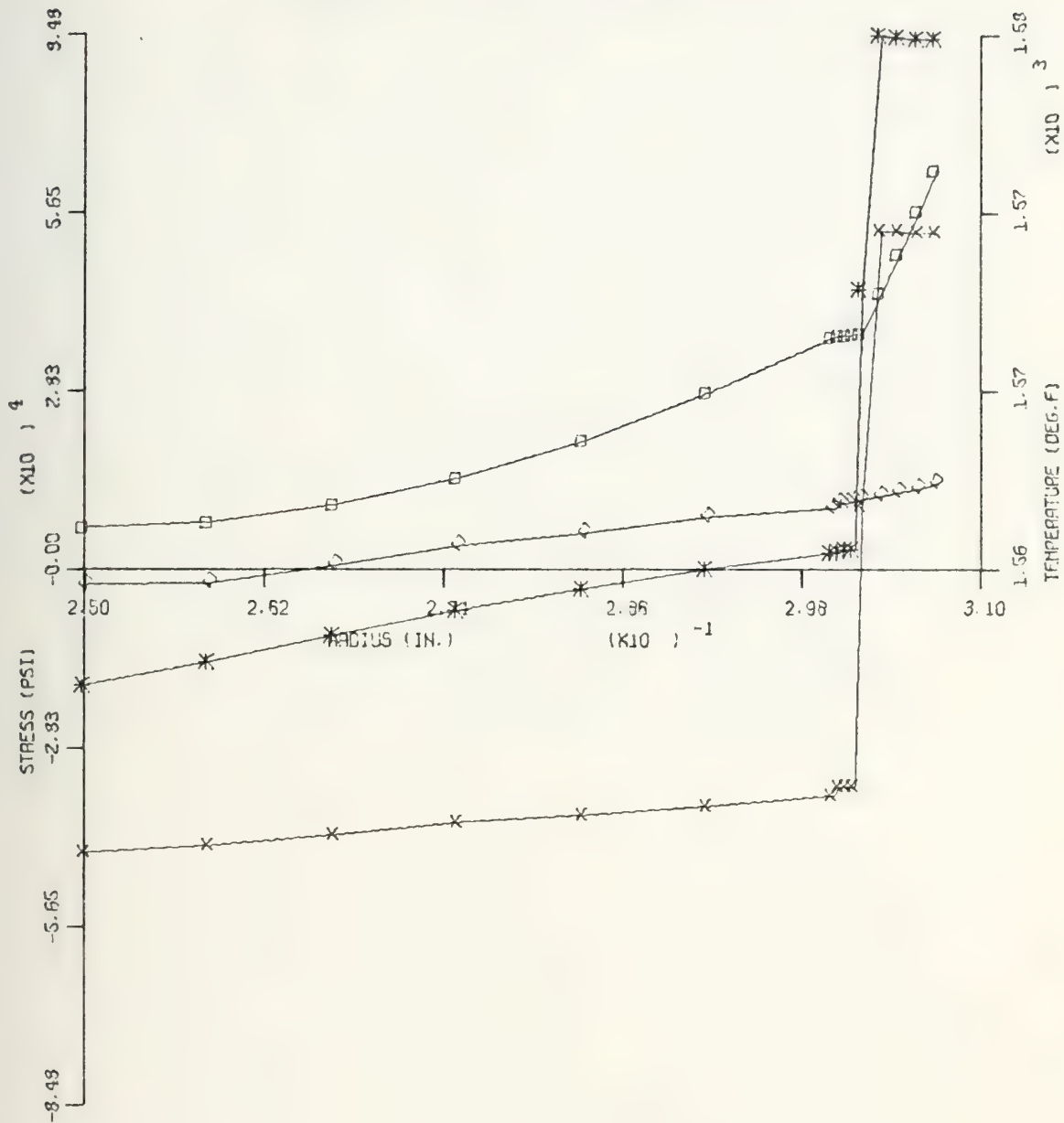
STRESS SCALE = 1.458×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 8(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.2-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

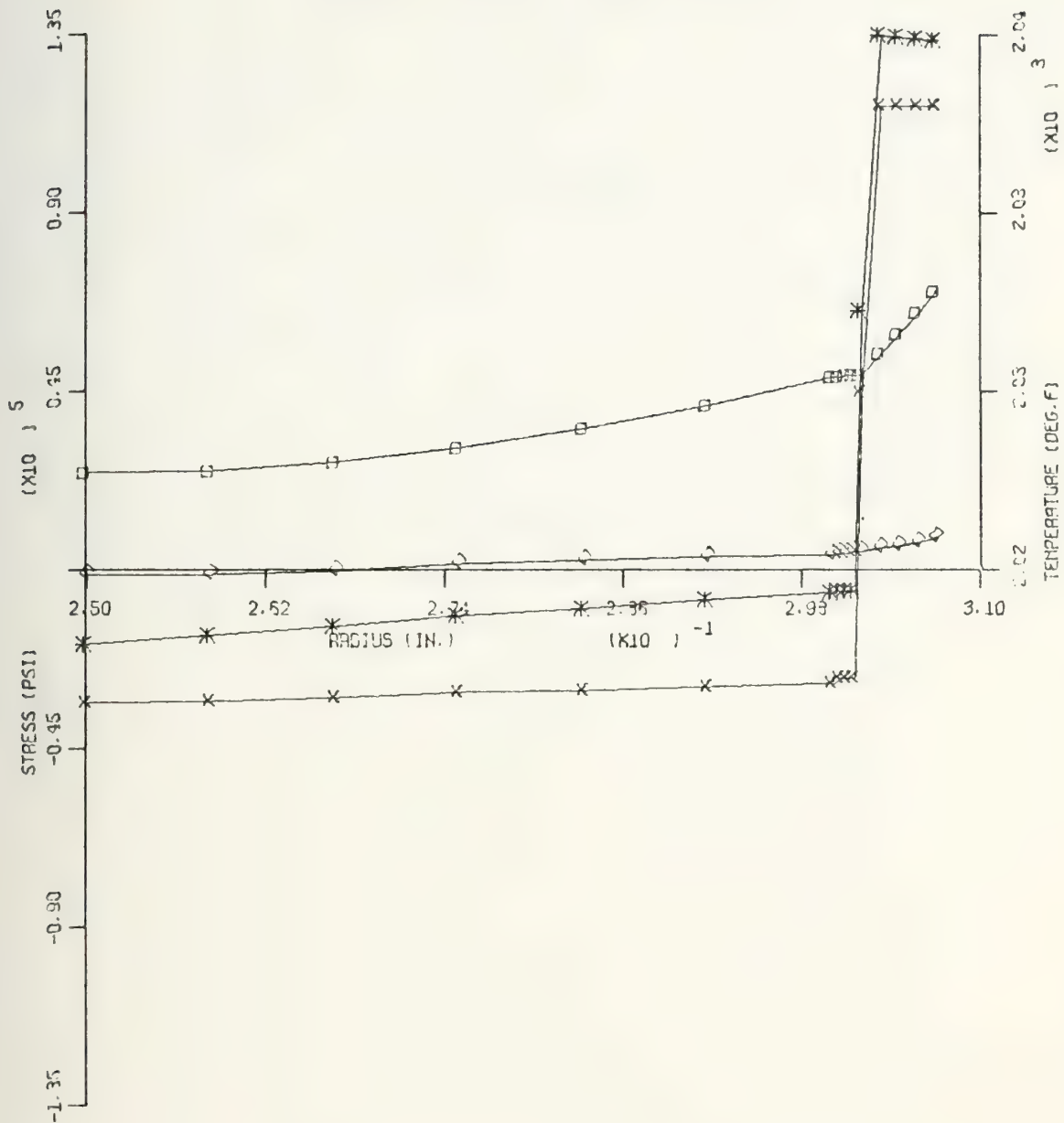
STRESS SCALE = 2.825×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 8(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF. 2-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

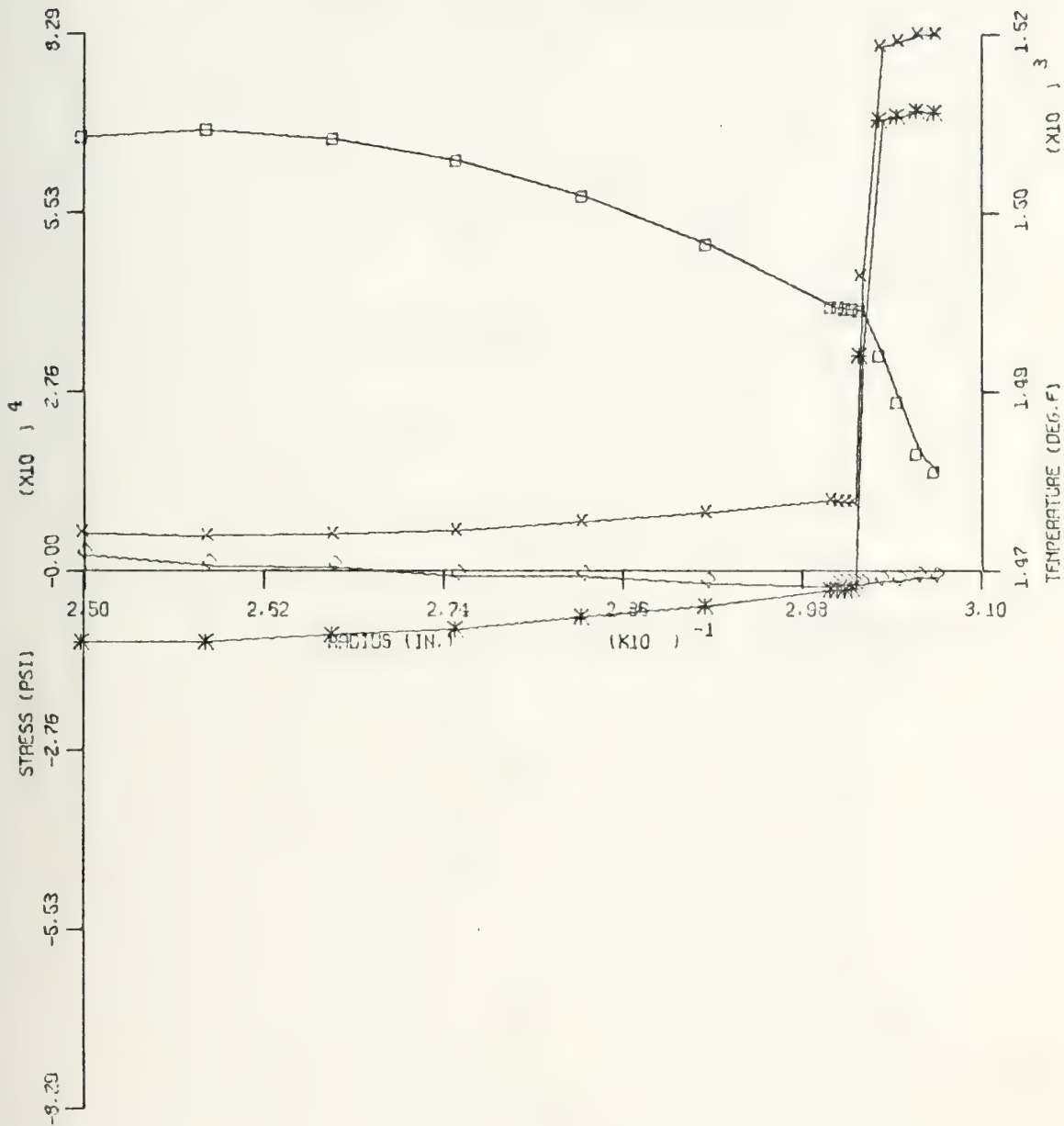
STRESS SCALE = 4.512×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 8(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.2-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

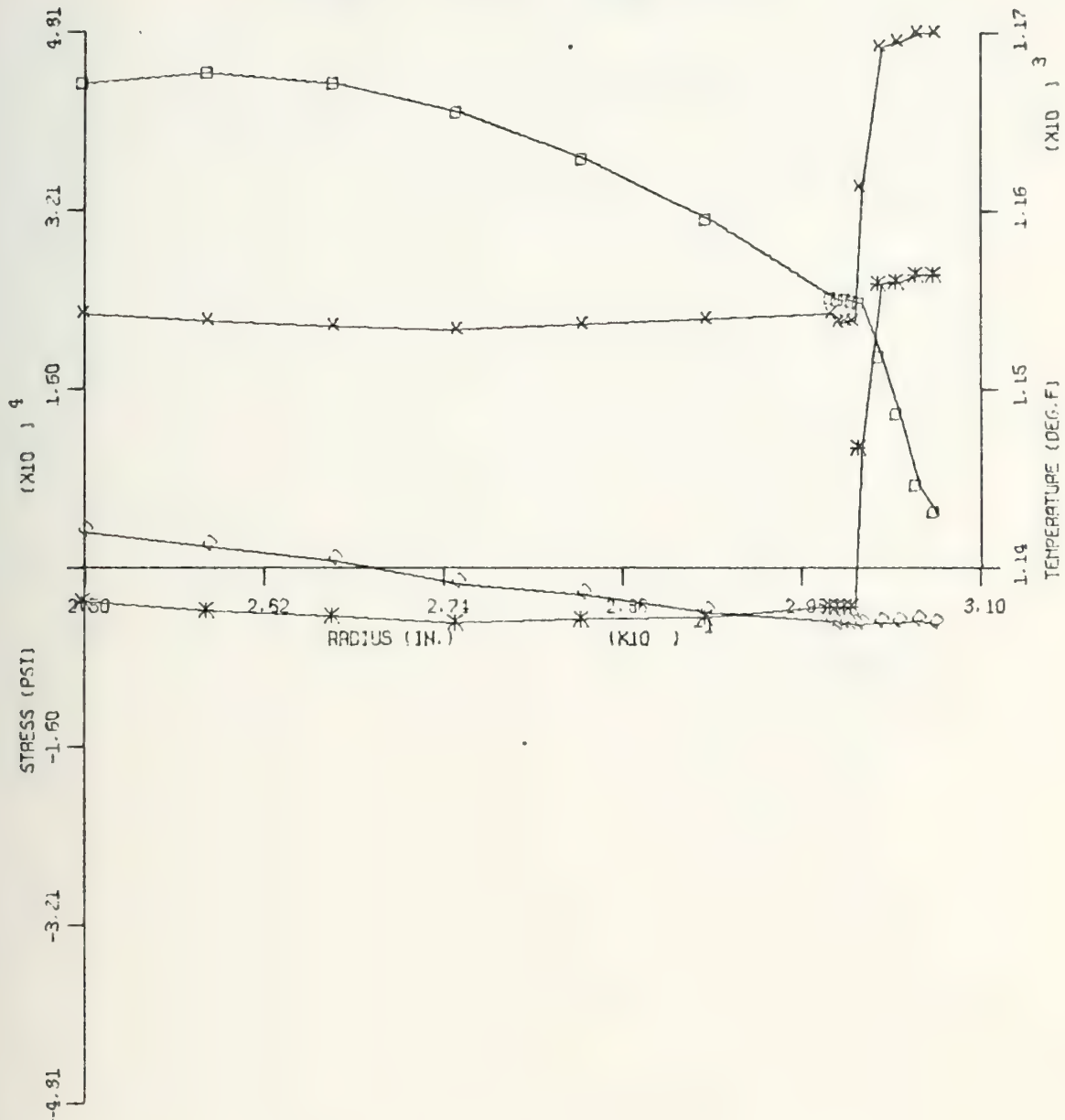
STRESS SCALE = 2.754×10^4 PSI/INCH

TEMPERATURE SCALE = 16.2 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 8(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.2-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

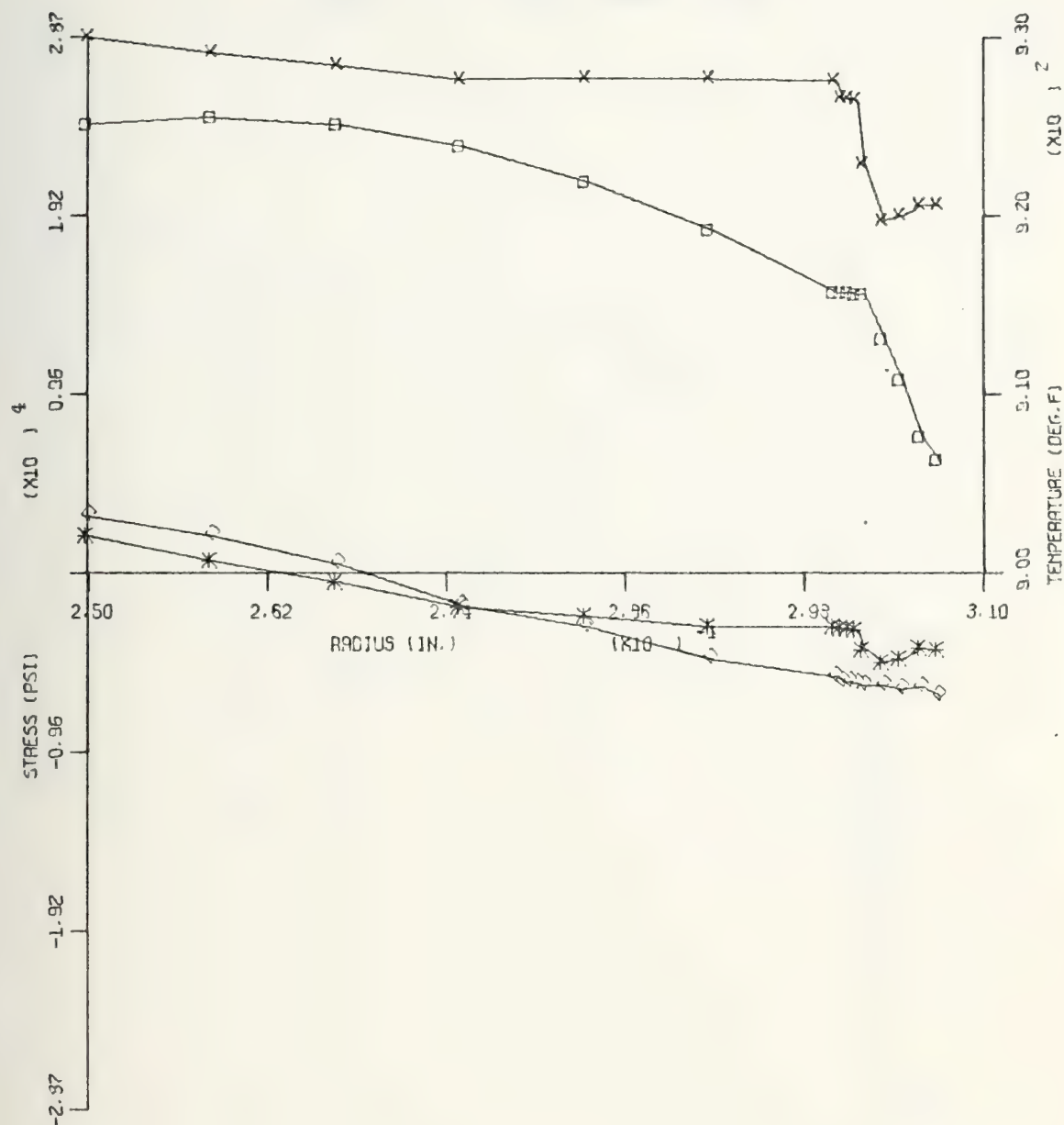
STRESS SCALE = 1.504×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 8(g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.2-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

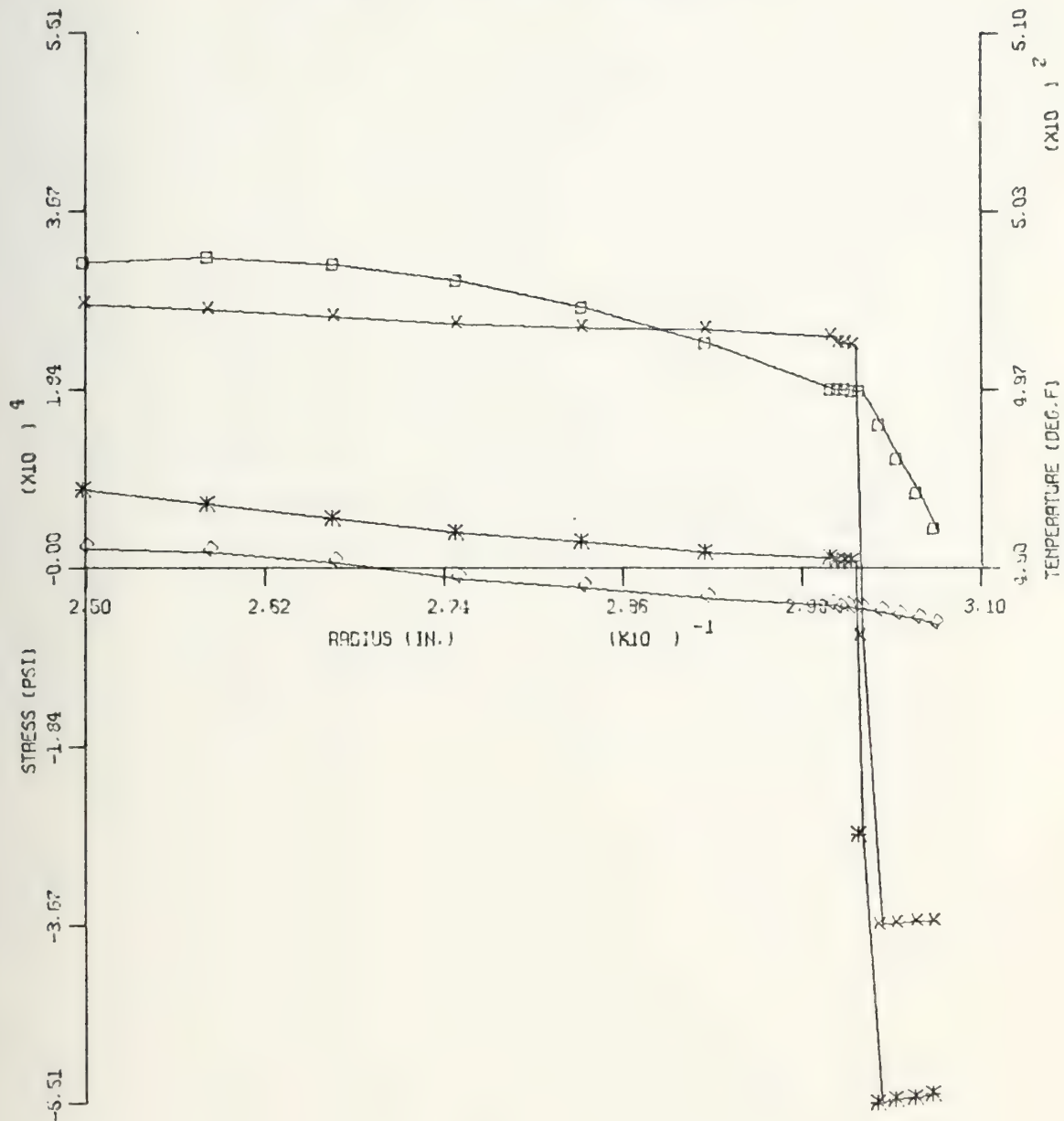
STRESS SCALE = 8.580×10^3 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 8(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.2-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

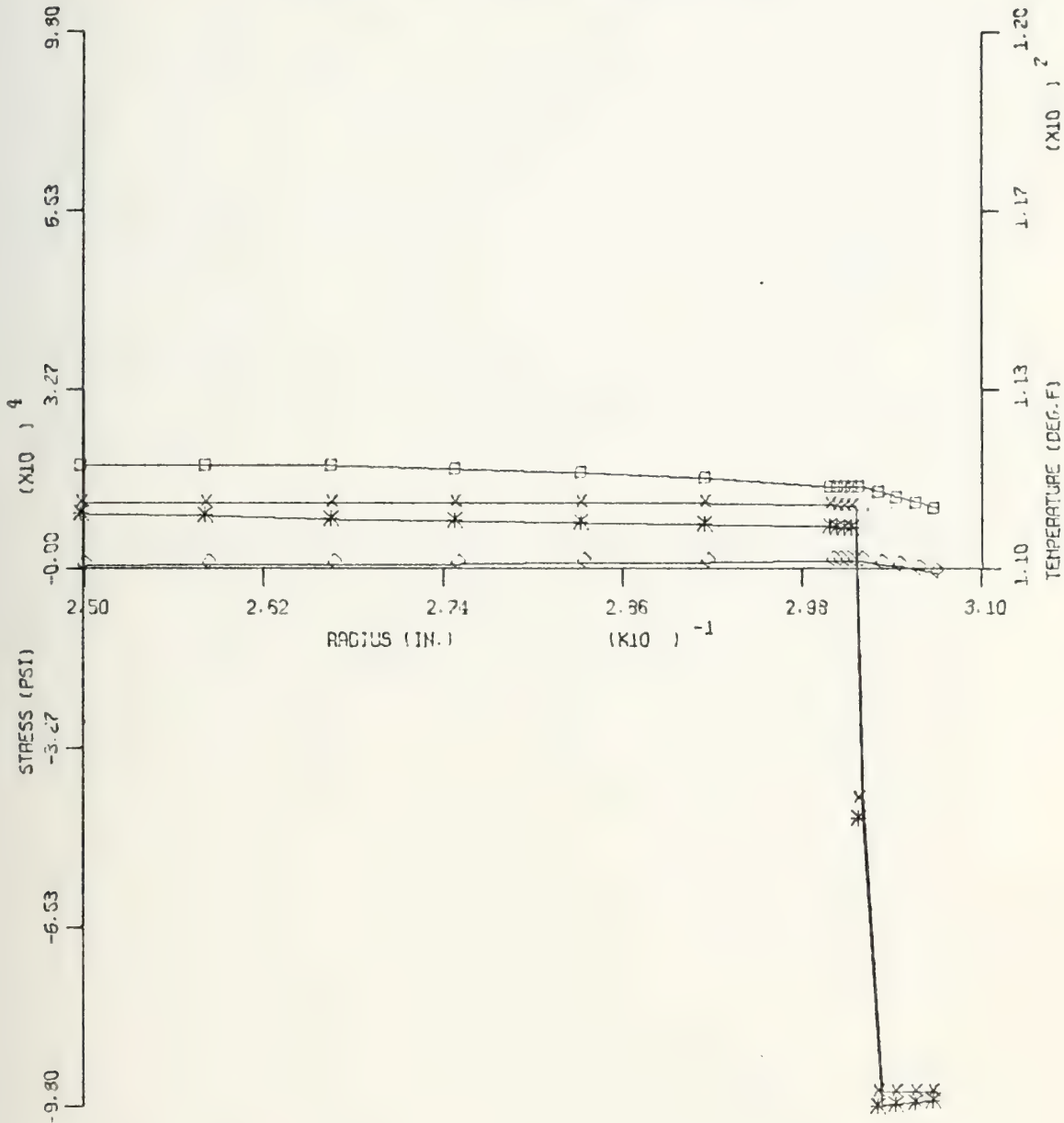
STRESS SCALE = 1.835×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 8(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.2-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

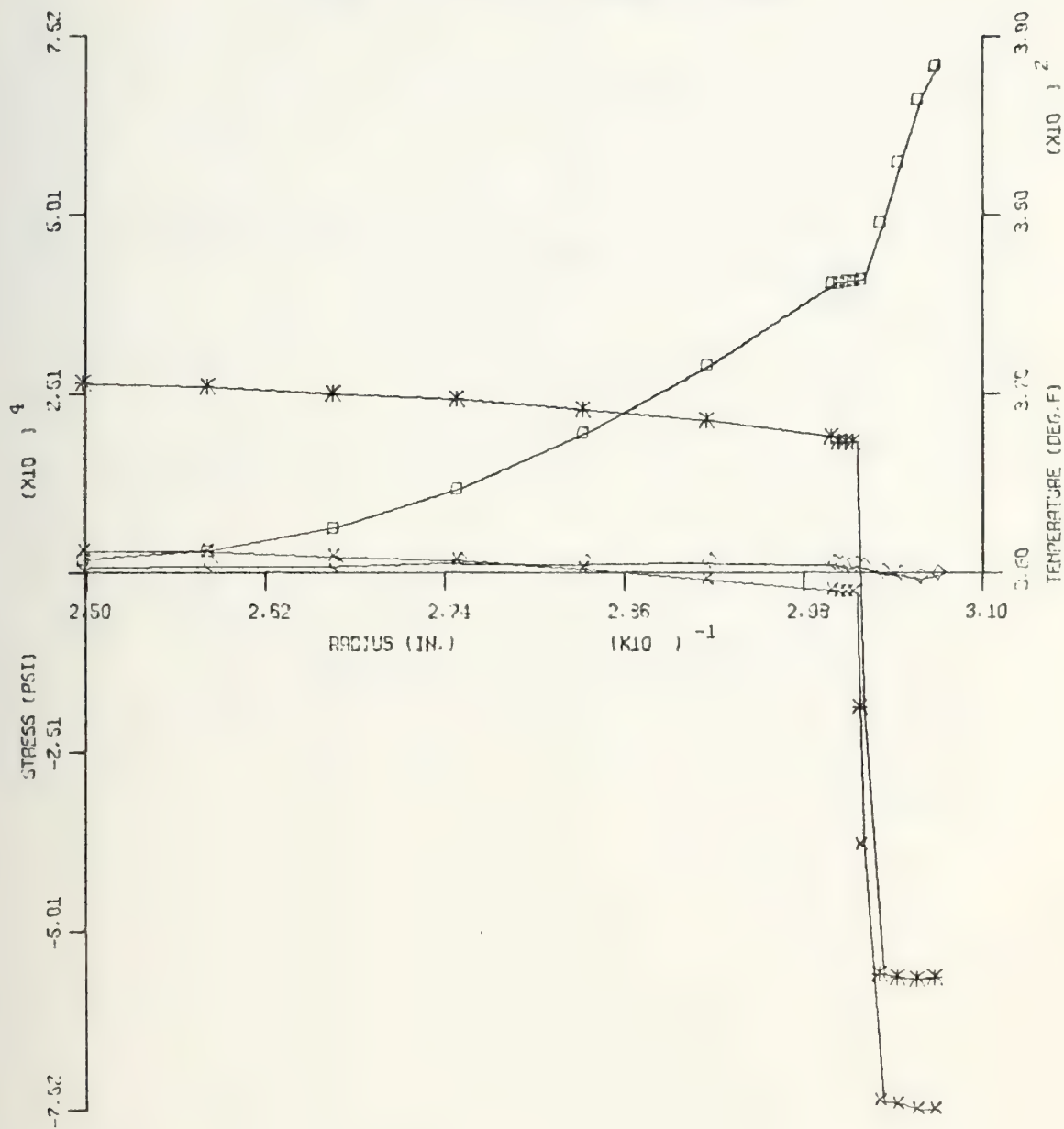
STRESS SCALE = 3.265×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 8(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.2-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

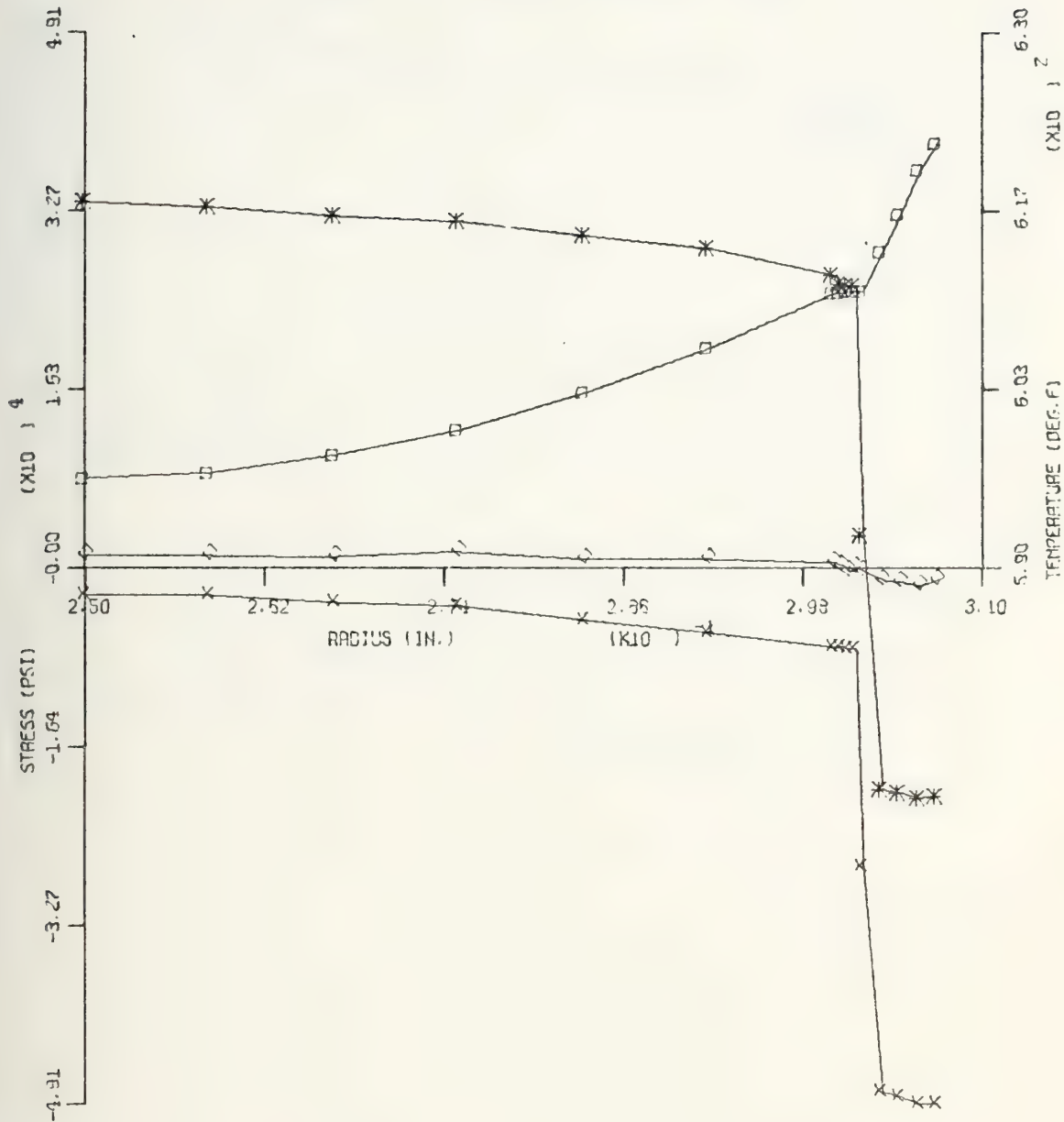
STRESS SCALE = 2.507×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 9(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.2-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

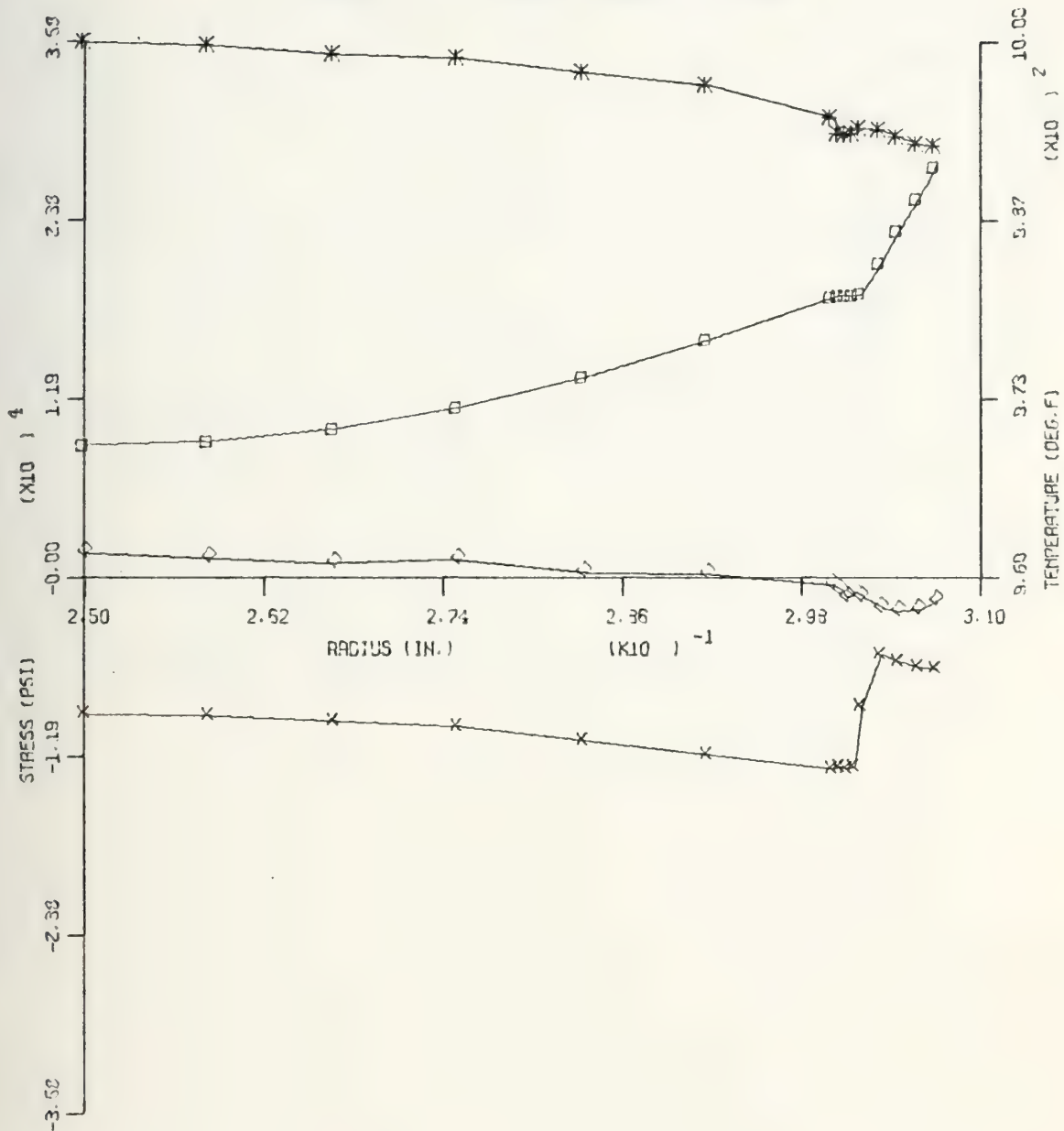
STRESS SCALE = 1.535×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 9(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.2-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

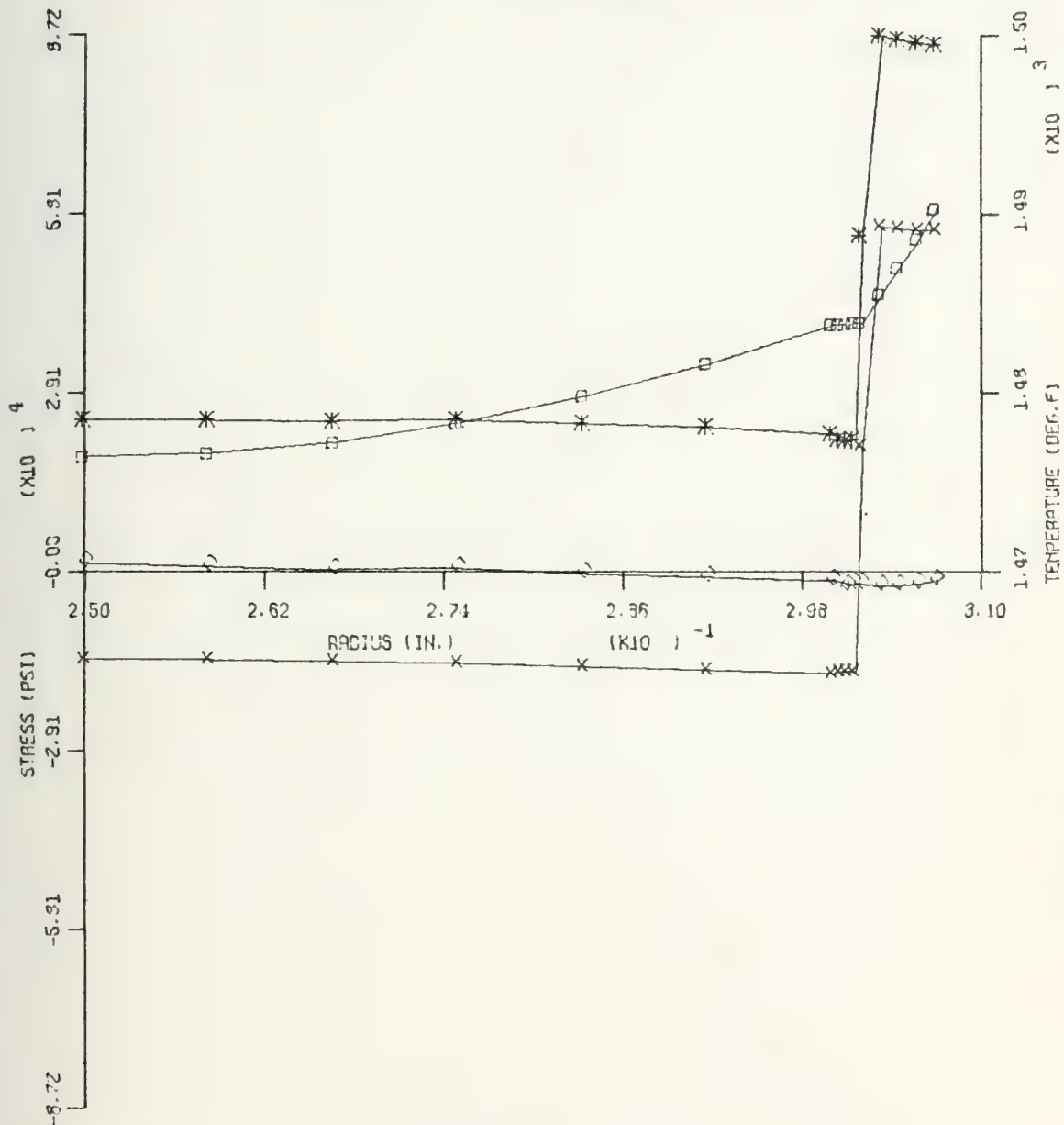
STRESS SCALE = 1.192×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 9(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.2-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

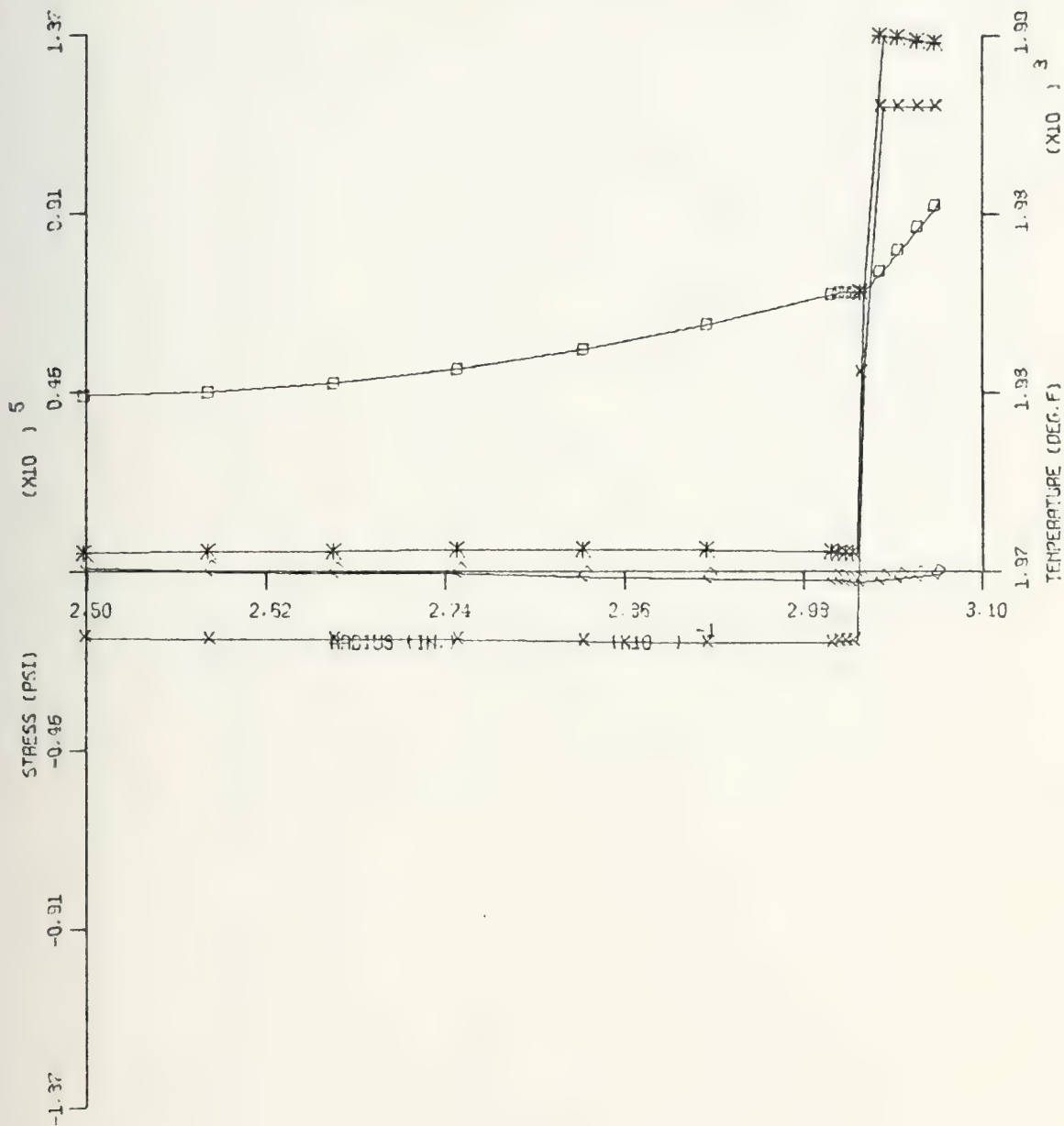
STRESS SCALE = 2.935×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 9(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.2-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

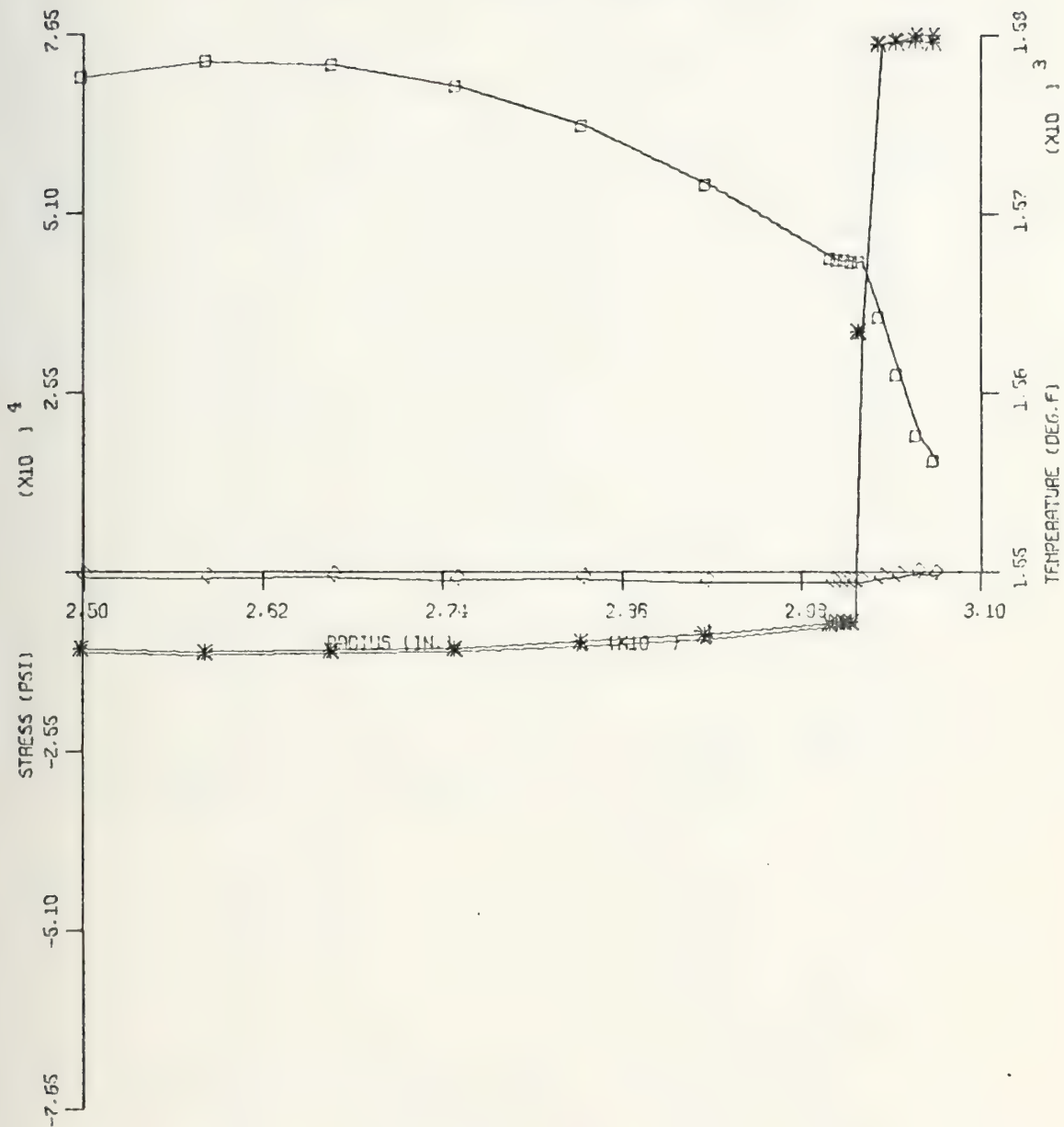
STRESS SCALE = 4.556×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 9(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.2-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

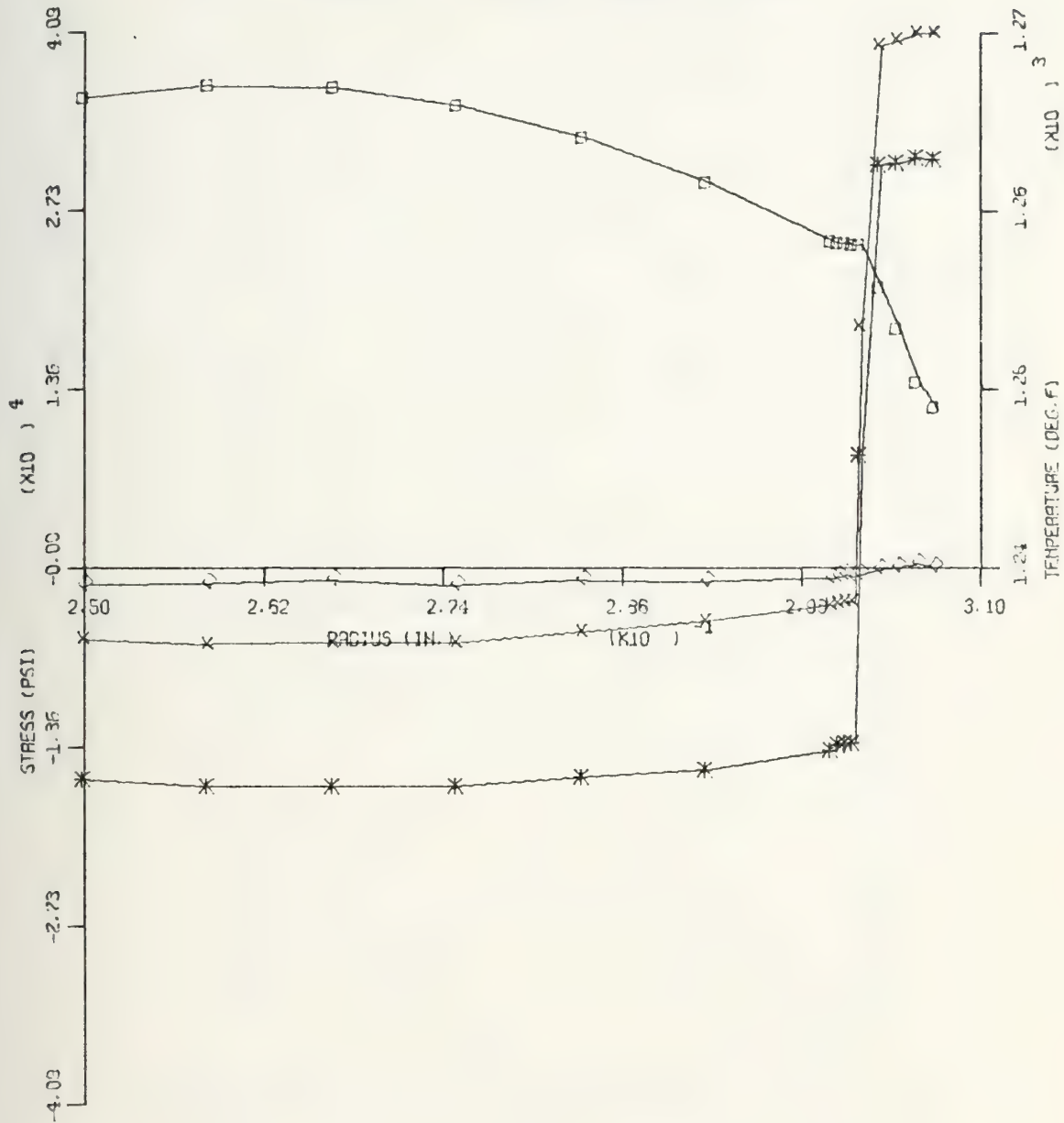
STRESS SCALE = 2.550×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 9(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.2-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

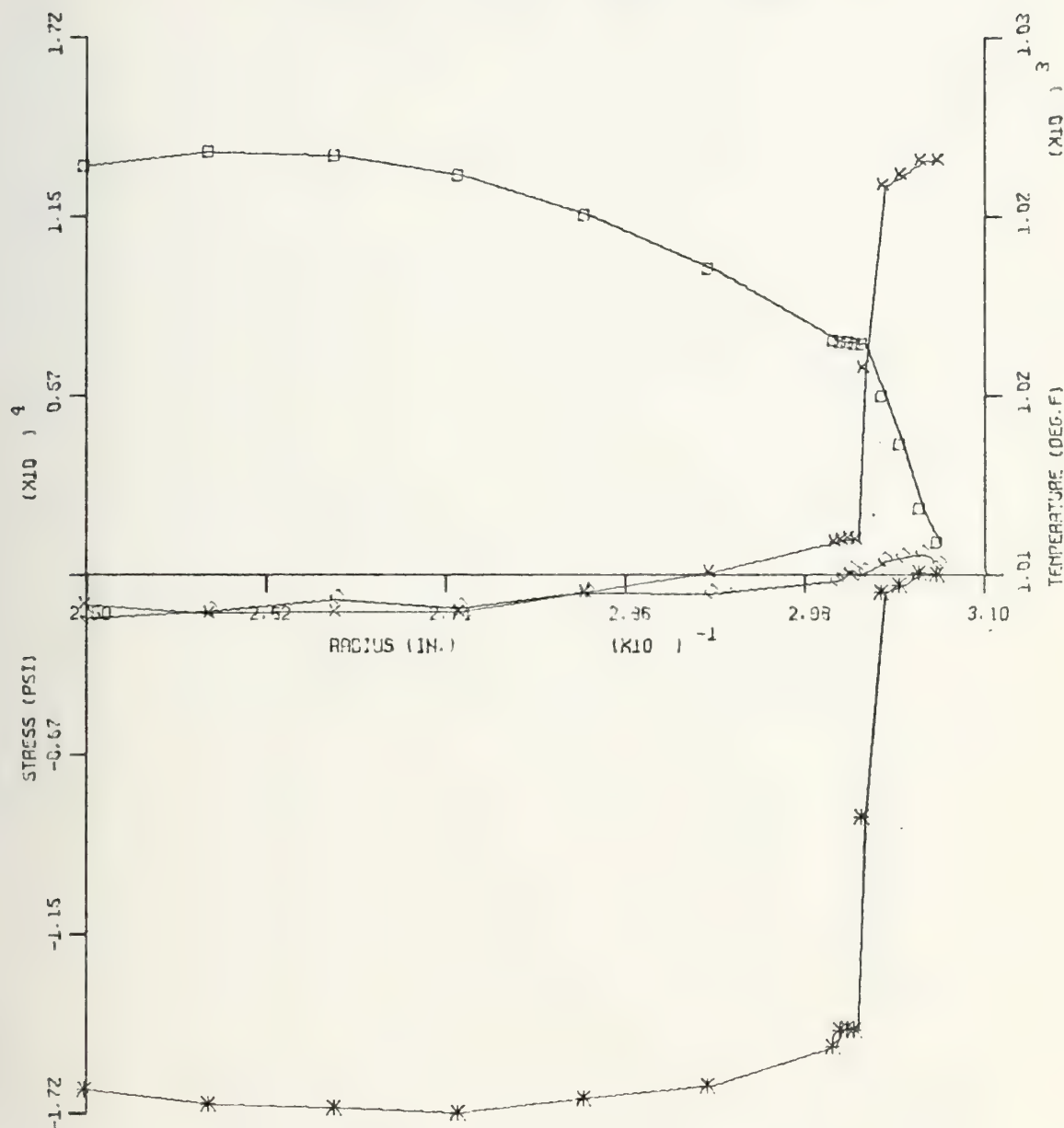
STRESS SCALE = 1.354×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 9(g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.2-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

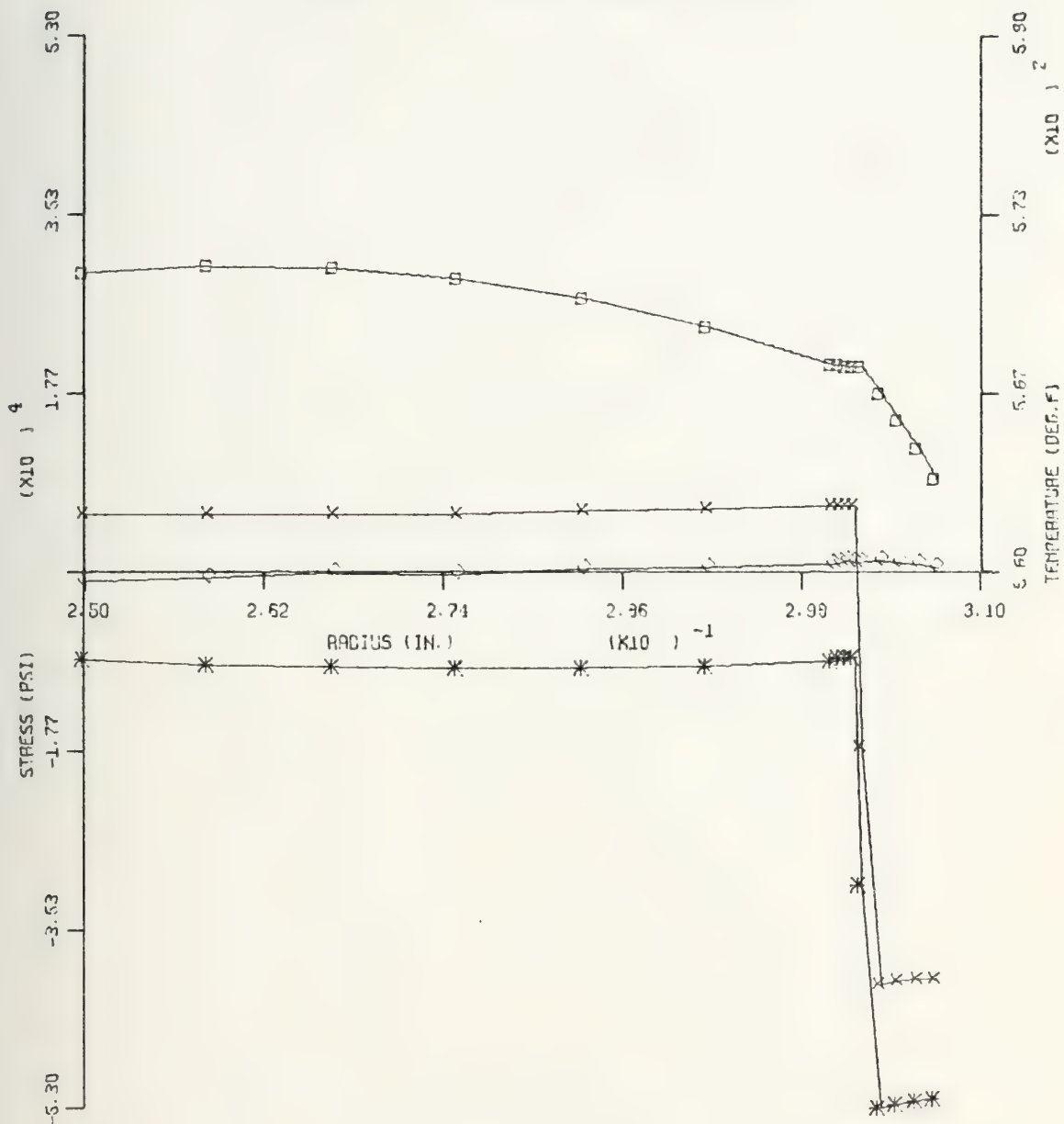
STRESS SCALE = 5.723×10^3 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 9(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.2-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

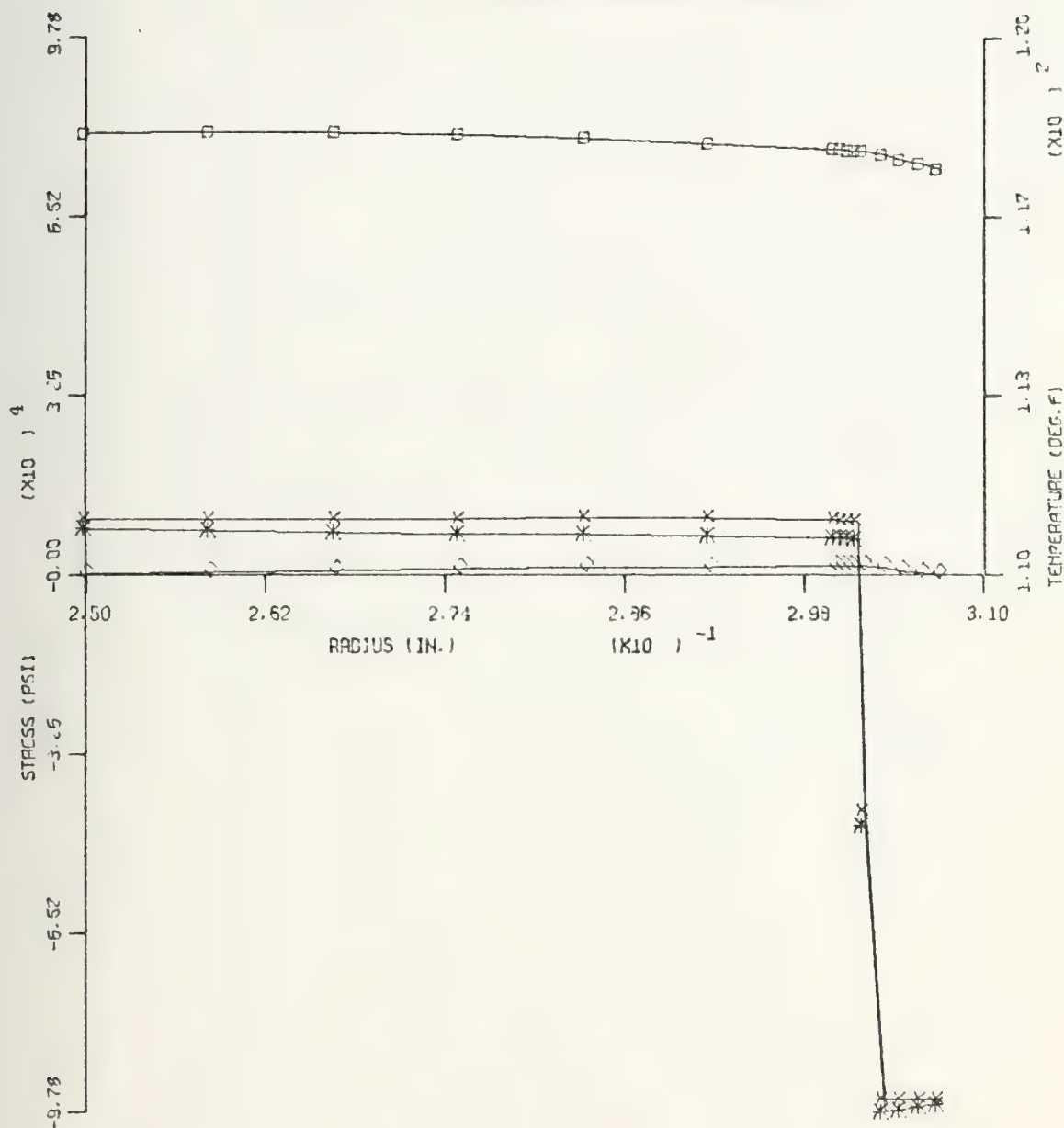
STRESS SCALE = 1.767×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 9(i)

STRESS AND TEMPERATURE VERSUS RADIAL POSITION TIME = 1.650 CONF.2-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

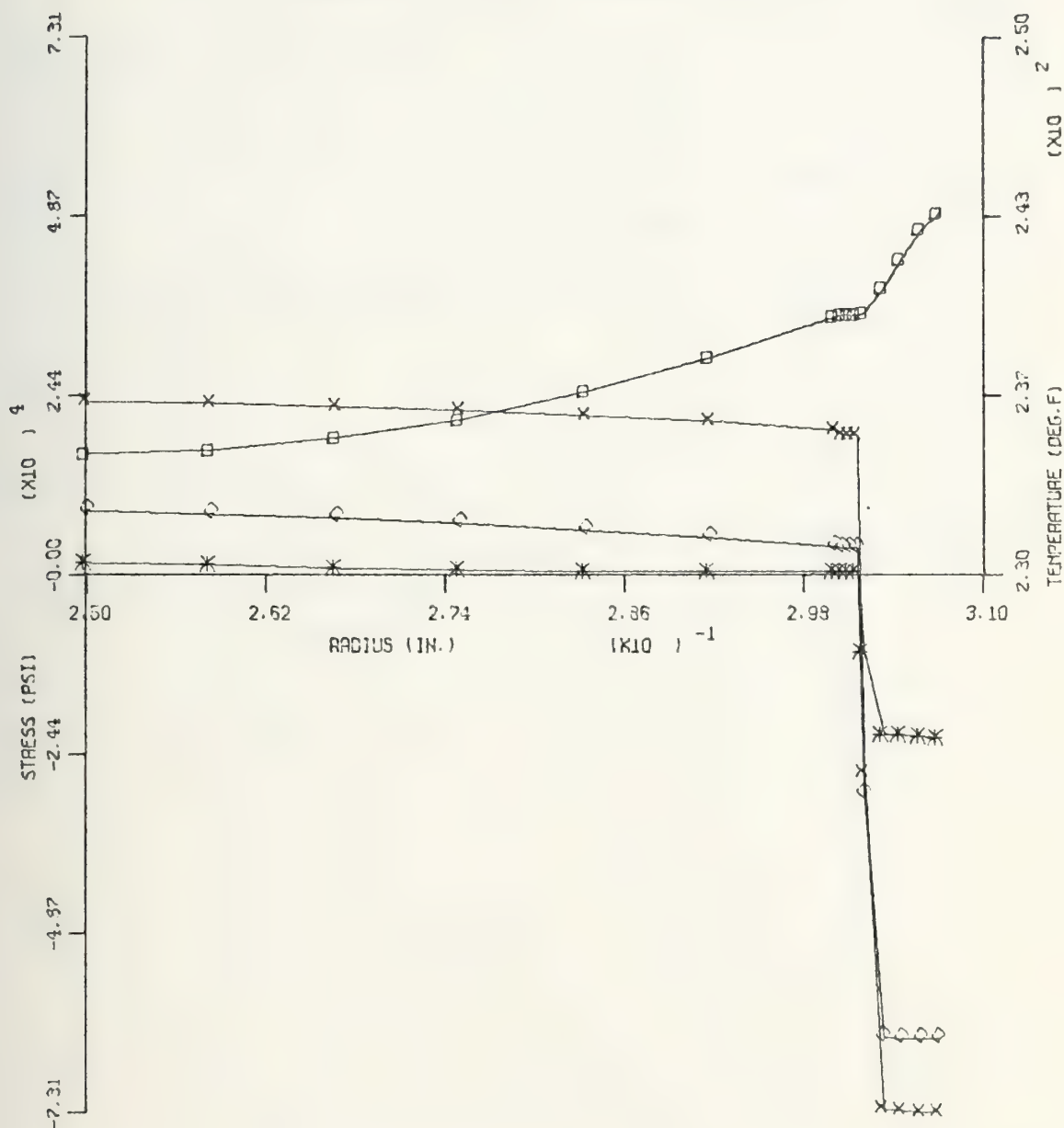
STRESS SCALE = 3.251×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 9(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.2-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

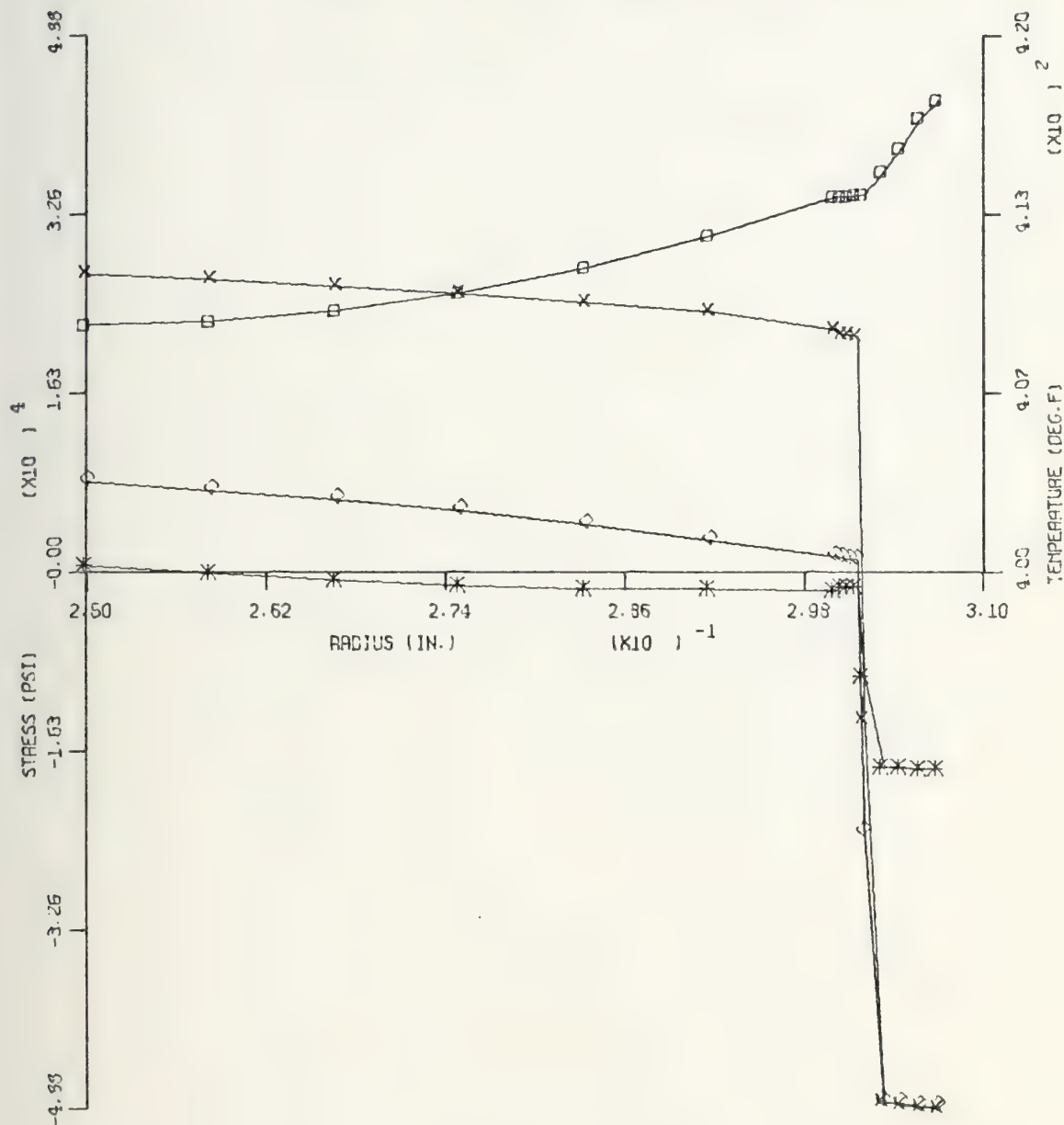
STRESS SCALE = 2.43×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 10(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.2-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

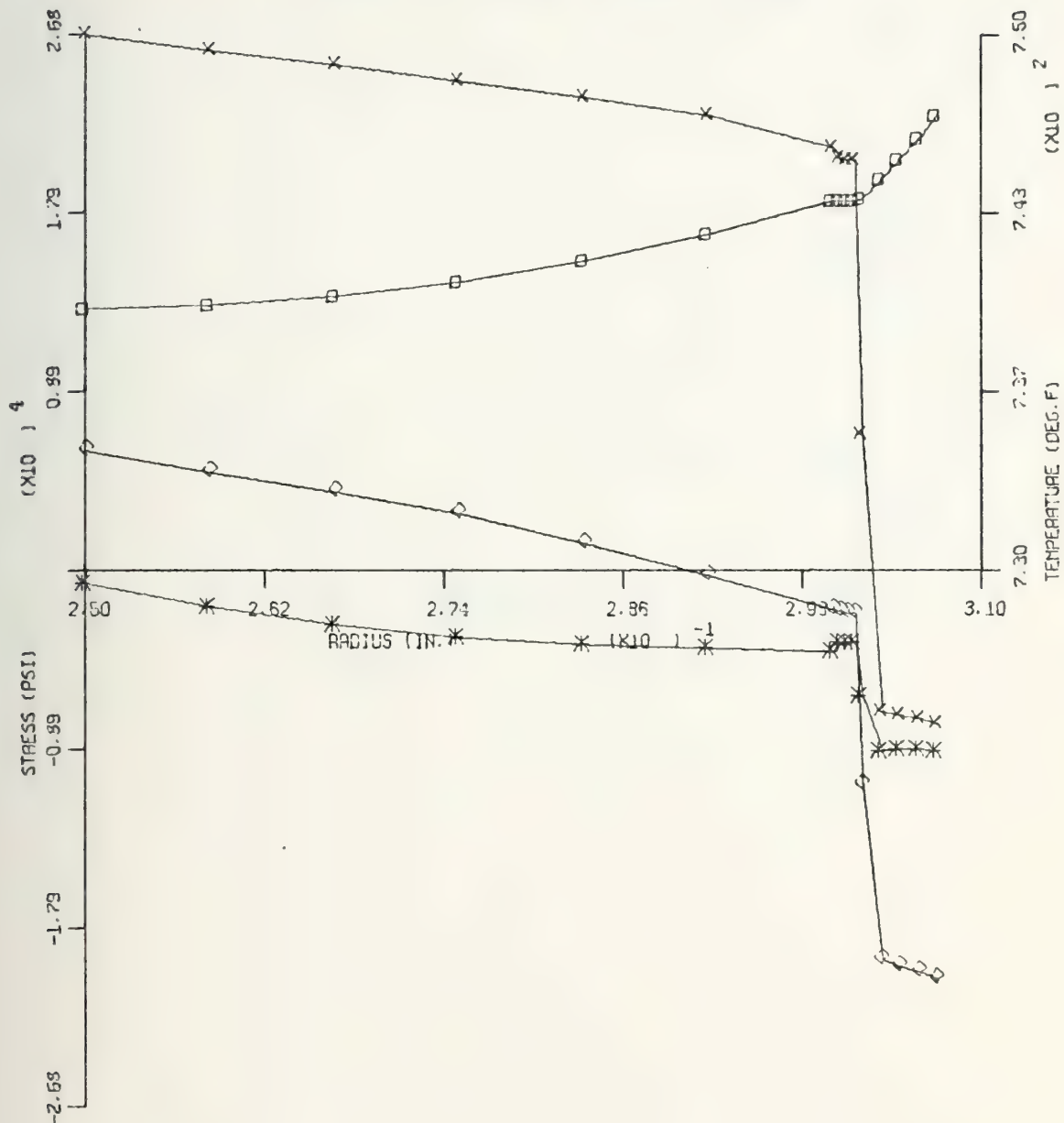
STRESS SCALE = 1.528×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 10(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.2-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

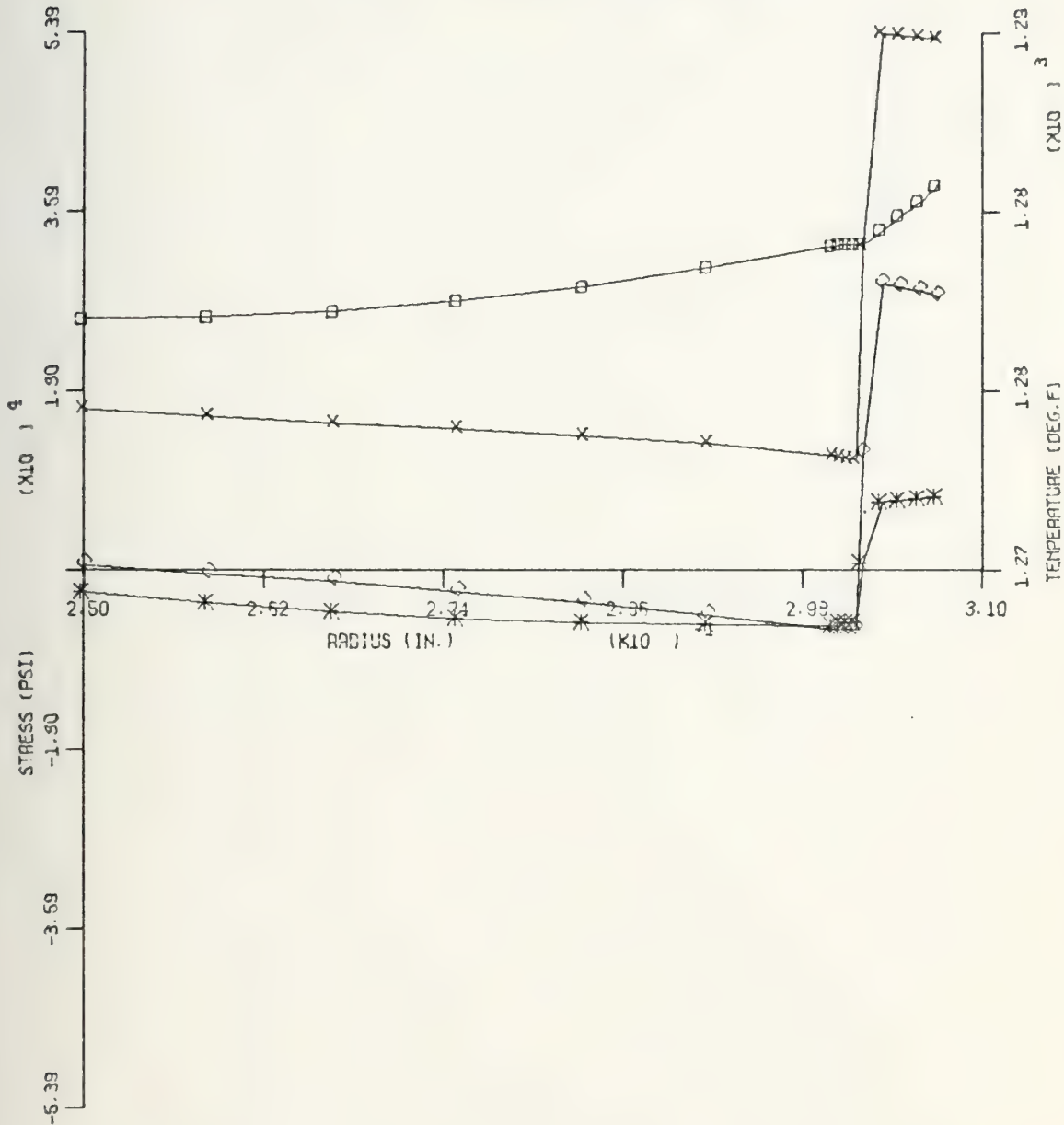
STRESS SCALE = 8.930×10^3 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 10(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.2-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

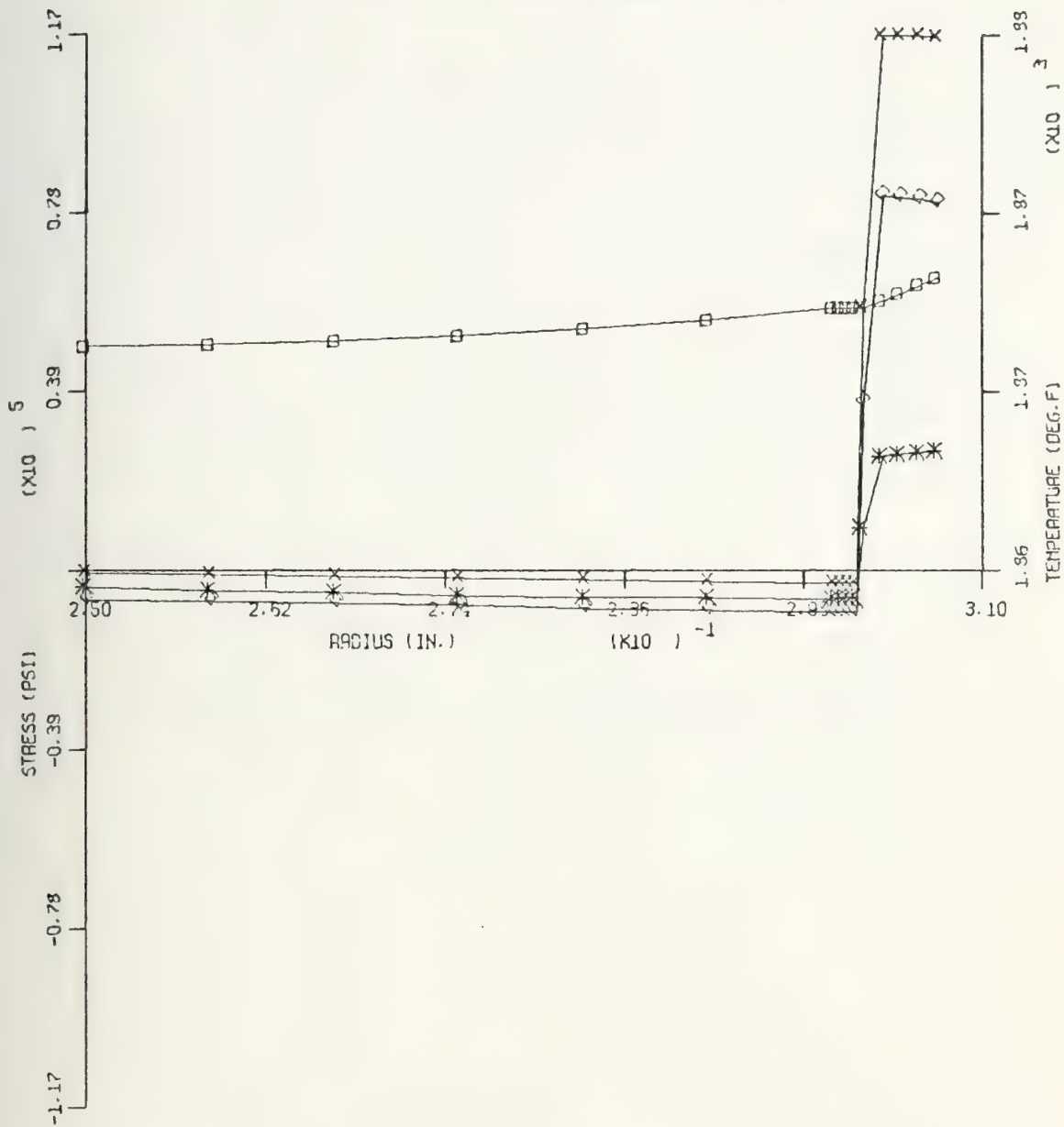
STRESS SCALE = 1.796×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 10(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.2-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

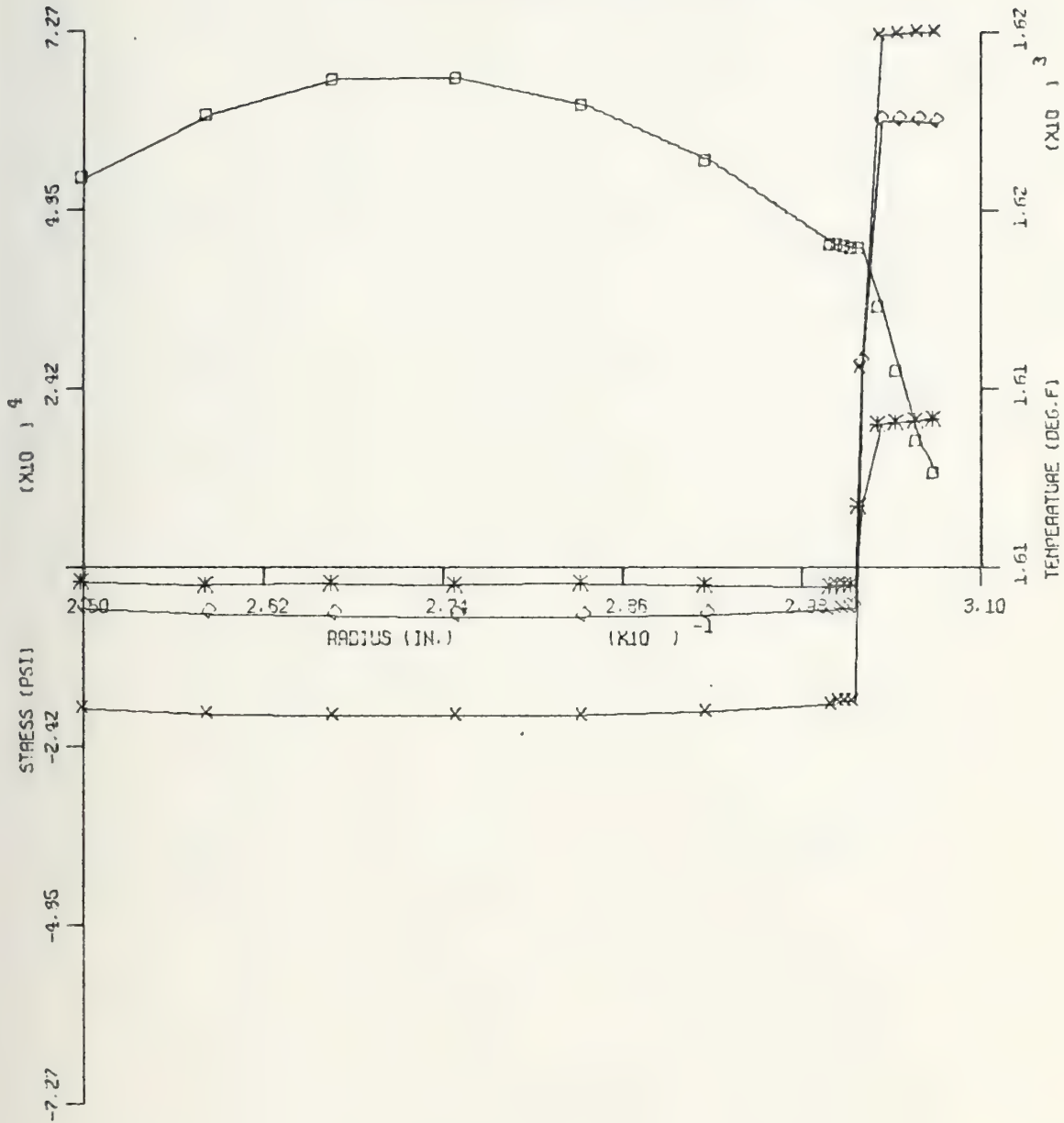
STRESS SCALE = 3.910×10^4 PSI/INCH

TEMPERATURE SCALE = 6.2 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 10(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.2-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

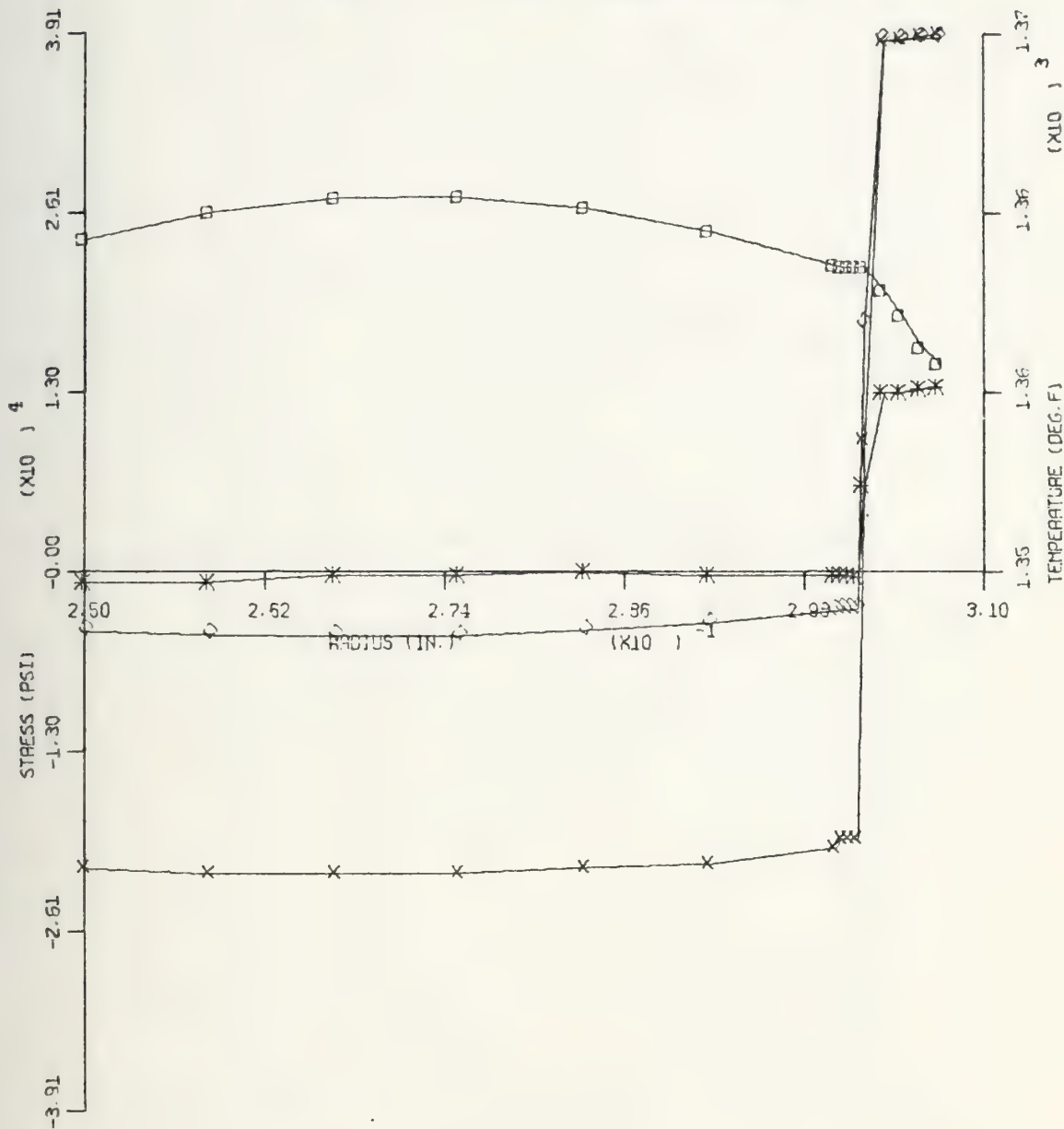
STRESS SCALE = 2.424×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 10(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.2-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

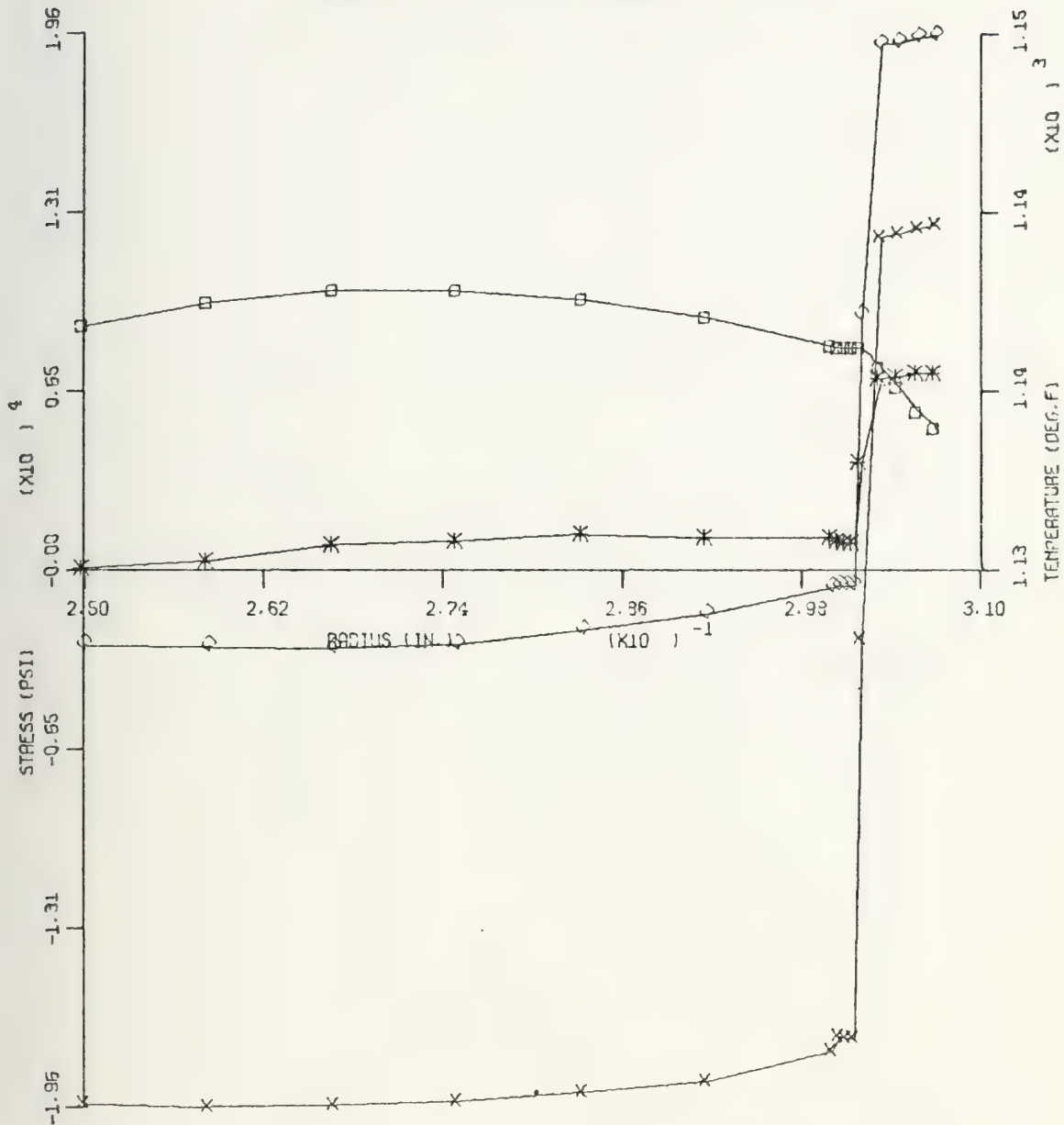
STRESS SCALE = 1.304×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 10(g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.2-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

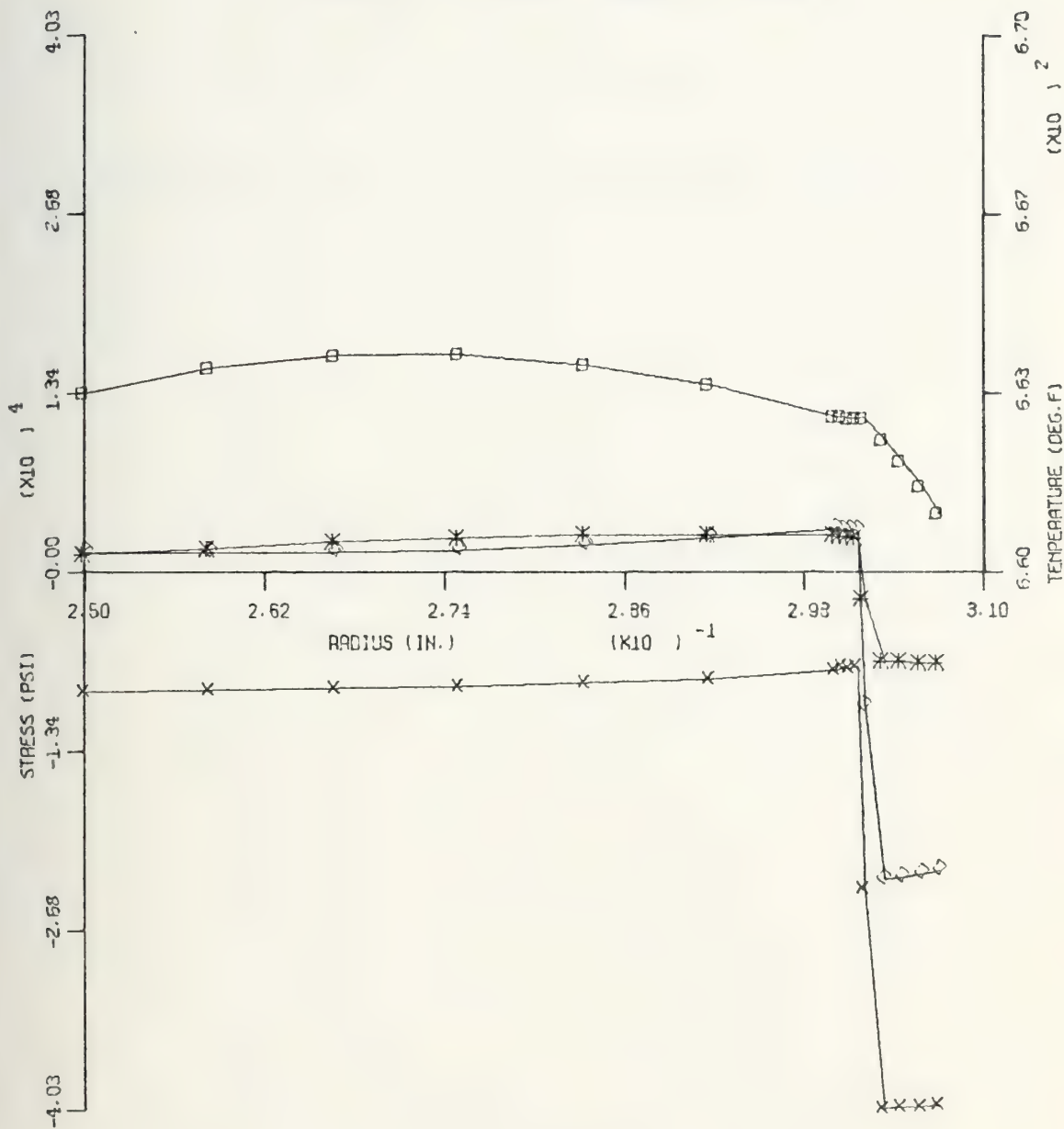
STRESS SCALE = 5.537×10^3 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 10(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.2-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

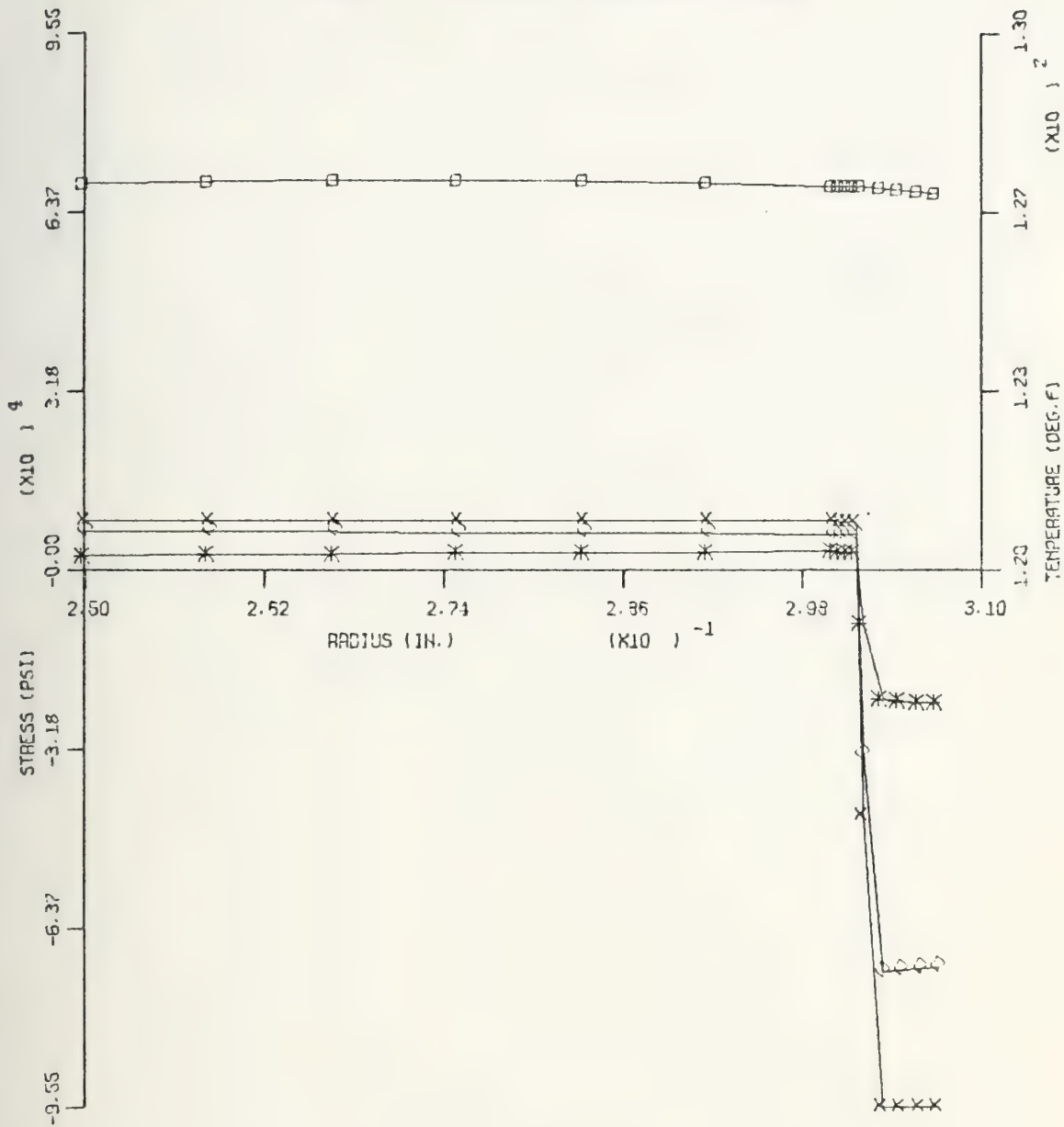
STRESS SCALE = 1.342×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 10 (i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.2-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

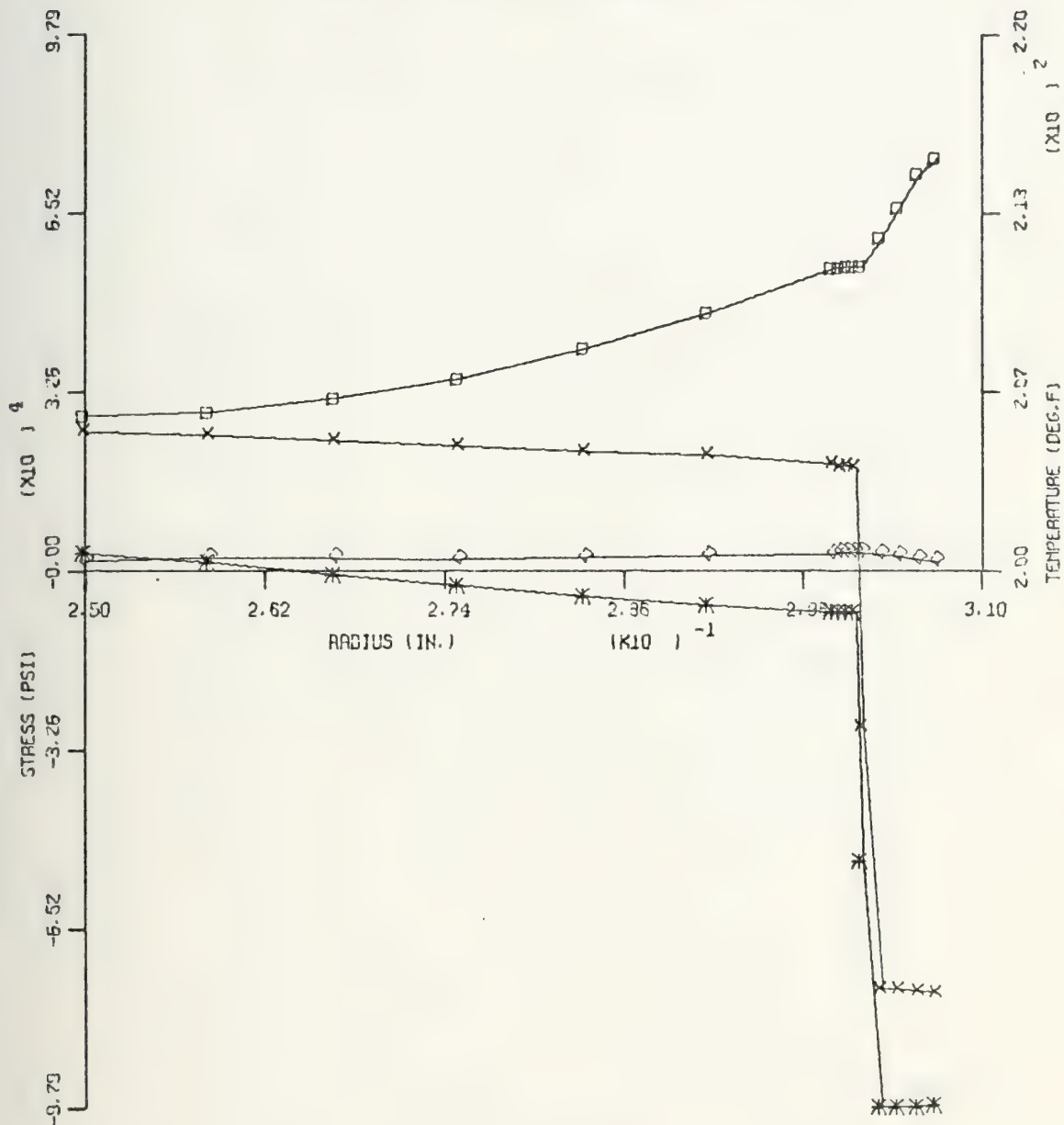
STRESS SCALE = 3.193×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 10(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.2-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

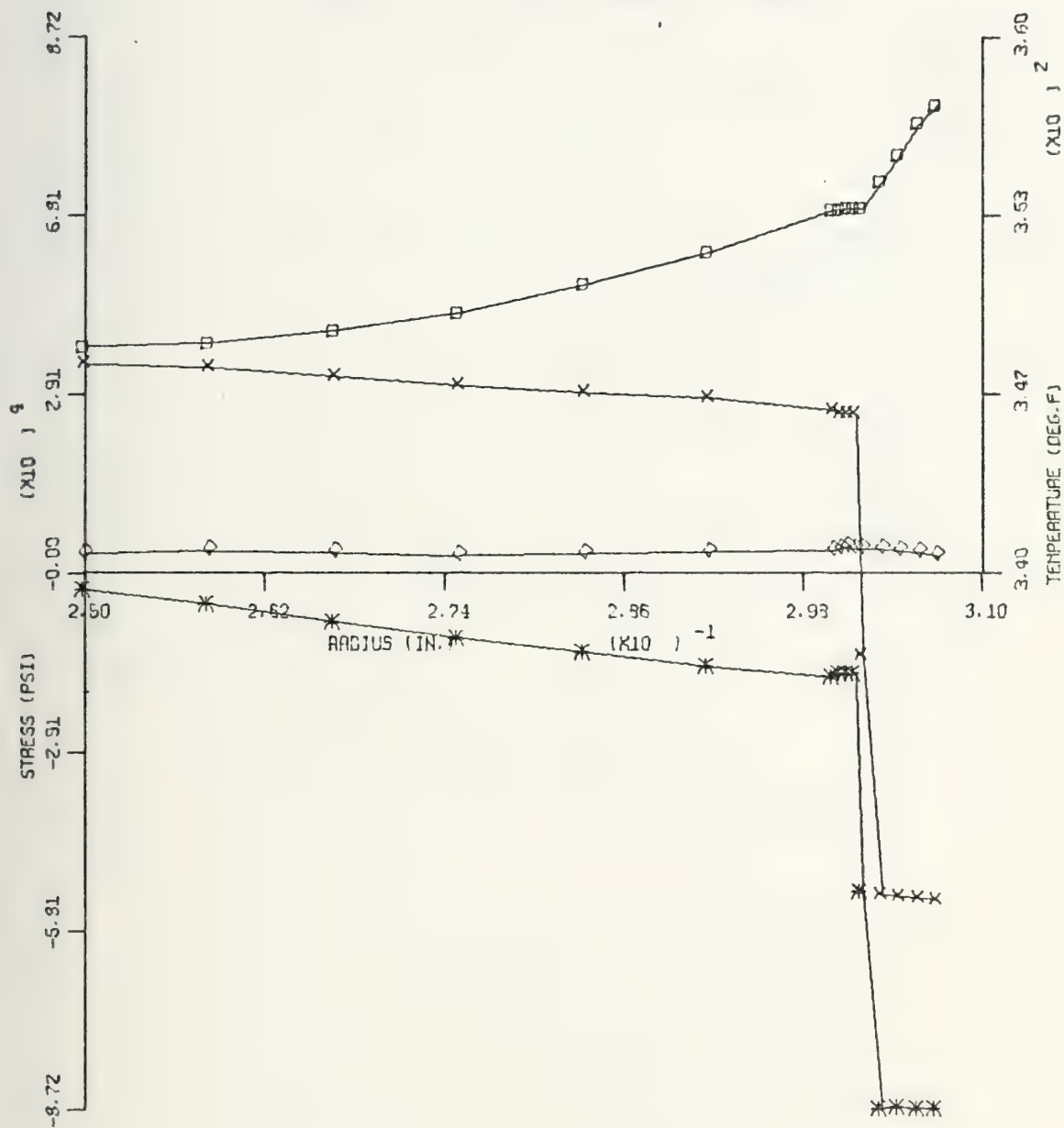
STRESS SCALE = 3.262×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 11(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.2-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

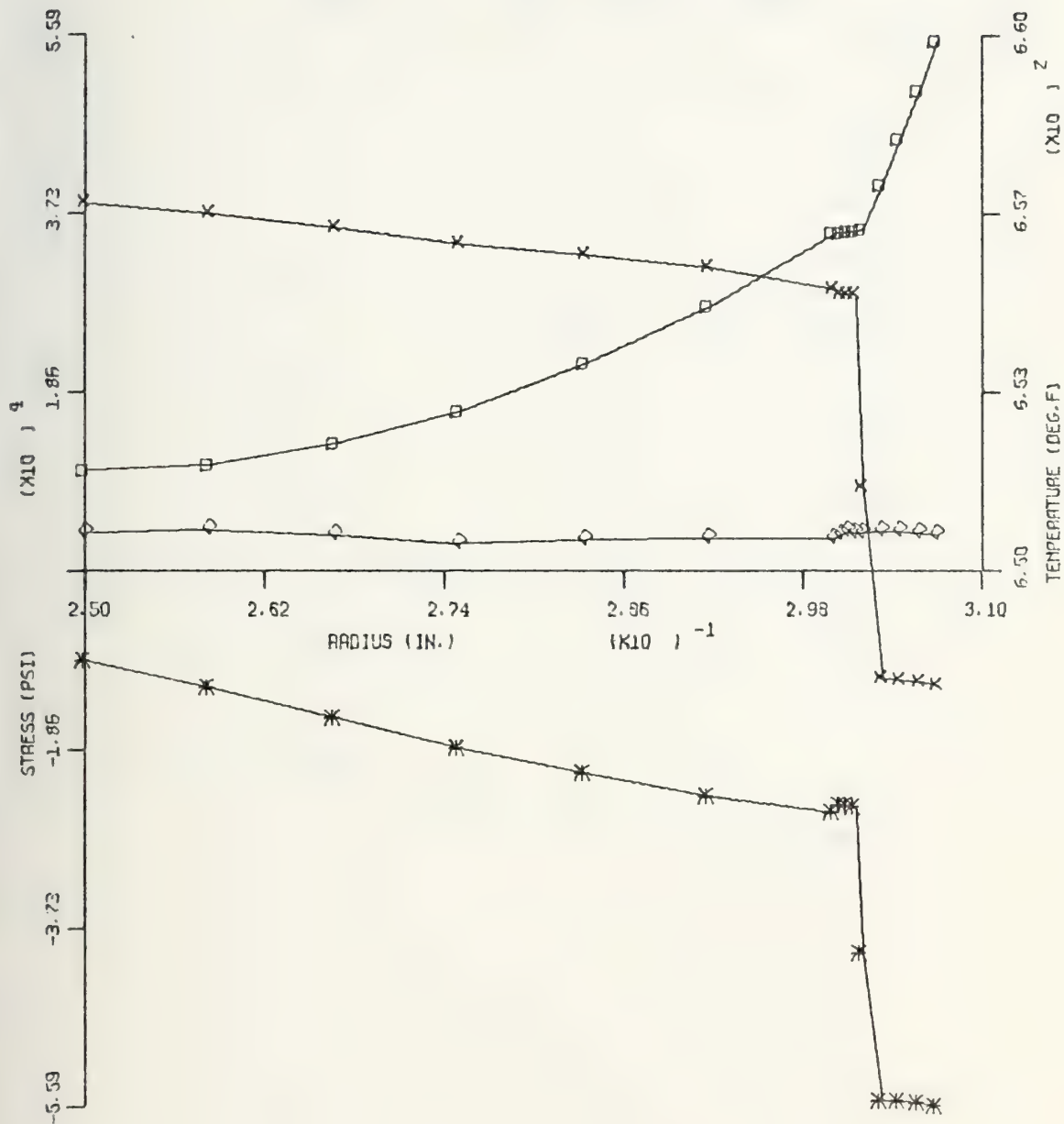
STRESS SCALE = 2.907×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 11(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.2-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

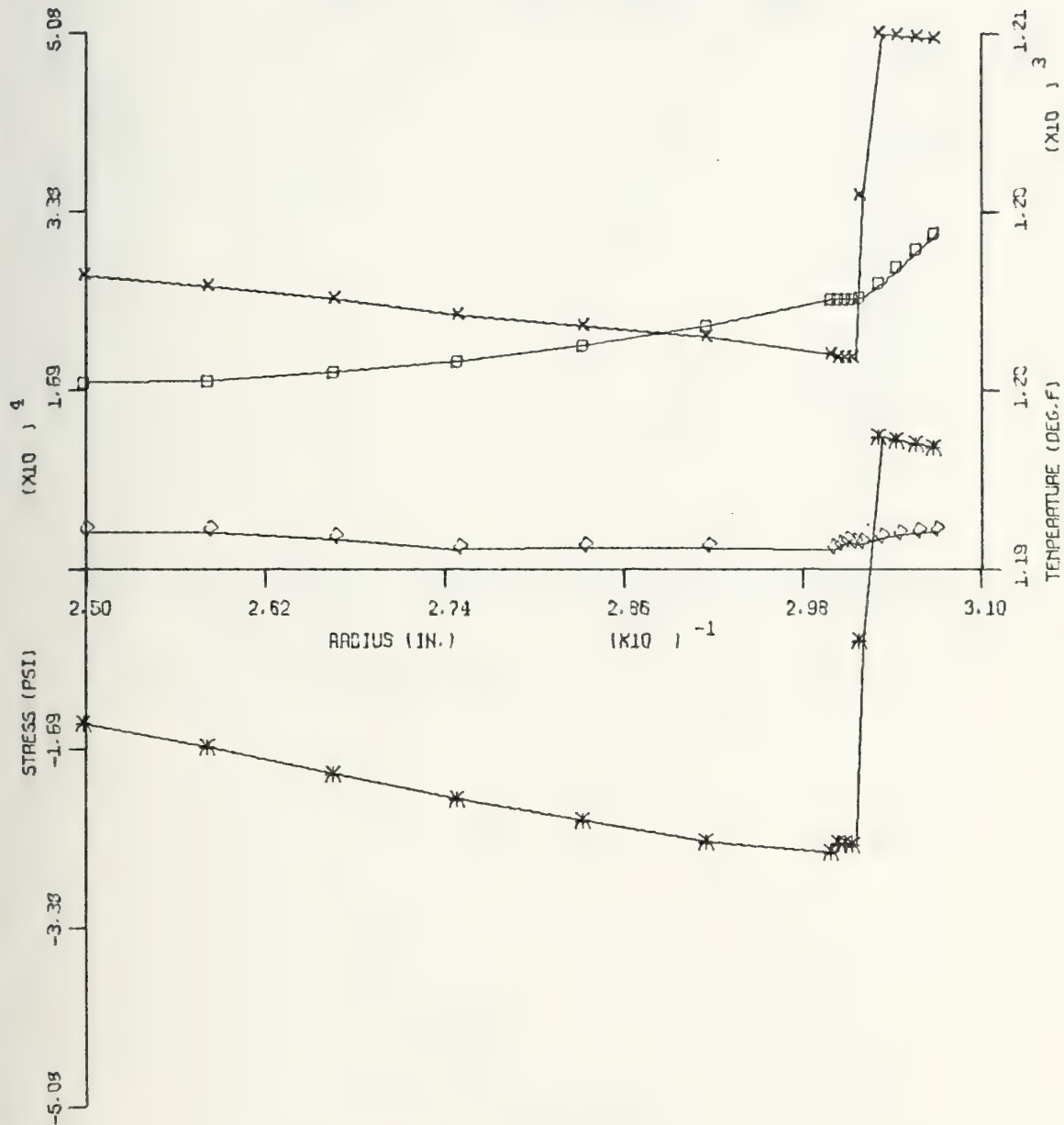
STRESS SCALE = 1.863×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 11(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.2-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

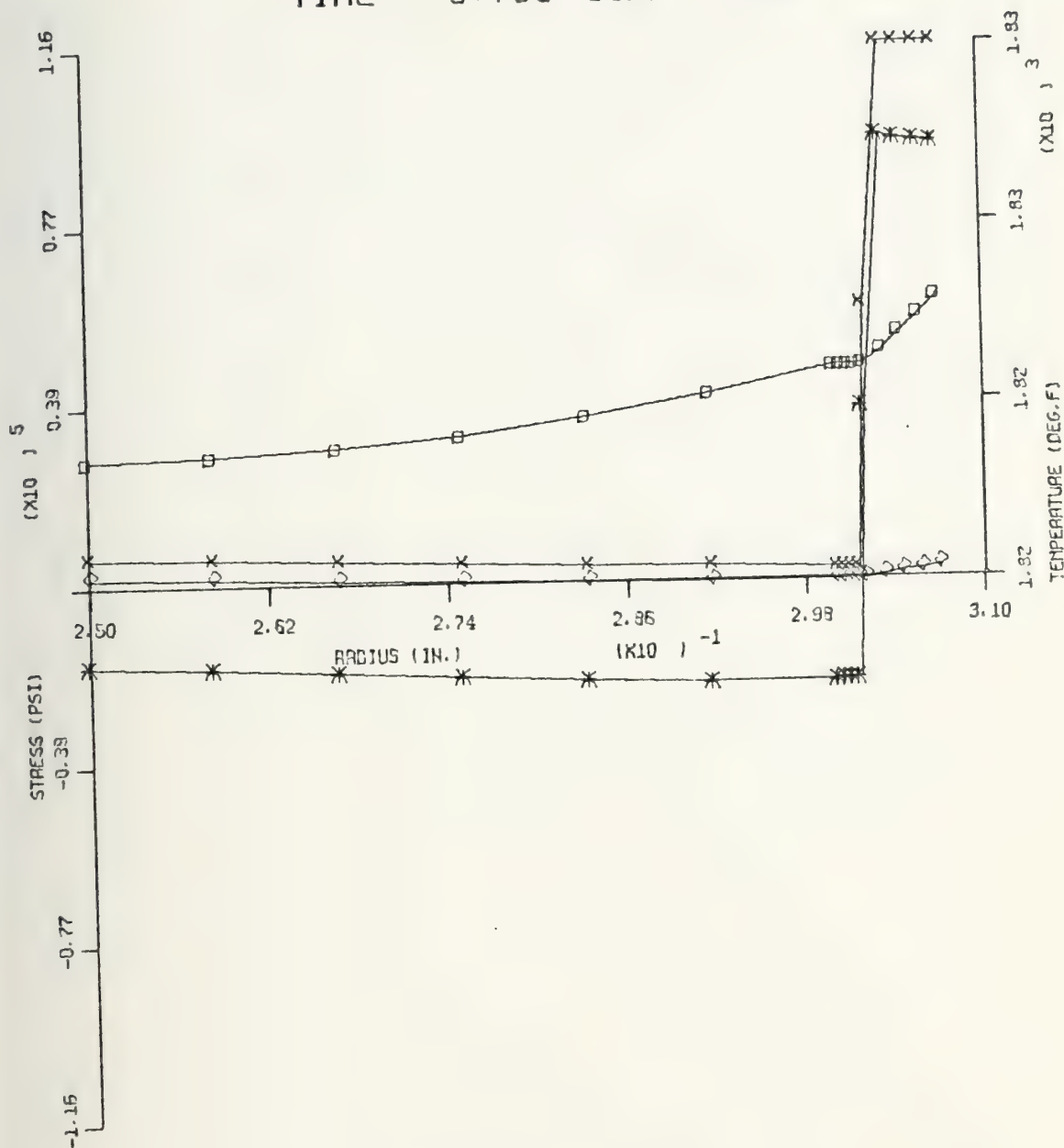
STRESS SCALE = 1.692×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 11(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.2-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

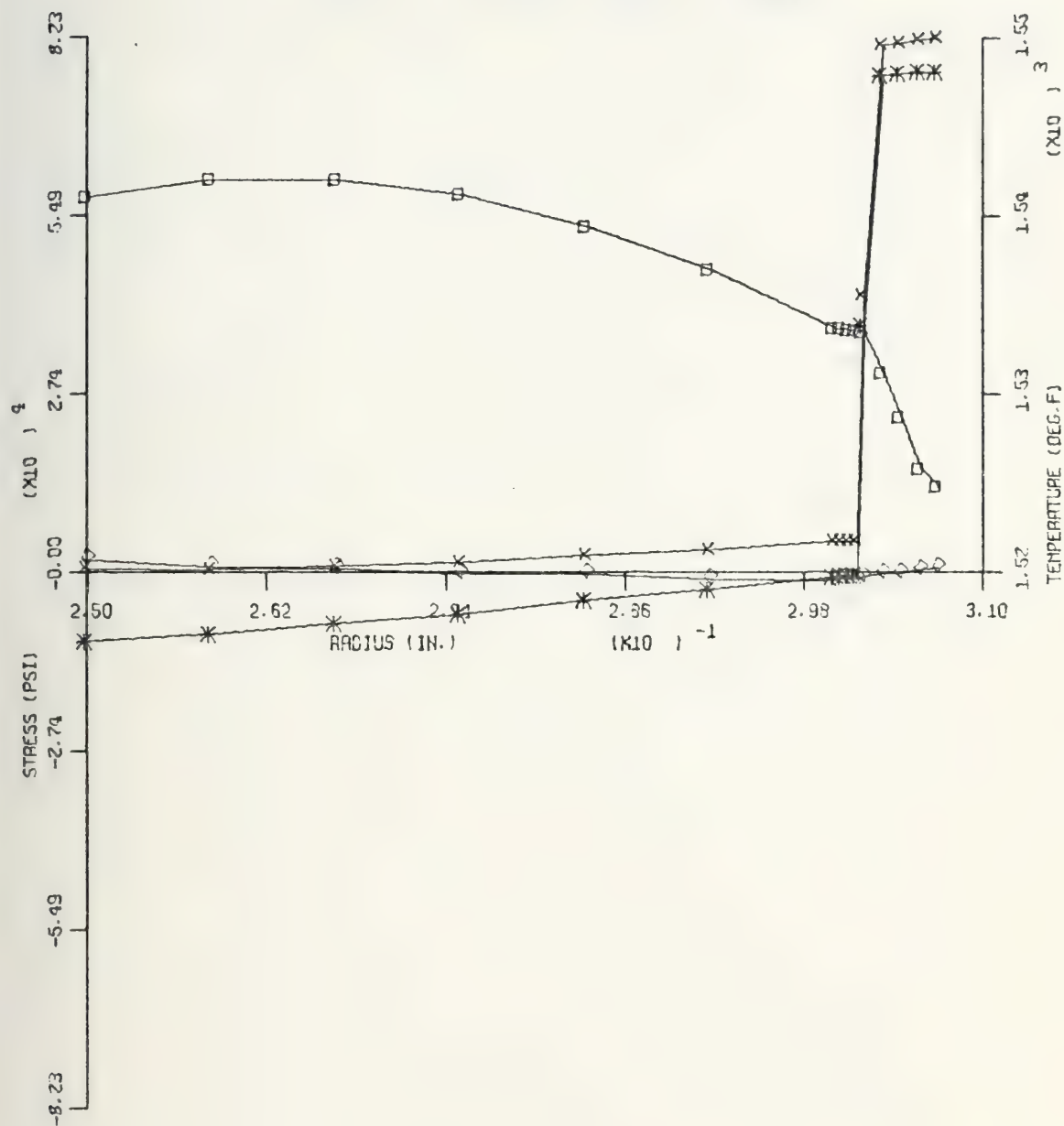
STRESS SCALE = 3.254×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 11(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.2-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

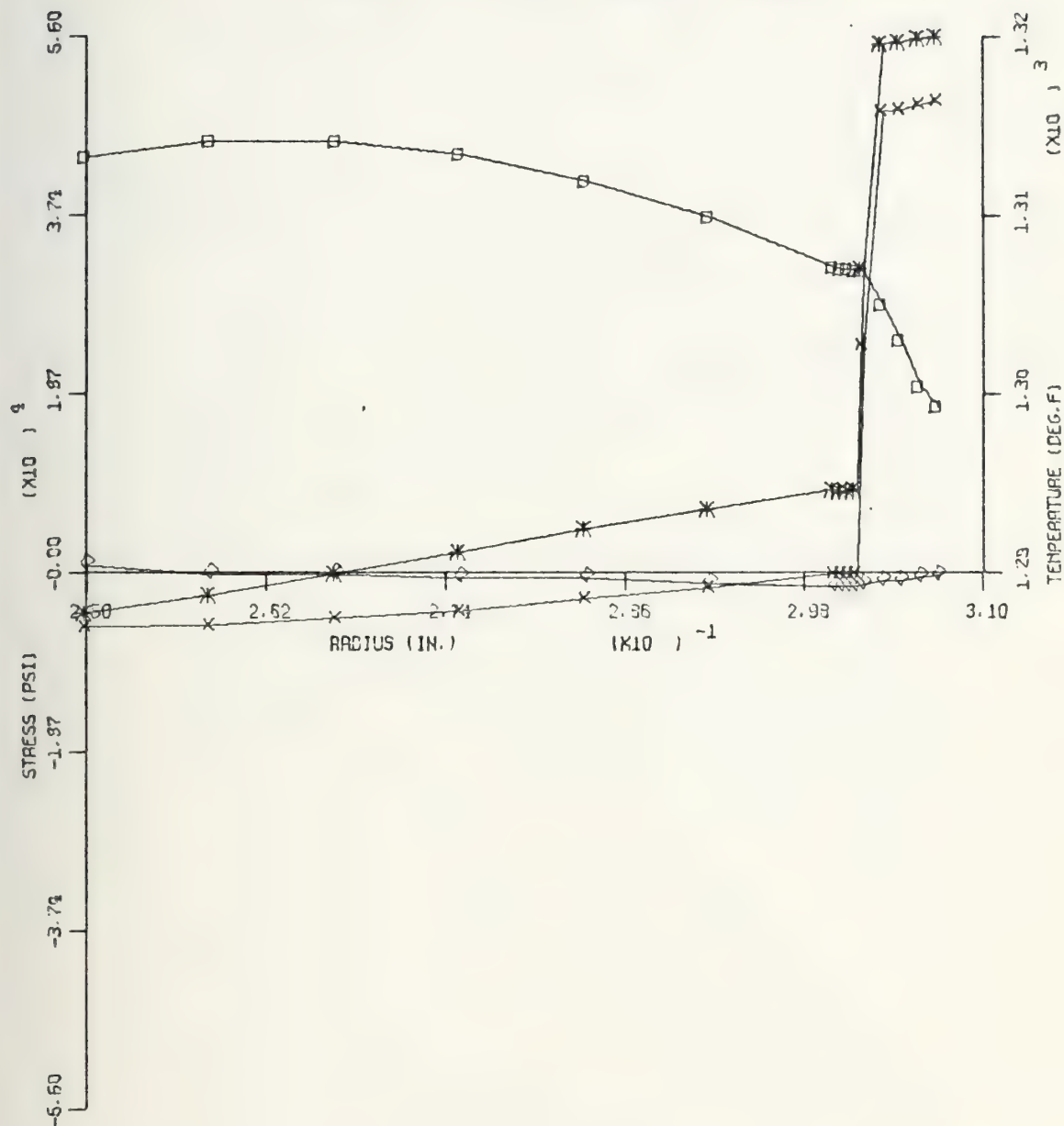
STRESS SCALE = 2.743×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 11(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.2-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

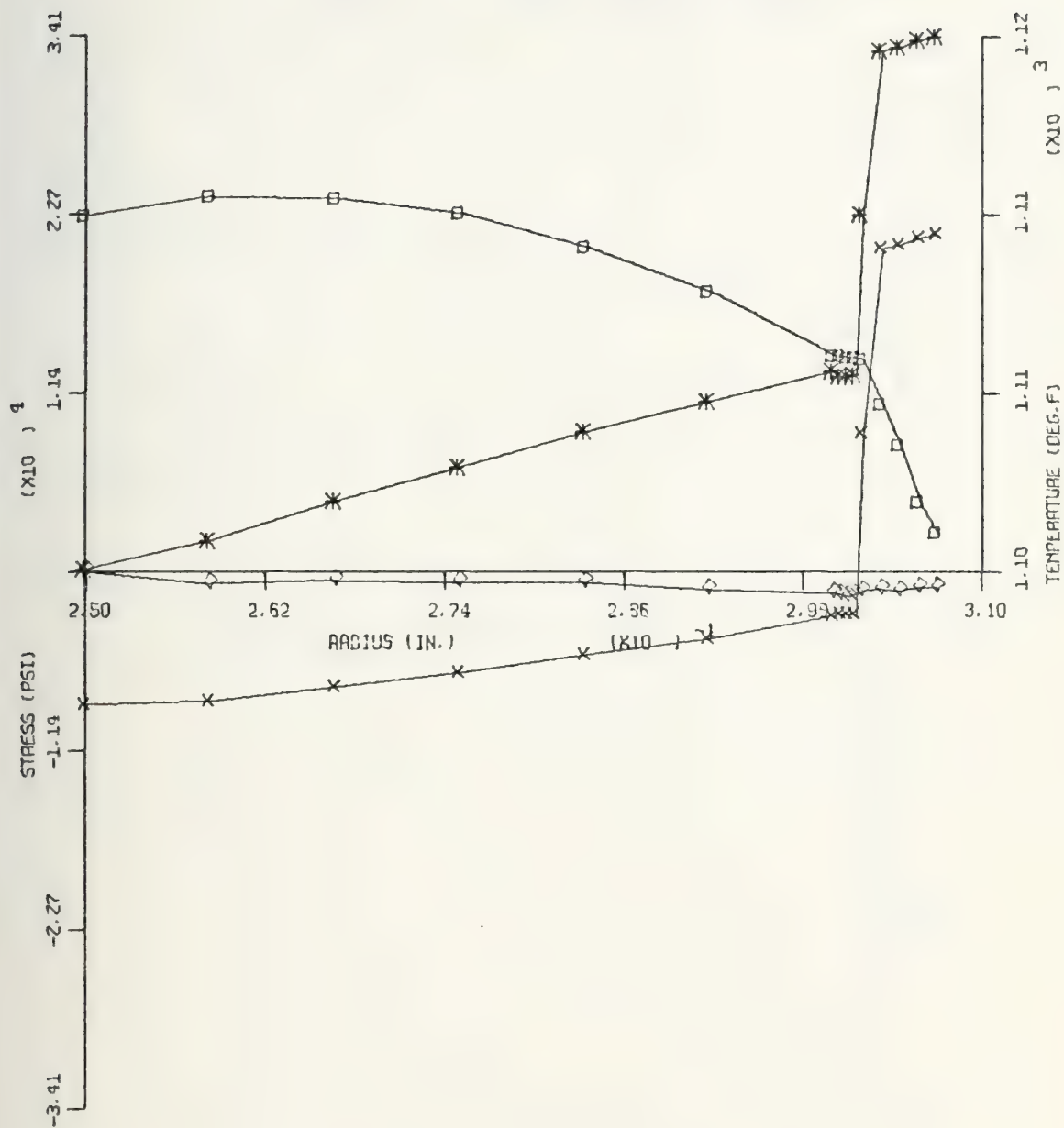
STRESS SCALE = 1.666×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 11(g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.2-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

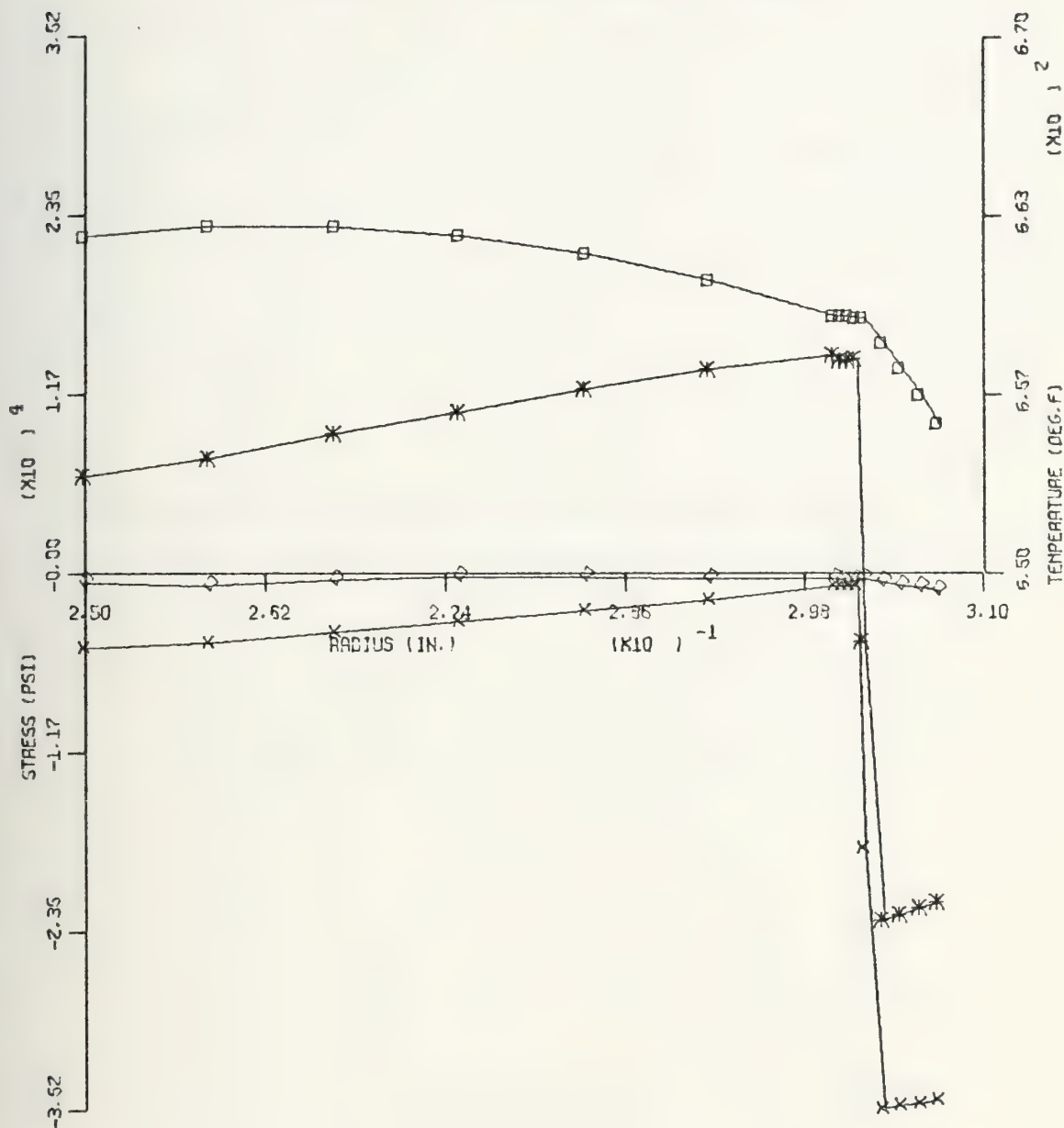
STRESS SCALE = 1.137×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 11(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.2-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

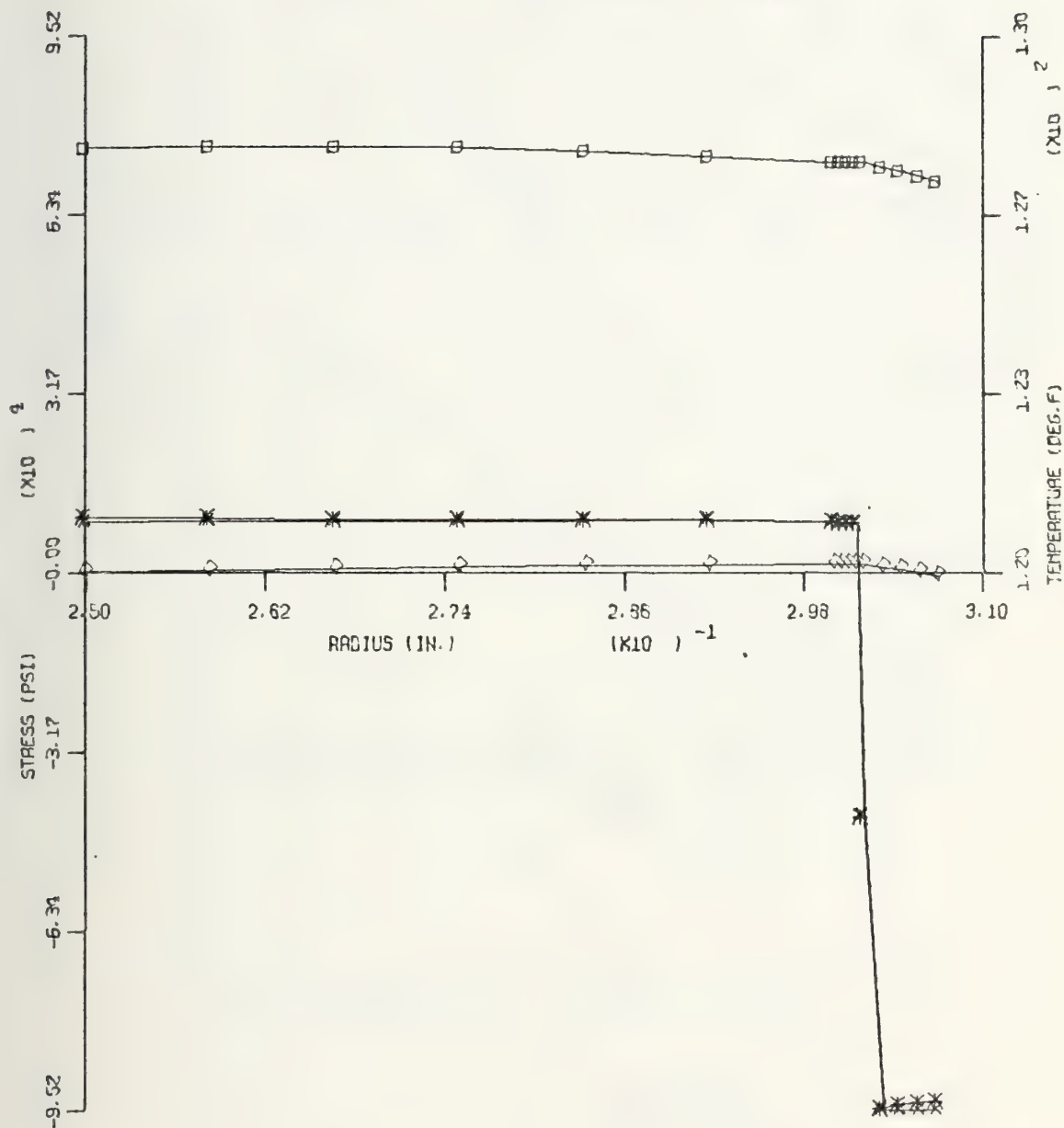
STRESS SCALE = 1.173×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 11(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.2-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

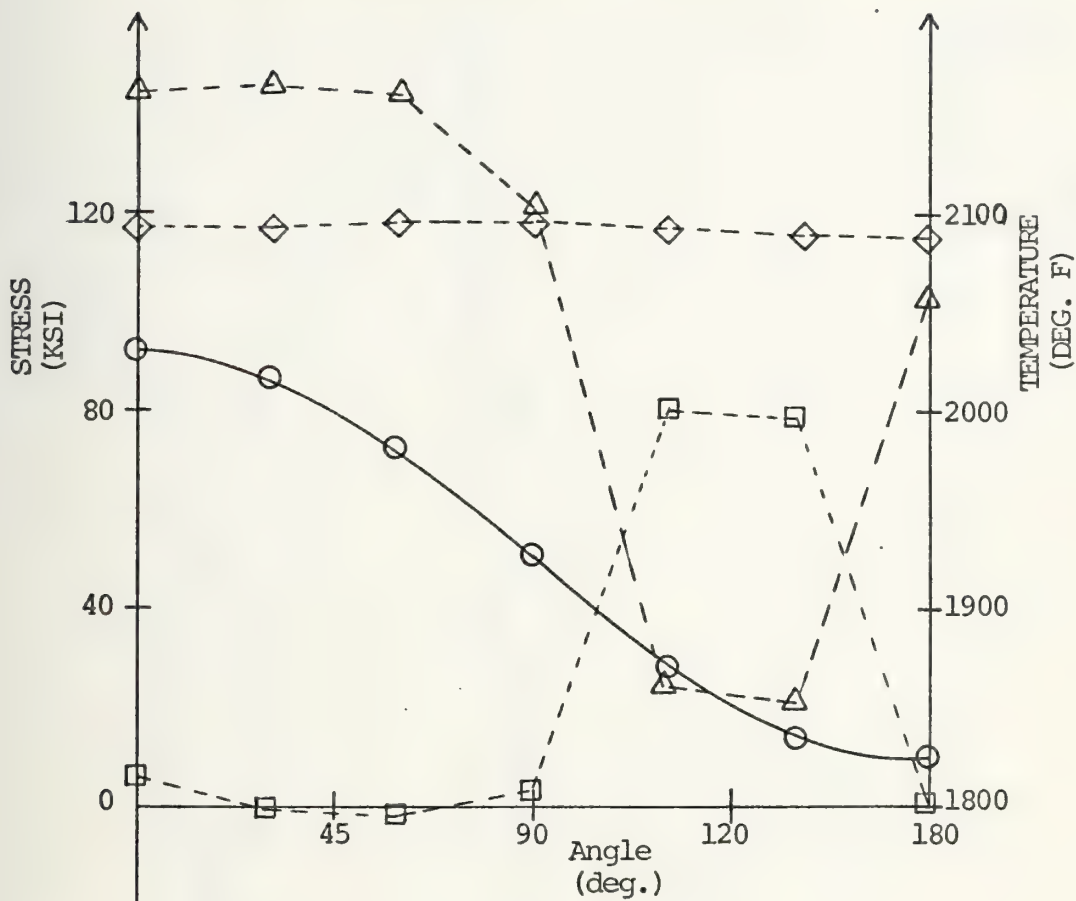
STRESS SCALE = 3.122×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 11(j)

STRESS AND TEMPERATURE
VERSUS
CIRCUMFERENTIAL POSITION



CONFIGURATION 2, $r=0.3045$ in., TIME=0.75 min.

- TEMPERATURE
- RADIAL STRESS
- ◇ AXIAL STRESS
- △ TANGENTIAL STRESS

FIGURE 12

STRESS AND TEMPERATURE
VERSUS TIME

CONFIGURATION 2, $r=0.3045$ in., $\text{ANGLE}=60^\circ$

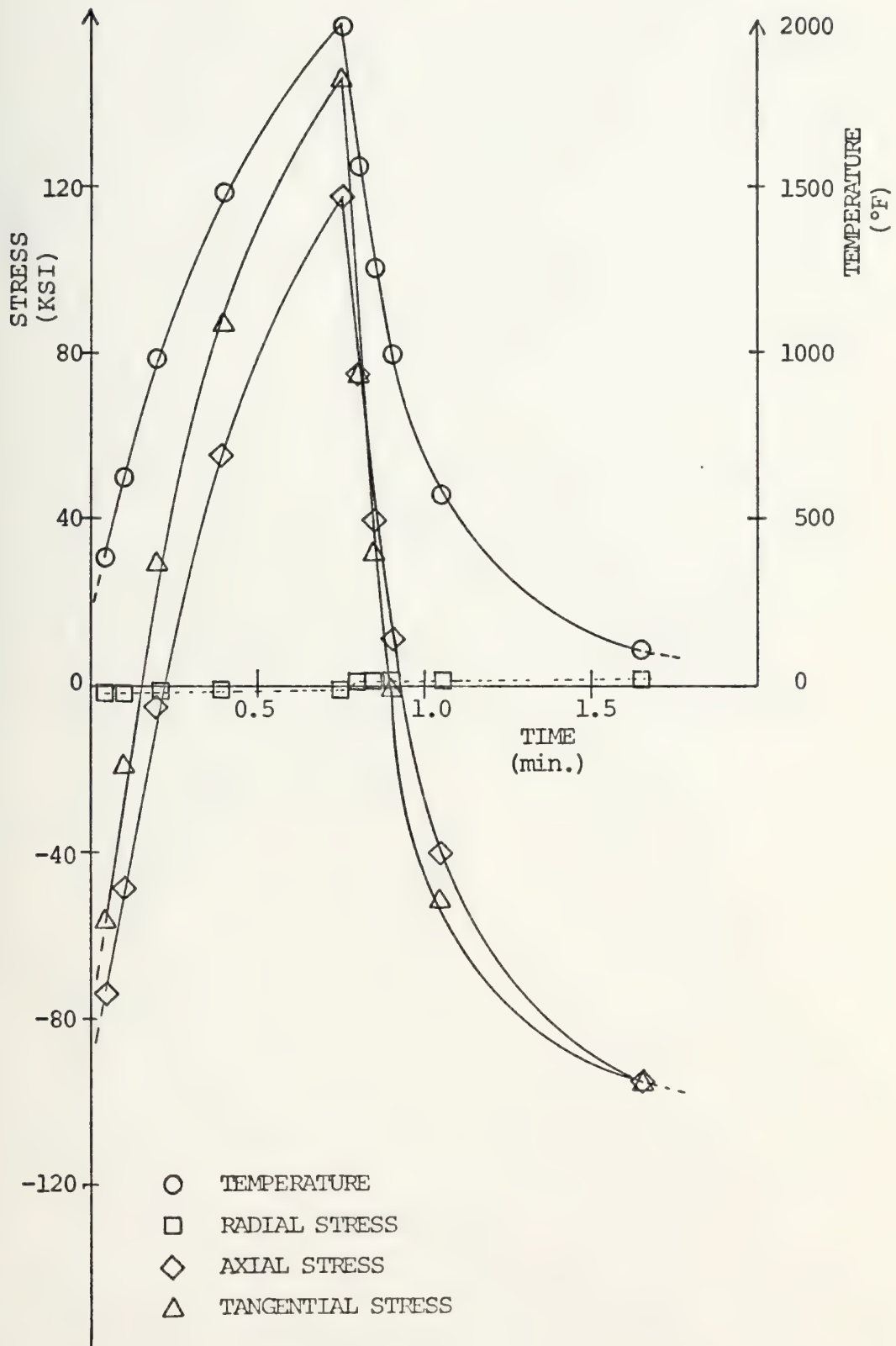


FIGURE 13



MAXIMUM AND MINIMUM PRINCIPAL STRESSES
VERSUS TIME
CONFIGURATION NUMBER TWO

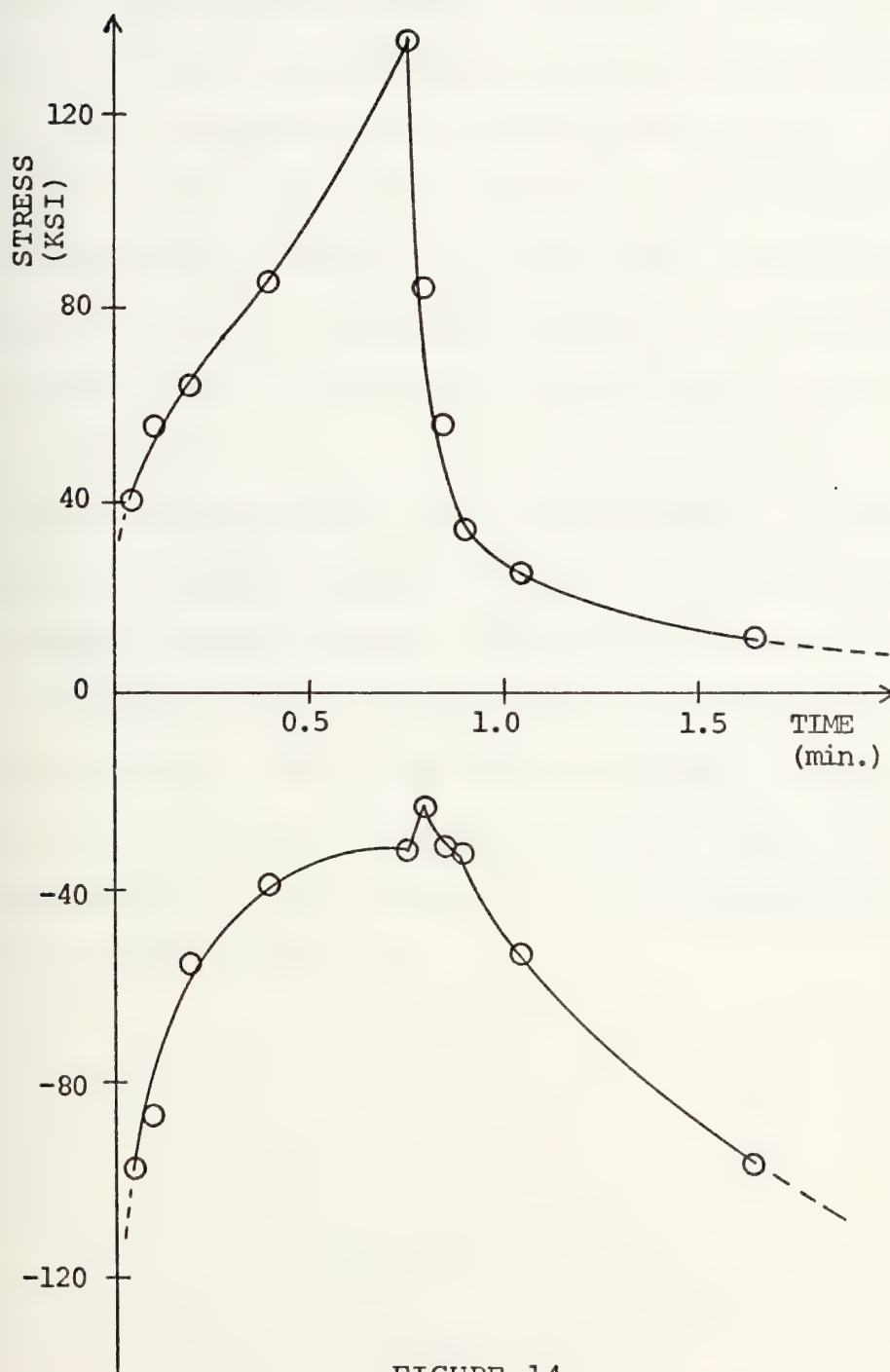


FIGURE 14



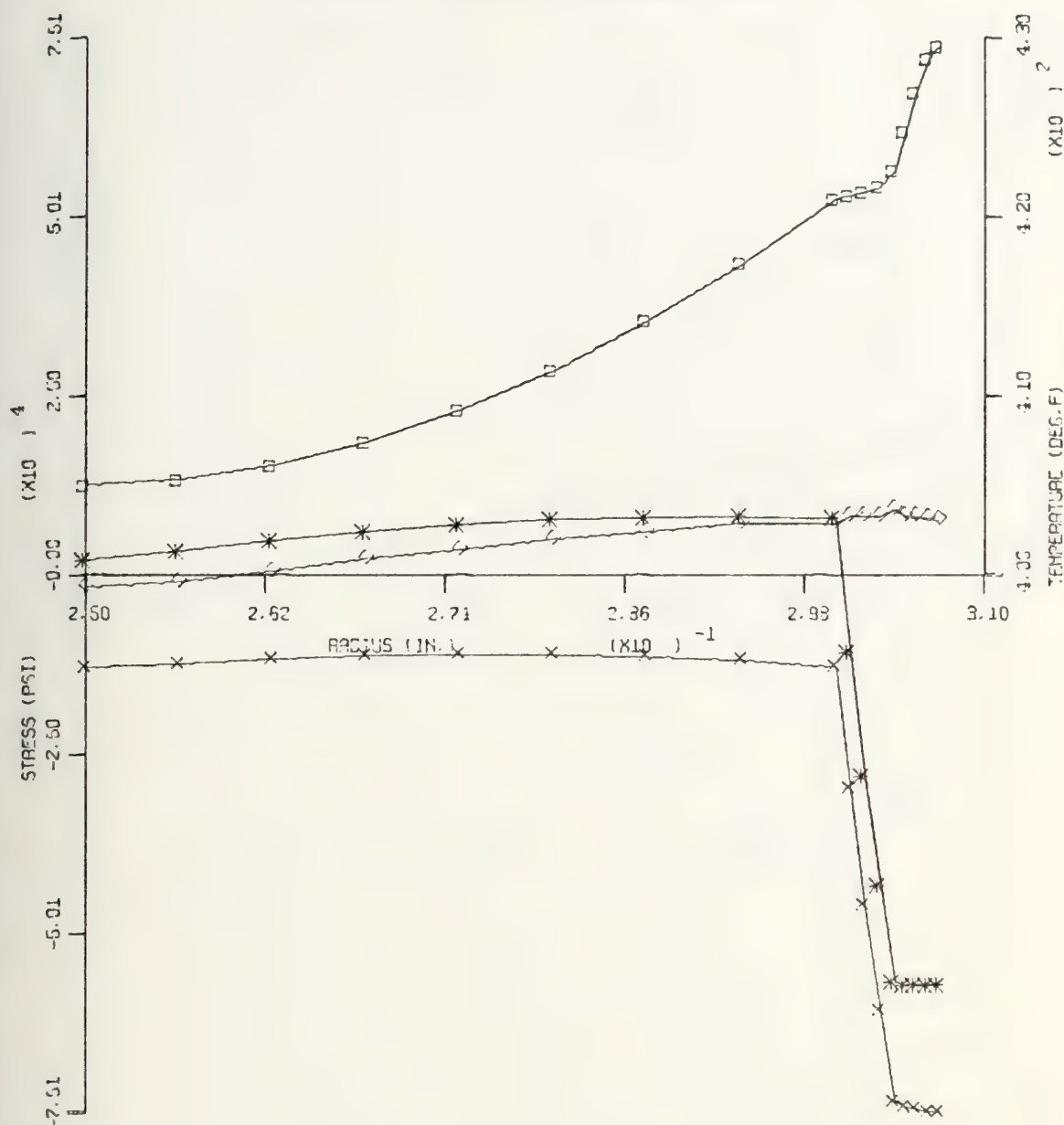
C. CONFIGURATION NUMBER THREE

This arrangement of coatings included a graded layer of nickel and zirconia. The effect of this layer was to eliminate the abrupt change in the stress levels seen in the first two coating arrangements. The maximum principal stress (134.9 KSI) occurred at the maximum temperature. The minimum principal stress (-97.2 KSI) occurred at the beginning and end of the thermal cycle, i.e., the room temperature condition. Since the tangential and axial stress levels are excessive for a major part of the cycle, failure would occur due to these stresses.

As with configuration two, a comparison of stress and temperature distributions as a function of angular position was prepared using data for the cylinder ends at a radius of 0.3055 inches and at a time of 0.75 minutes. The graphical representation of this comparison is shown in Figure 19. The function of stress and temperature versus time for the 60° angular position, at a radius of 0.3005 inches at the cylinder ends is shown in Figure 20.



STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.3-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 2.503×10^4 PSI/INCH

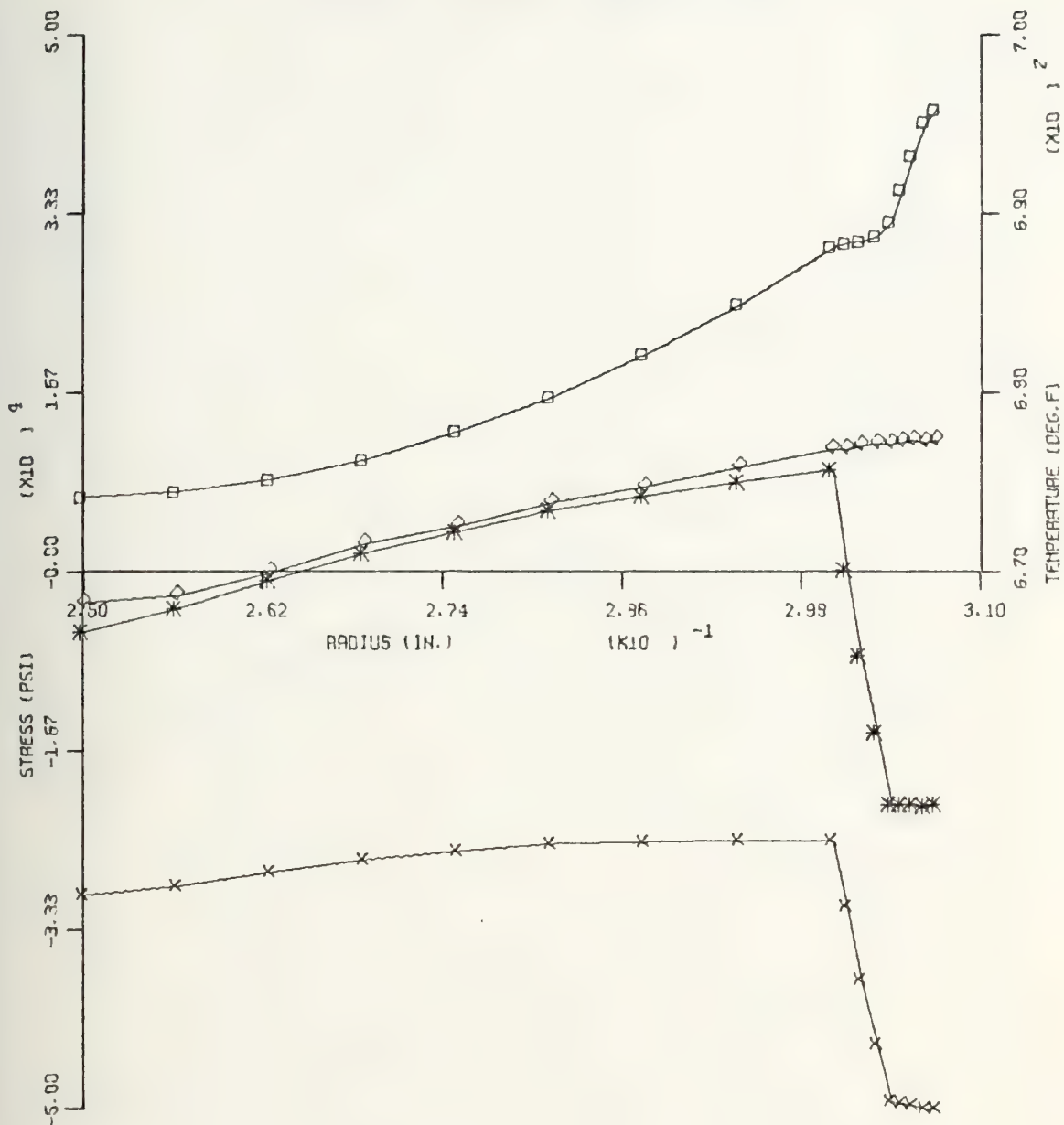
TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 15(a)



STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.3-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 1.565×10^4 PSI/INCH

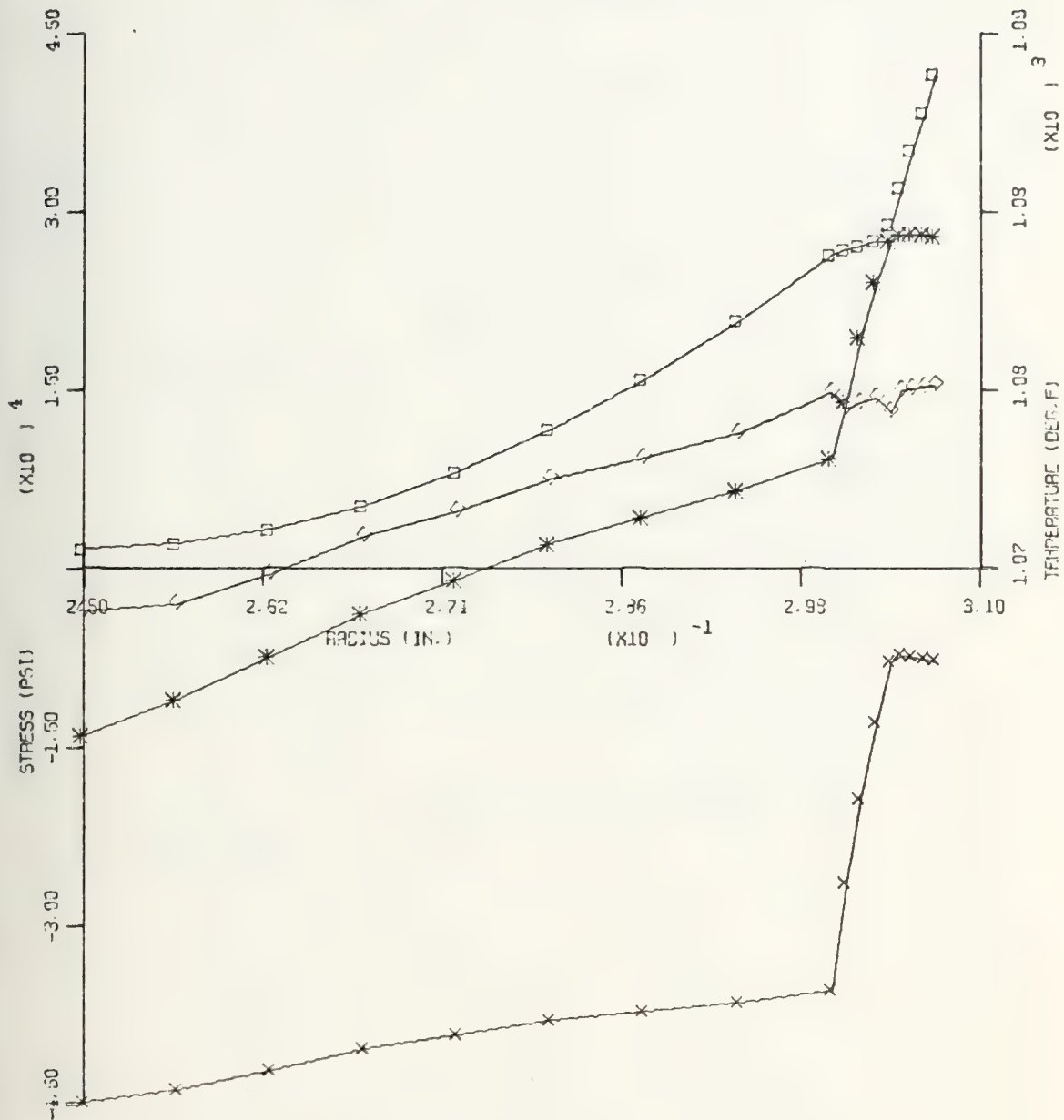
TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 15(b)



STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.3-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

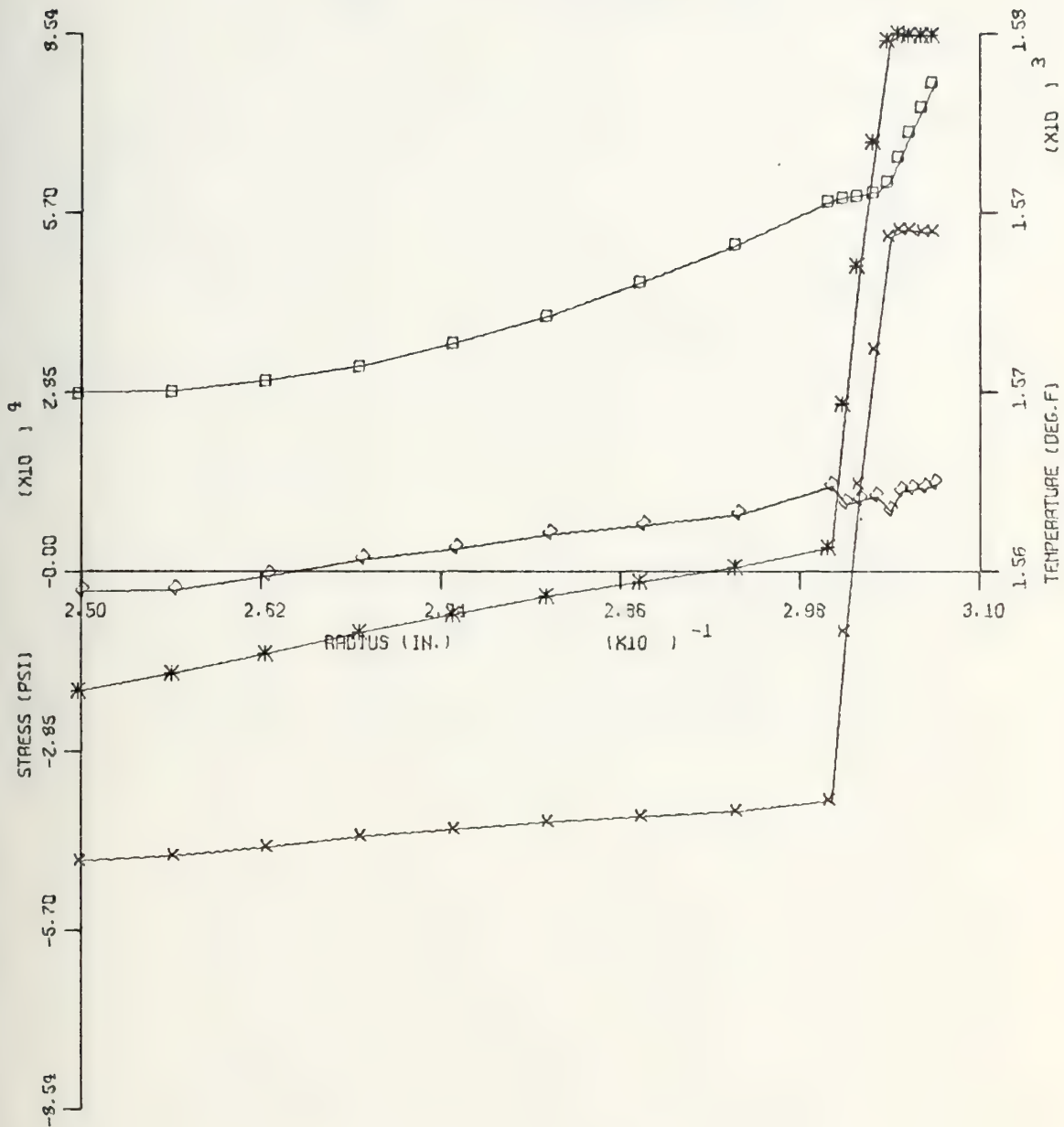
STRESS SCALE = 1.500×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 15(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.3-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

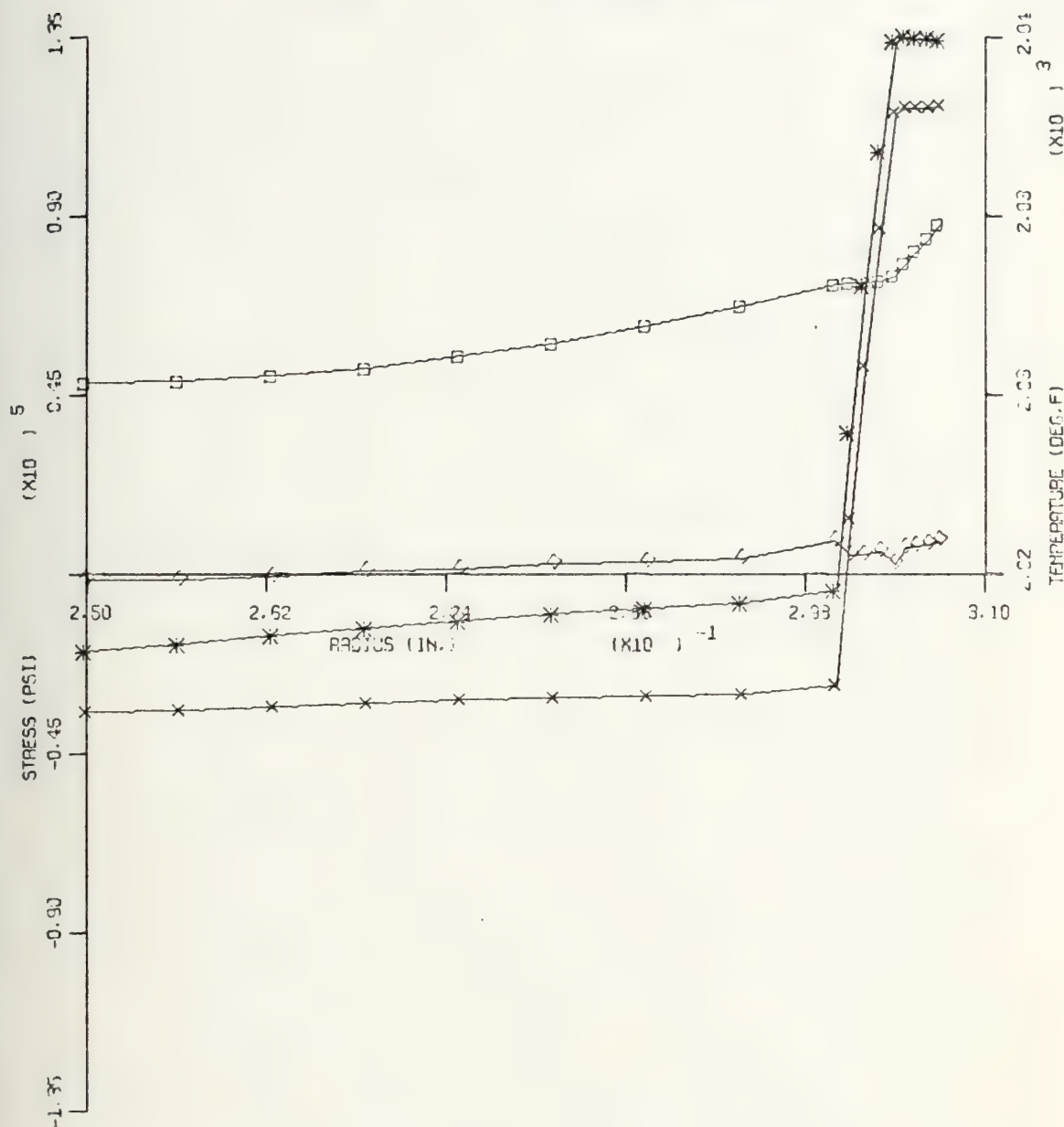
STRESS SCALE = 2.543×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 15(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.3-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

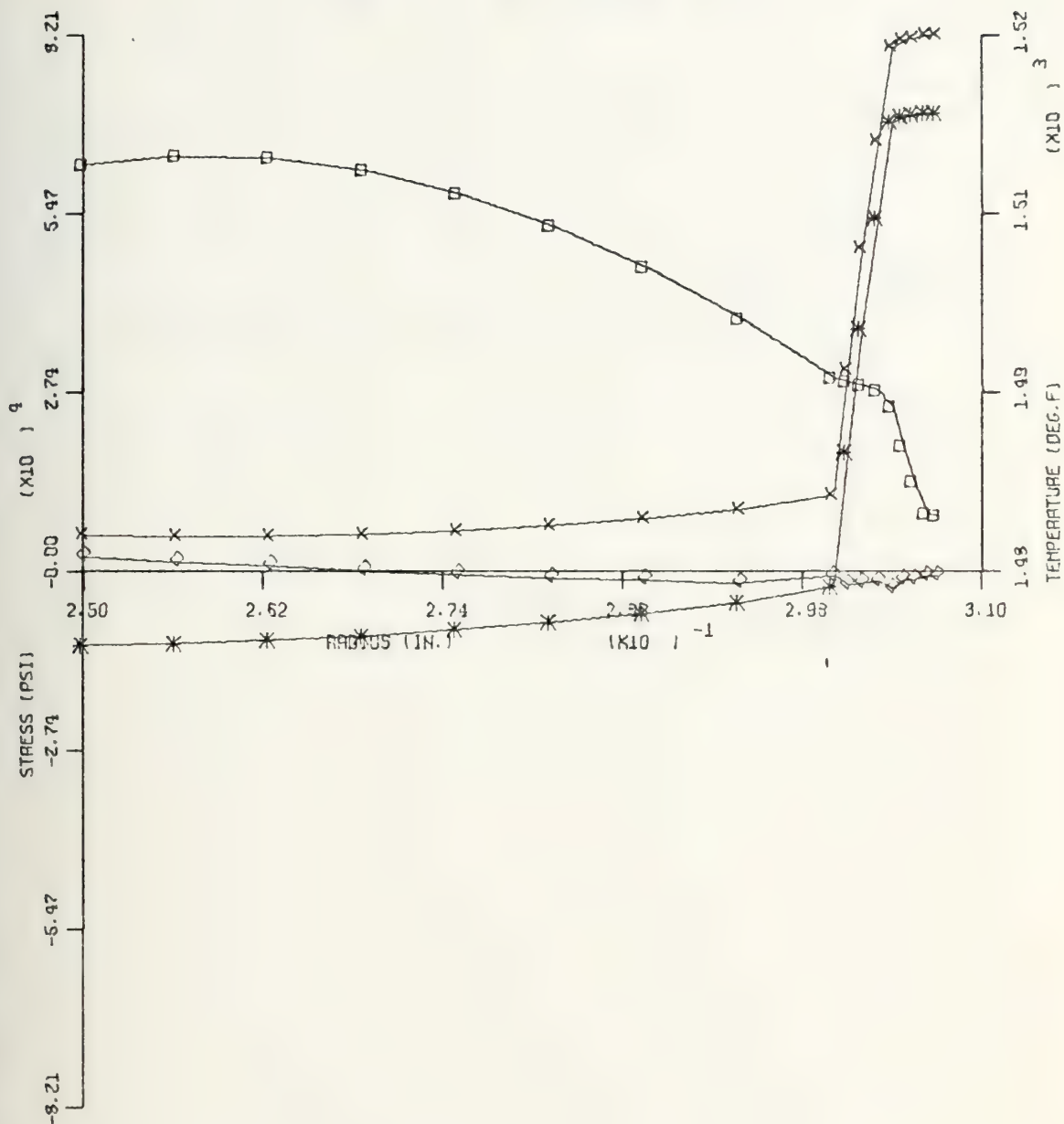
STRESS SCALE - 4.509×10^4 PSI/INCH

TEMPERATURE SCALE - 6.7 DEGREES/INCH

RADIAL POSITION SCALE - 0.01 INCHES/INCH

FIGURE 15(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.3-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

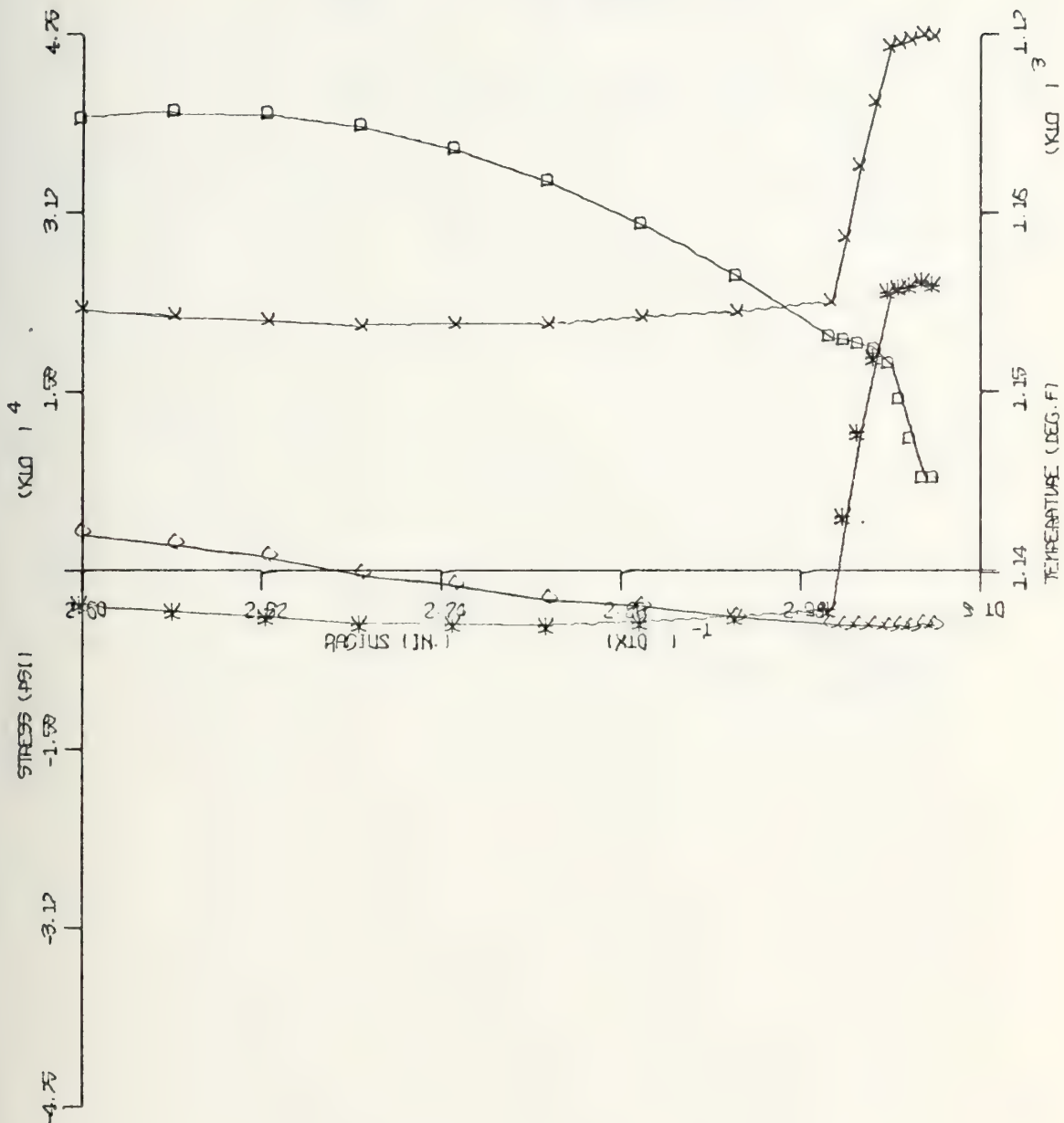
STRESS SCALE = 2.737×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 15(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.3-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

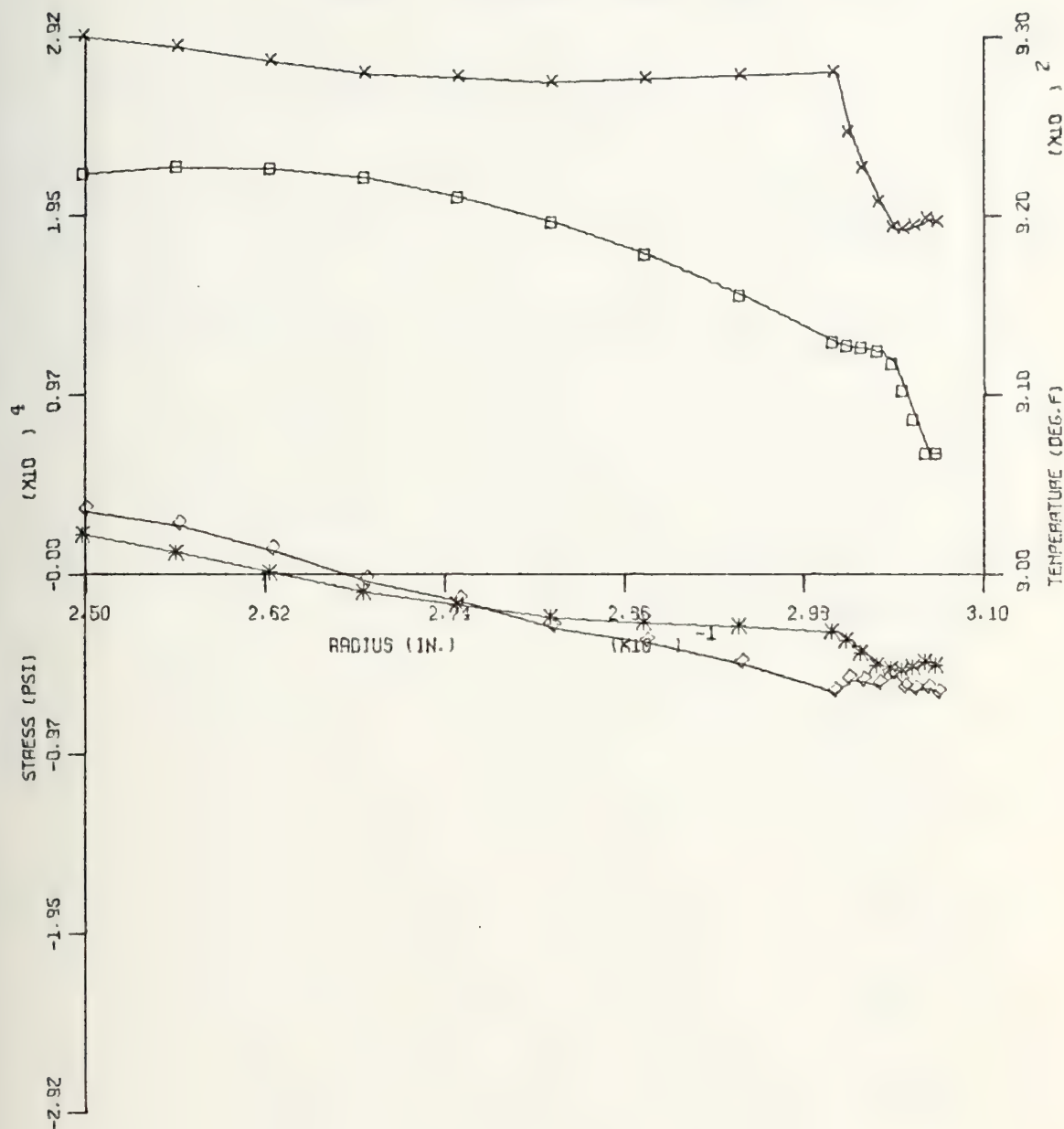
STRESS SCALE = 1.505×10^8 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 15 (g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.3-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

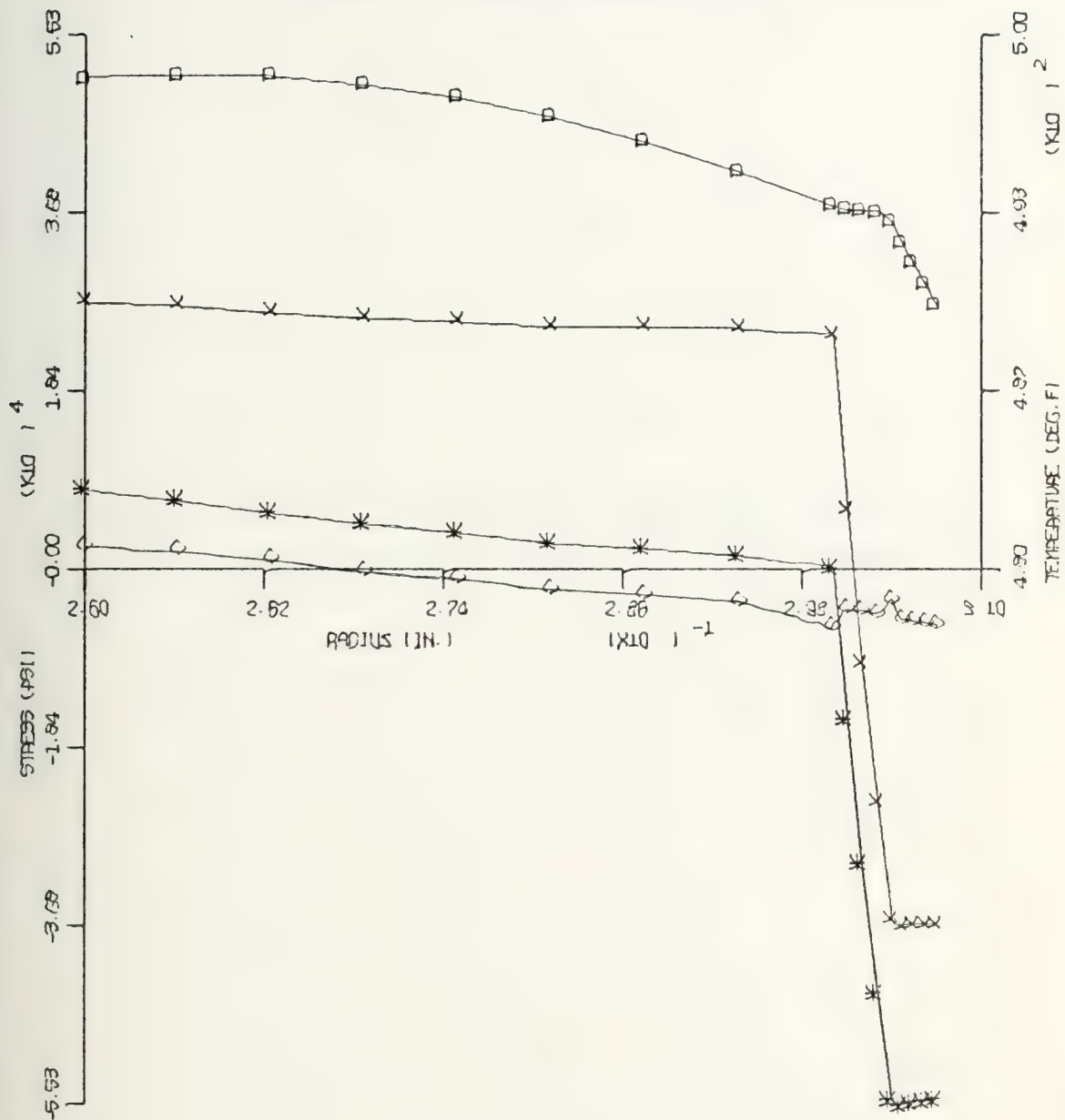
STRESS SCALE = 9.747×10^3 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 15(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.3-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

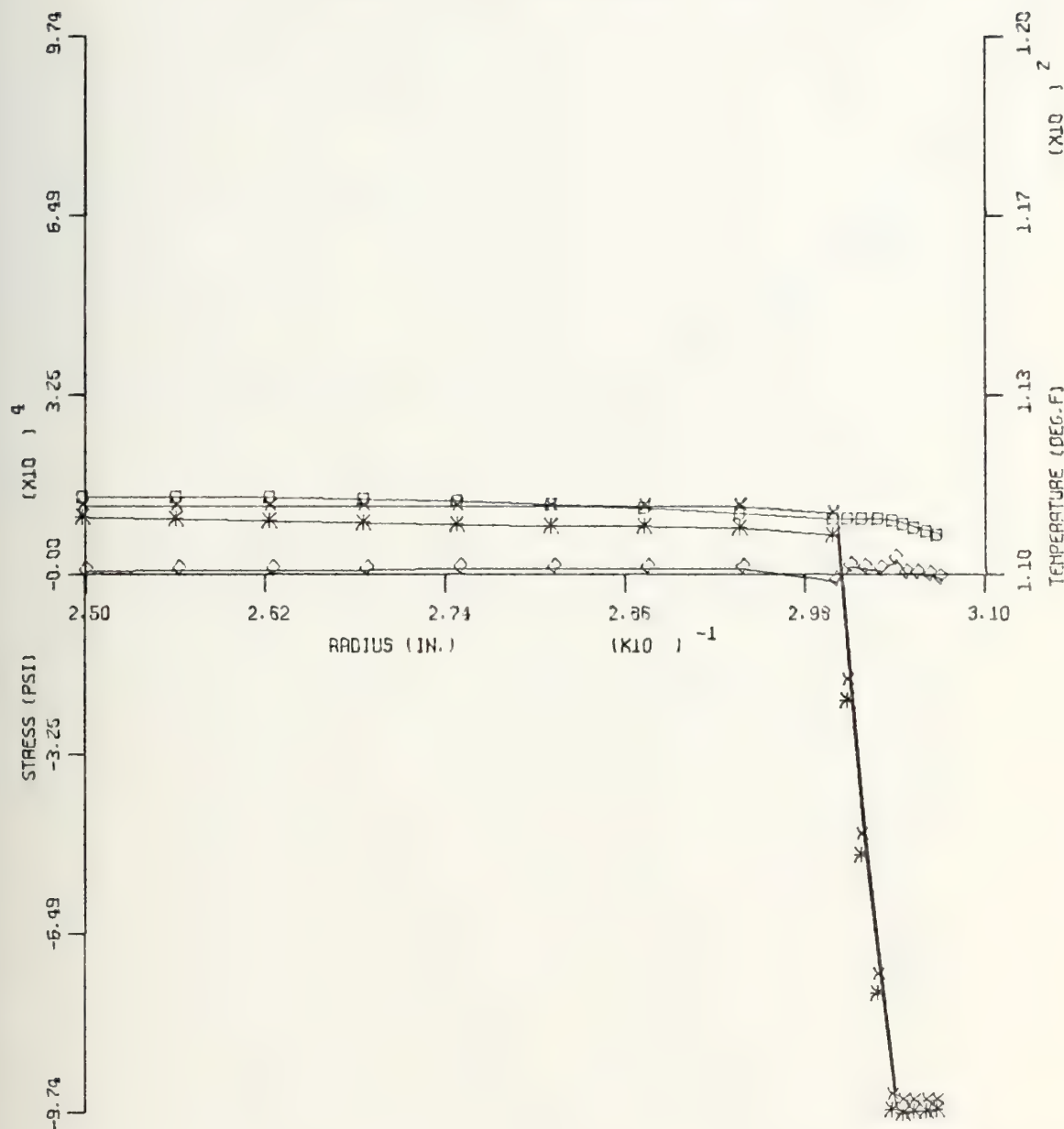
STRESS SCALE = 1.842×10^3 PSI/INCH

TEMPERATURE SCALE = 57 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 15(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.3-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

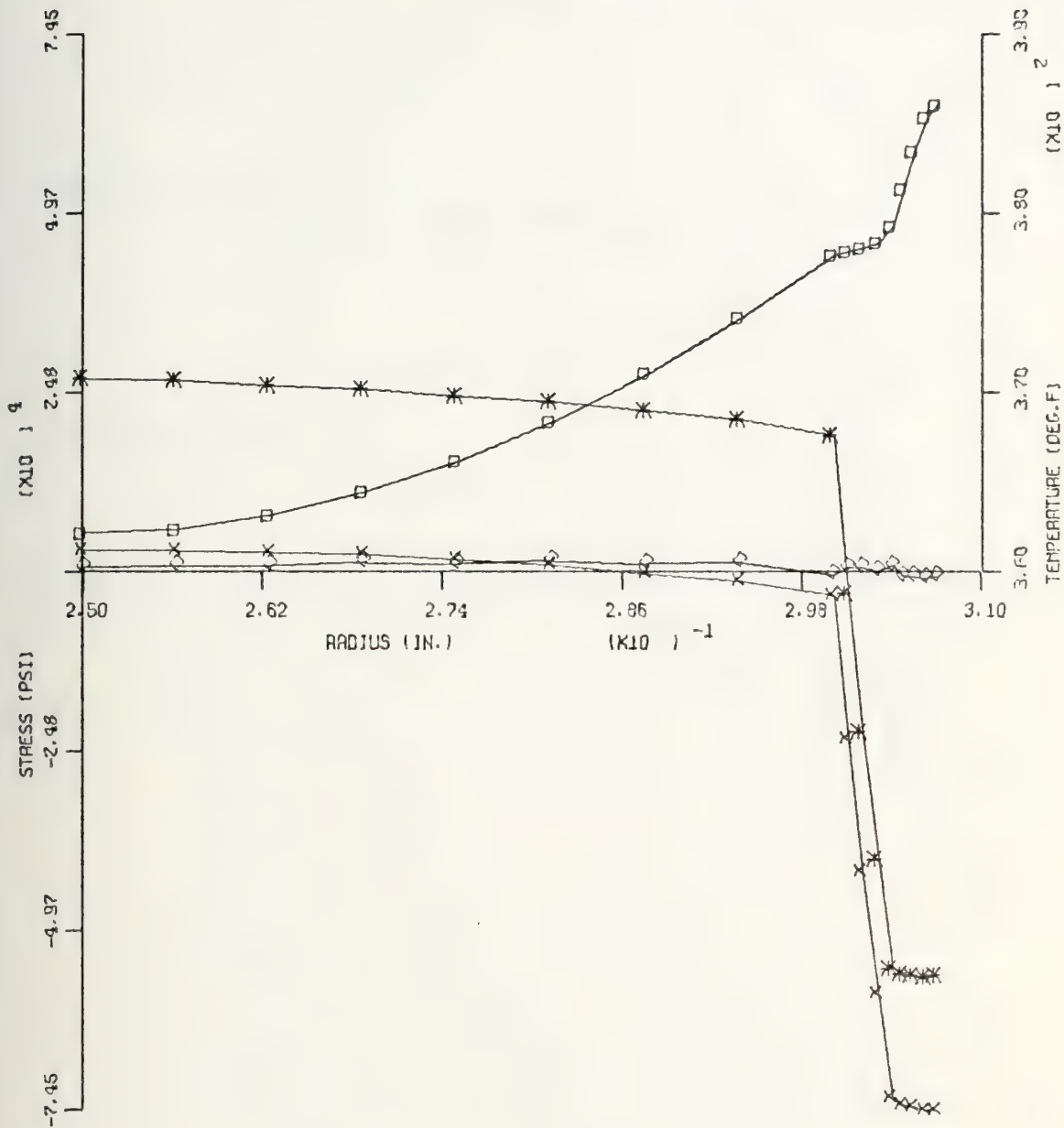
STRESS SCALE = 3.247×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 15(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.3-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

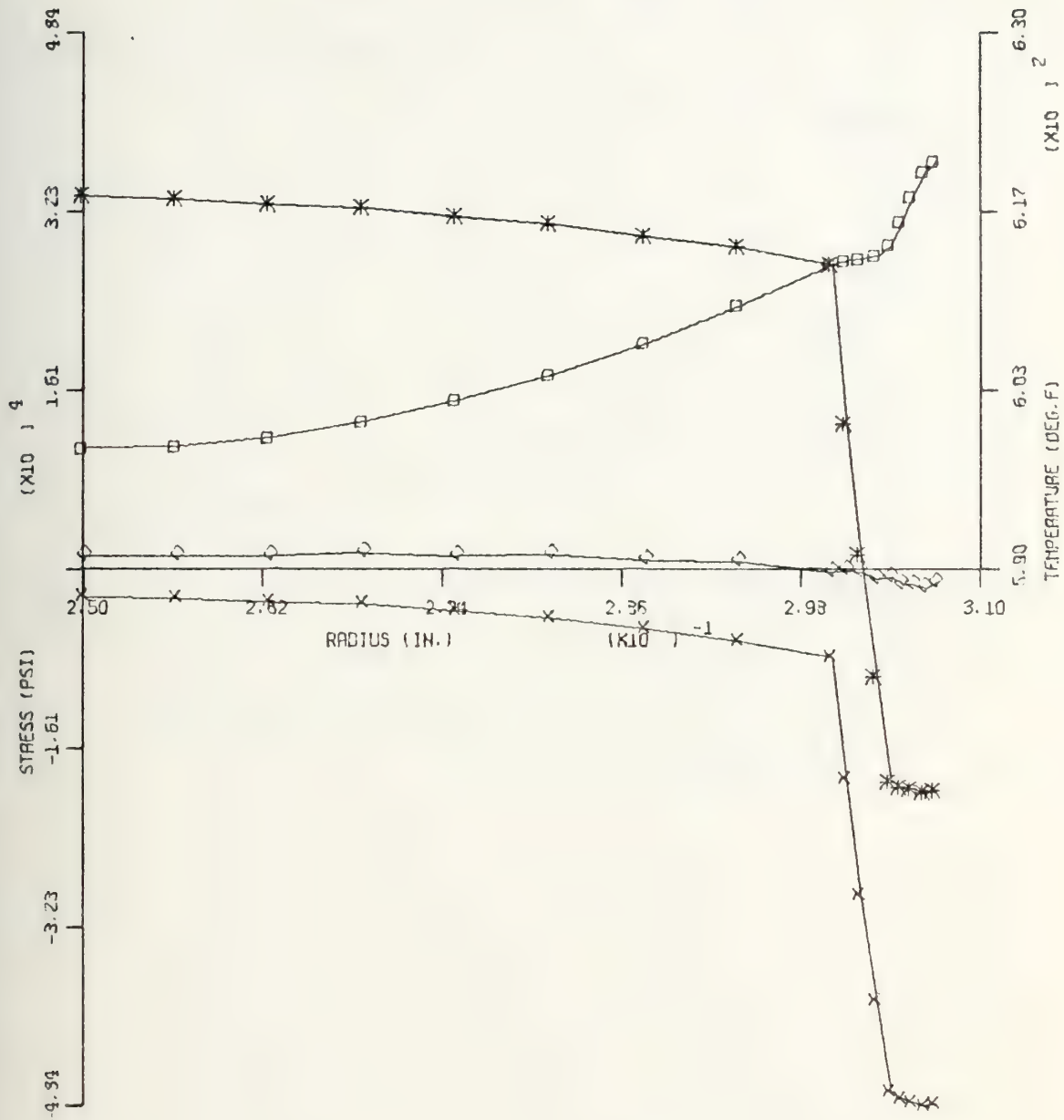
STRESS SCALE = 2.493×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 16(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.3-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

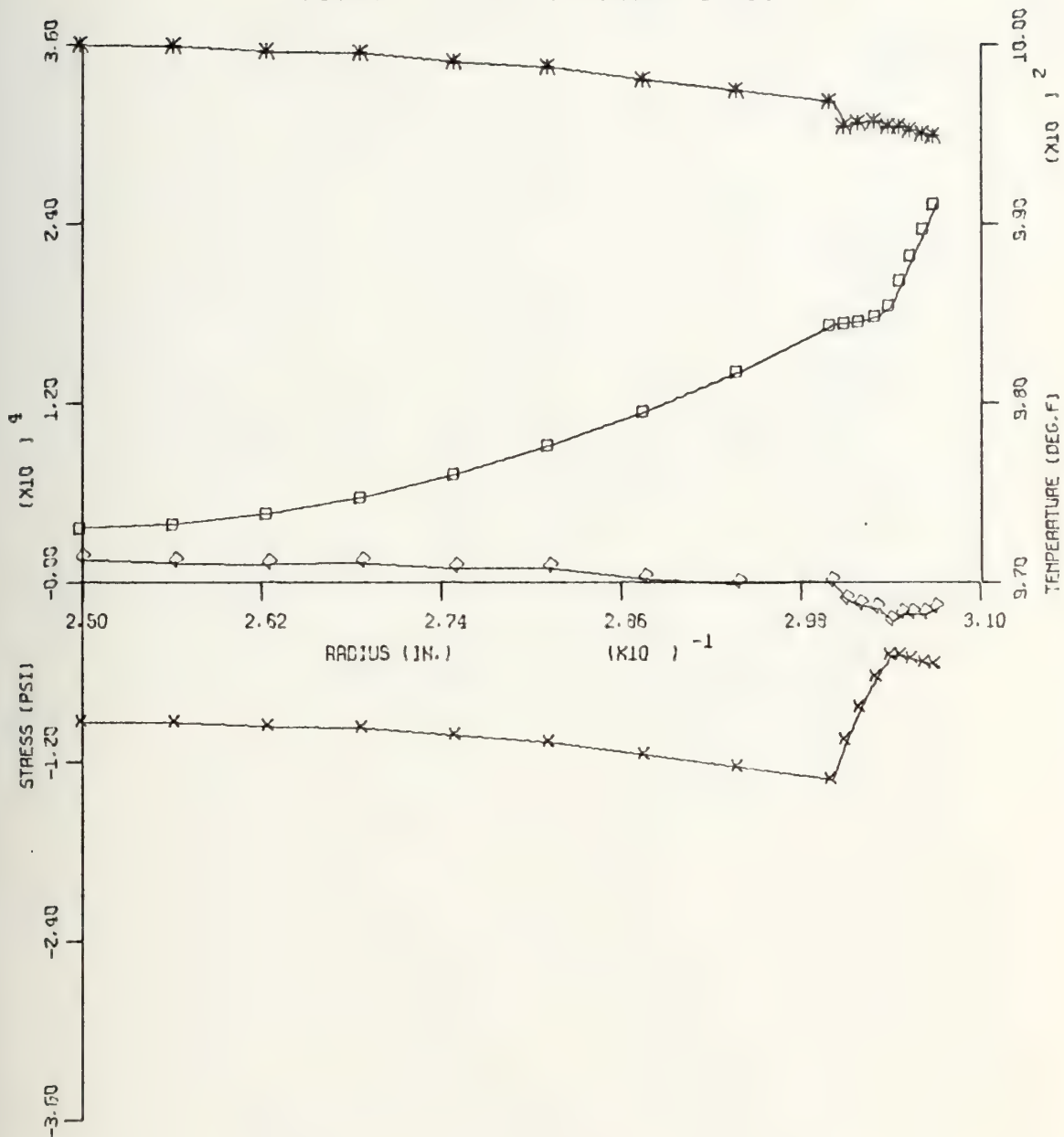
STRESS SCALE = 1.613×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 16(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.3-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

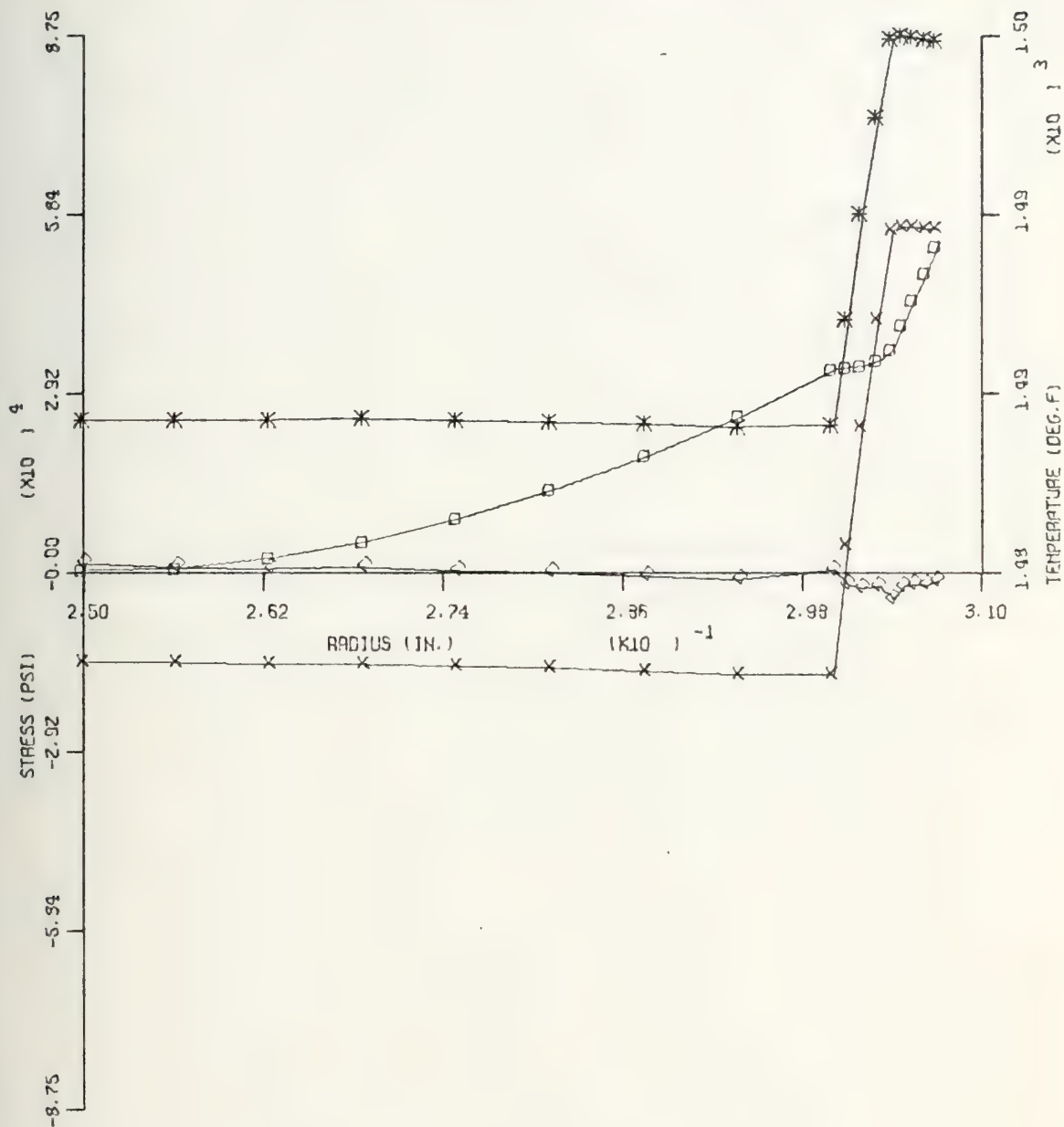
STRESS SCALE = 1.203×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 16(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.3-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

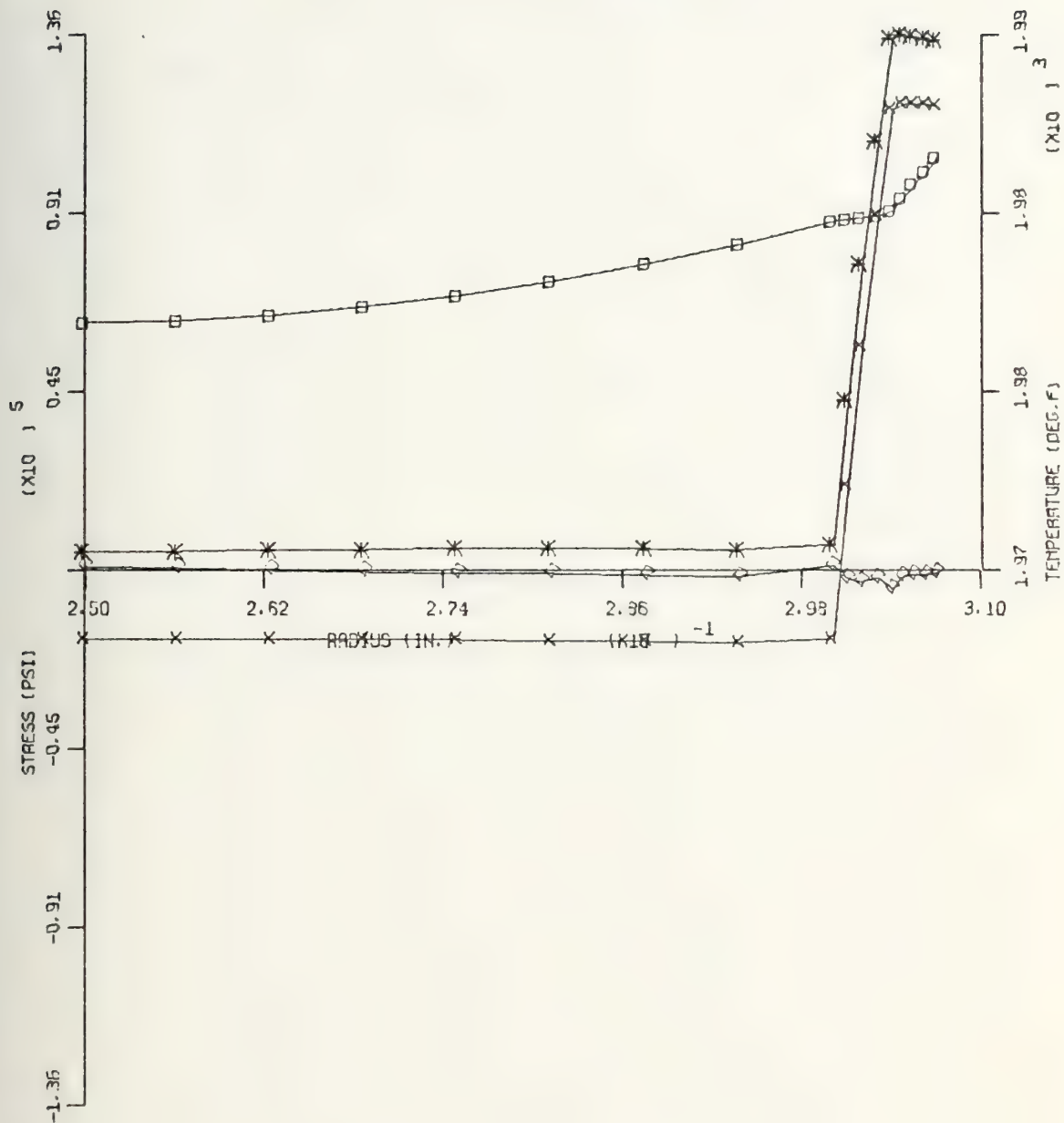
STRESS SCALE = 2.918×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 16(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.3-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

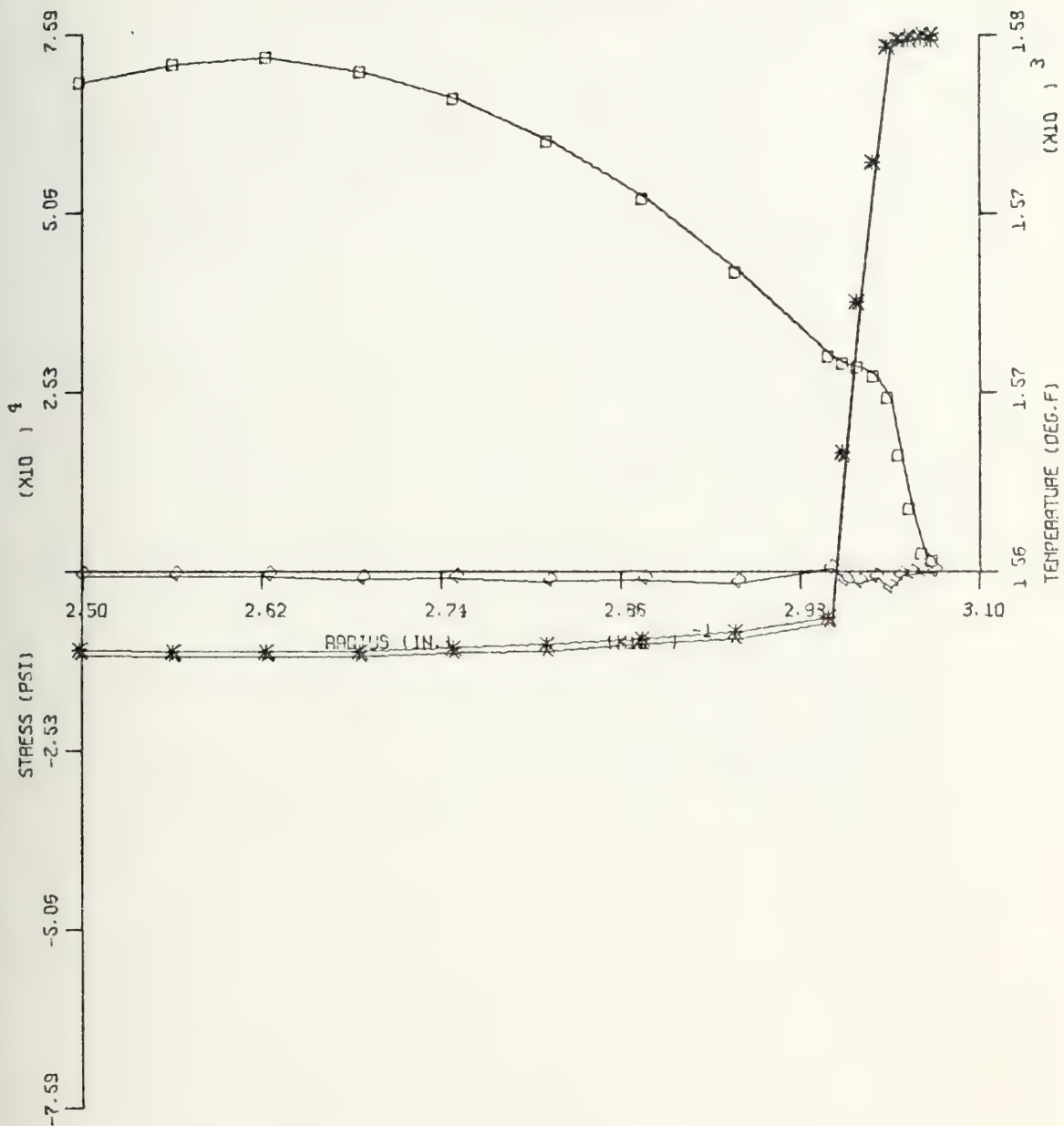
STRESS SCALE = 4.542×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 16(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.3-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

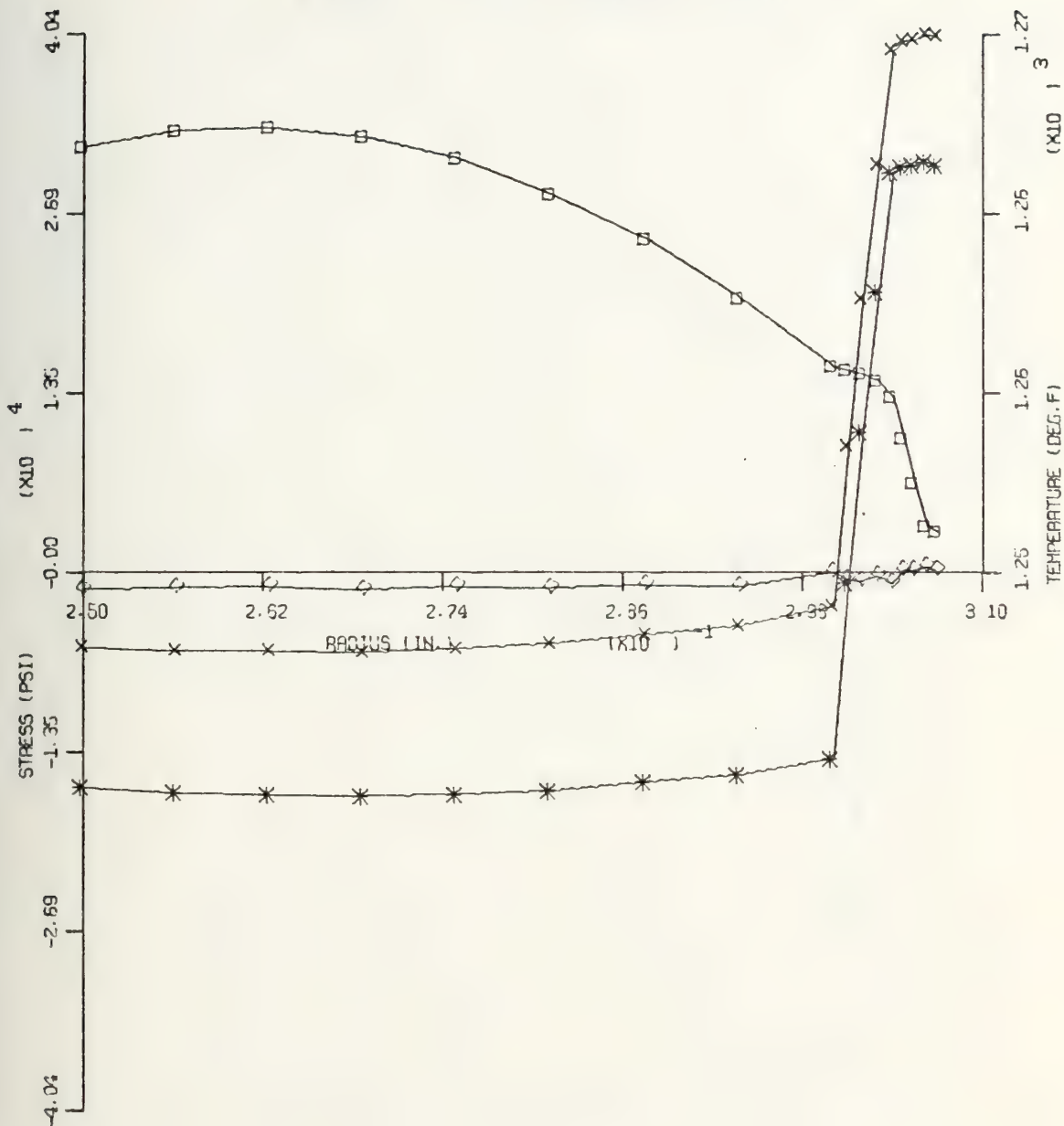
STRESS SCALE = 2.529×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 16(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.3-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

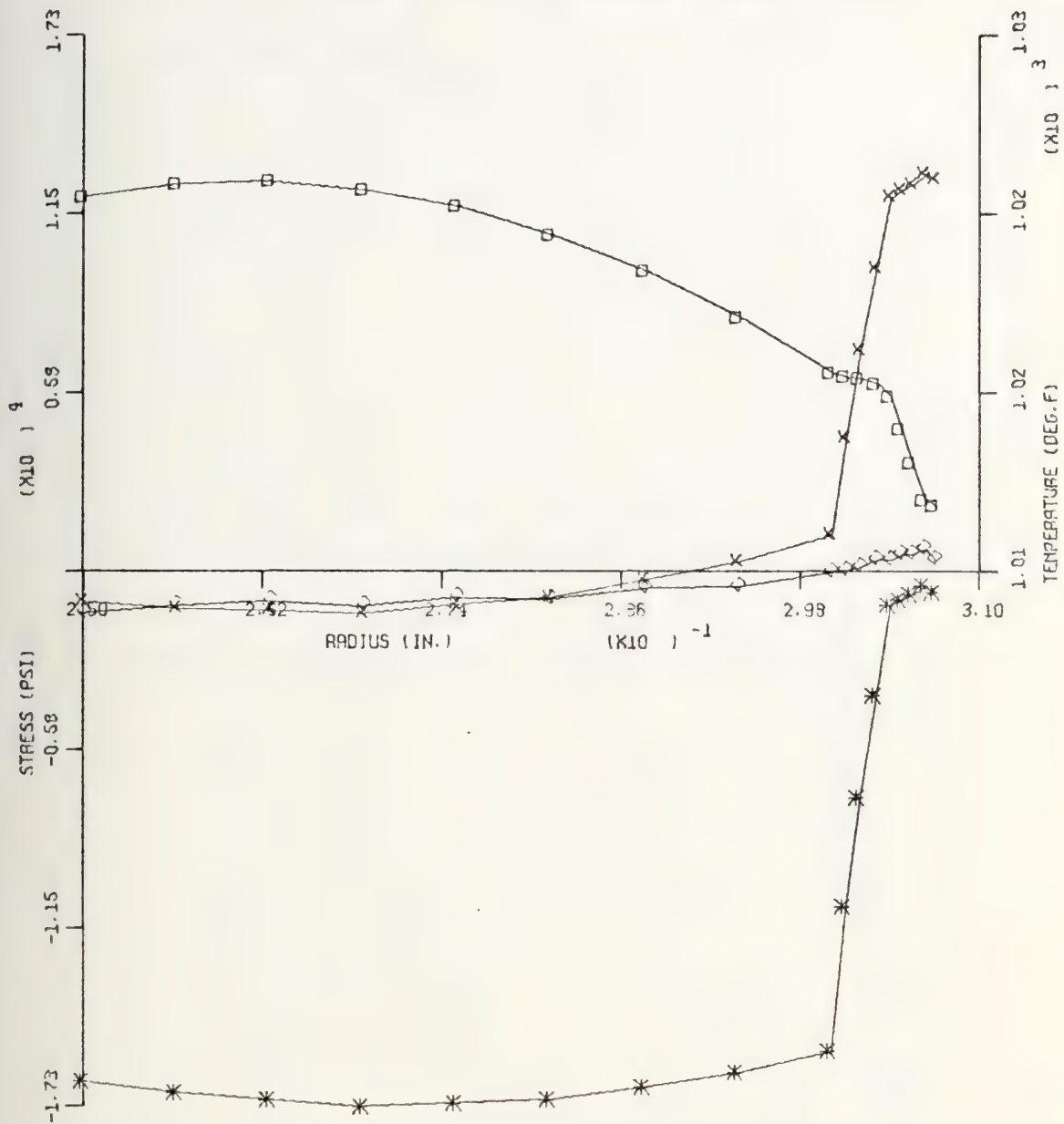
STRESS SCALE = 1.345×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 16 (g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.3-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

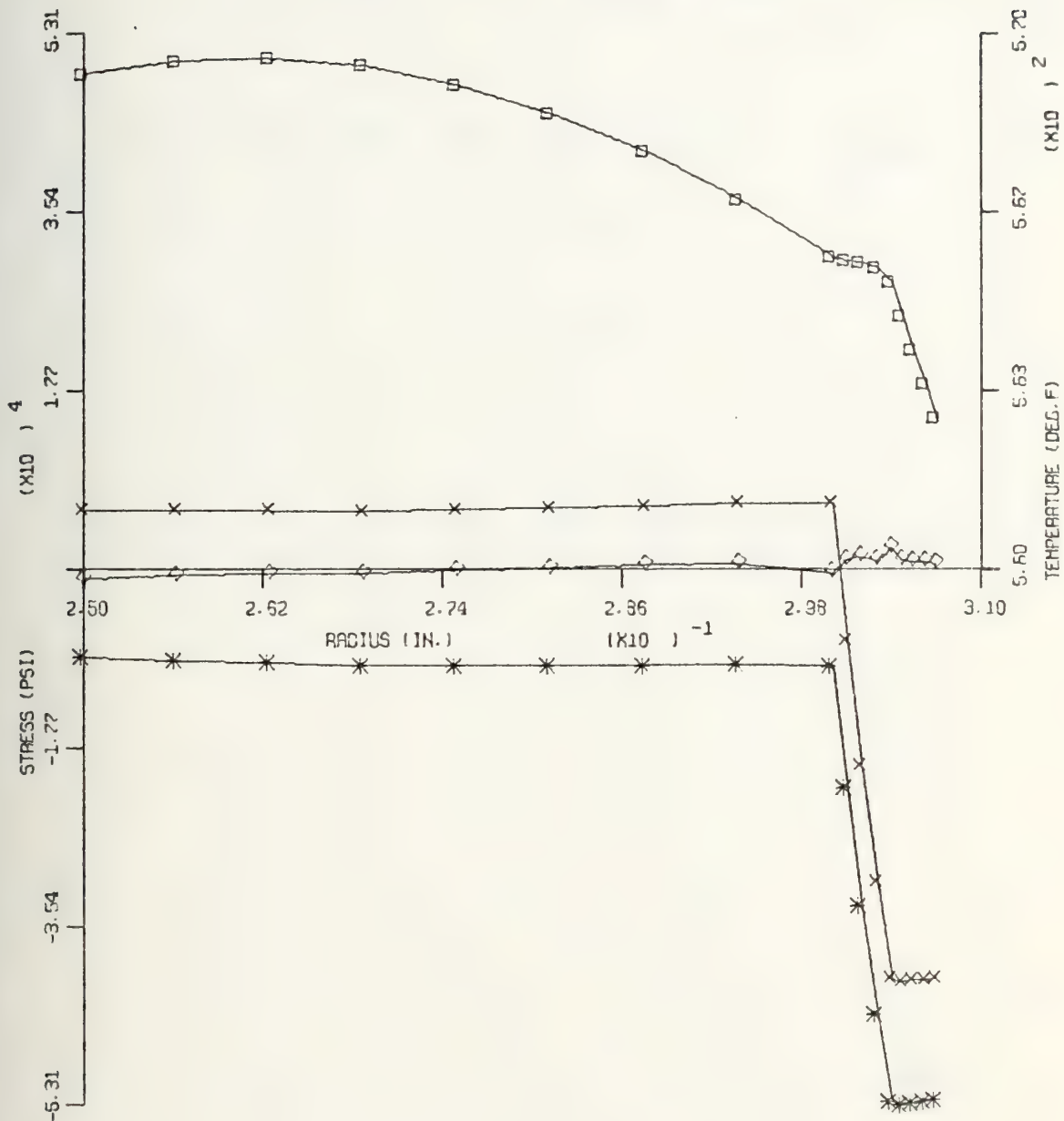
STRESS SCALE = 5.772×10^3 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 16(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.3-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

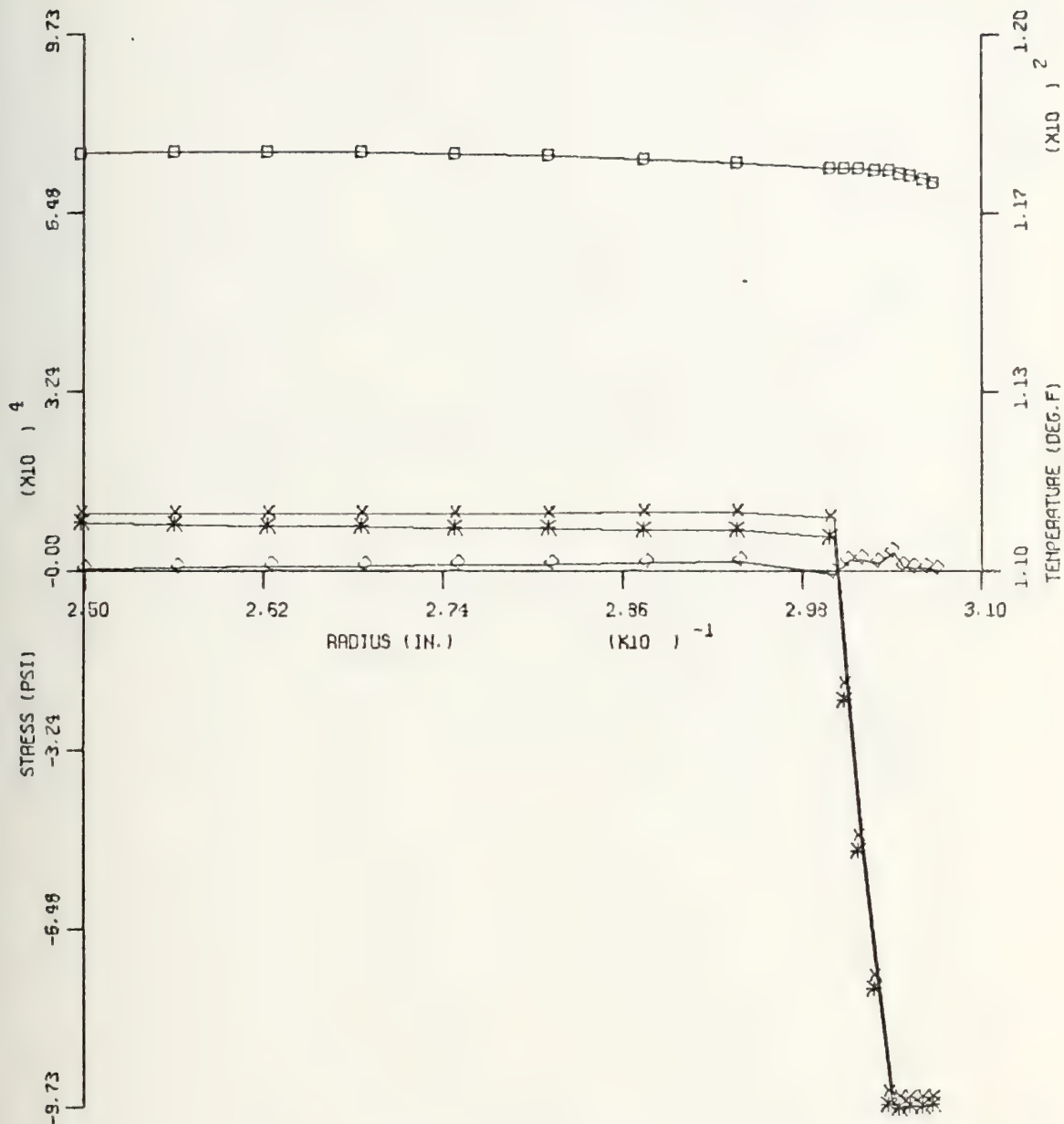
STRESS SCALE = 1.269×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 16(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.3-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

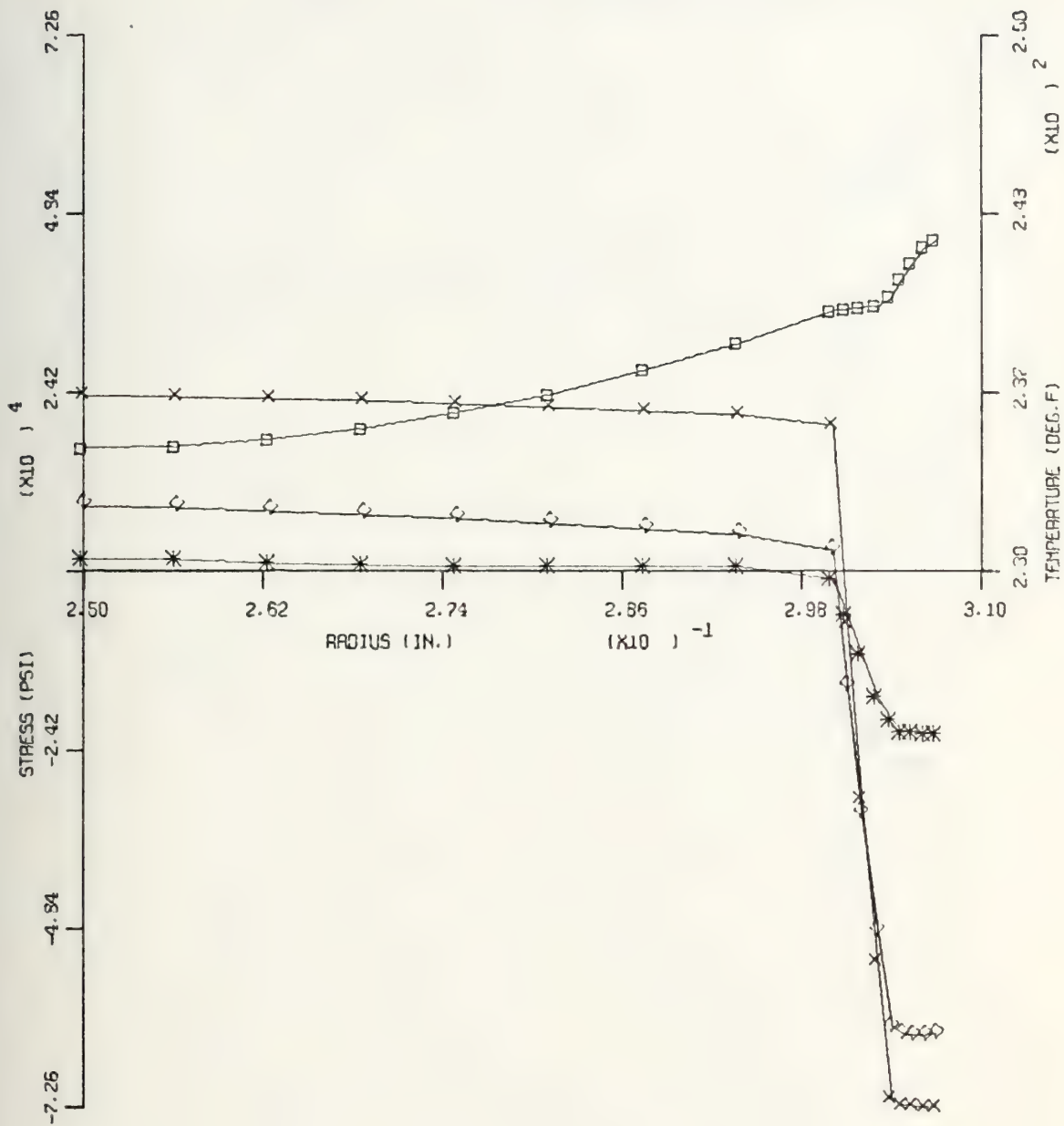
STRESS SCALE = 3.242×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 16(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.3-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

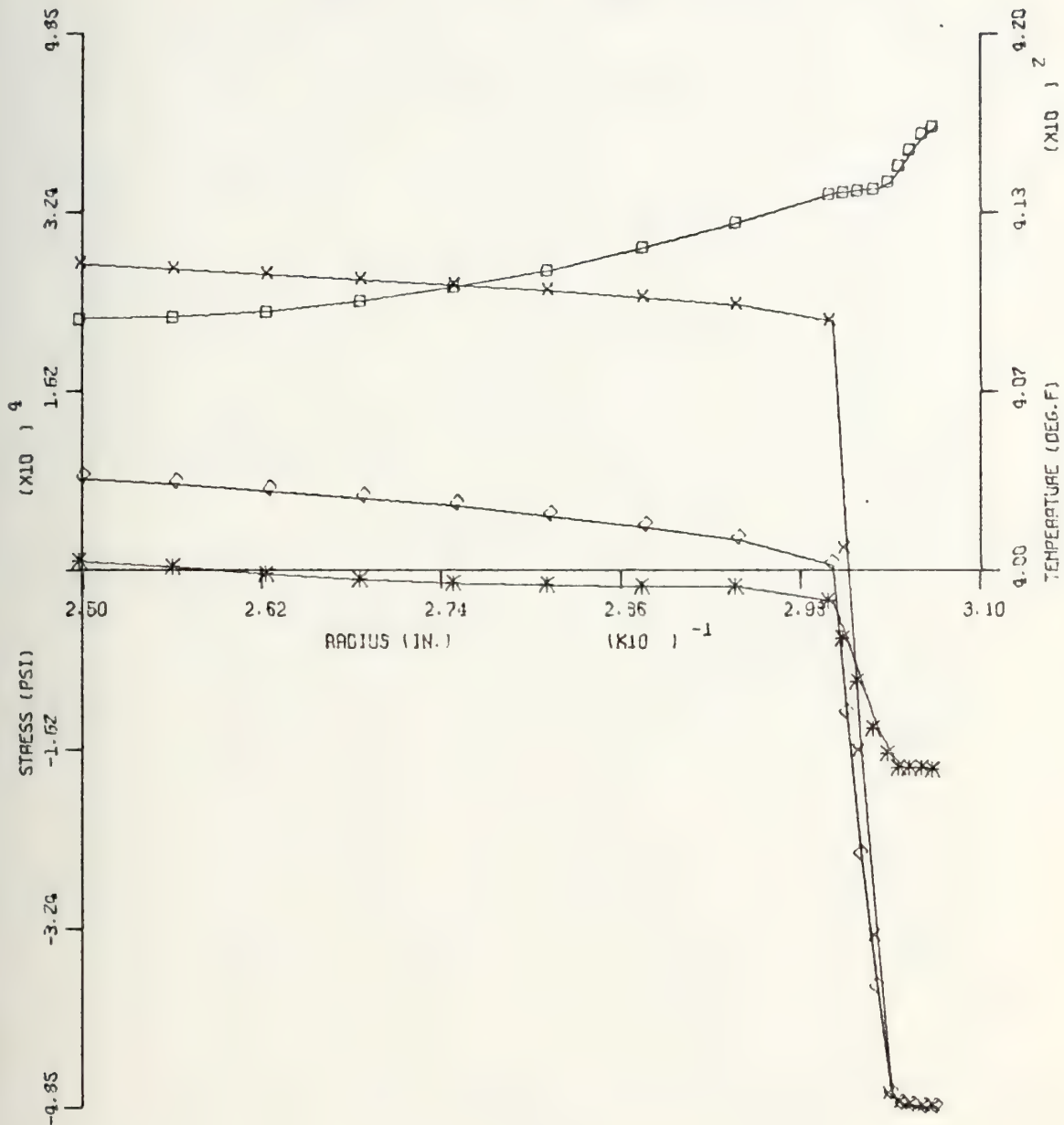
STRESS SCALE = 2.421×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 17(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.3-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

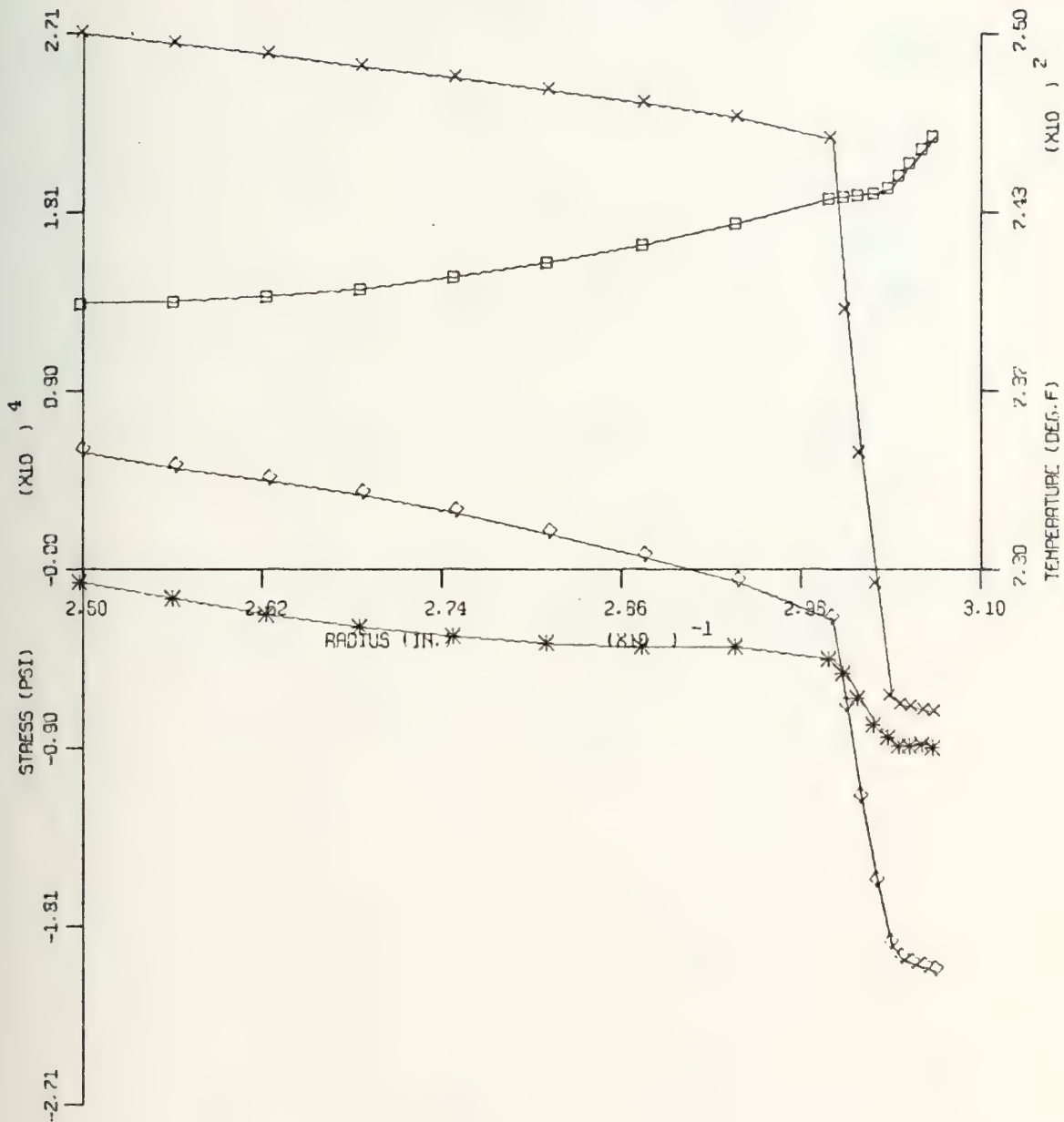
STRESS SCALE = 1.518×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 17(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.3-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 9.033×10^3 PSI/INCH

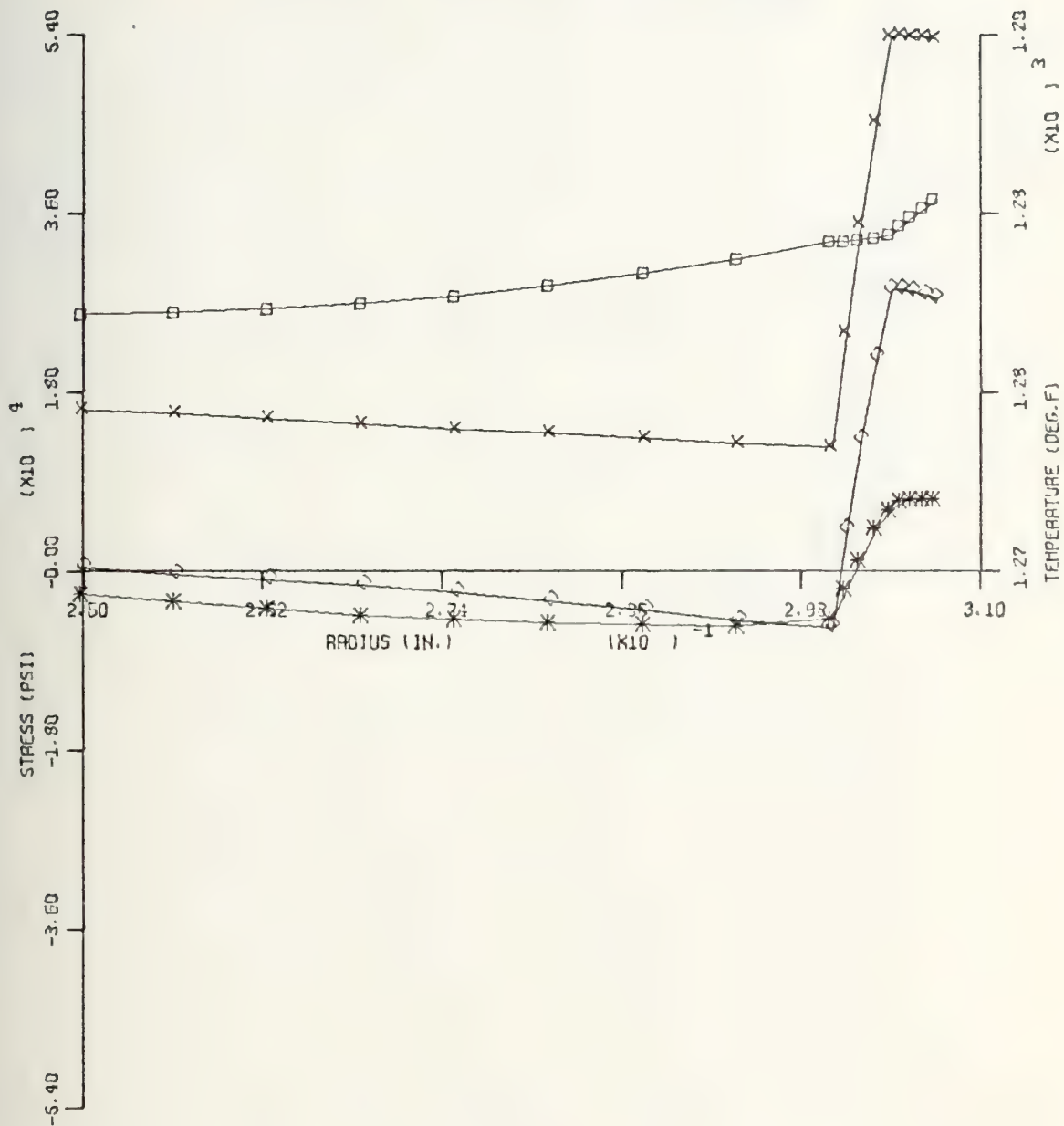
TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 17(c)



STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.3-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

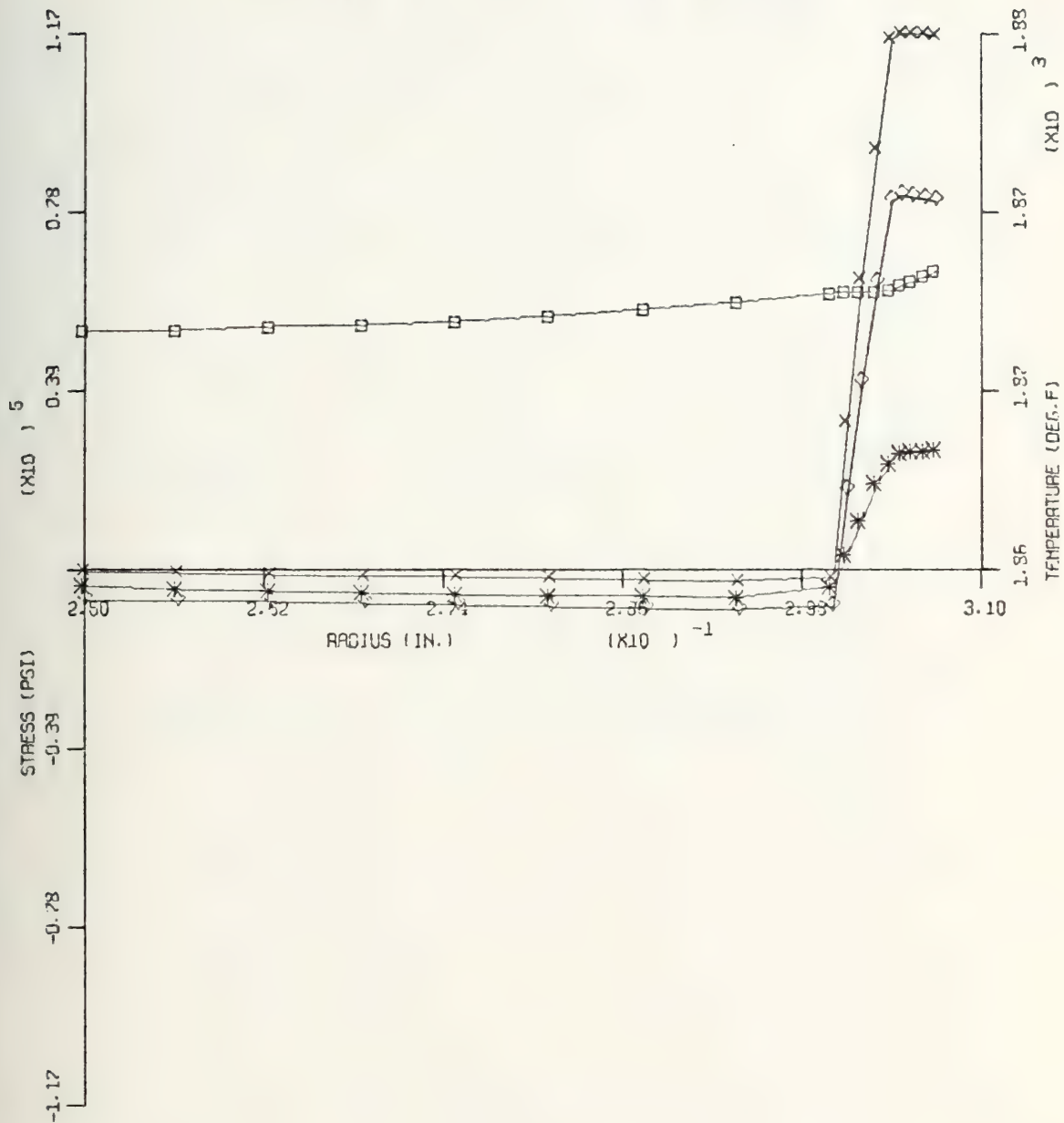
STRESS SCALE = 1.793×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 17(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.3-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

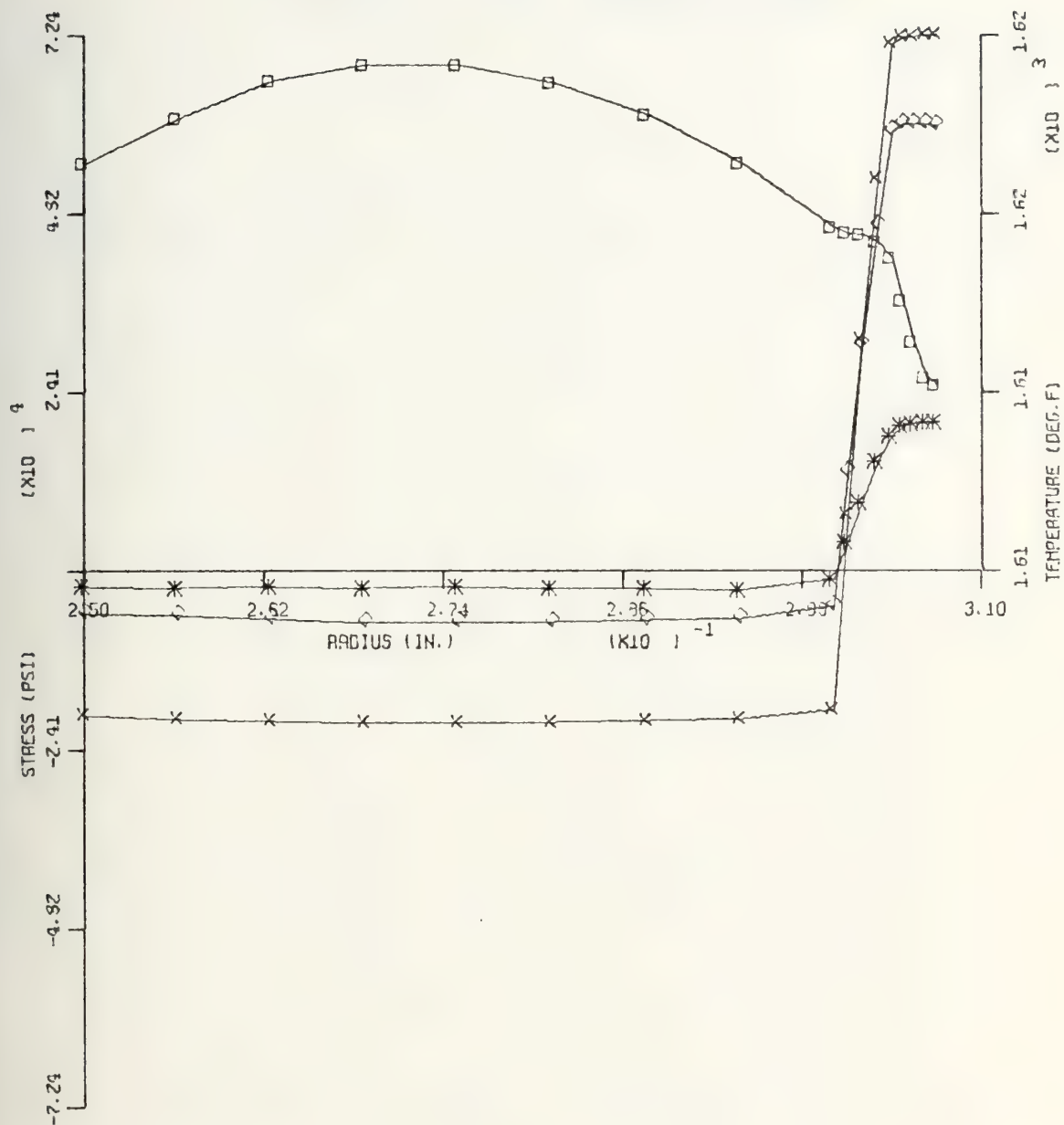
STRESS SCALE = 3.909×10^9 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 17(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.3-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

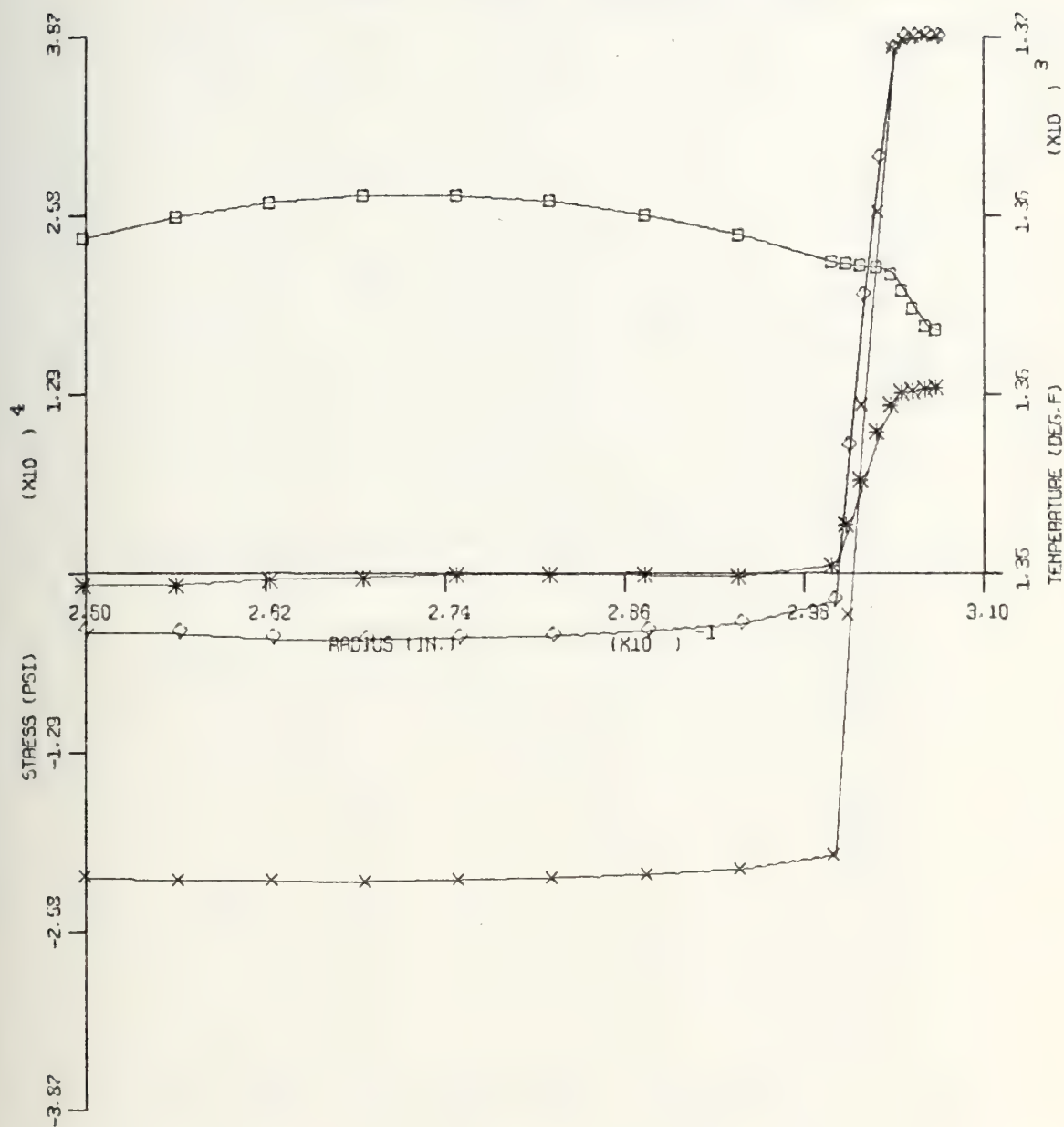
STRESS SCALE = 2.412×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 17(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.3-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 1.230×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 17(g)



TIME = 0.900 CONF.3-120



SIGMA Z - CROSS TEMPERATURE - SQUARE

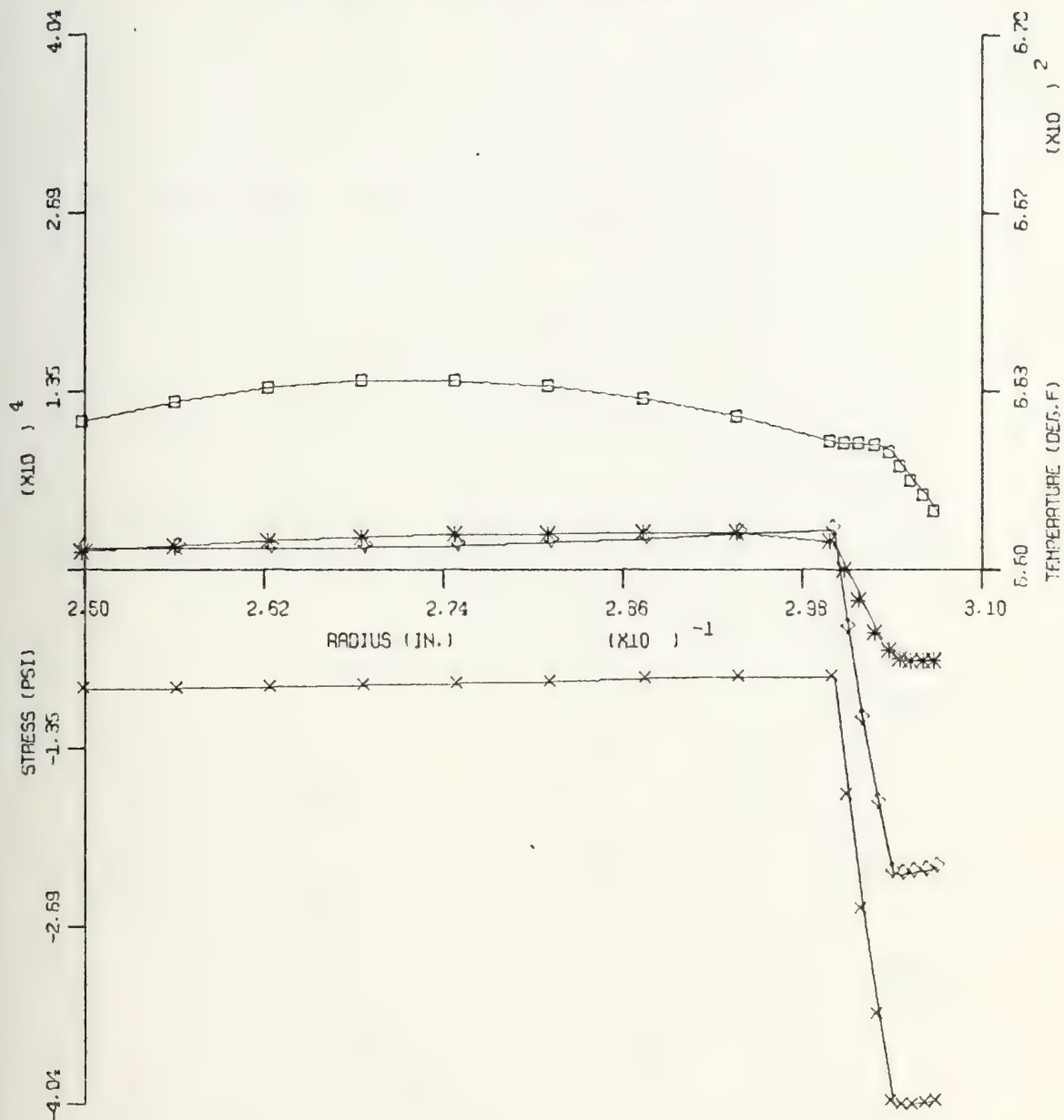
STRESS SCALE = 5.52×10^3 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 17(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.3-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

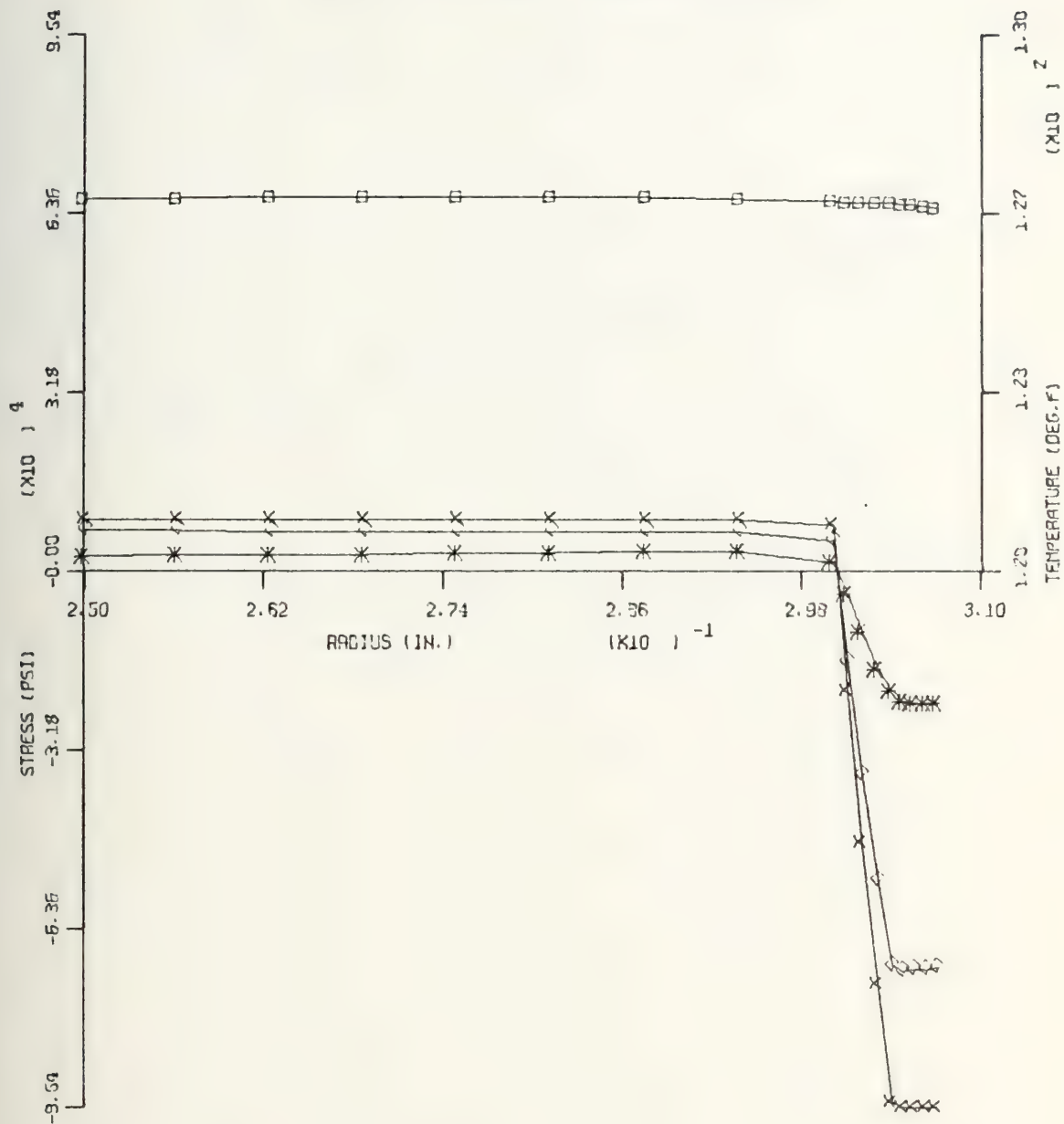
STRESS SCALE = 1.345×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 17(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.3-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

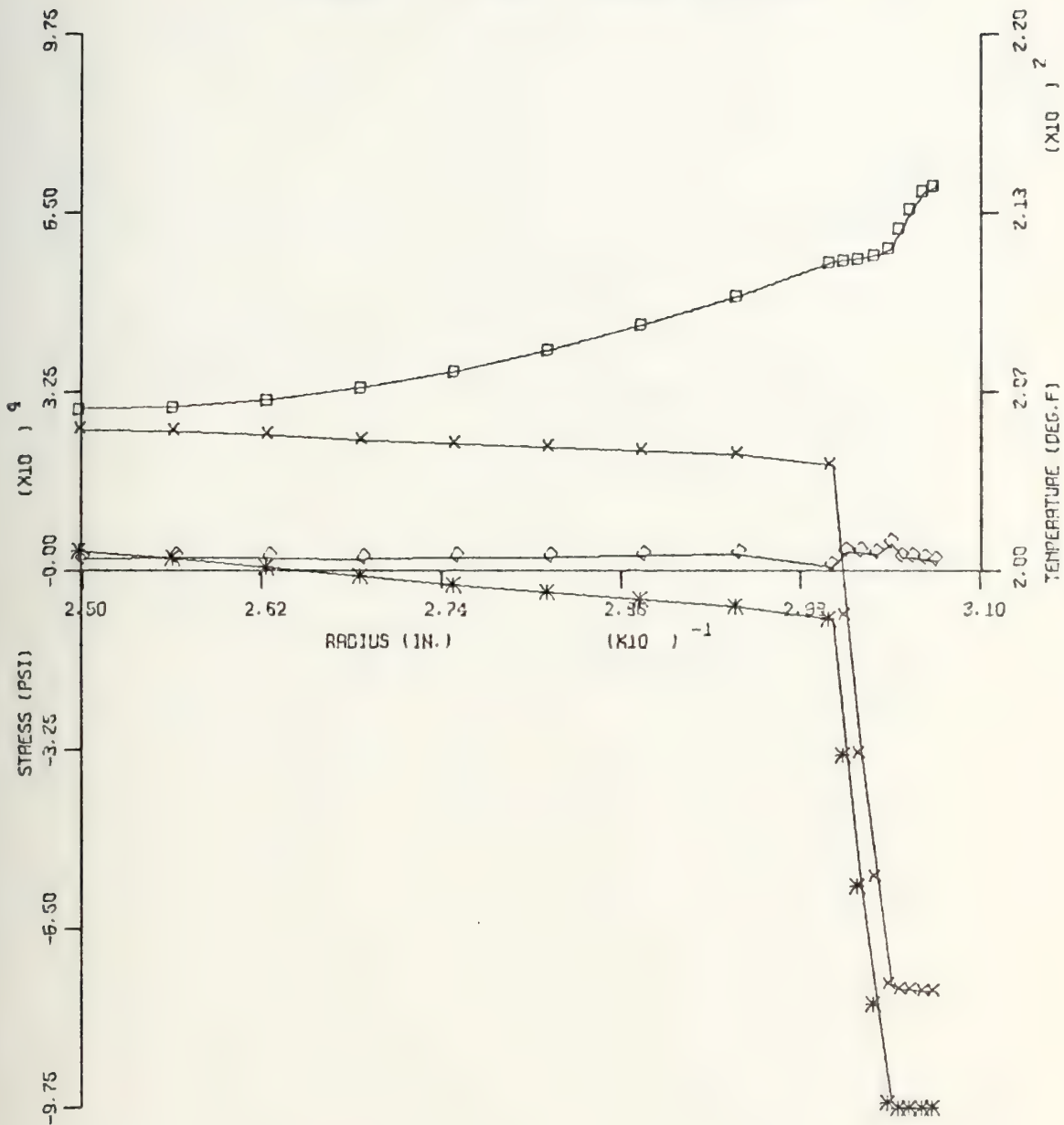
STRESS SCALE = 3.178×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 17(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.3-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

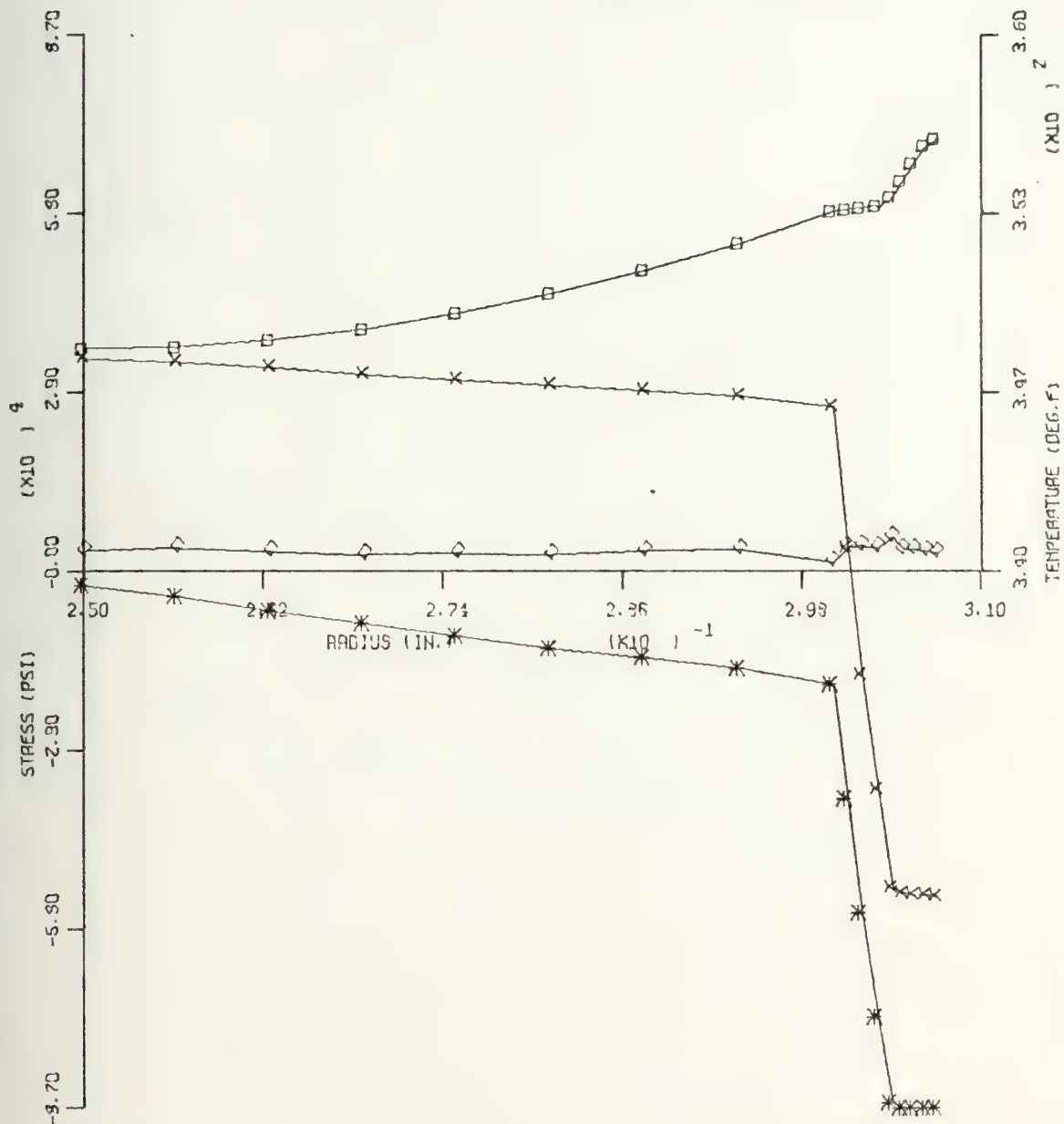
STRESS SCALE = 3.249×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 18(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.3-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

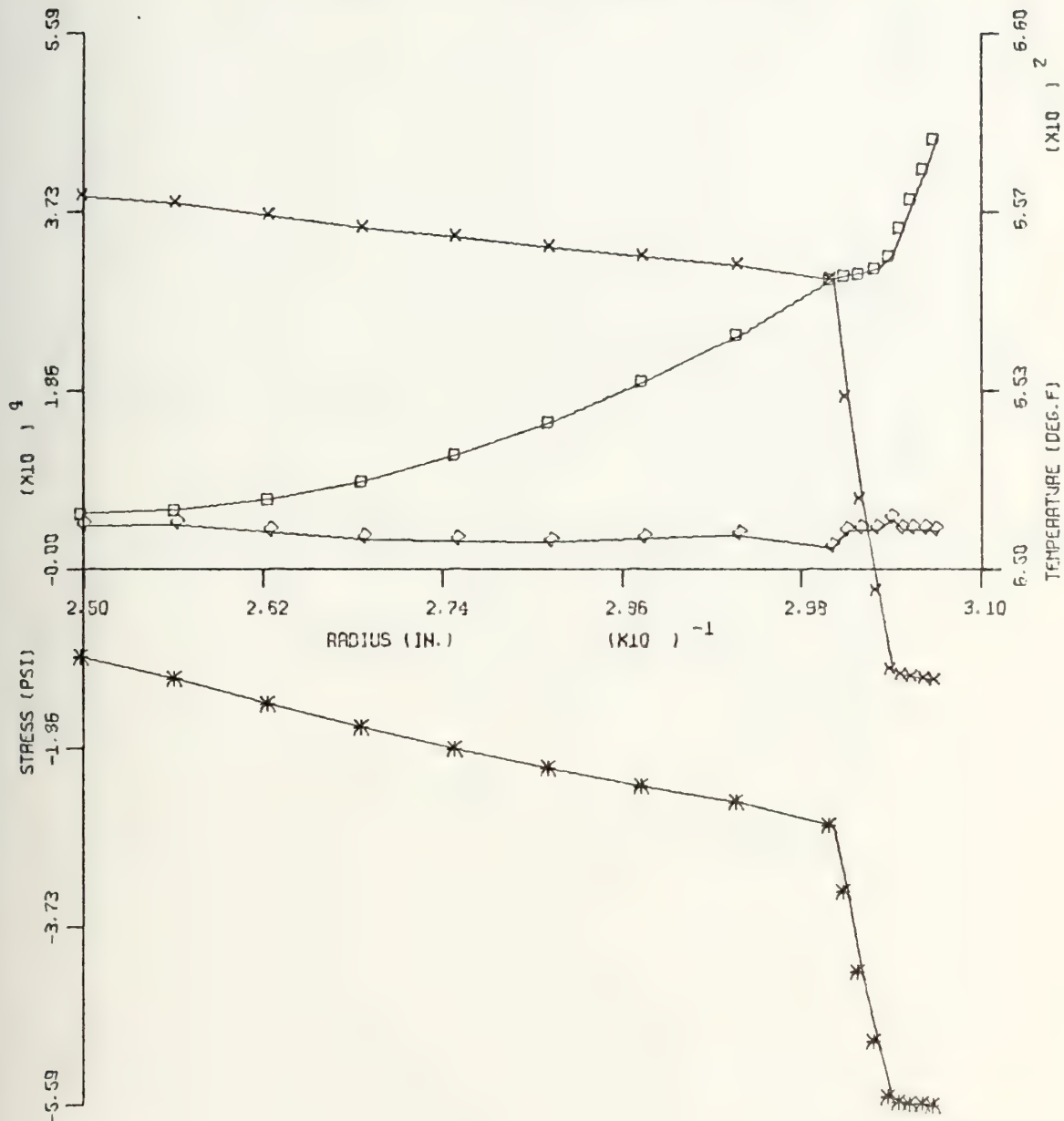
STRESS SCALE = 2.993×10^4 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 18(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.3-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

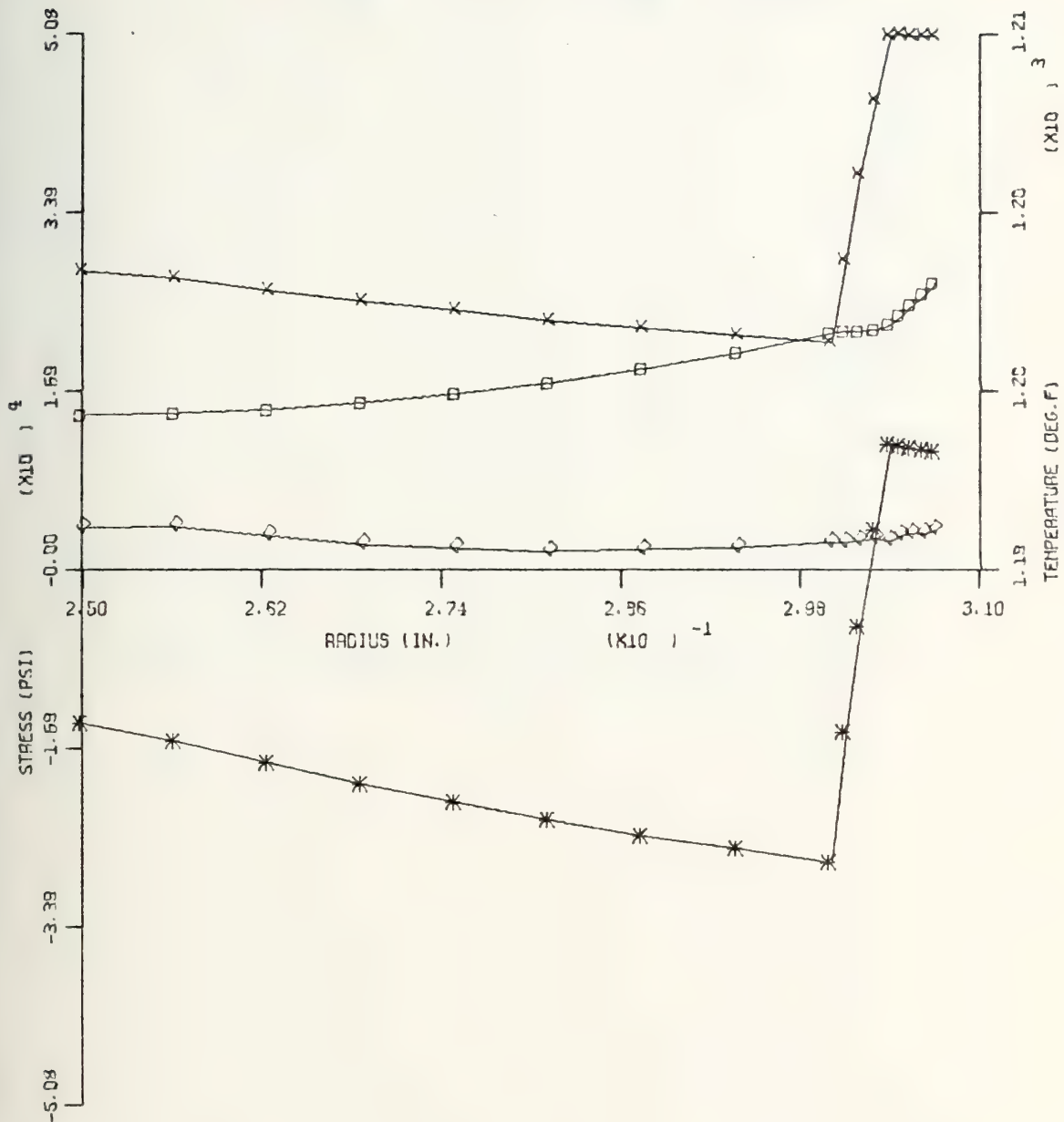
STRESS SCALE = 1.865×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 18(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.3-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

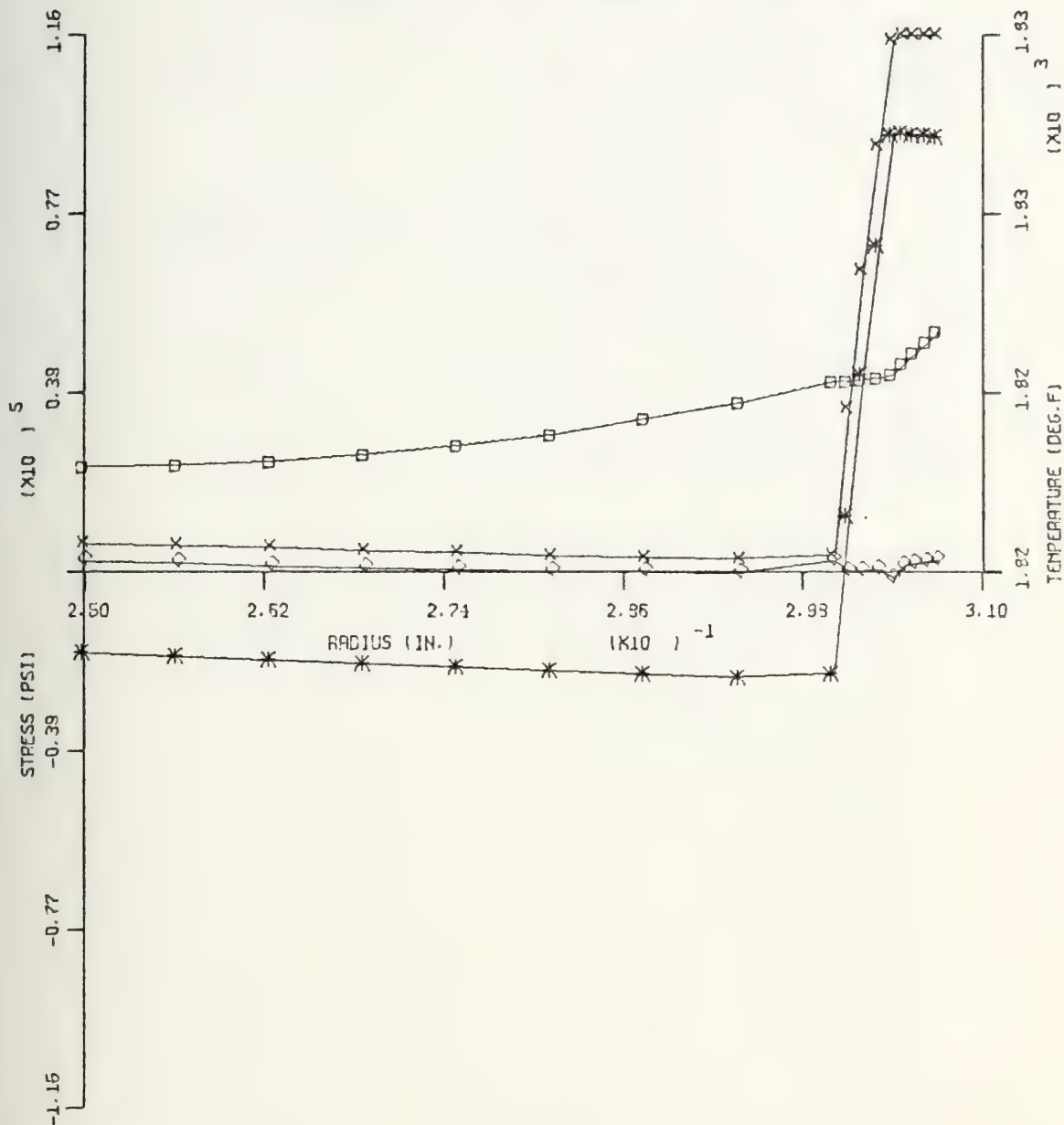
STRESS SCALE = 1.595×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 18(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.3-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

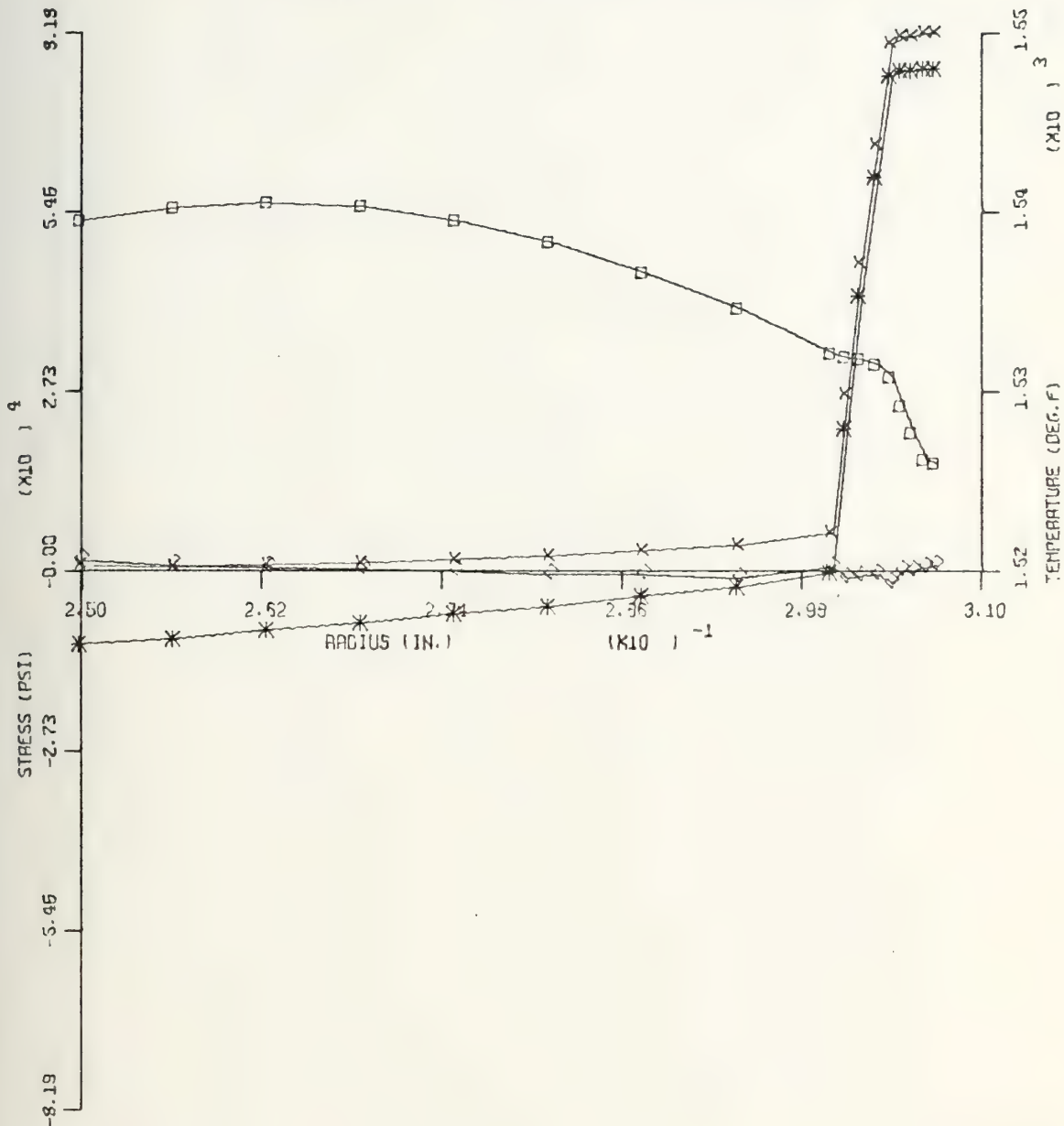
STRESS SCALE = 3.852×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 18(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.3-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

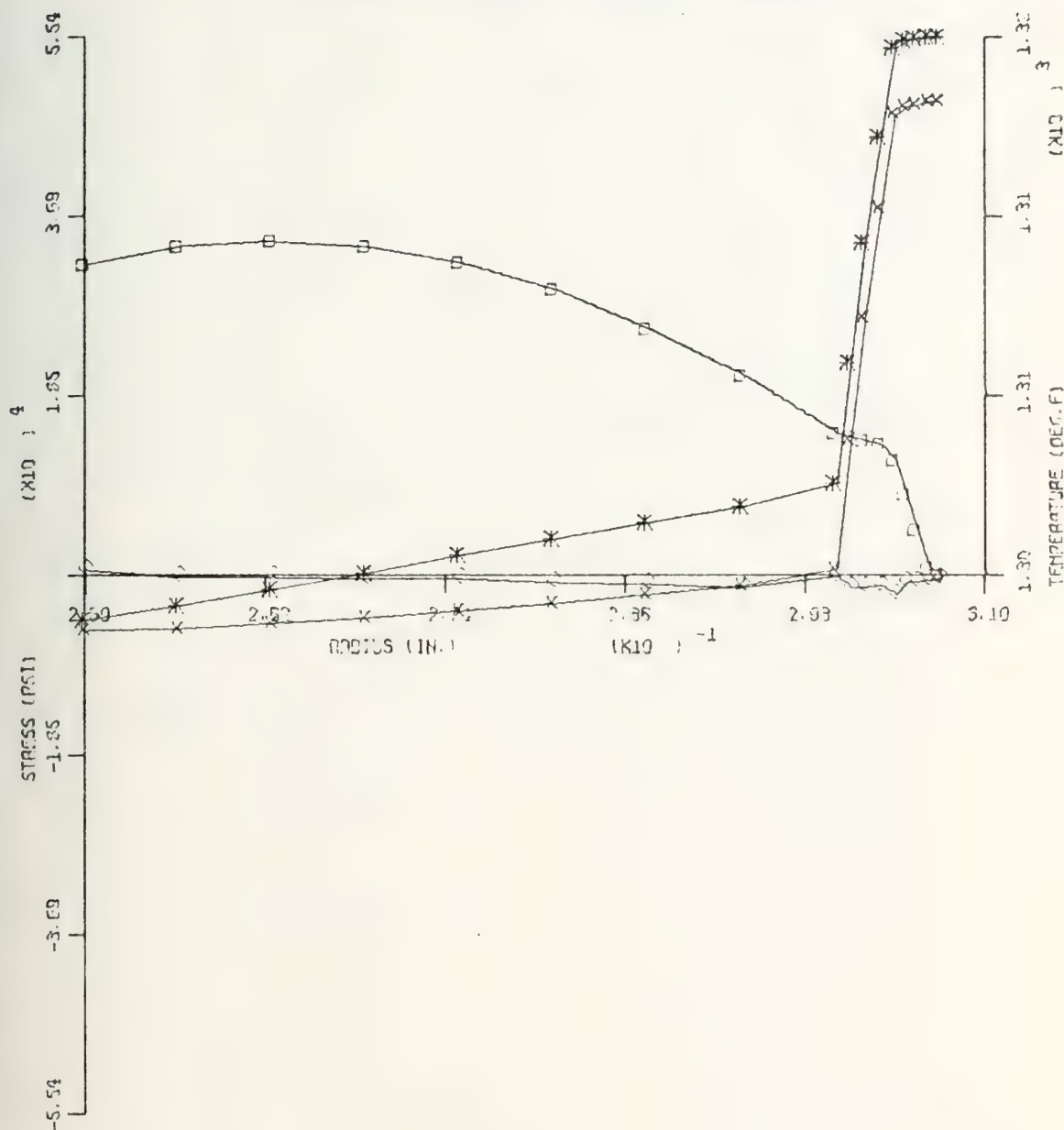
STRESS SCALE = 2.723×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 18(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.3-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 1.847×10^4 PSI/INCH

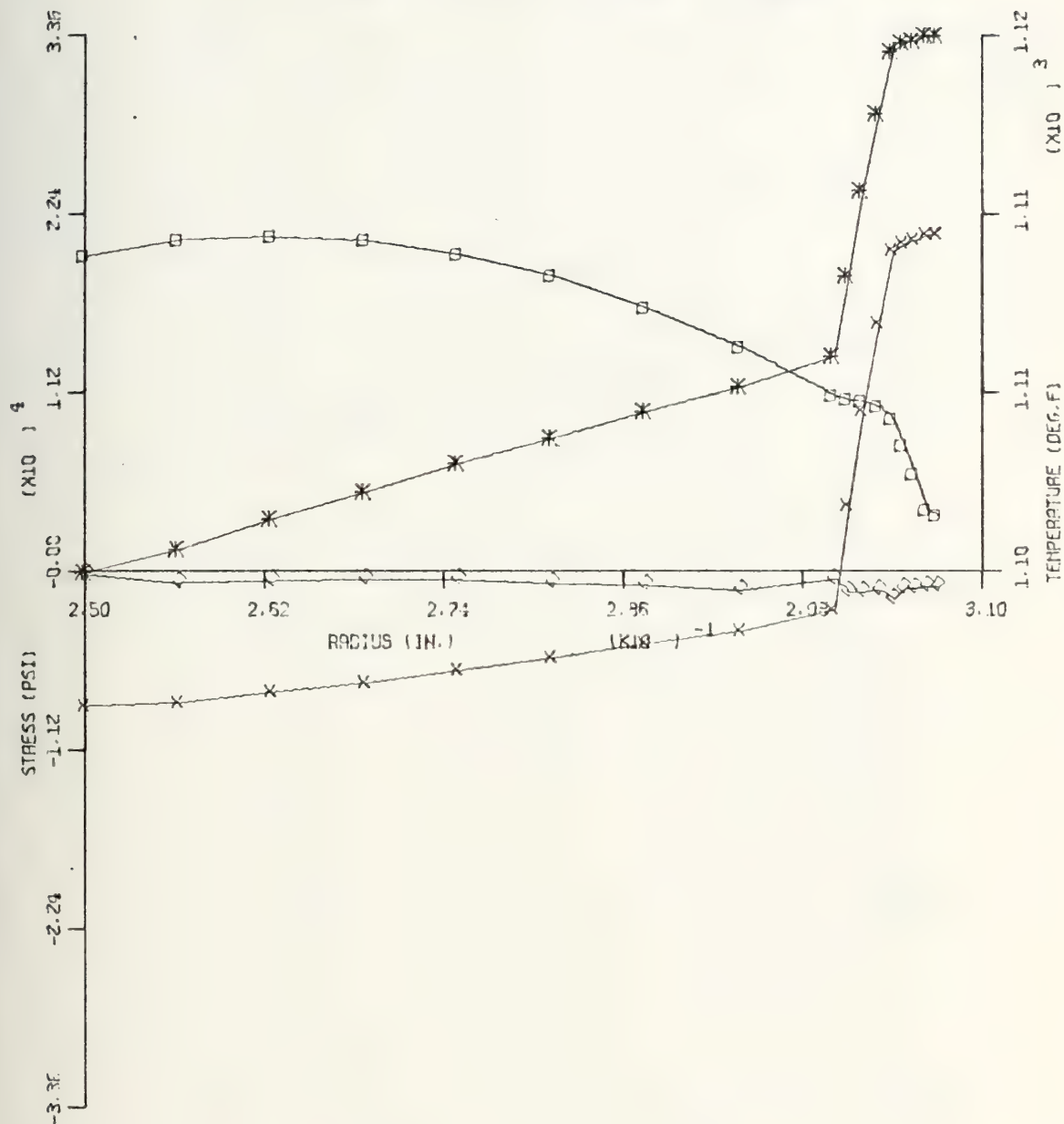
TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 18(g)



STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.3-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 1.12×10^4 PSI/INCH

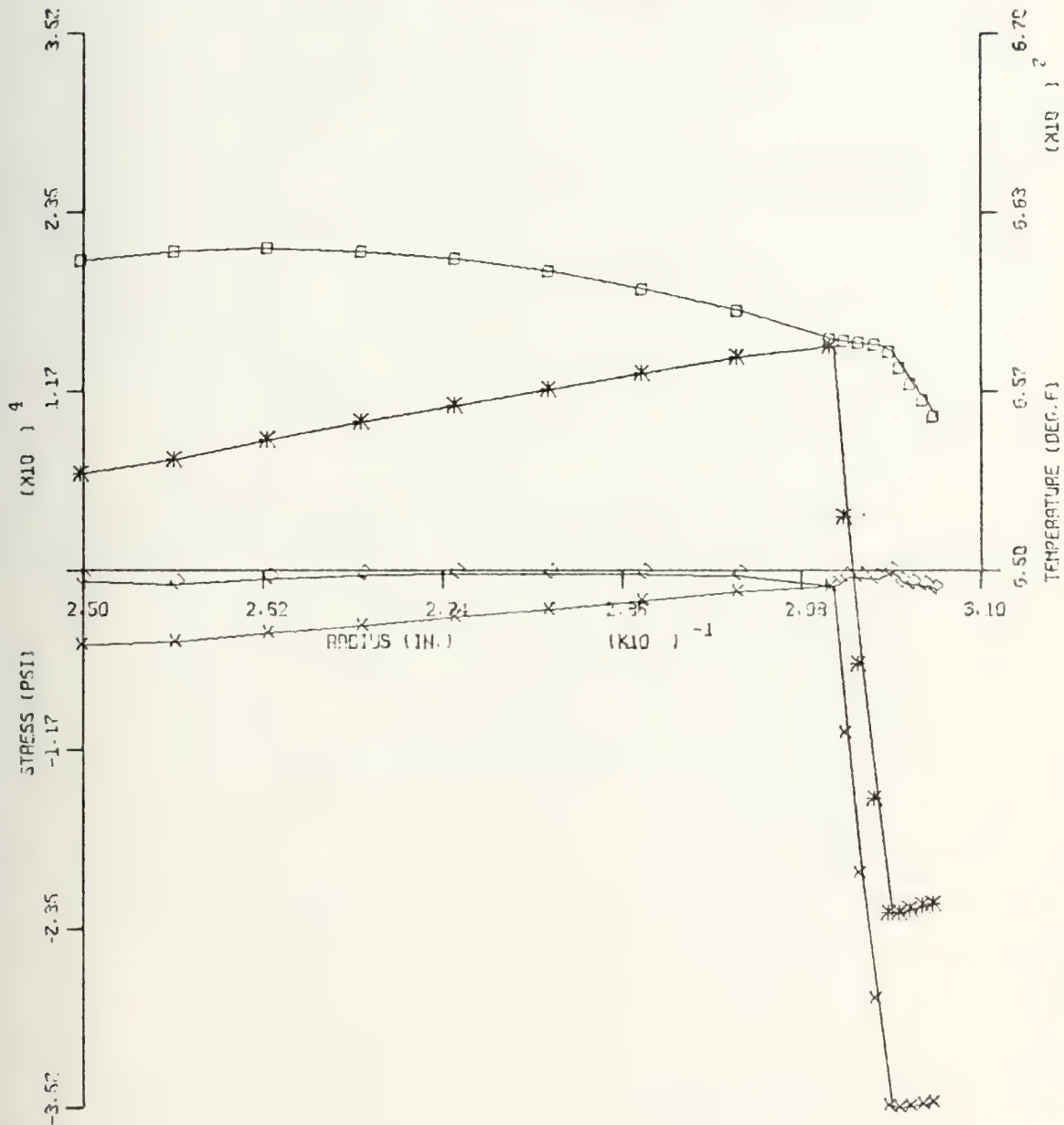
TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 18(h)



STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.3-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 1.173×10^4 PSI/INCH

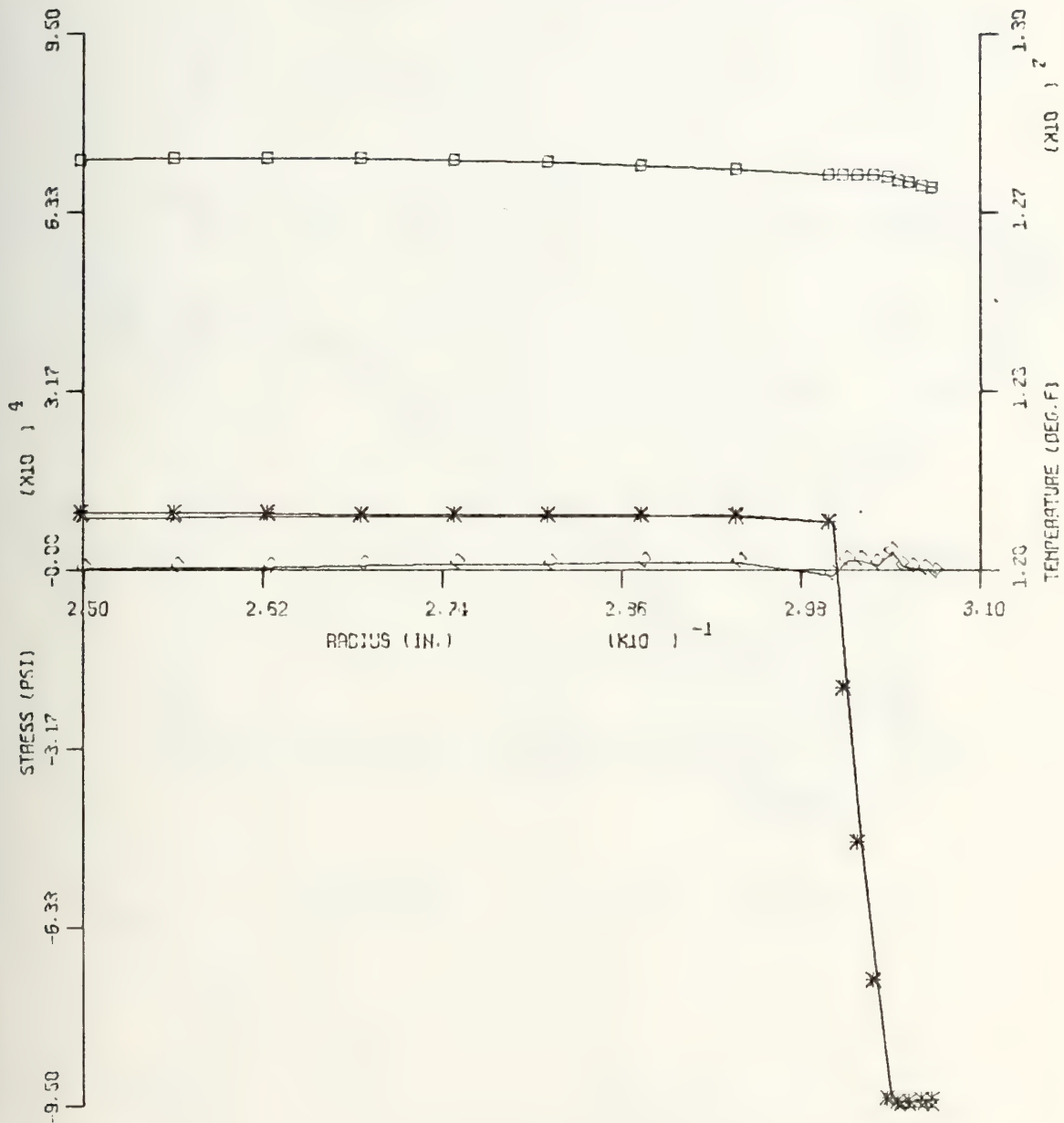
TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 18(i)



STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.3-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

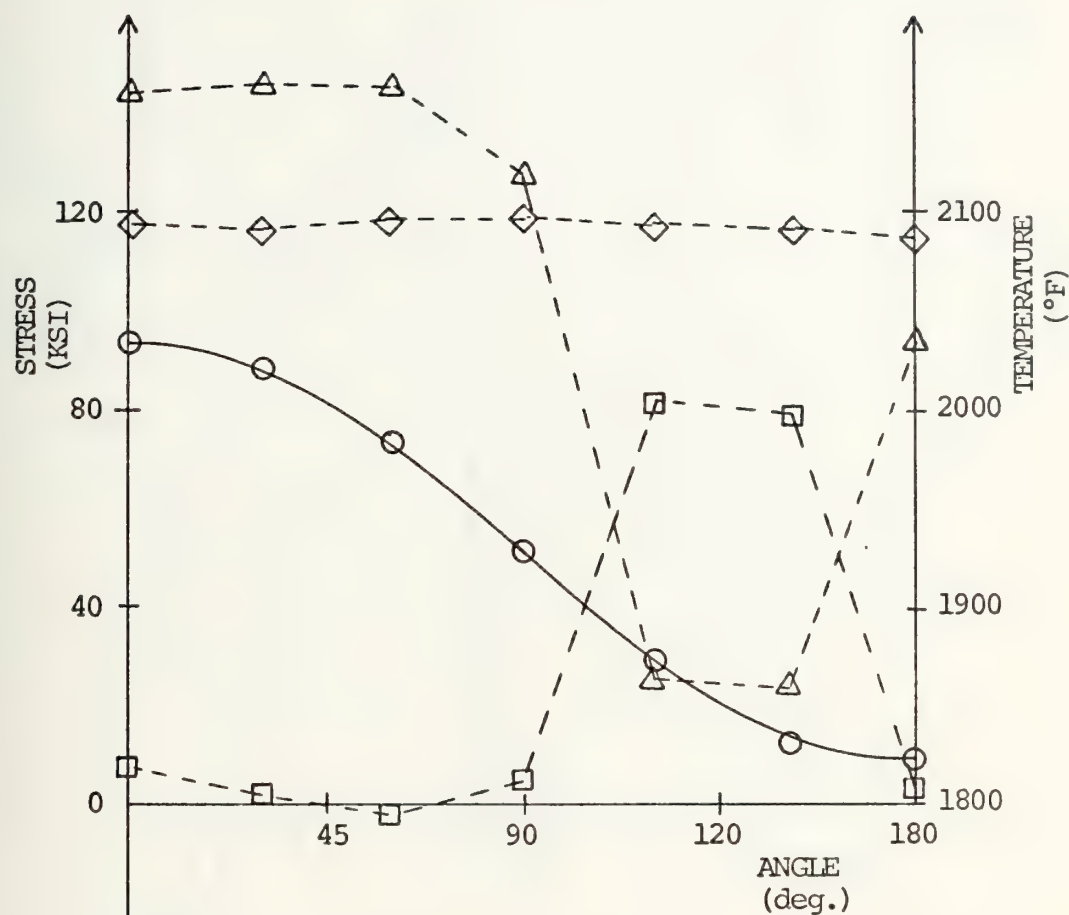
STRESS SCALE = 3.157×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 18(j)

STRESS AND TEMPERATURE
VERSUS
CIRCUMFERENTIAL POSITION



CONFIGURATION 3, $r=0.3055$ in., TIME=0.75 min.

- TEMPERATURE
- RADIAL STRESS
- ◇ AXIAL STRESS
- △ TANGENTIAL STRESS

FIGURE 19



STRESS AND TEMPERATURE
VERSUS TIME

CONFIGURATION 3, $r=0.3055$ in., $\text{ANGLE}=60^\circ$

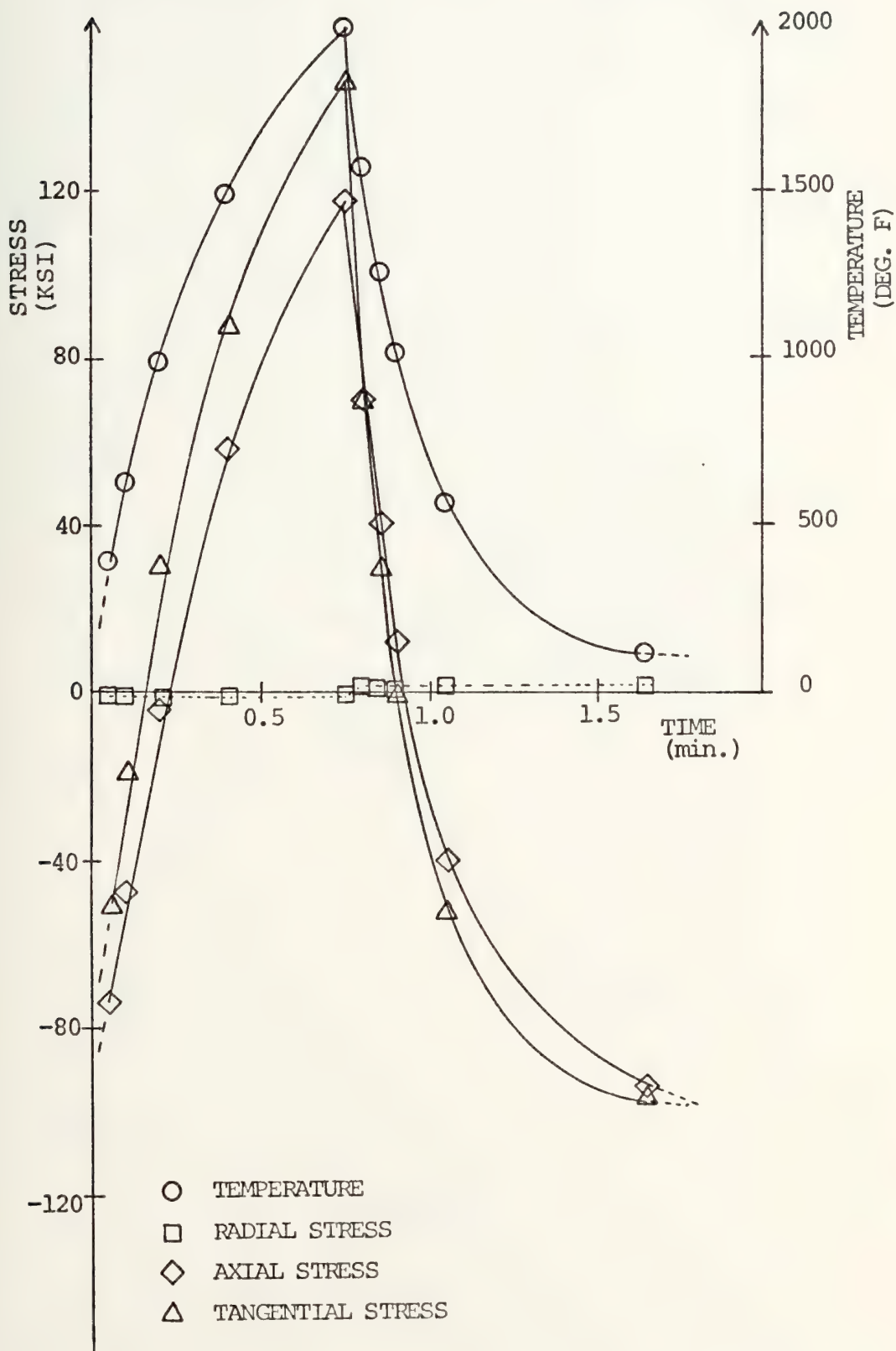


FIGURE 20

MAXIMUM AND MINIMUM PRINCIPAL STRESSES
VERSUS TIME
CONFIGURATION NUMBER THREE

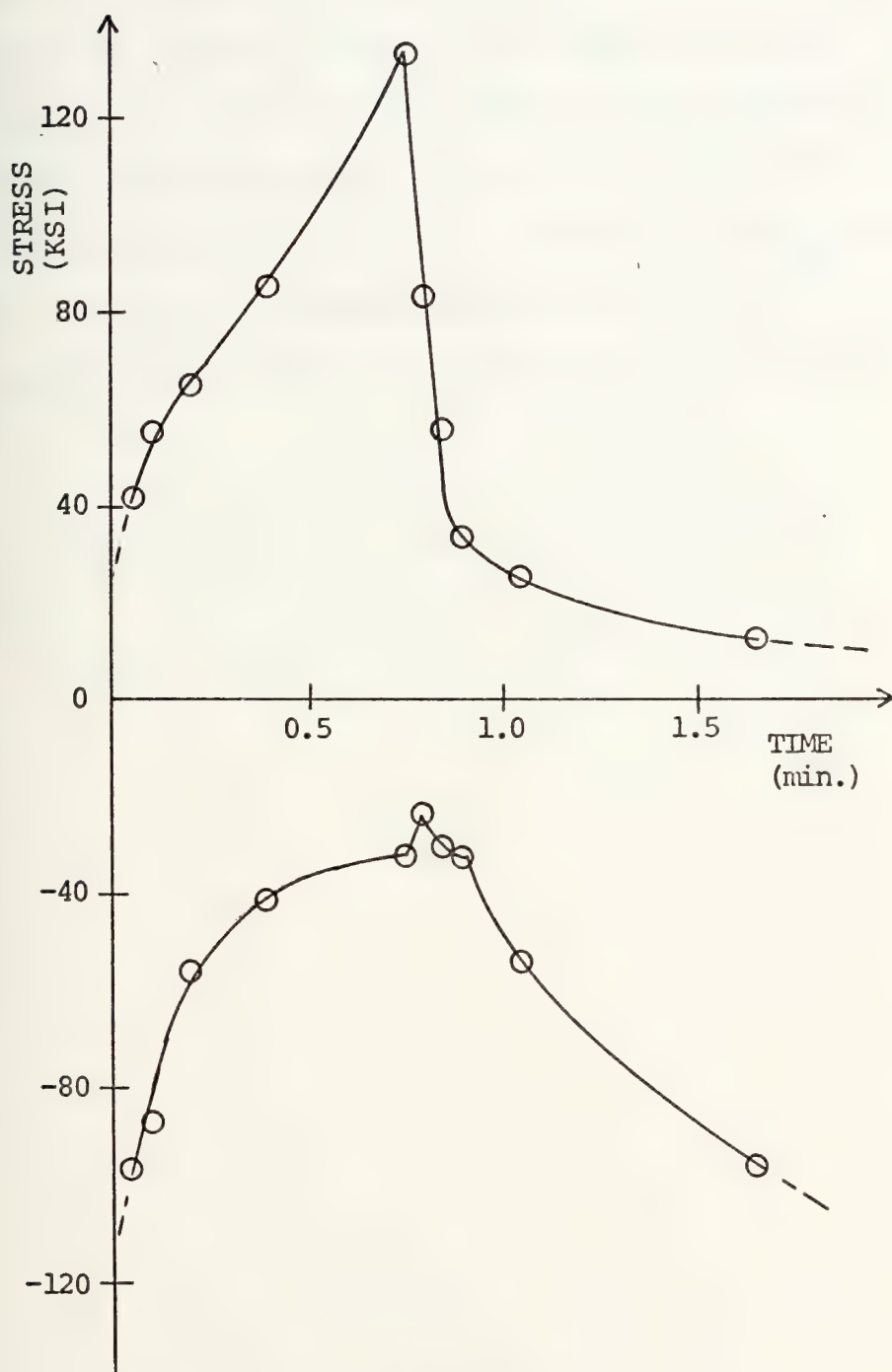


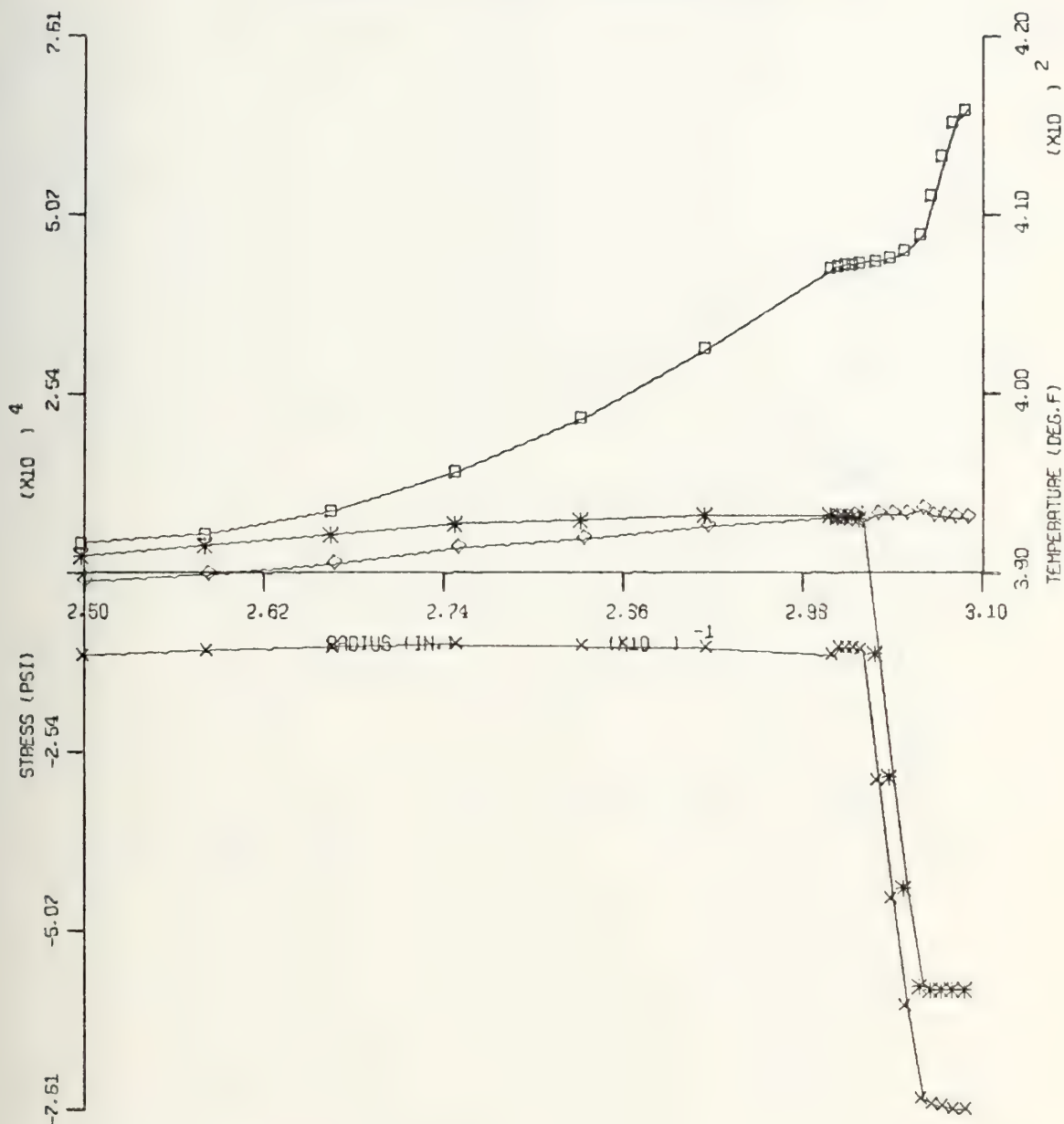
FIGURE 21



D. CONFIGURATION NUMBER FOUR

This coating arrangement included a layer of pure nickel between the substratum and the graded composition of configuration number three. The maximum principal stress (130.3 KSI) which occurred at the maximum temperature condition, was located at a radius of 0.3075 inches, the zero angle position, and at the midpoint of the cylinder length. As with the other configurations, the tangential and axial stresses were the largest for this configuration.

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF. 4-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 2.537×10^4 PSI/INCH

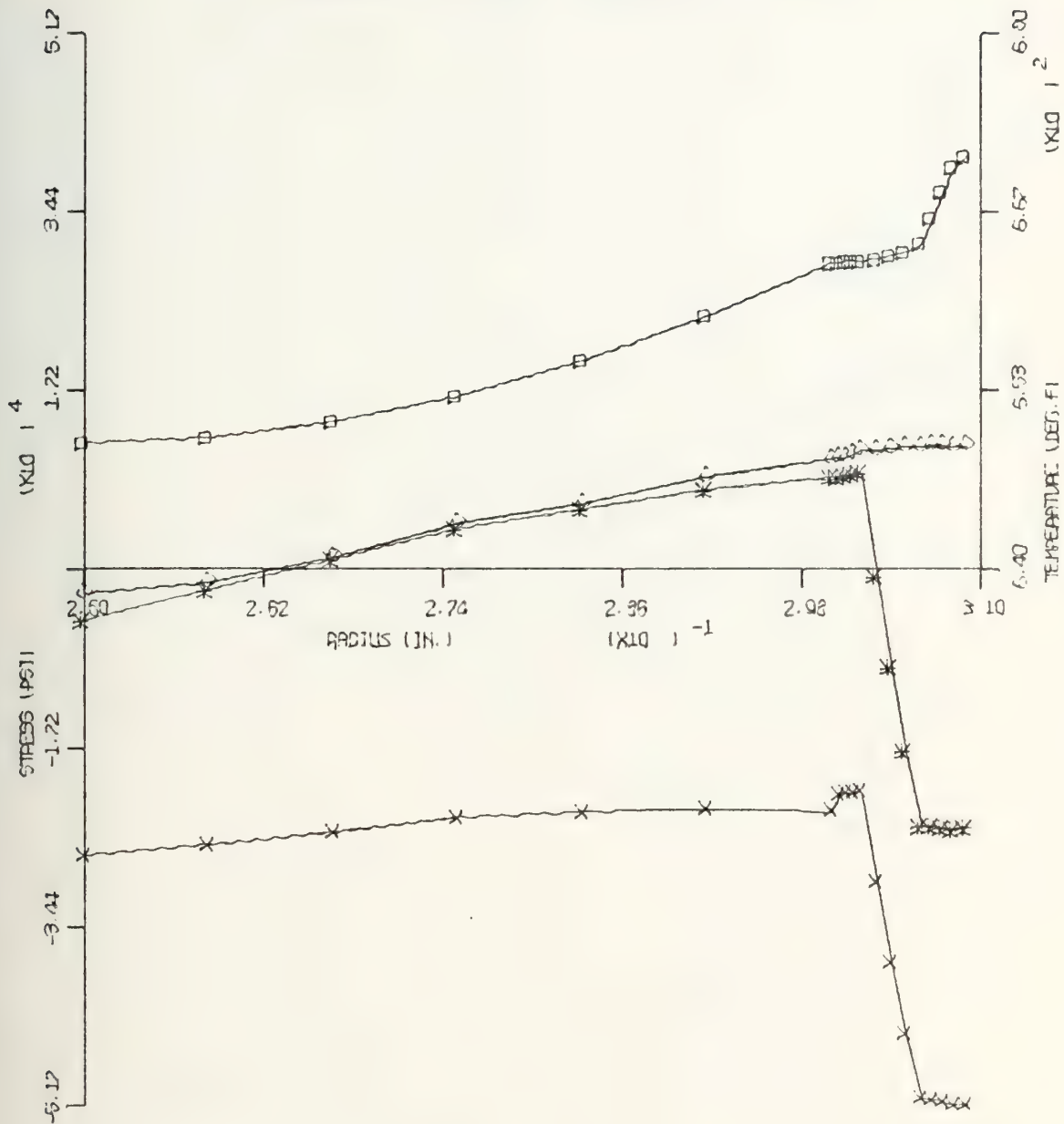
TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 22(a)



STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF. 4-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 1.722×10^4 PSI/INCH

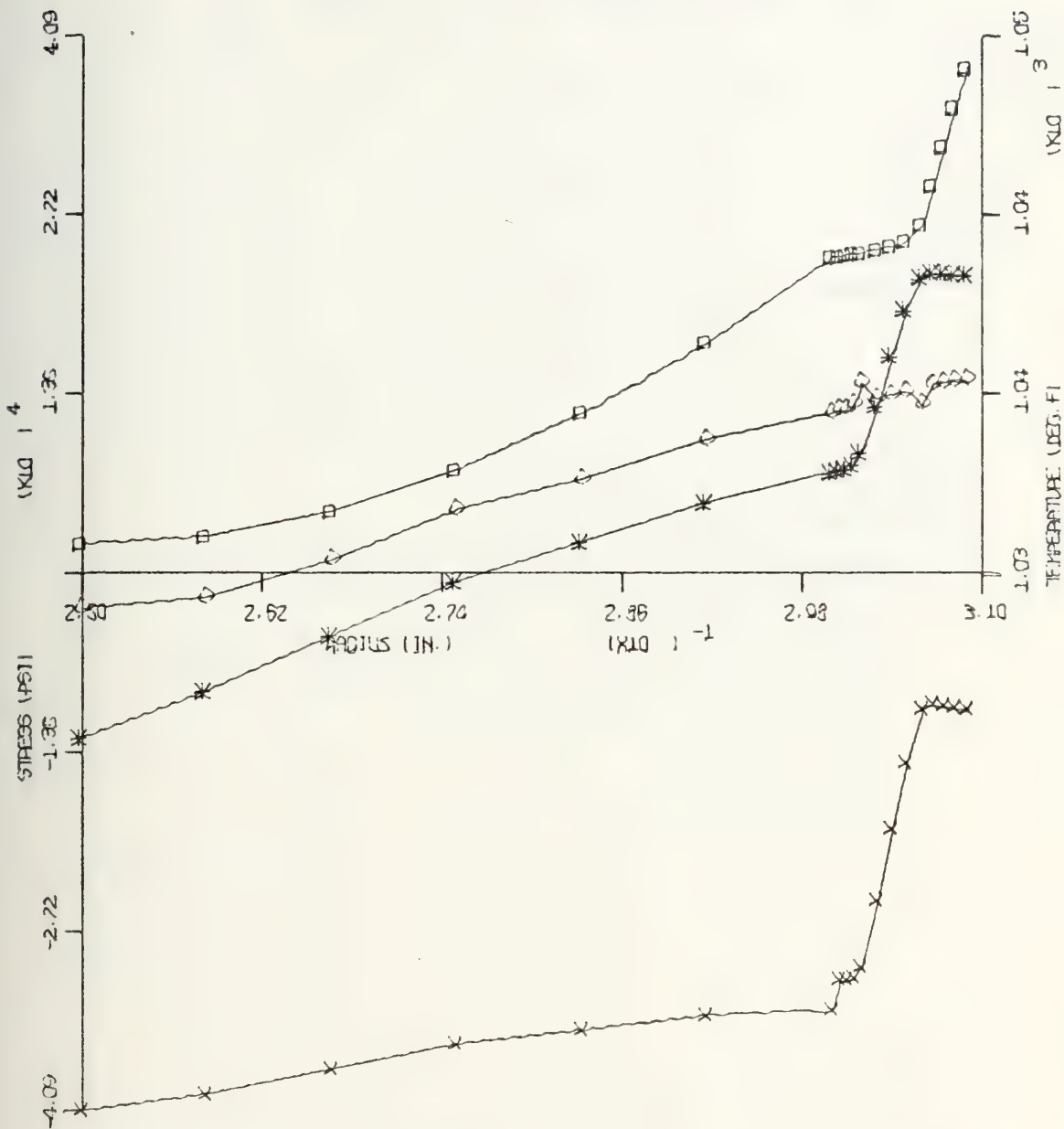
TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 22(b)



STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF. 4-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

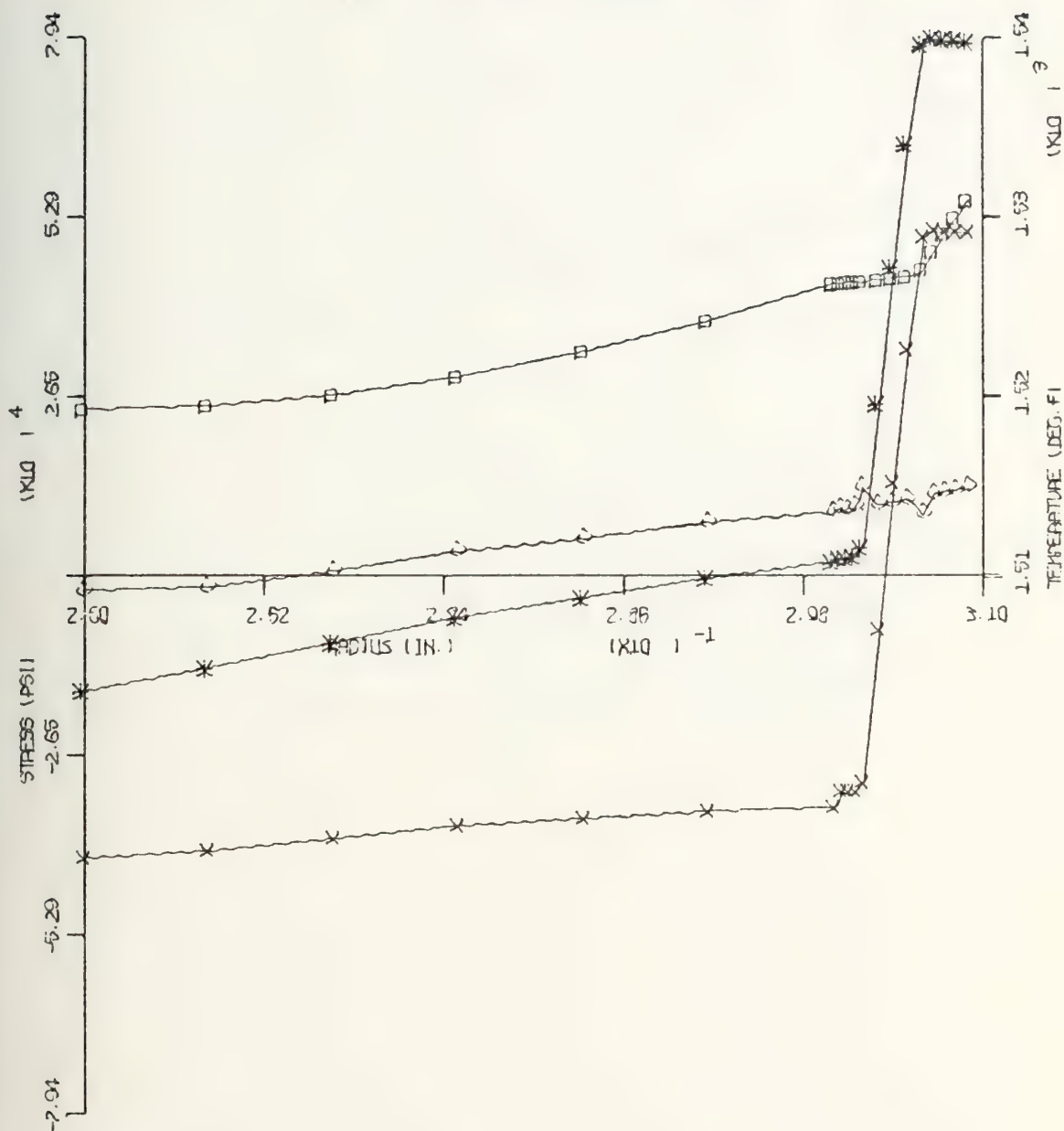
STRESS SCALE = 1.352×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 22(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF. 4-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

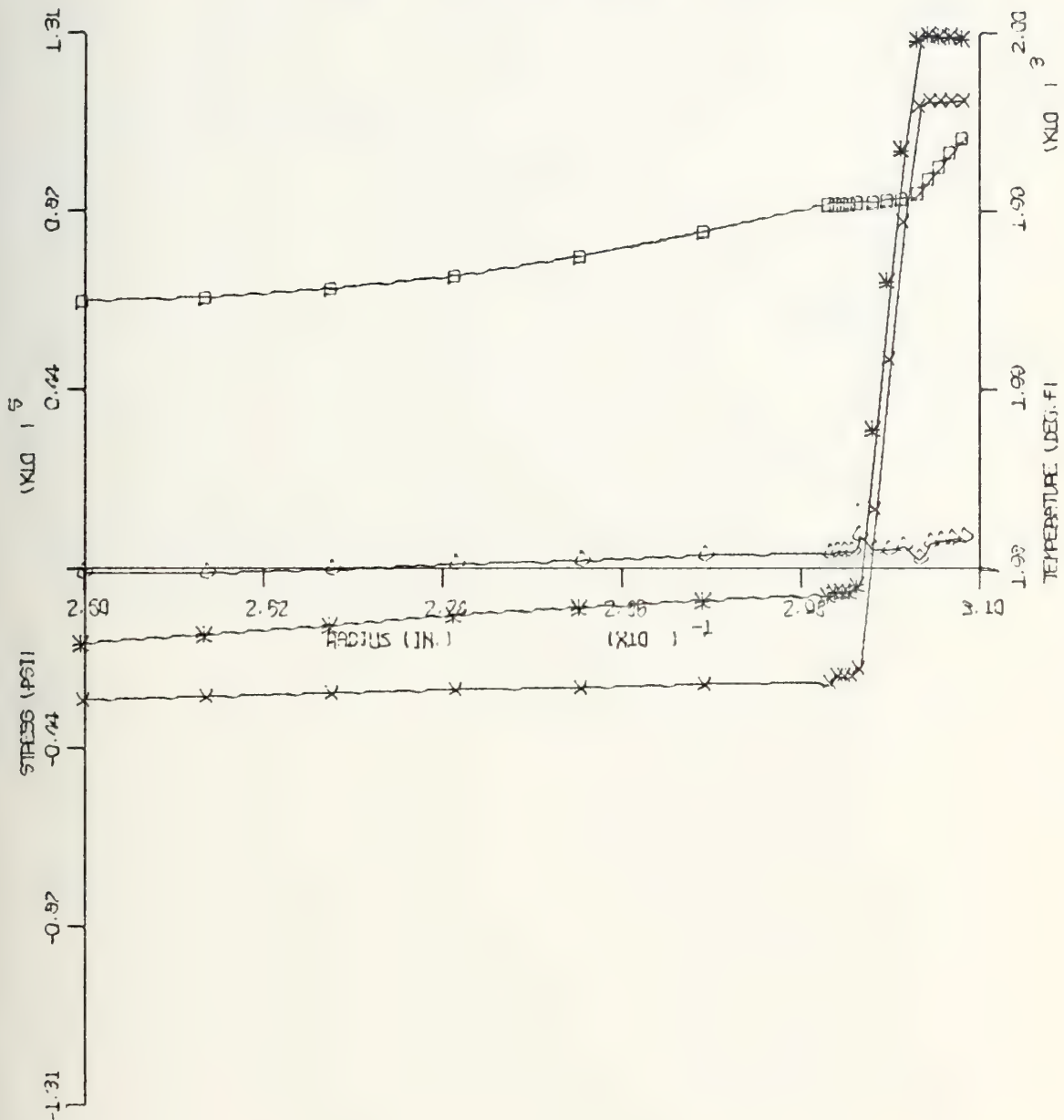
STRESS SCALE = 2.509×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 22(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF. 4-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

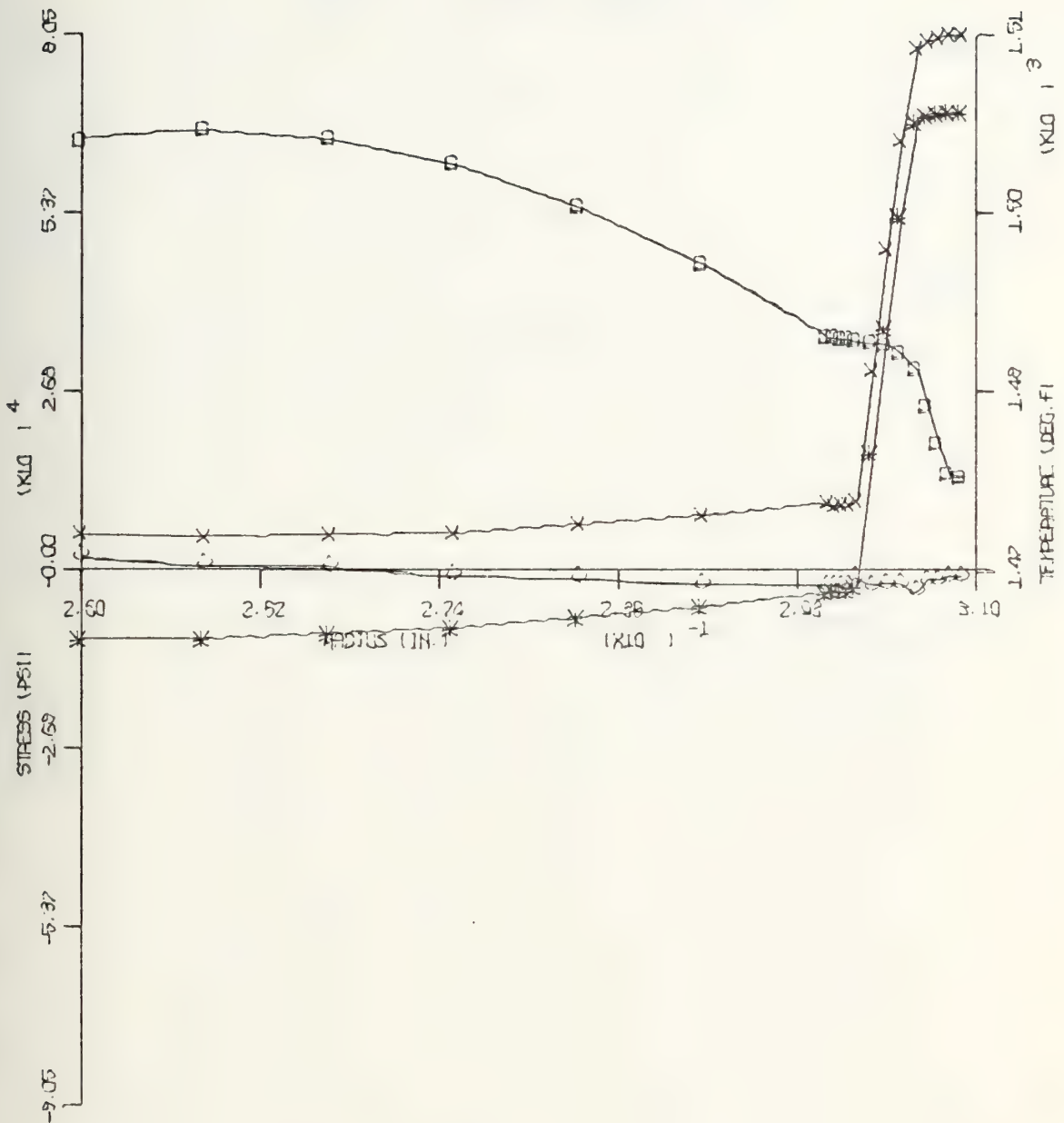
STRESS SCALE = 2.35×10^6 PSI/INCH

TEMPERATURE SCALE = 5.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 22 (e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF. 4-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

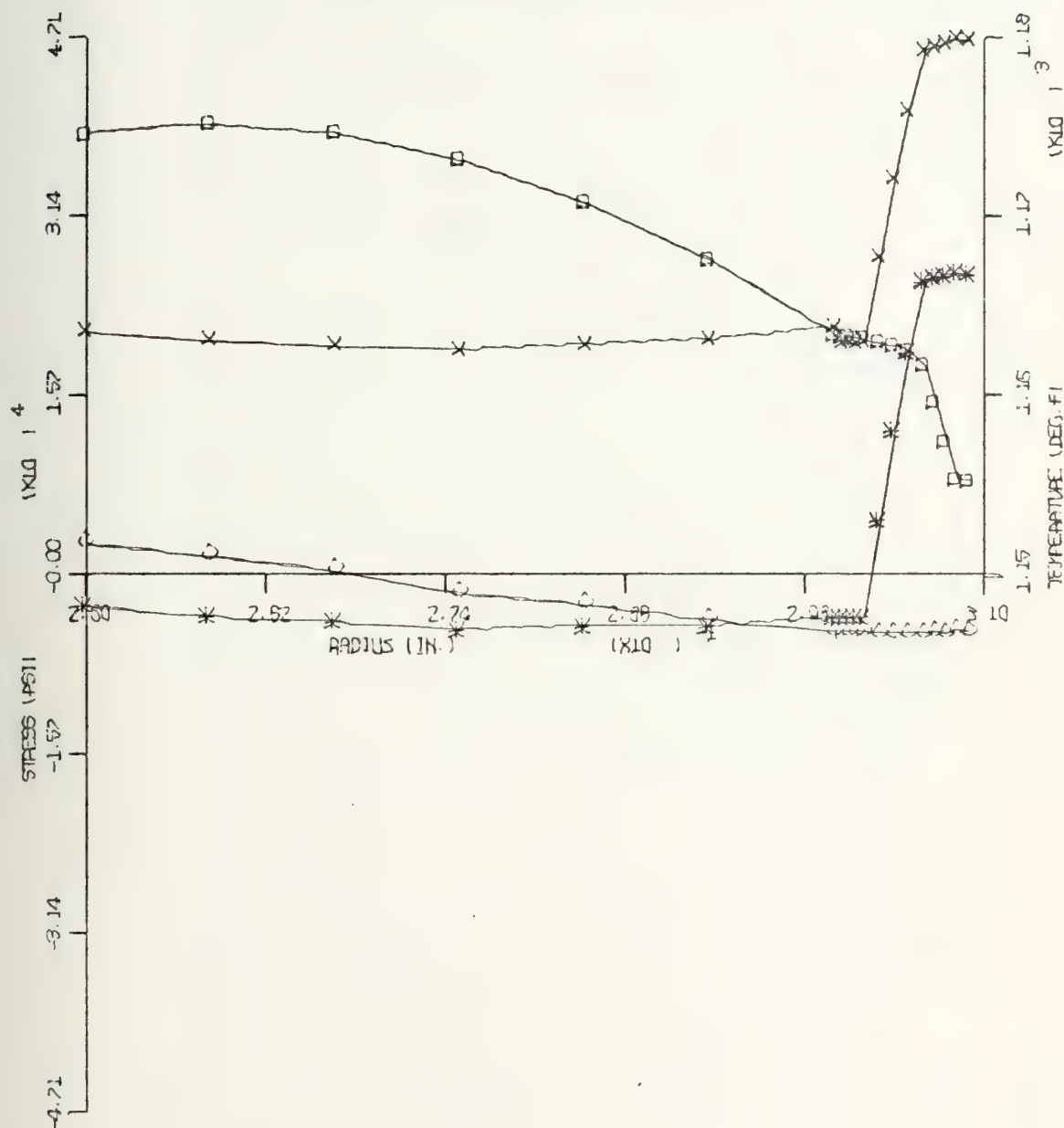
STRESS SCALE = 2.583×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 22(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.4-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

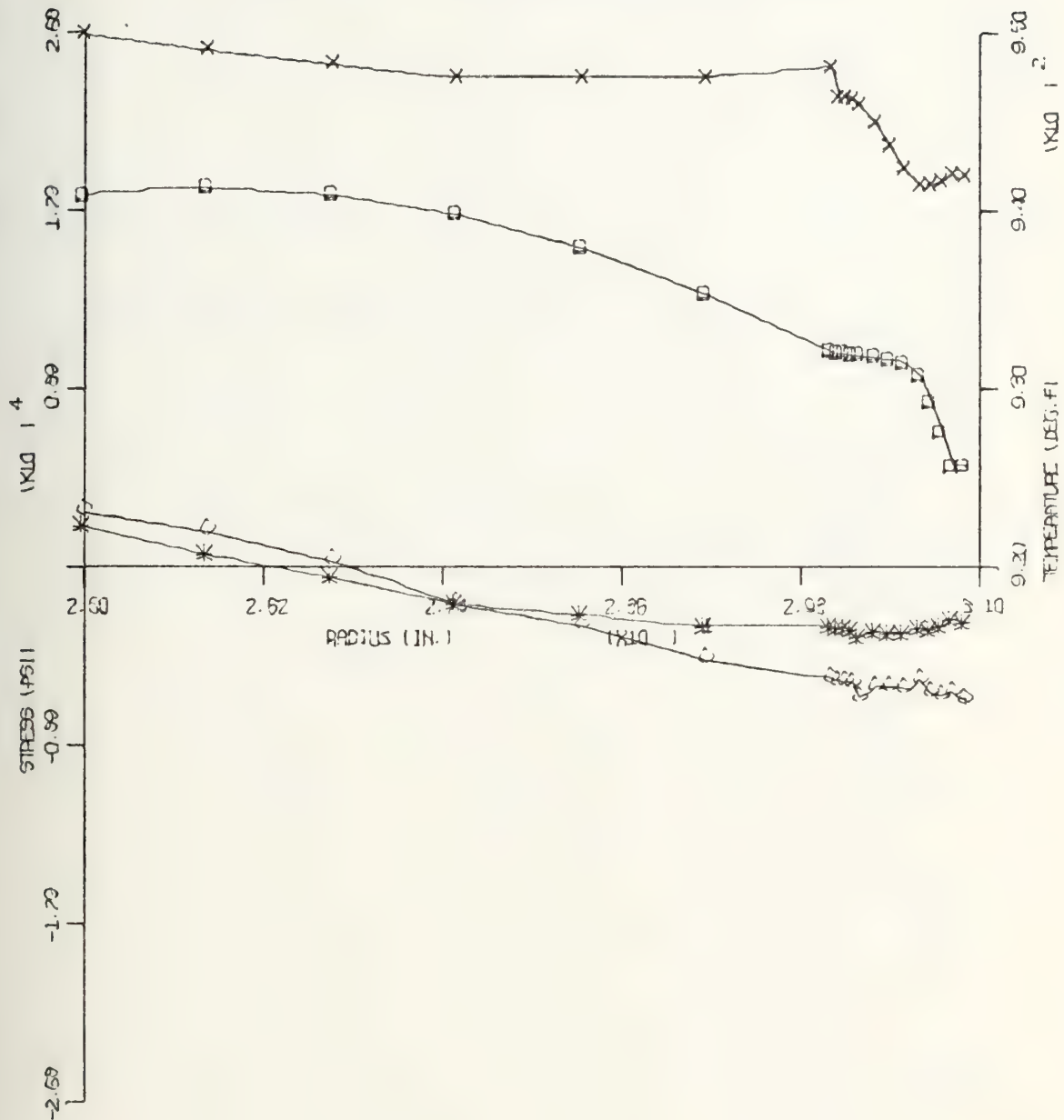
STRESS SCALE = 1.50×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 22(g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.4-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

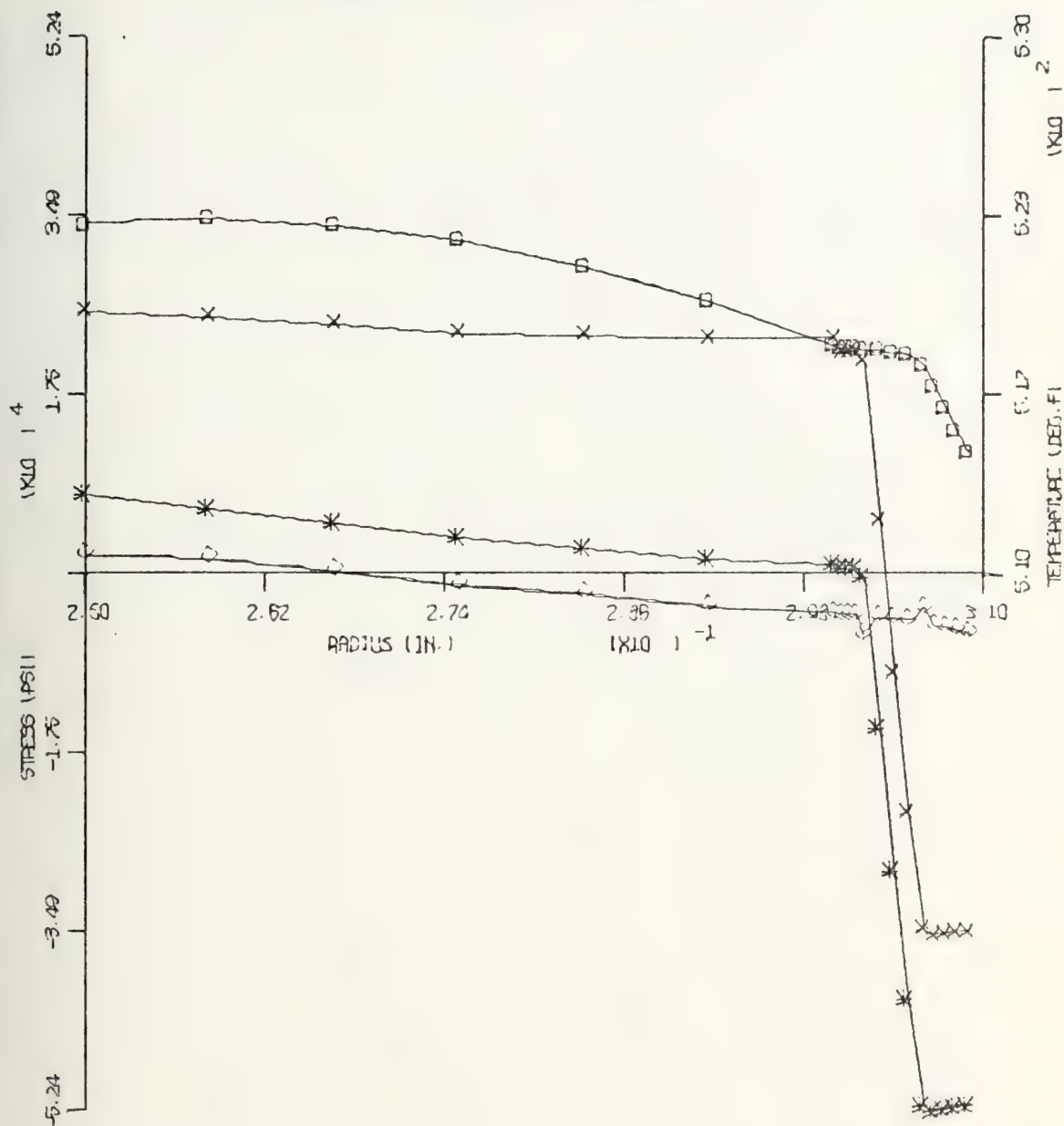
STRESS SCALE = 3.033×10^3 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 22(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF. 4-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

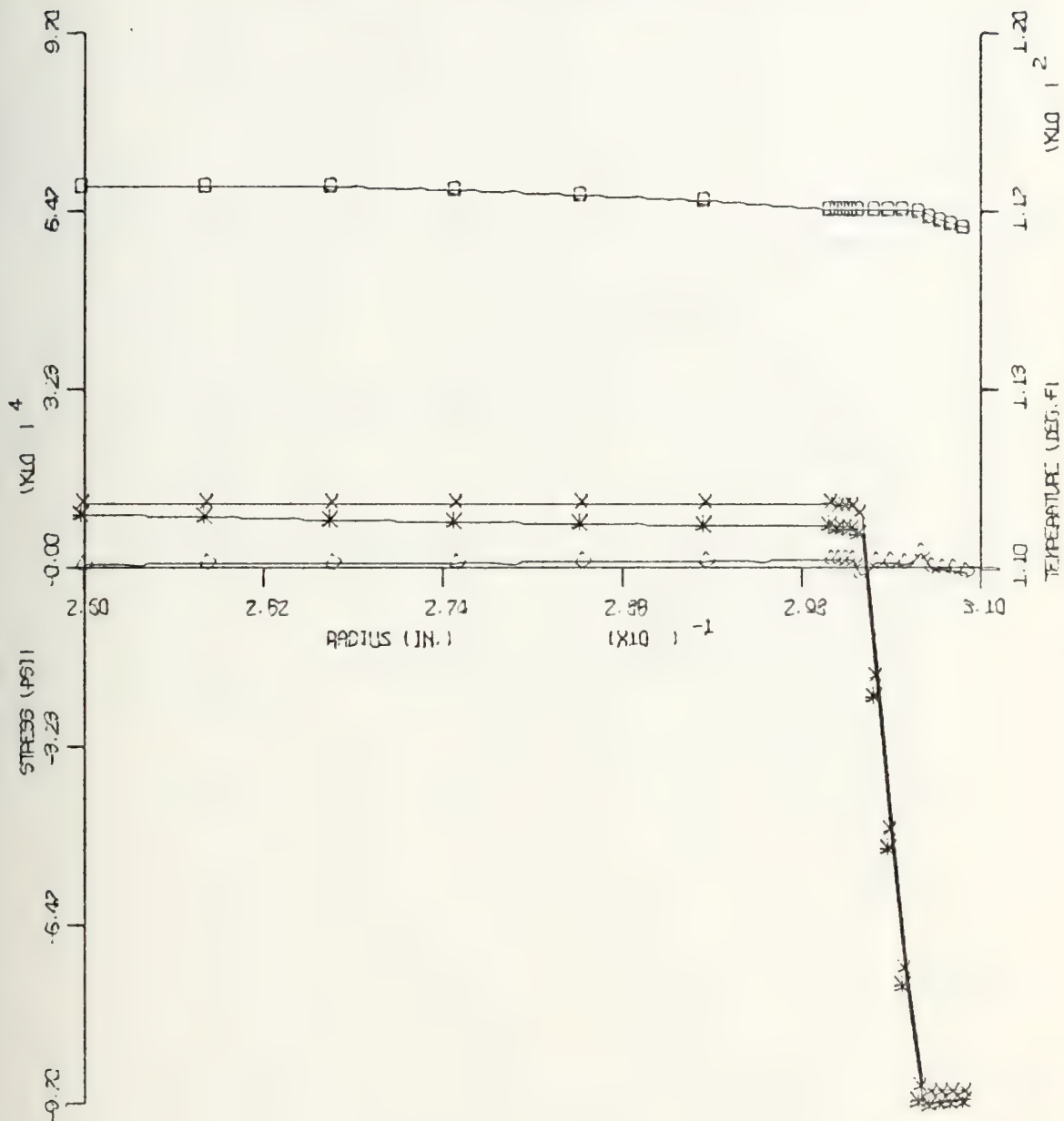
STRESS SCALE = 1.727×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 22(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF. 4-0



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

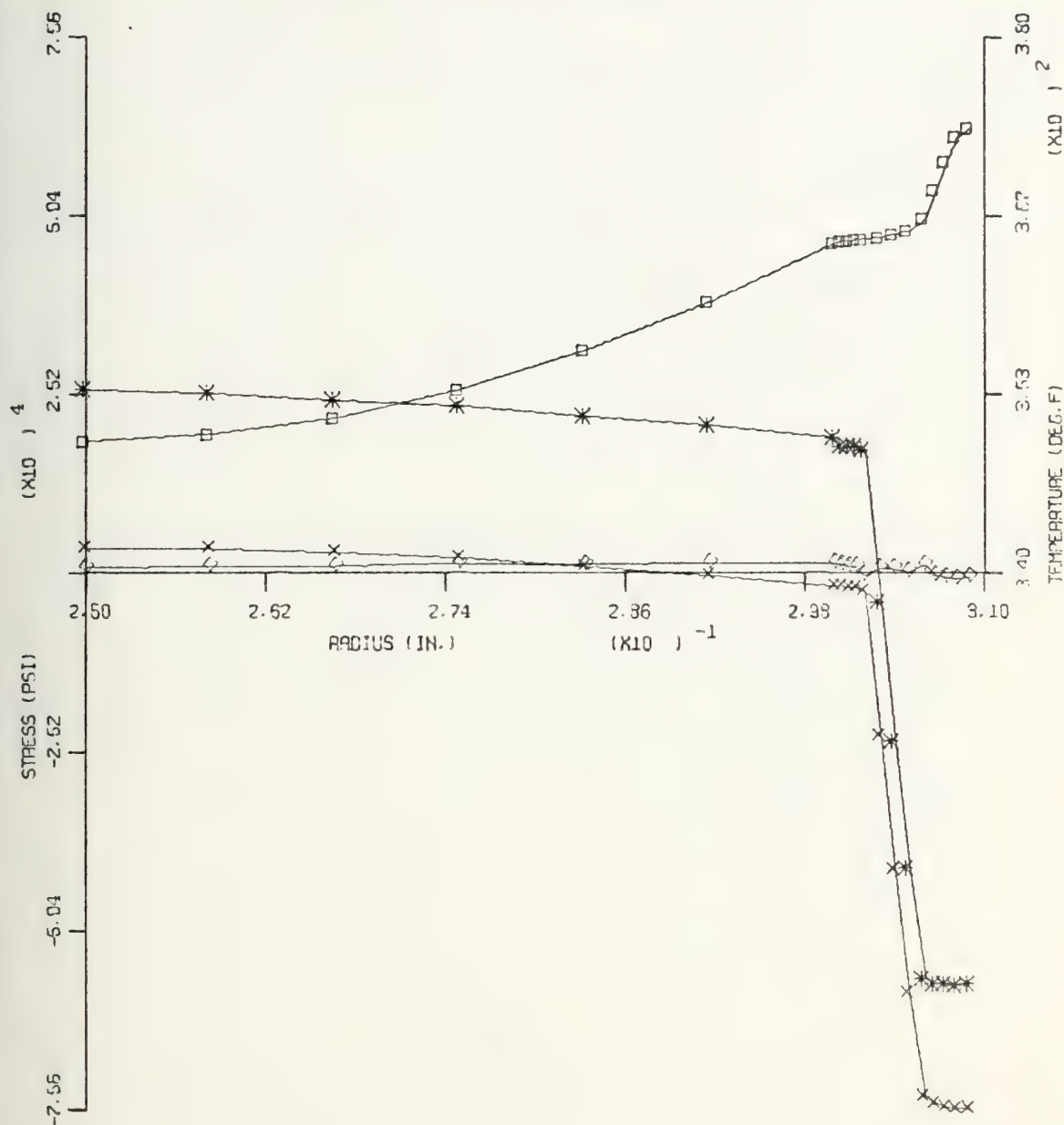
STRESS SCALE = 3.25×10^3 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 22(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.4-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

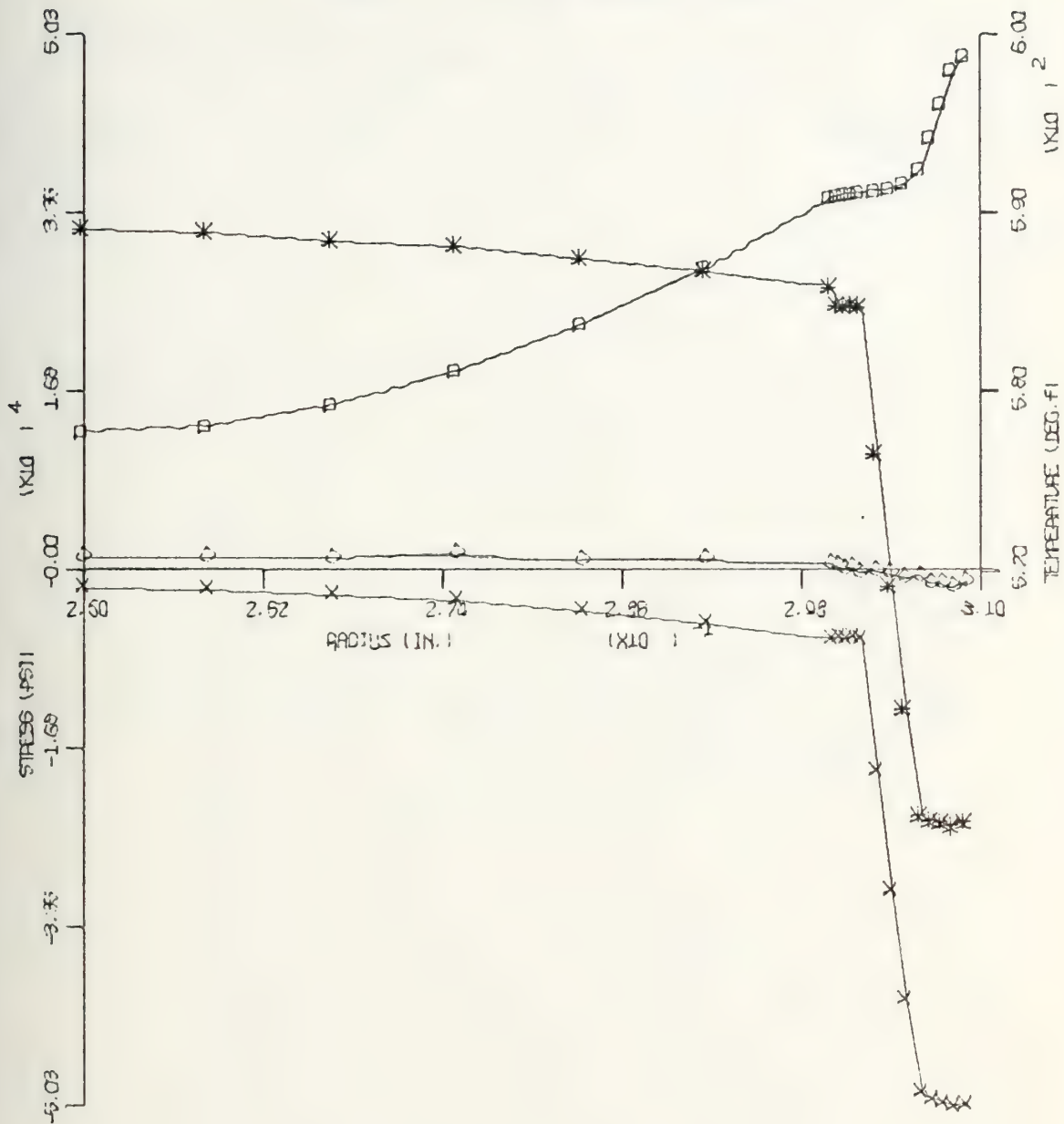
STRESS SCALE = 2.518×10^4 PSI/INCH

TEMPERATURE SCALE = 13.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 23(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF. 4-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

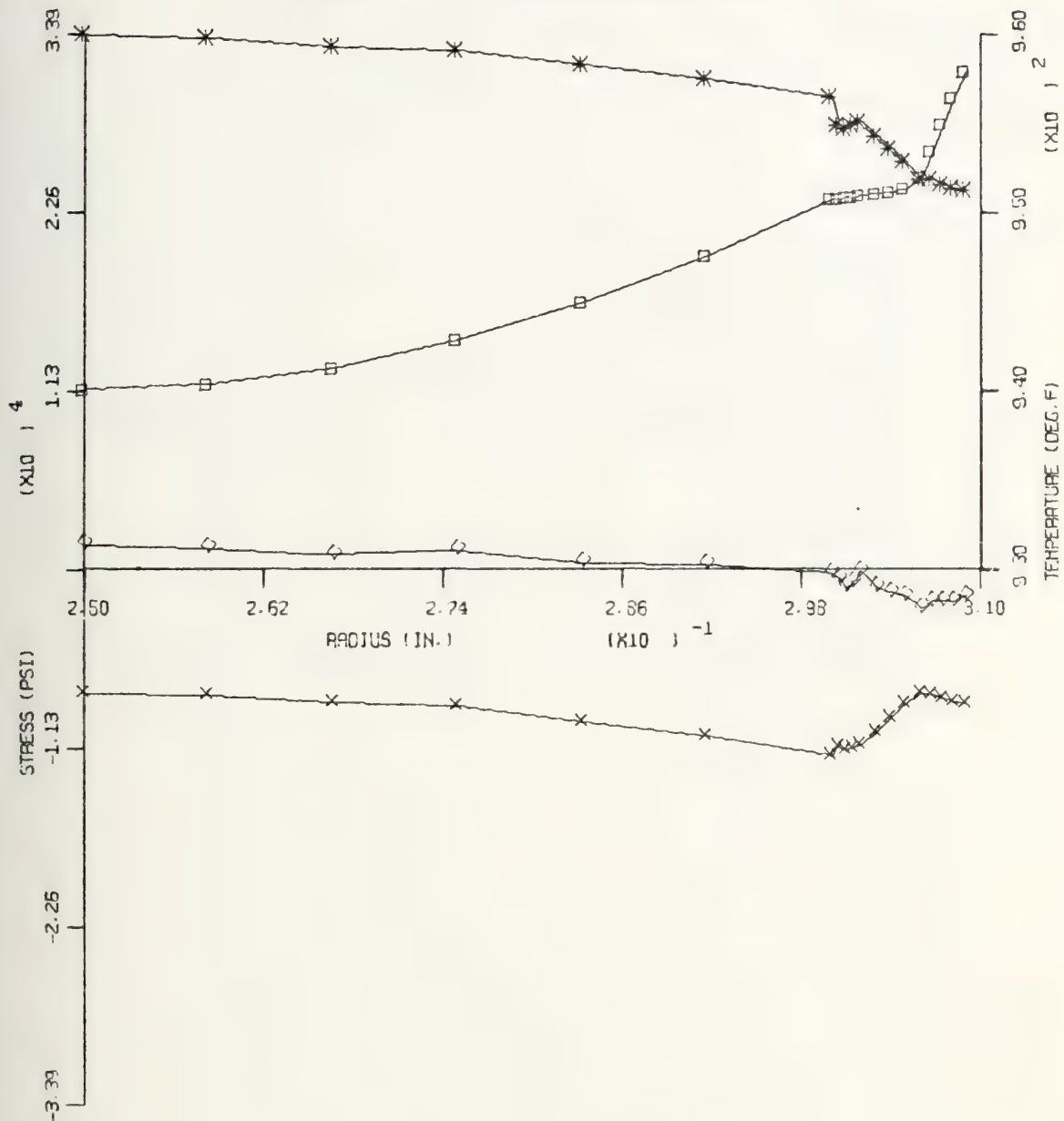
STRESS SCALE = 1.575×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 23(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.4-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

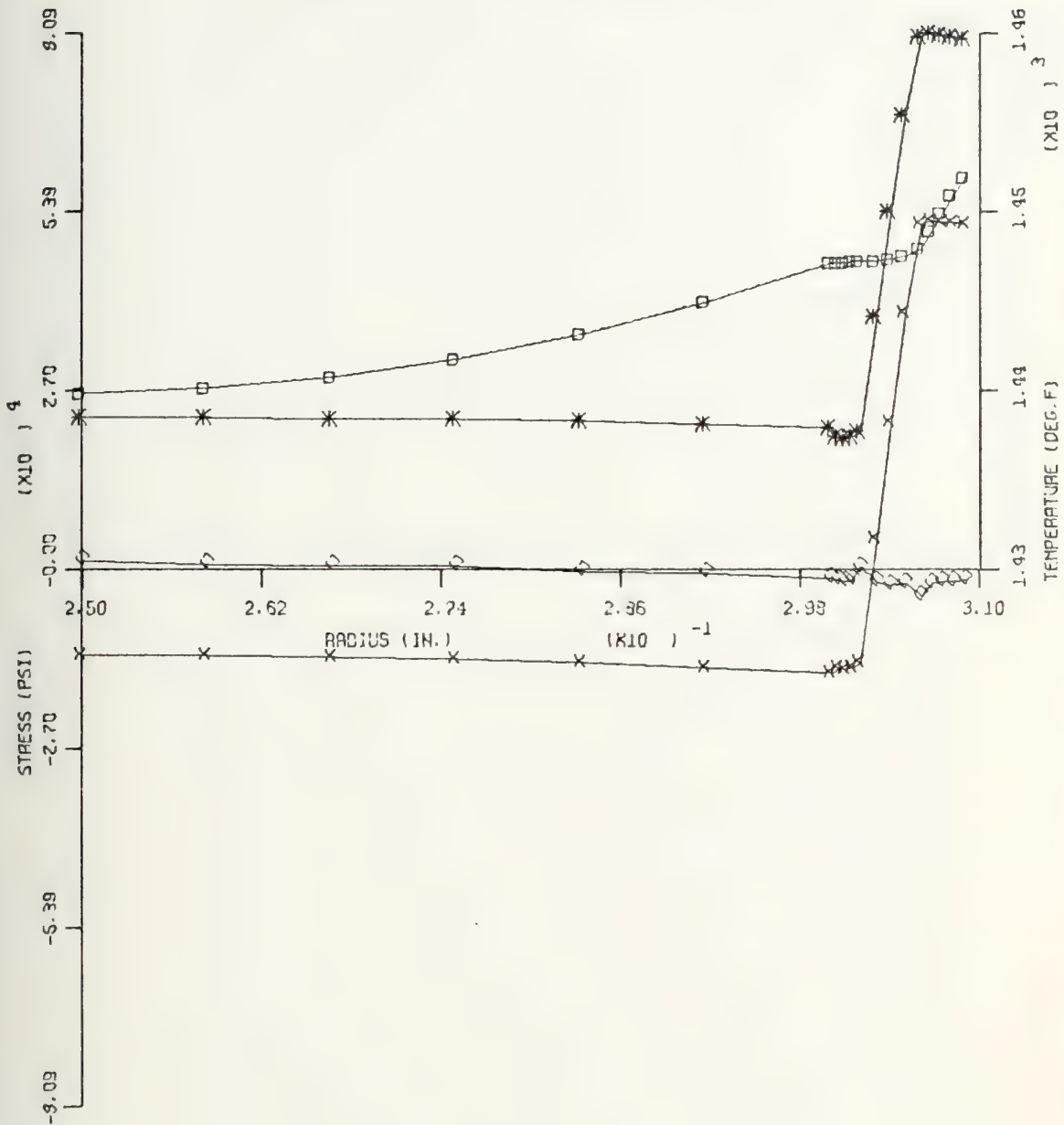
STRESS SCALE = 1.13×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 23(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF. 4-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

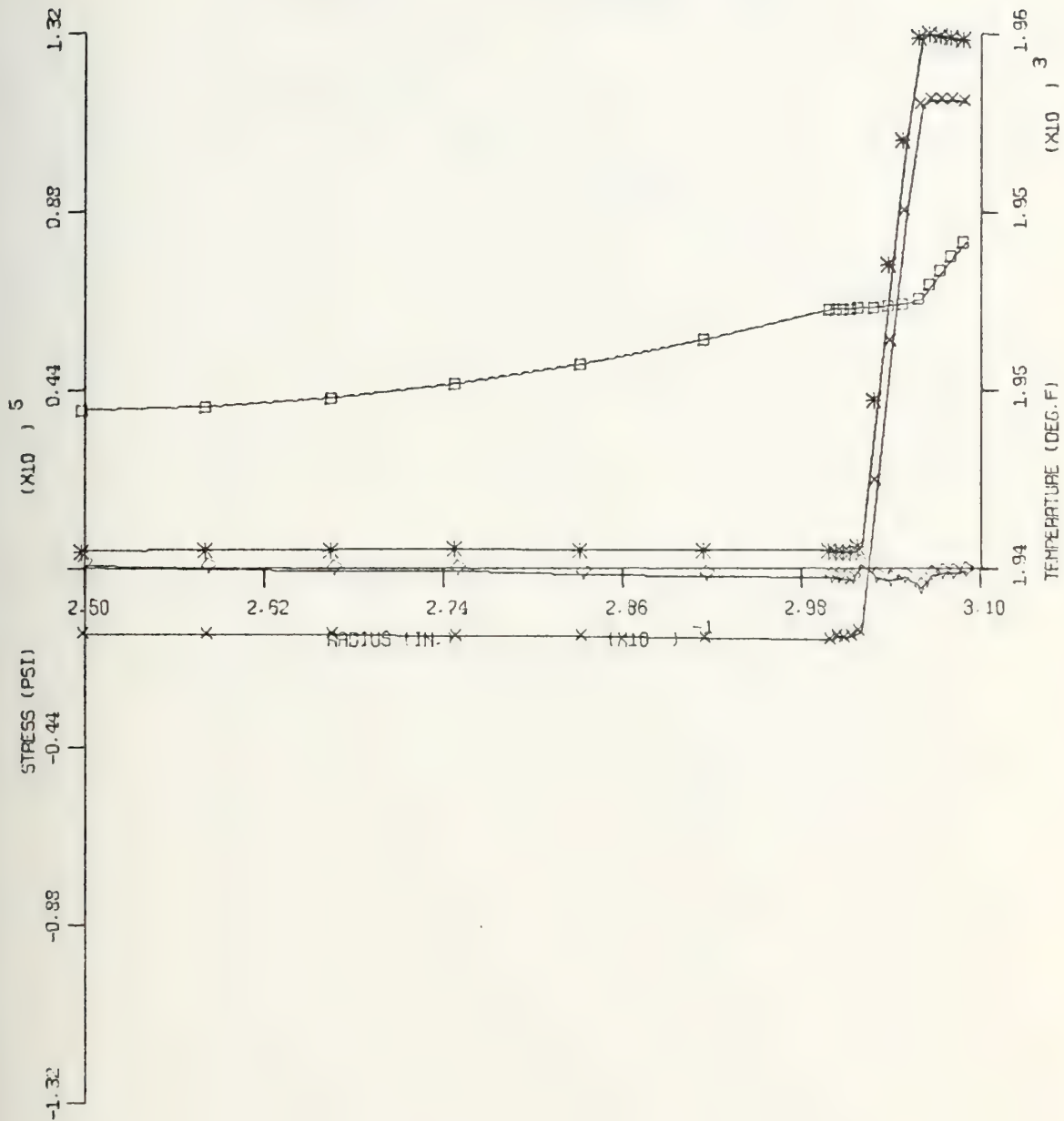
STRESS SCALE = 2.597×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 23(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF. 4-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

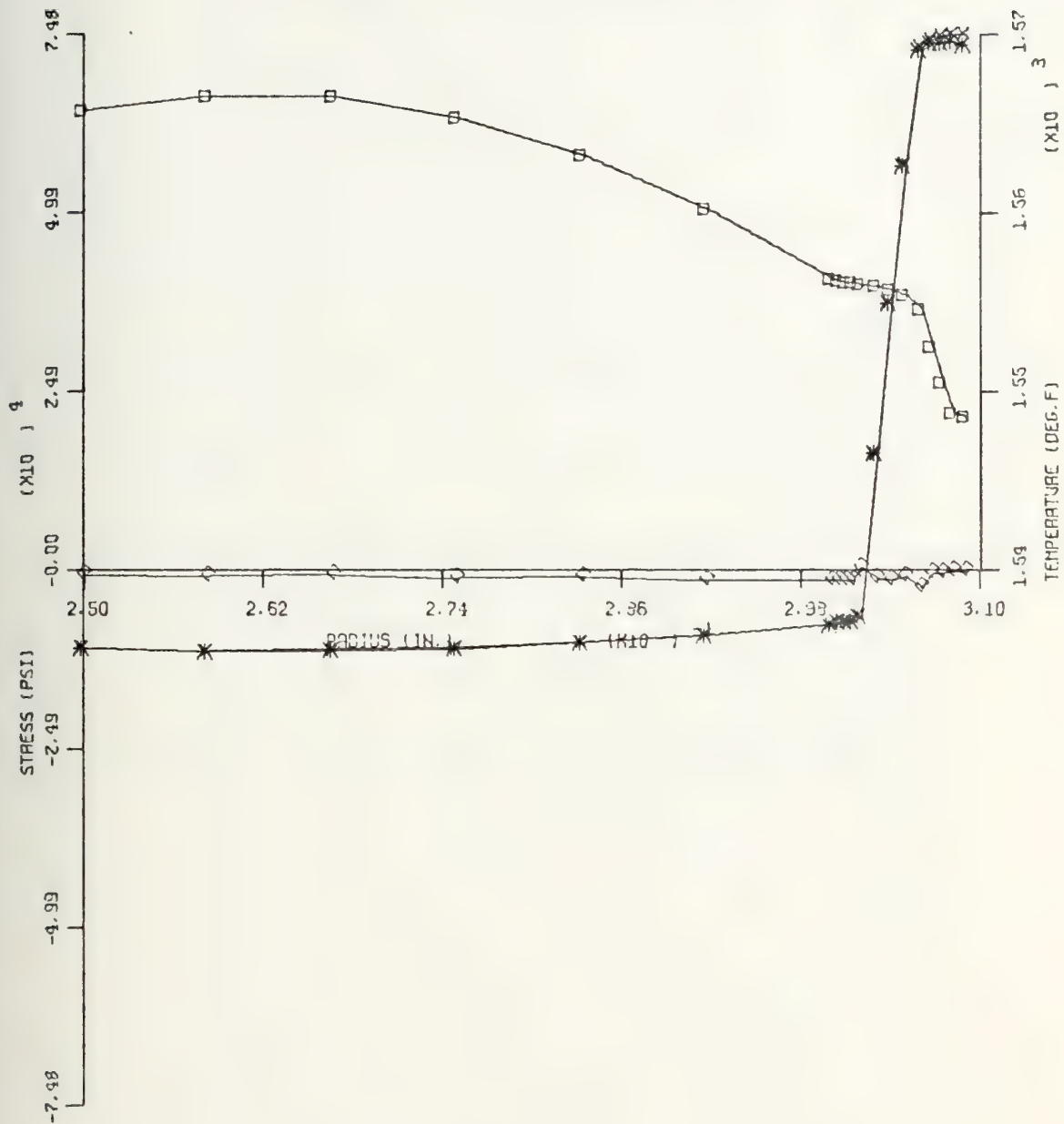
STRESS SCALE = 4.369×10^3 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 23(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF. 4-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

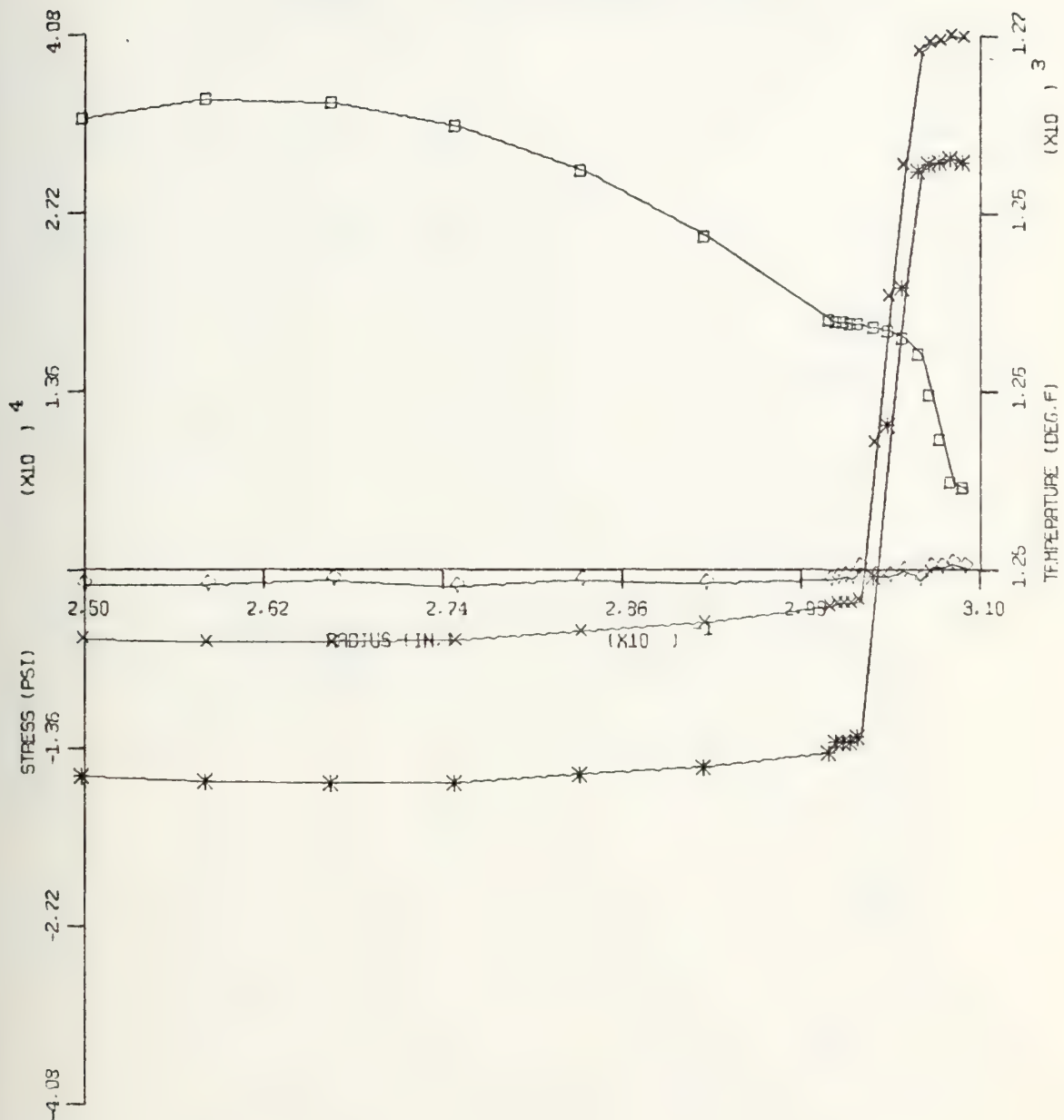
STRESS SCALE = 2.493×10^4 PSI/INCH

TEMPERATURE SCALE = 10.0 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 23(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF. 4-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

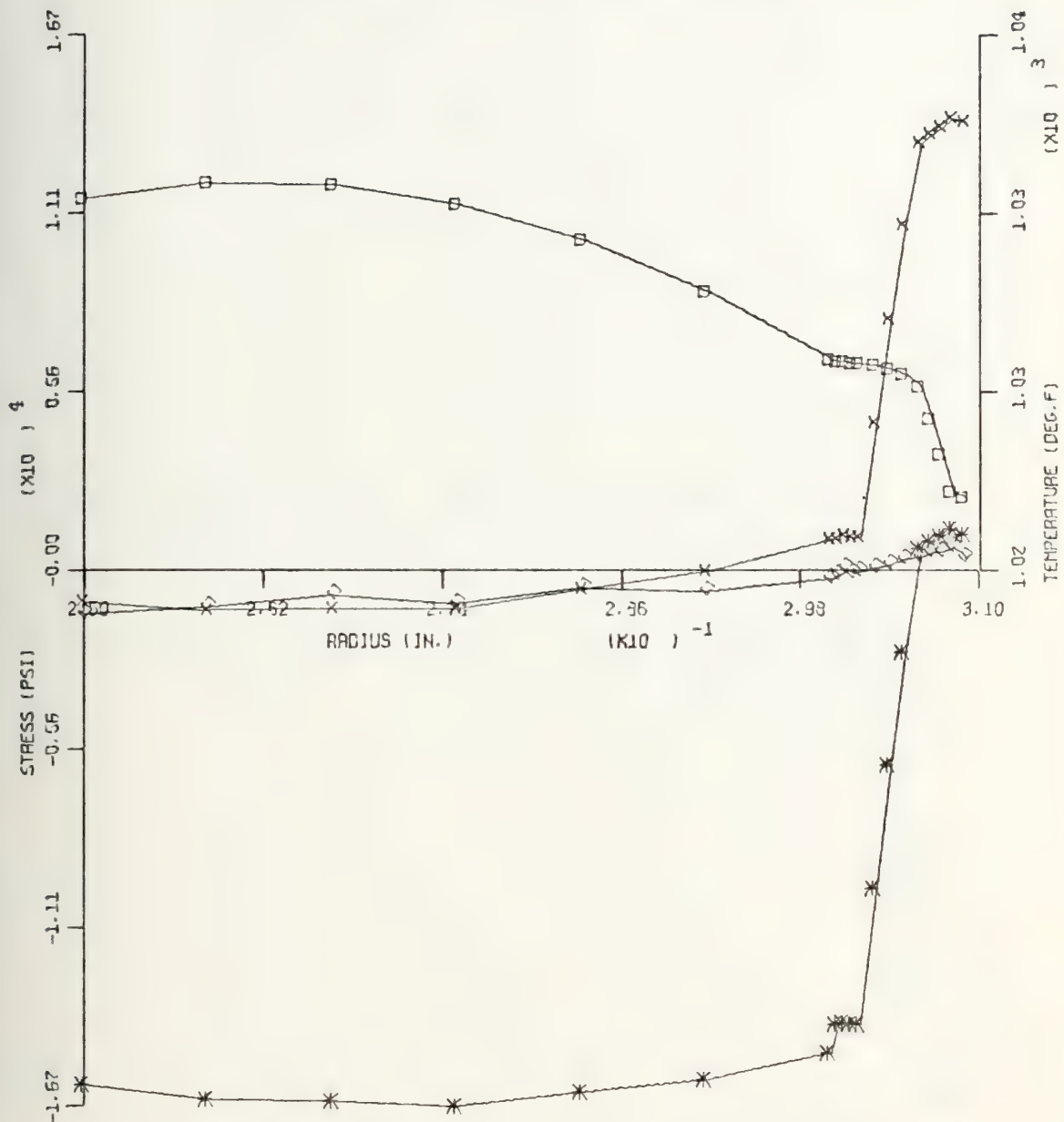
STRESS SCALE = 1.358×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 23(g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.4-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

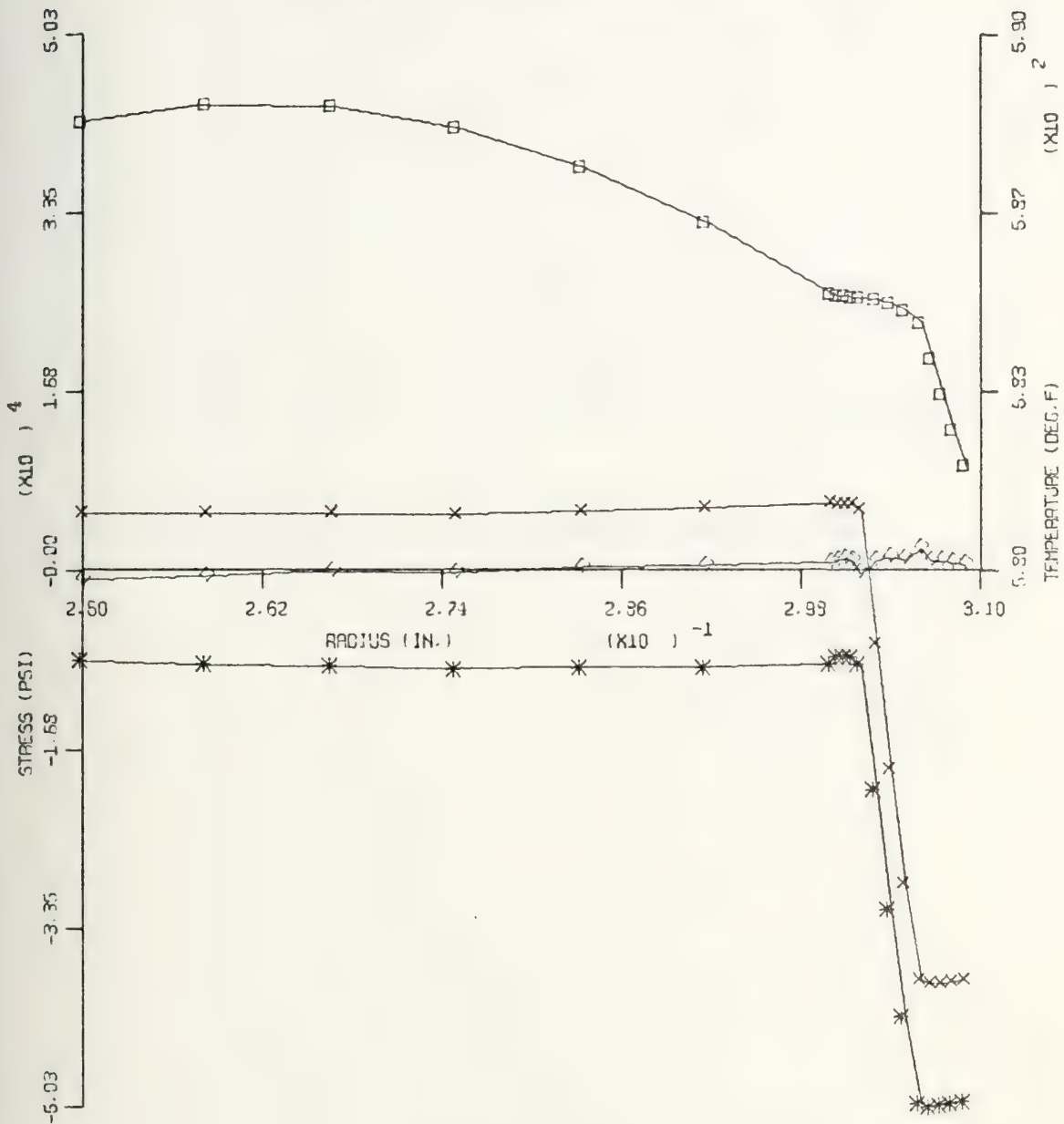
STRESS SCALE = 5.553×10^3 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 23(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.4-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

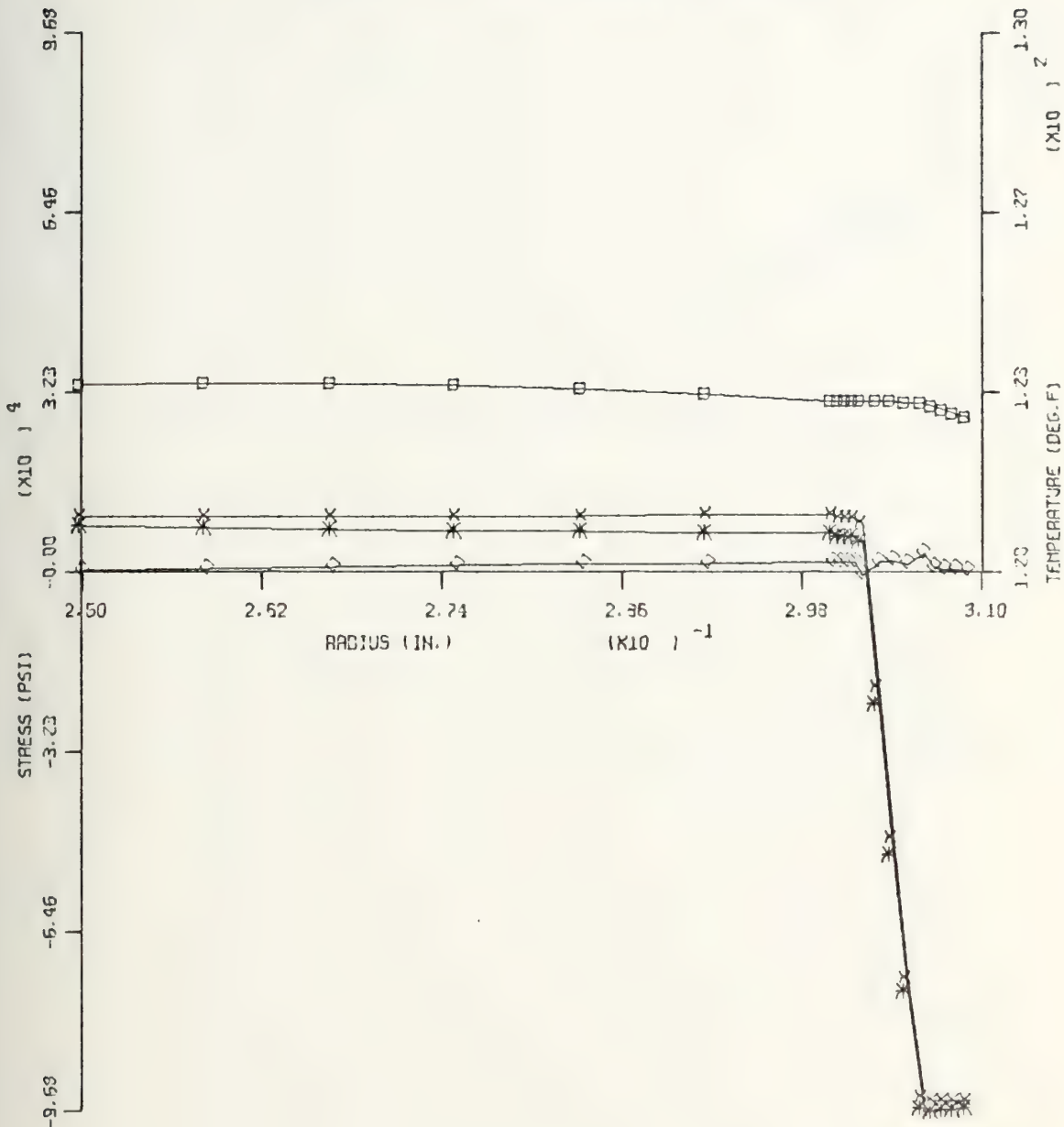
STRESS SCALE = 1.675×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 23(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.4-60



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

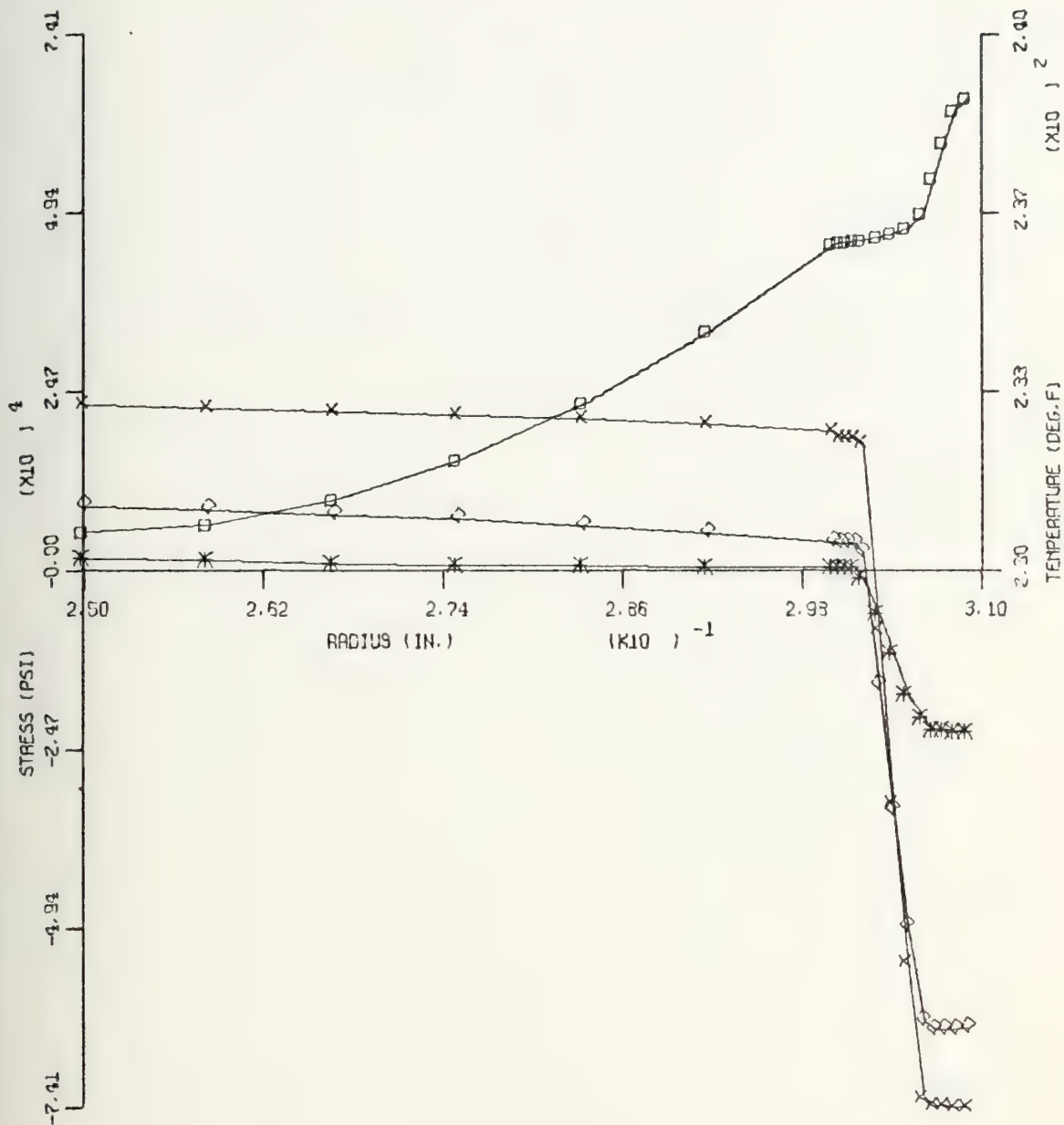
STRESS SCALE = 3.229×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 23(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.4-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

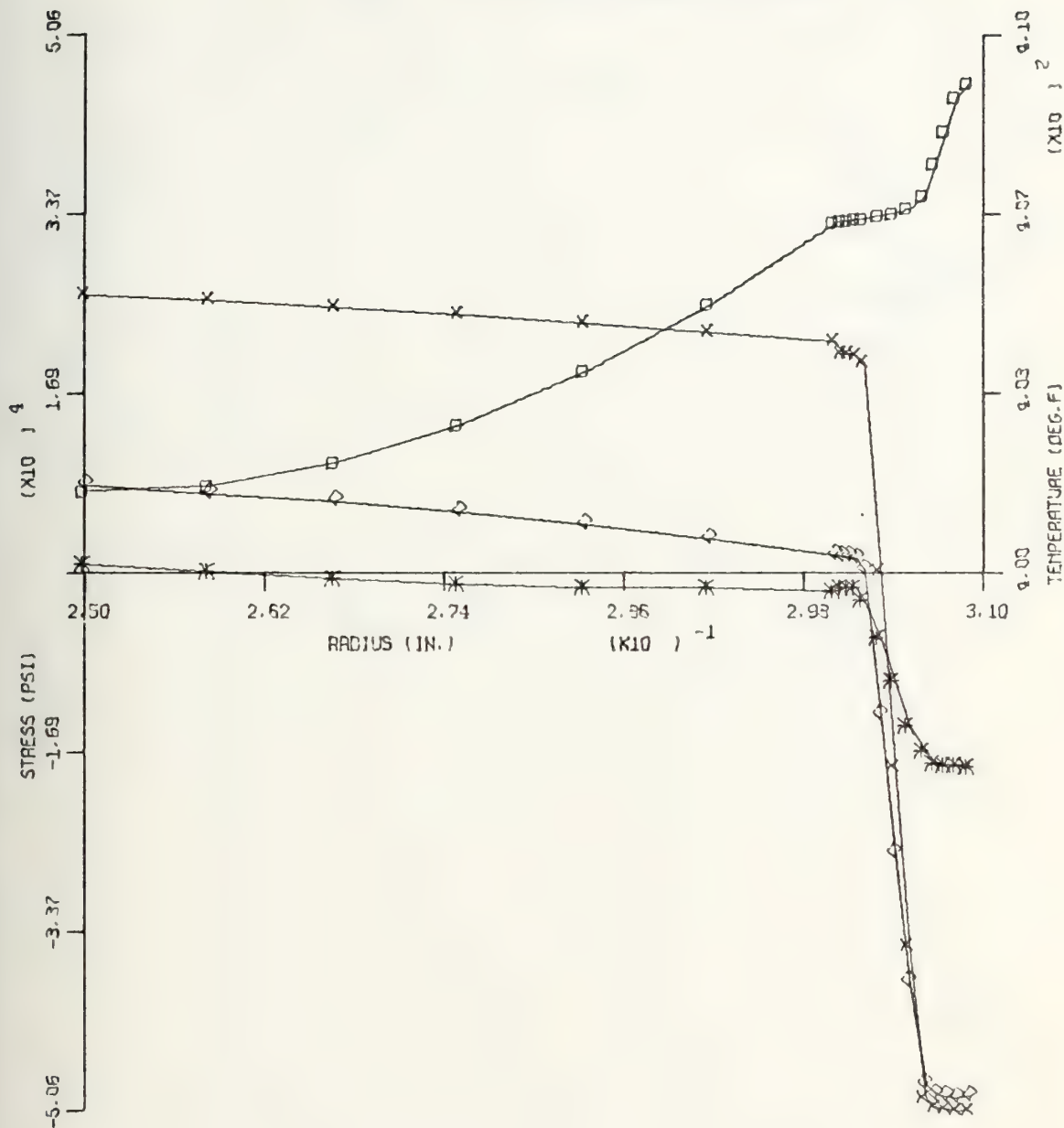
STRESS SCALE = 2.472×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 24(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.4-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

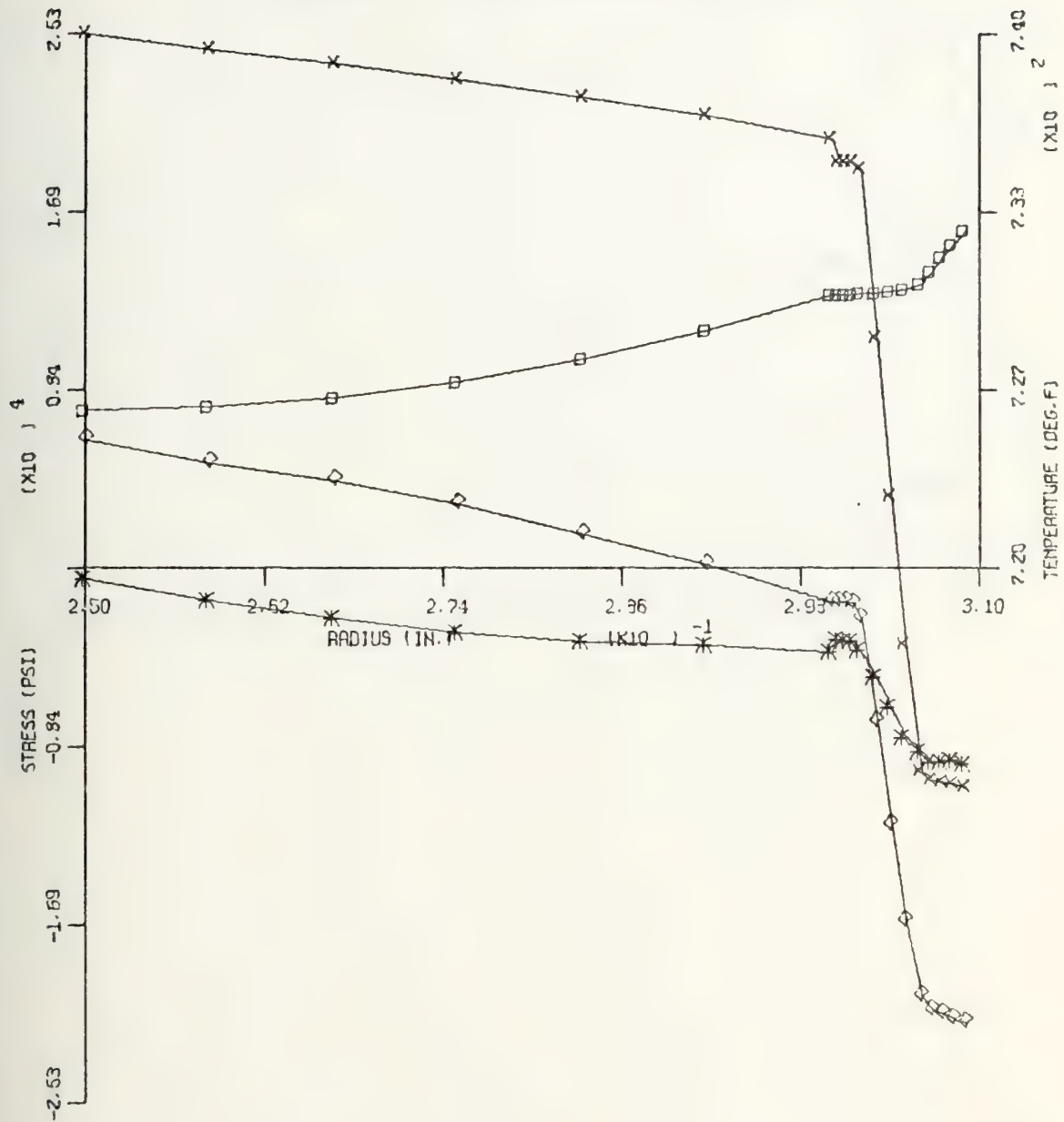
STRESS SCALE = 1.667×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 24(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.4-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

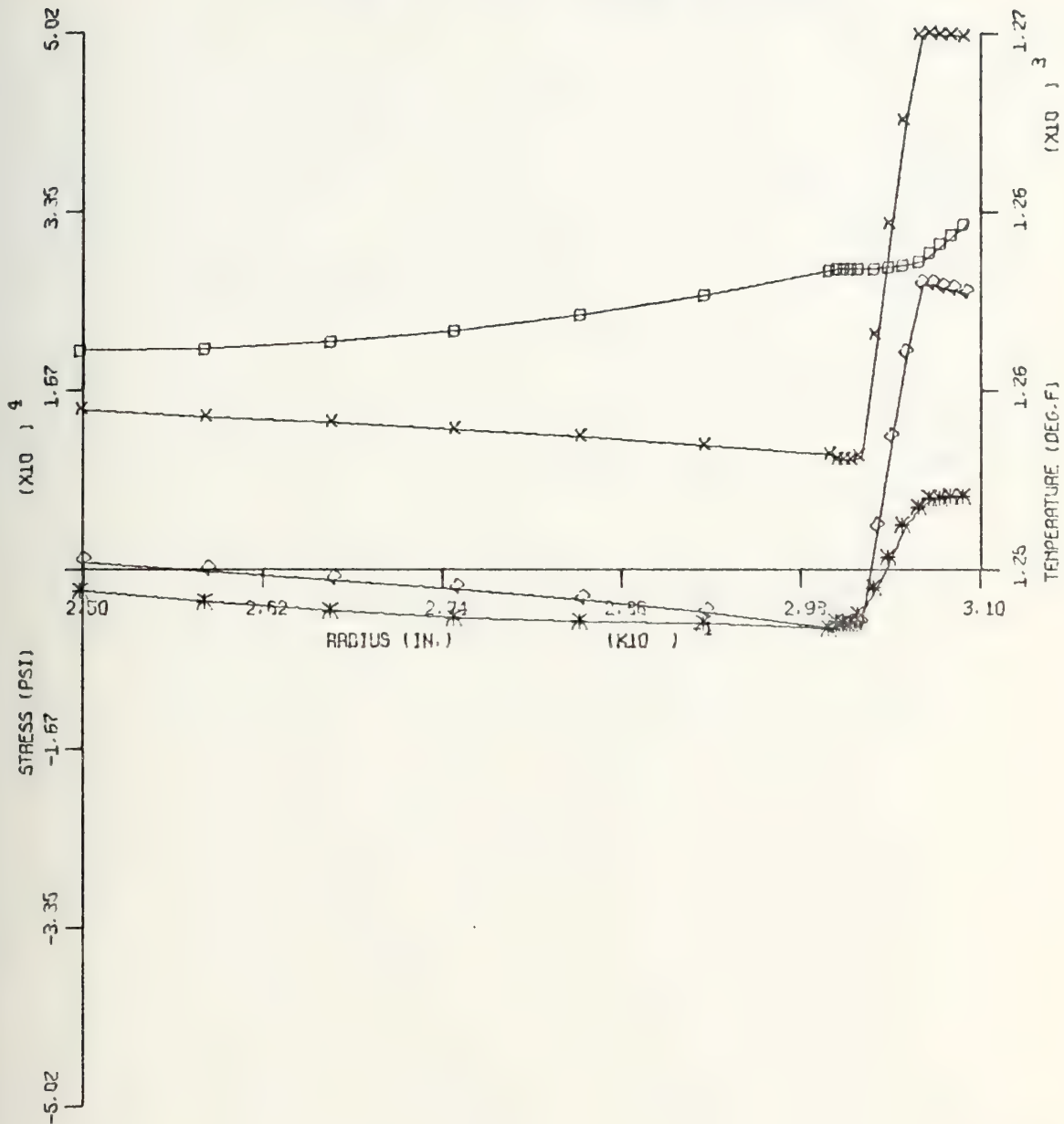
STRESS SCALE = 8.447×10^3 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 24(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.4-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

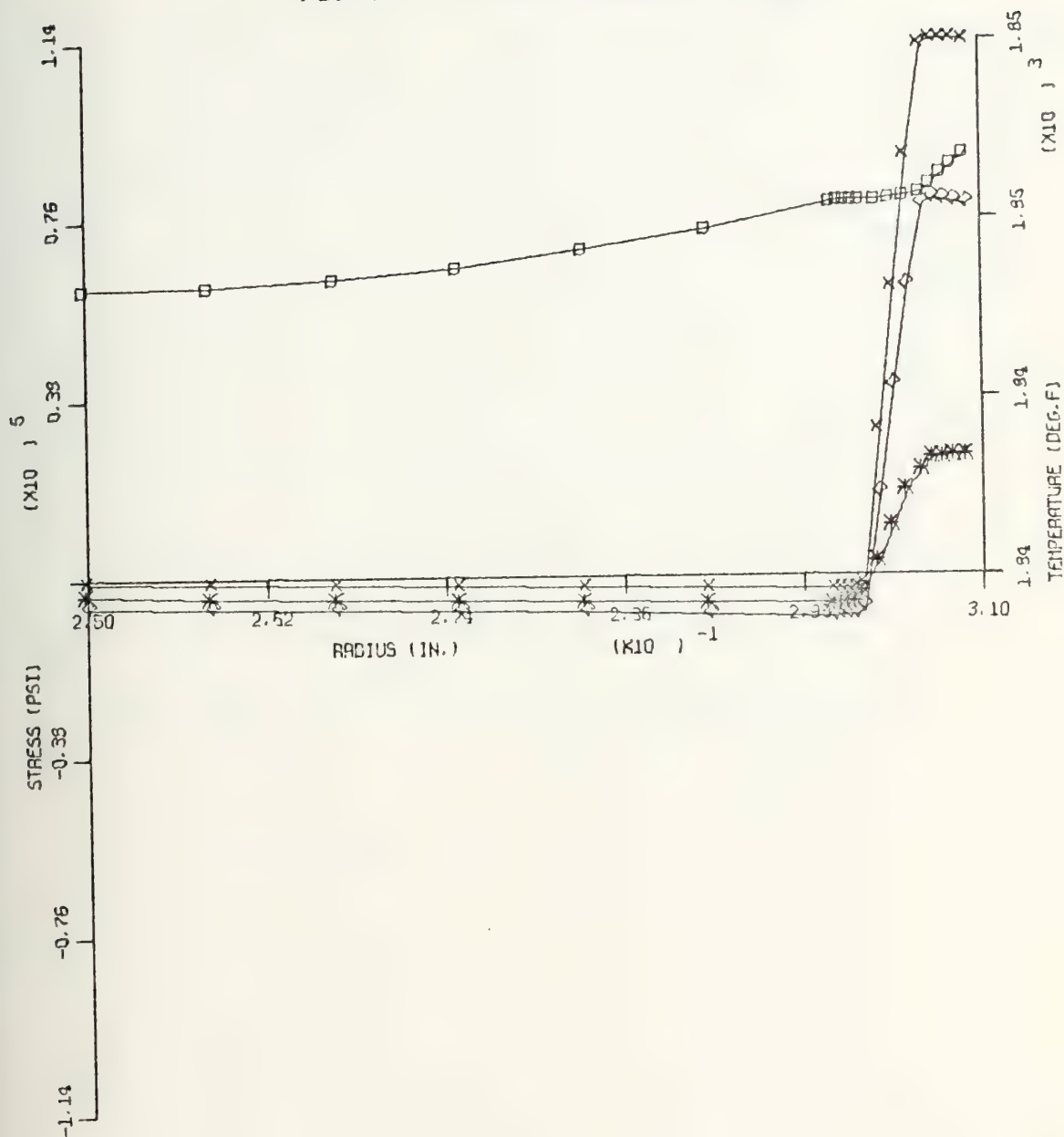
STRESS SCALE = 1.674×10^4 PSI/INCH

TEMPERATURE SCALE = 6.2 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 24(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.4-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

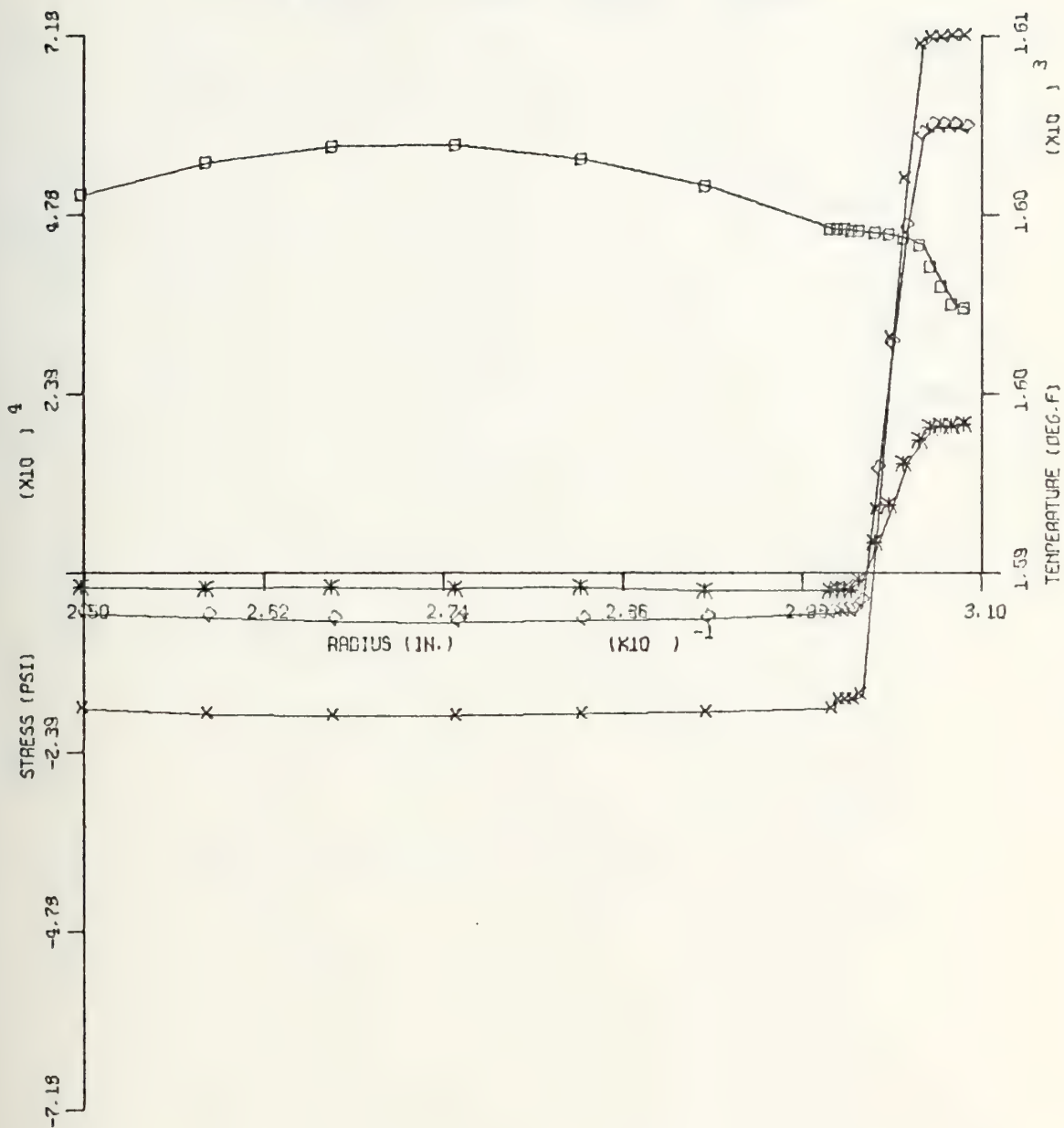
STRESS SCALE = 3.502×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 24(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.4-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

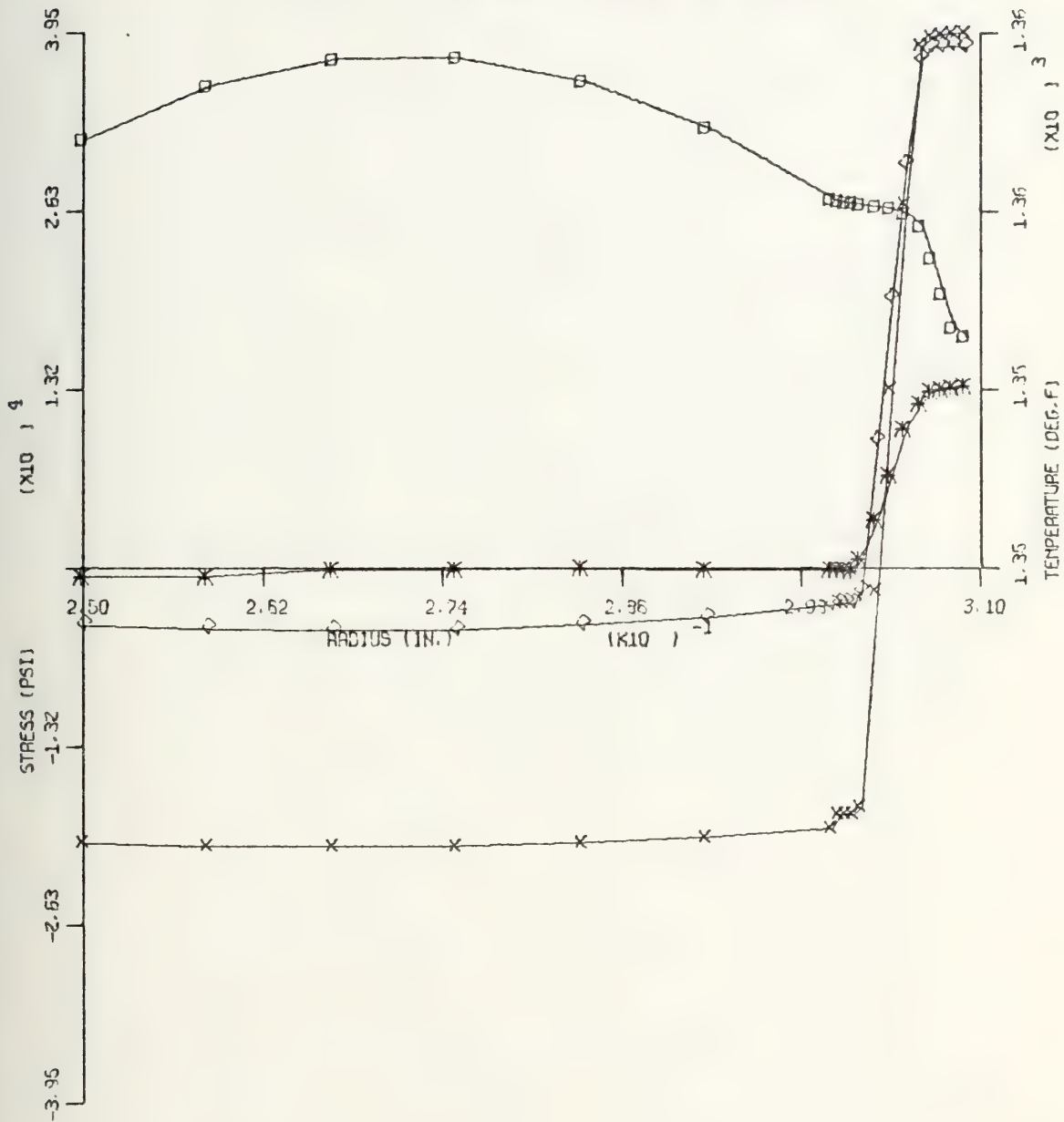
STRESS SCALE = 2.332×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 24(f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF.4-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

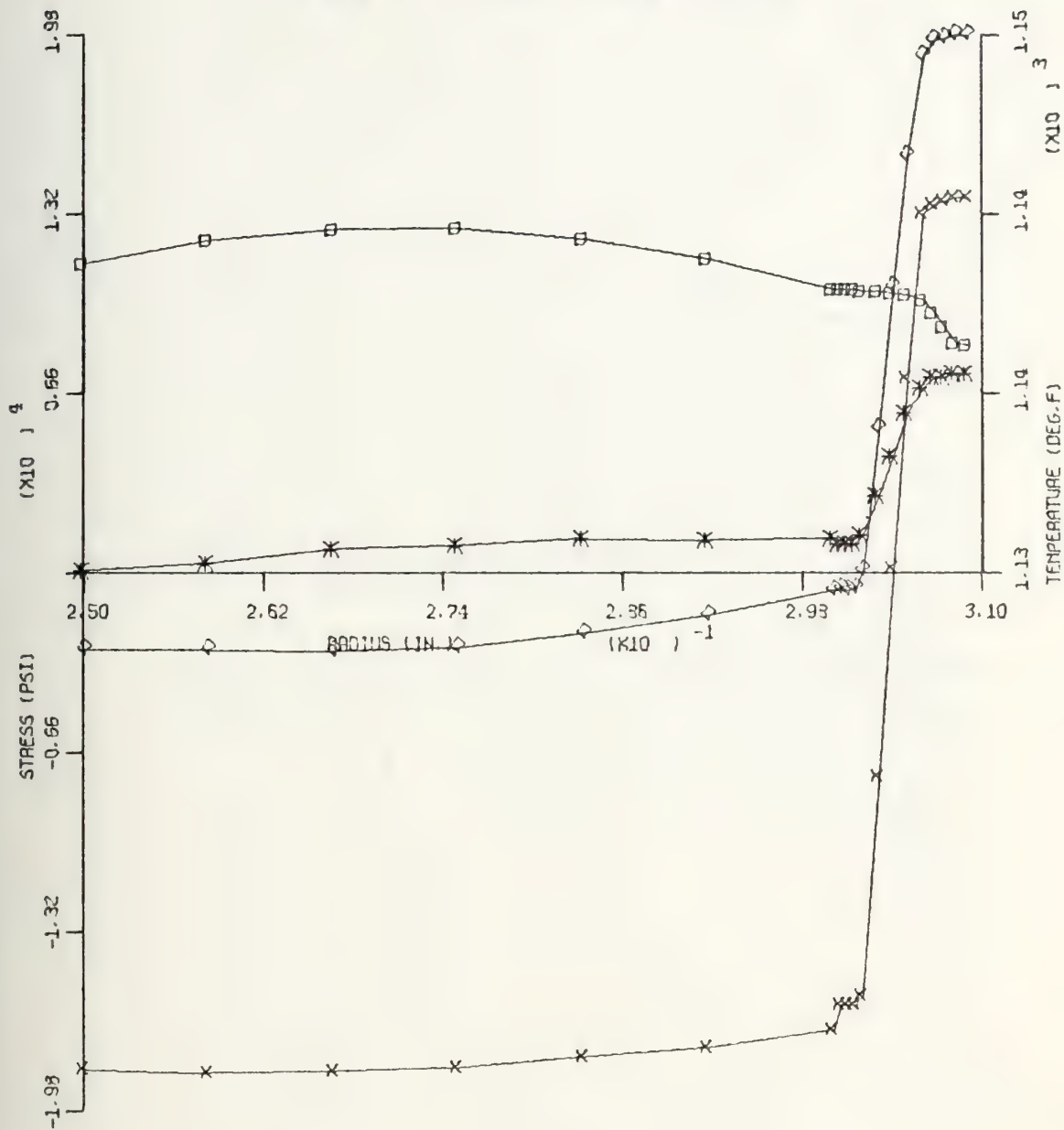
STRESS SCALE = 1.312×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 24(g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.4-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

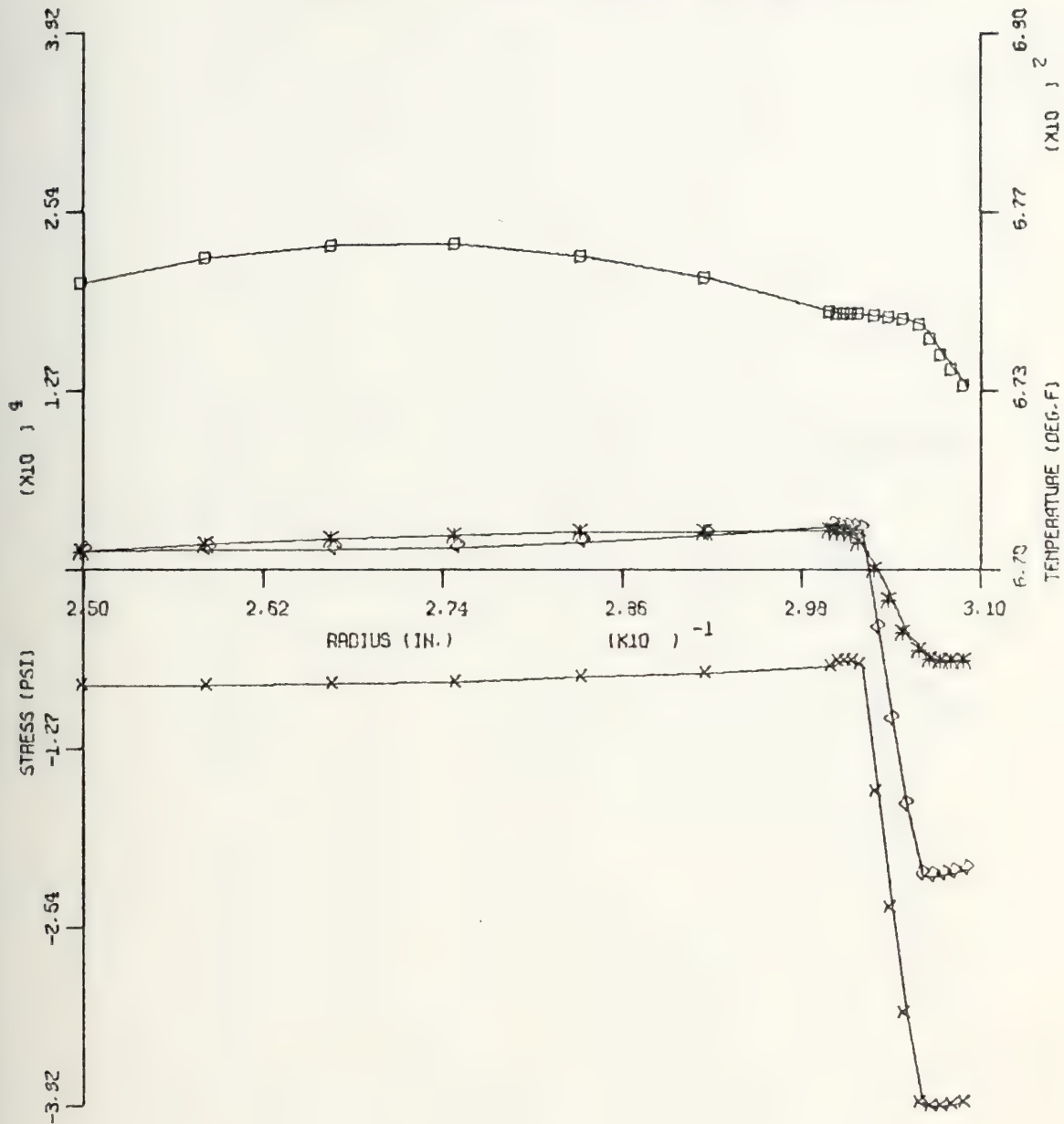
STRESS SCALE = 6.590×10^3 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 24(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.4-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

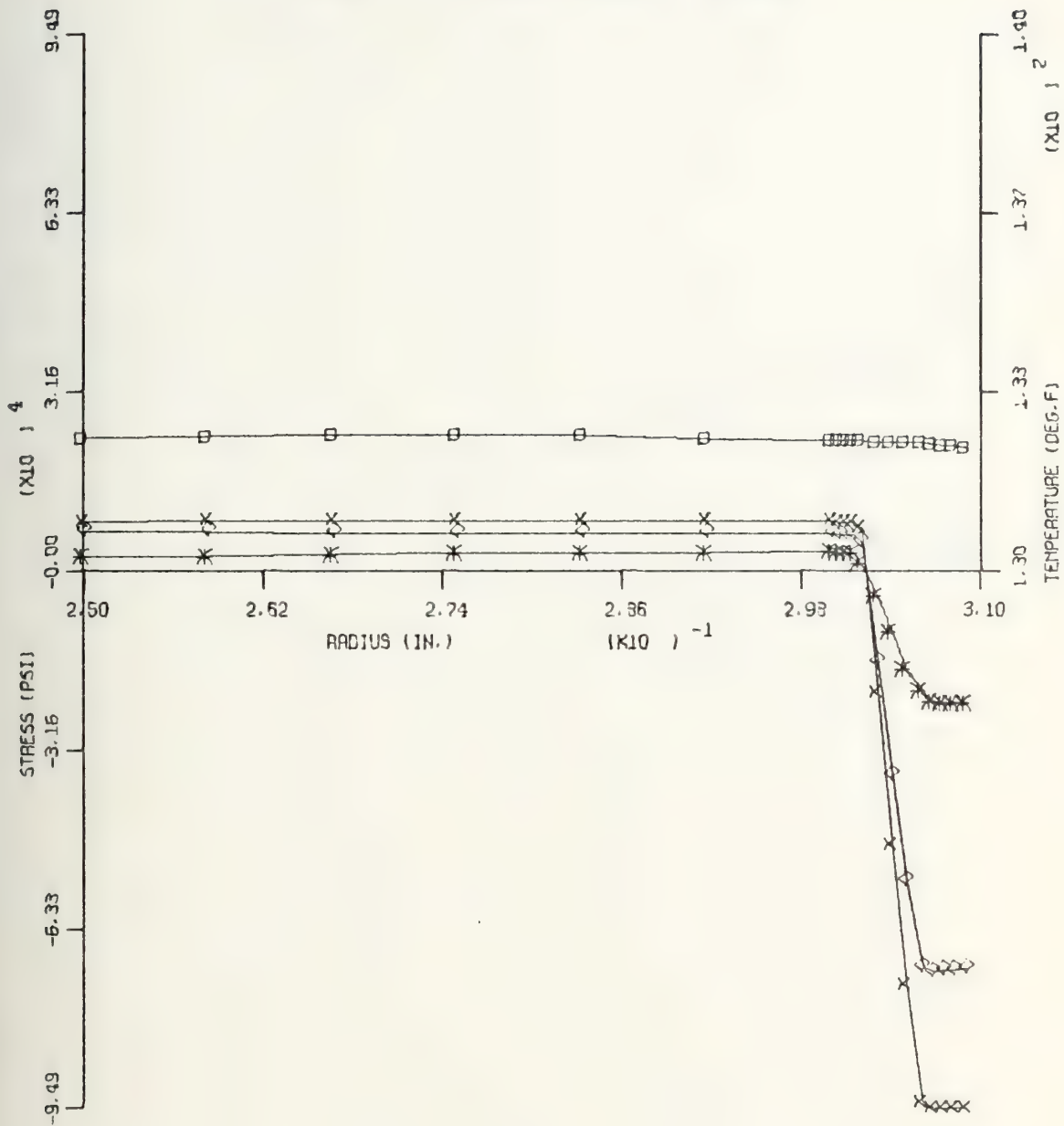
STRESS SCALE = 1.272×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 24(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.4-120



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

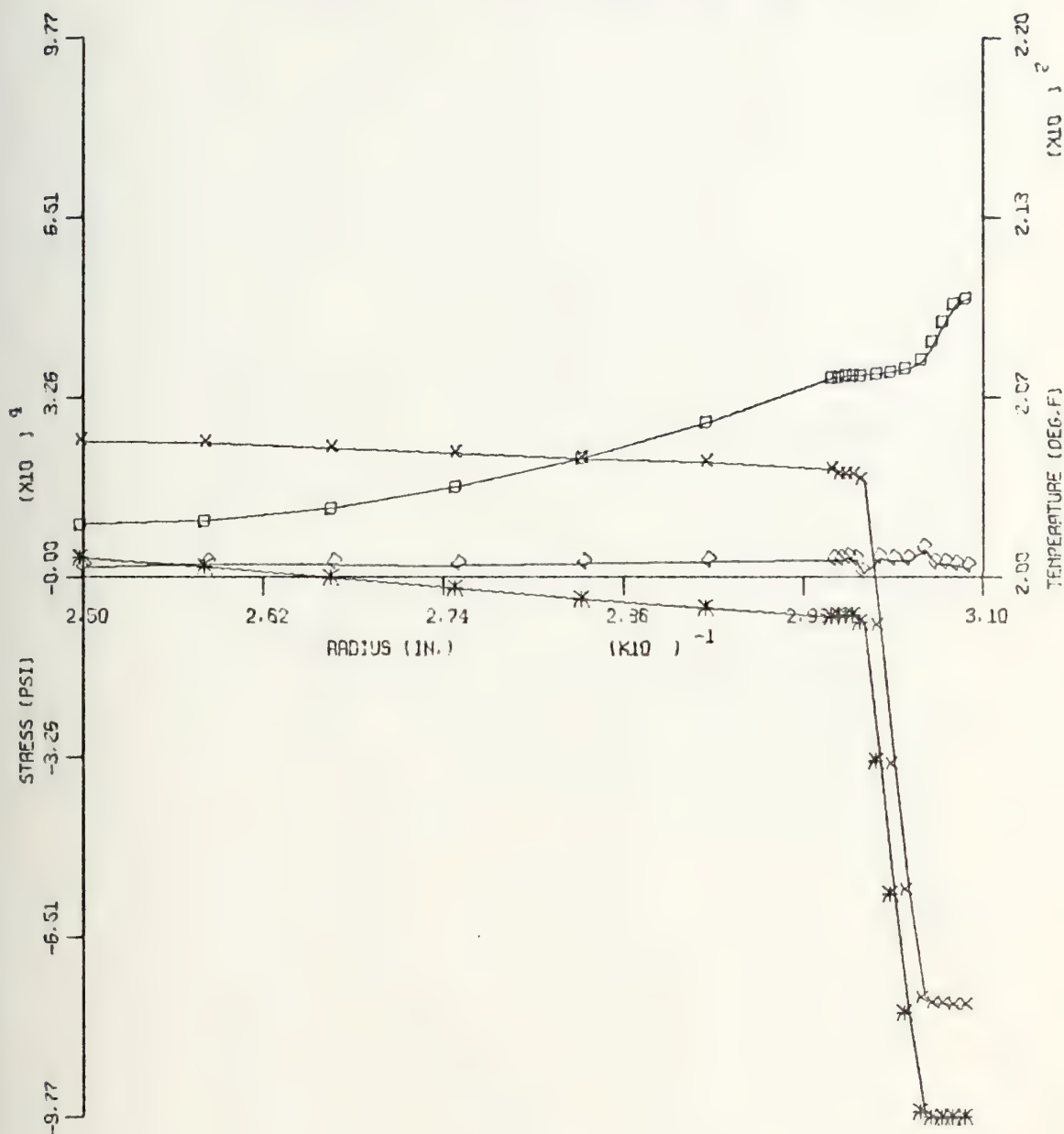
STRESS SCALE = 3.164×10^3 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 24(j)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.050 CONF.4-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

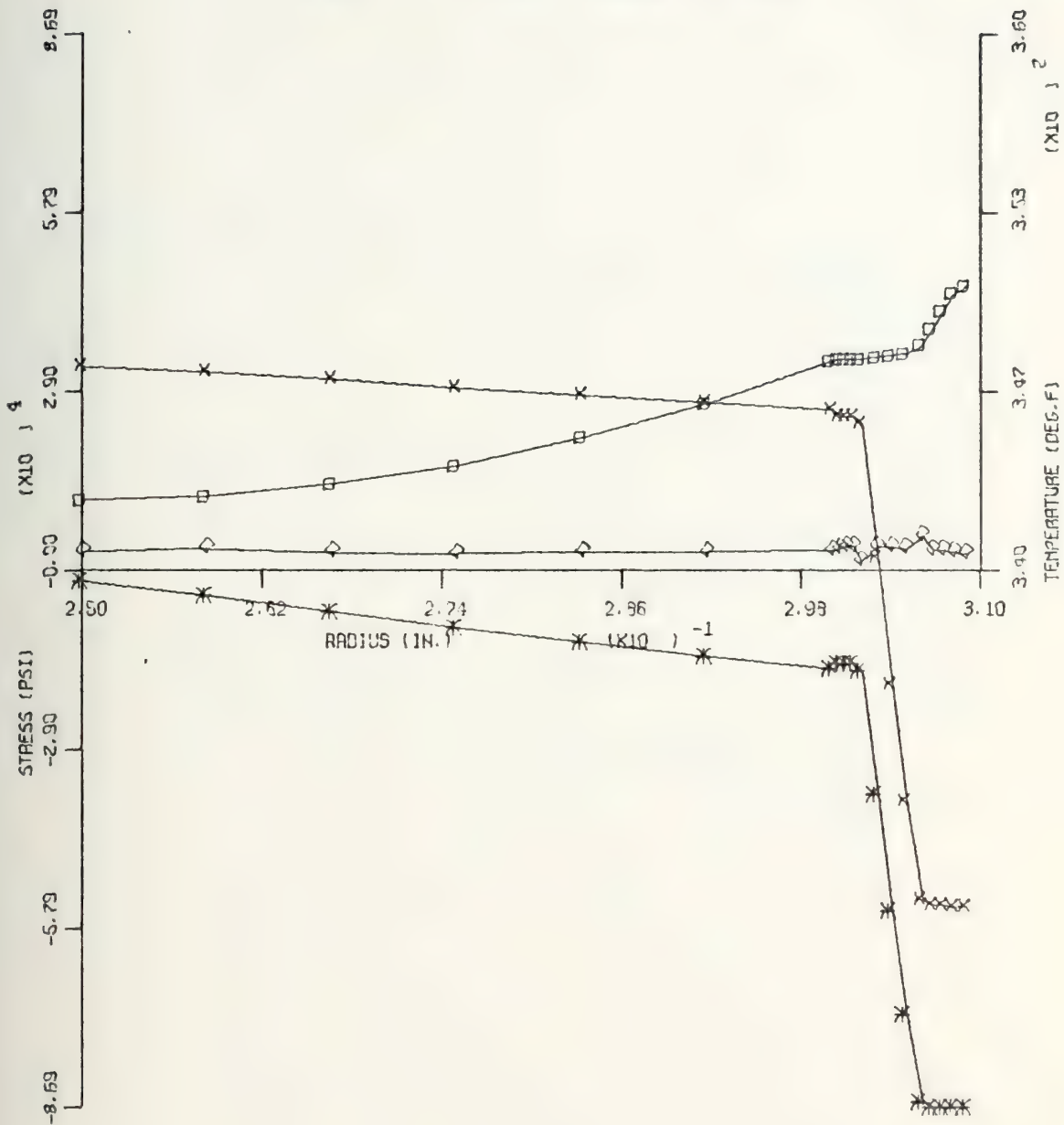
STRESS SCALE = 3.256×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 25(a)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.100 CONF.4-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

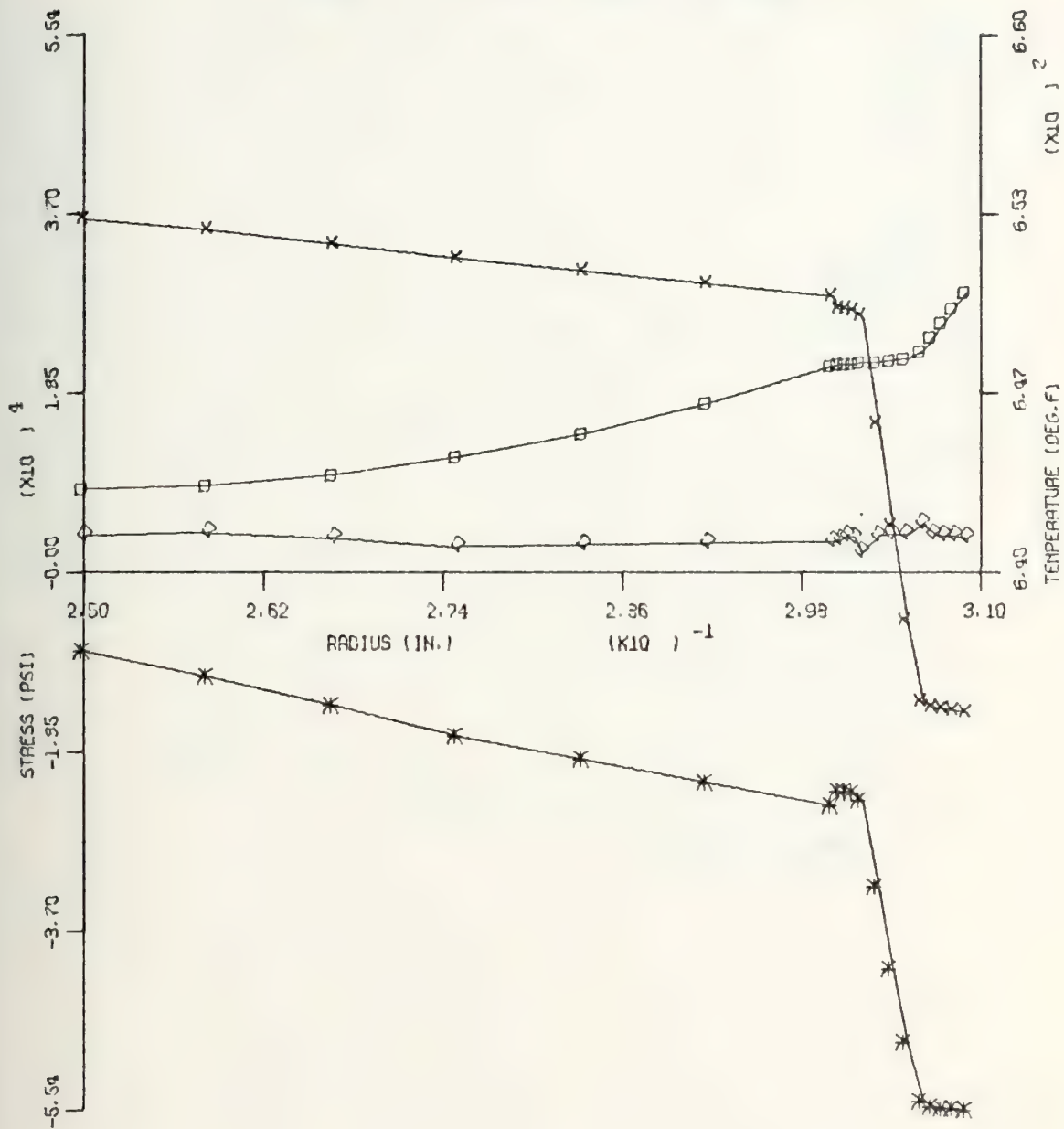
STRESS SCALE = 2.996×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 25(b)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.200 CONF.4-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

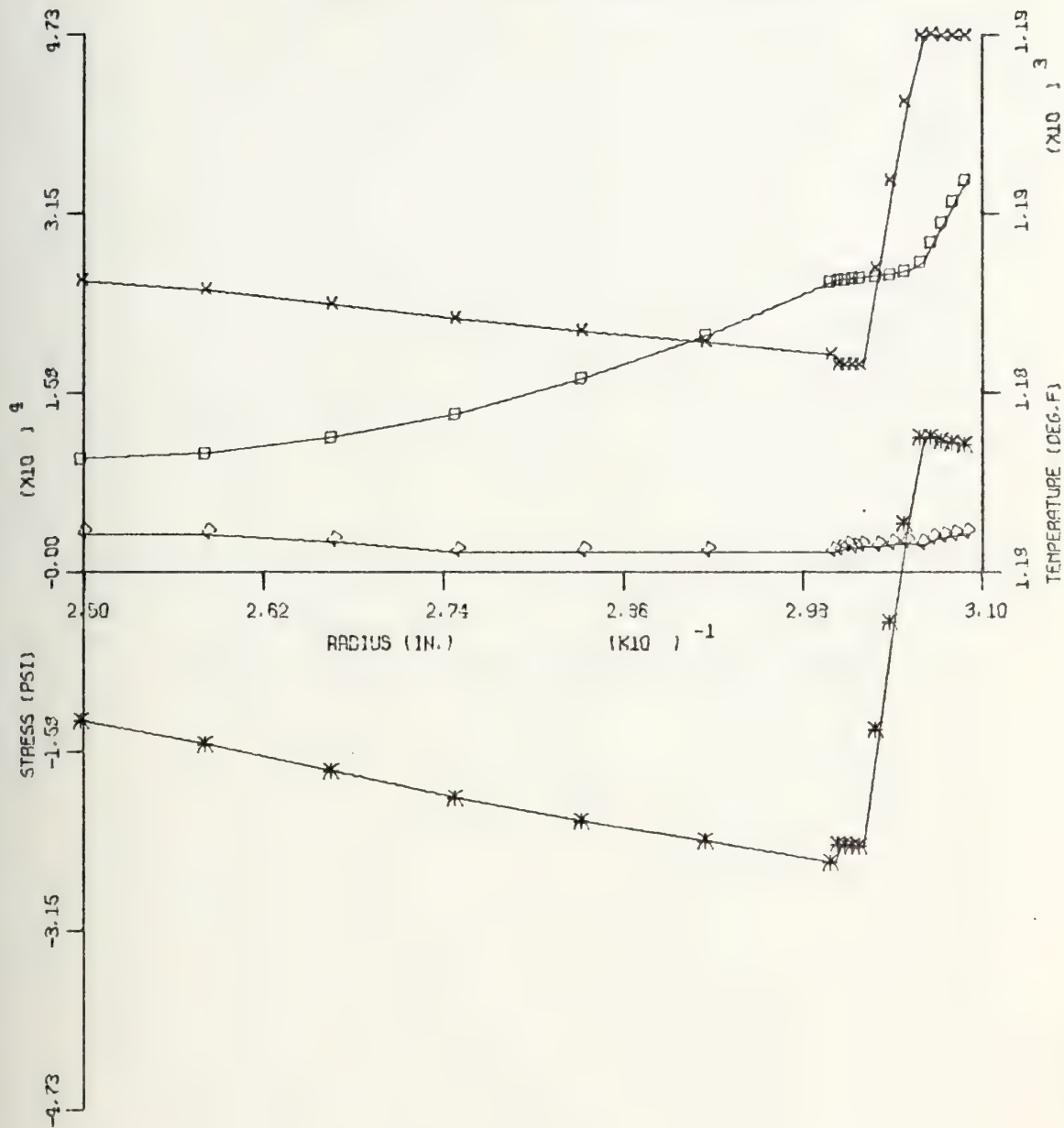
STRESS SCALE = 1.843×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 25(c)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.400 CONF.4-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

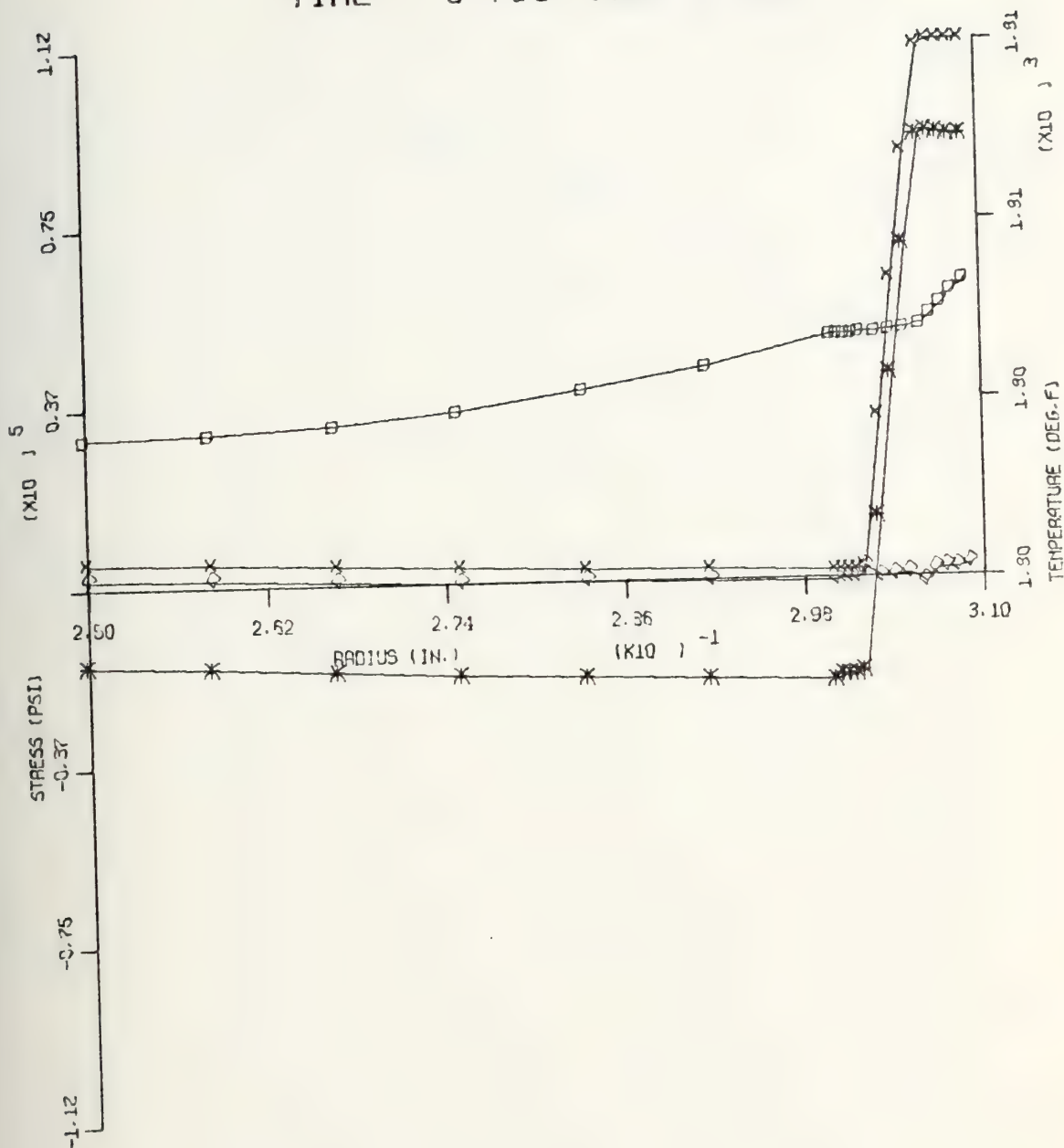
STRESS SCALE = 1.577×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 25(d)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.750 CONF.4-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

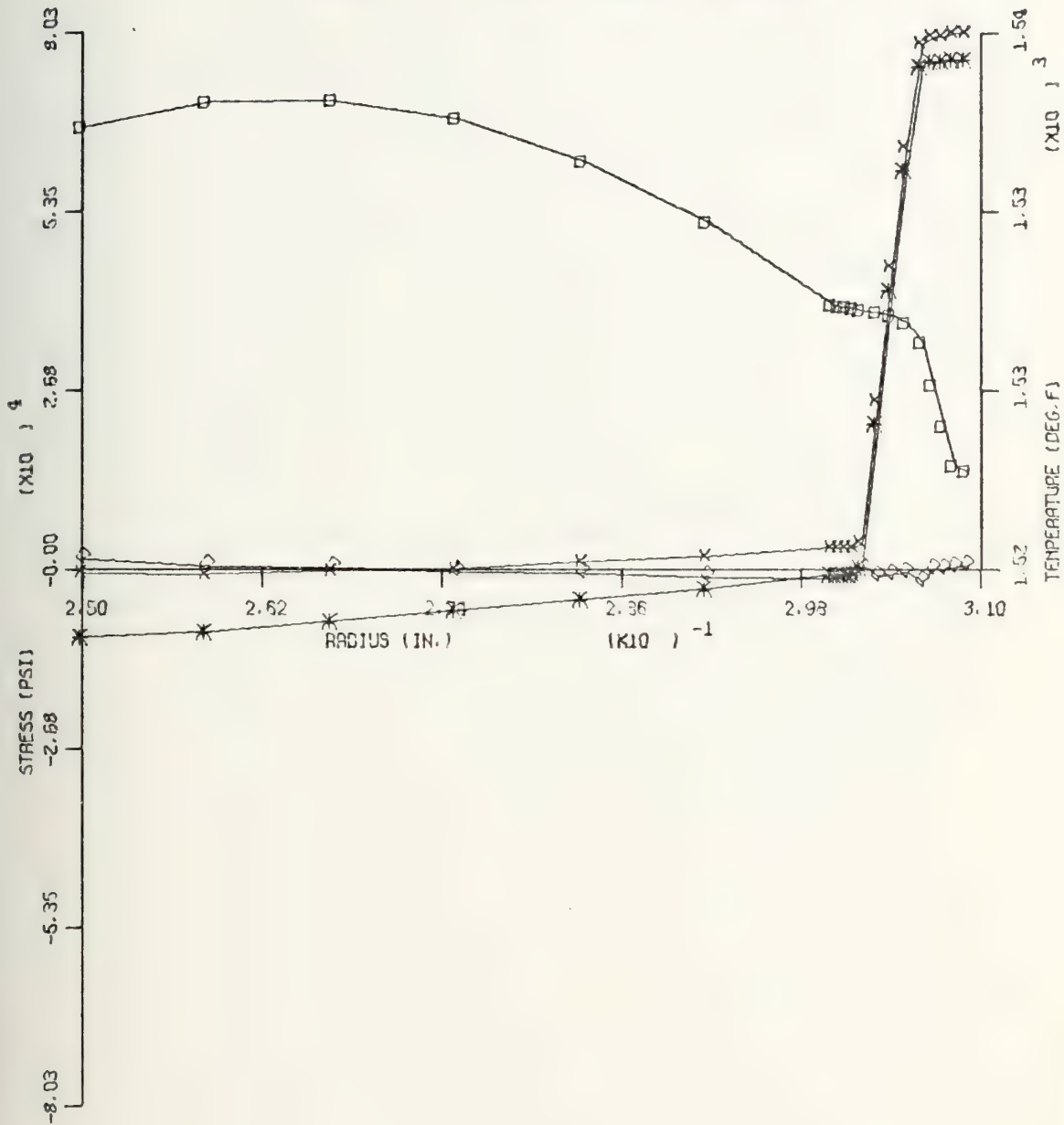
STRESS SCALE = 3.75×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 25(e)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.800 CONF.4-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

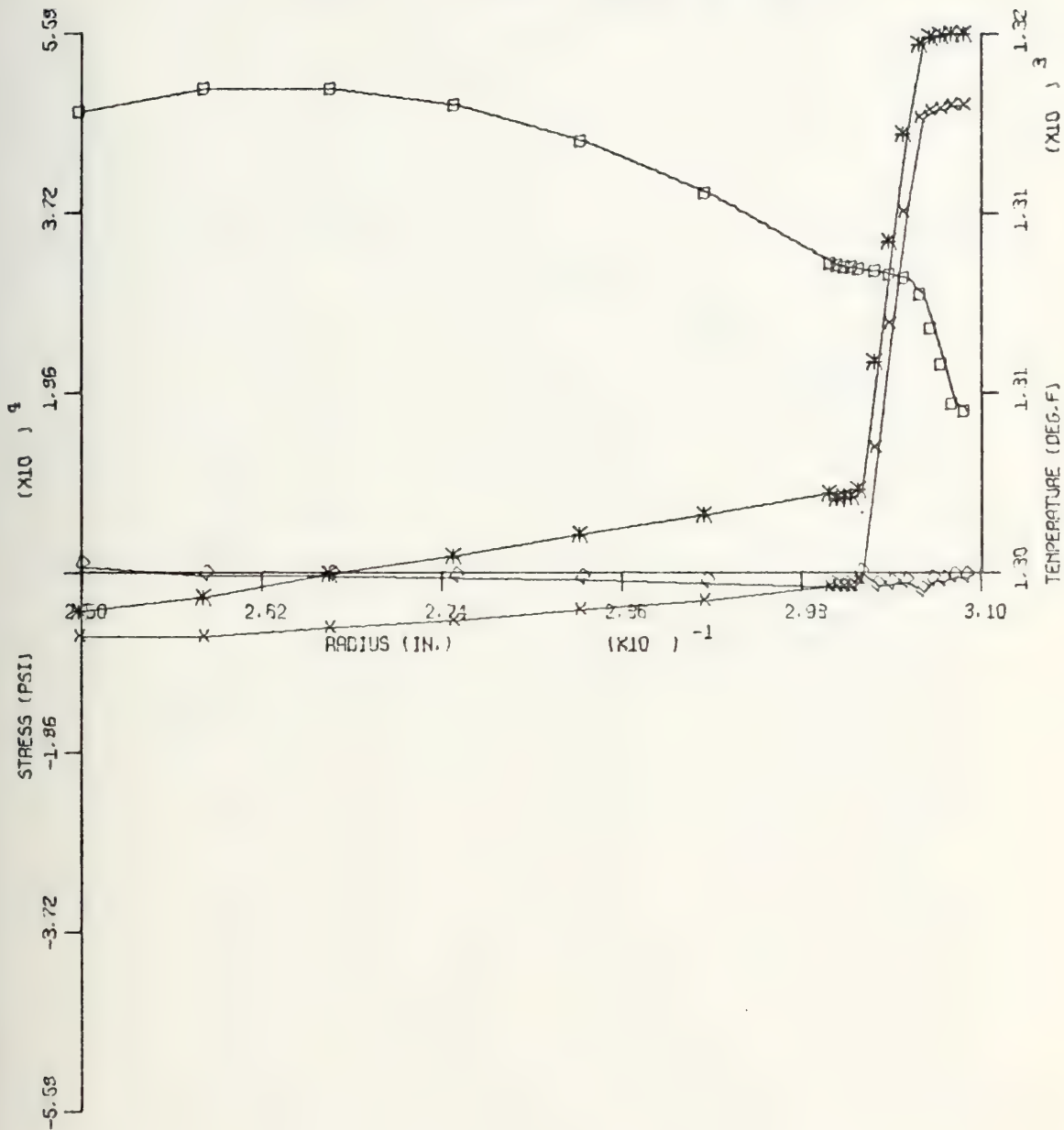
STRESS SCALE = 2.575×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 25 (f)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.850 CONF. 4-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

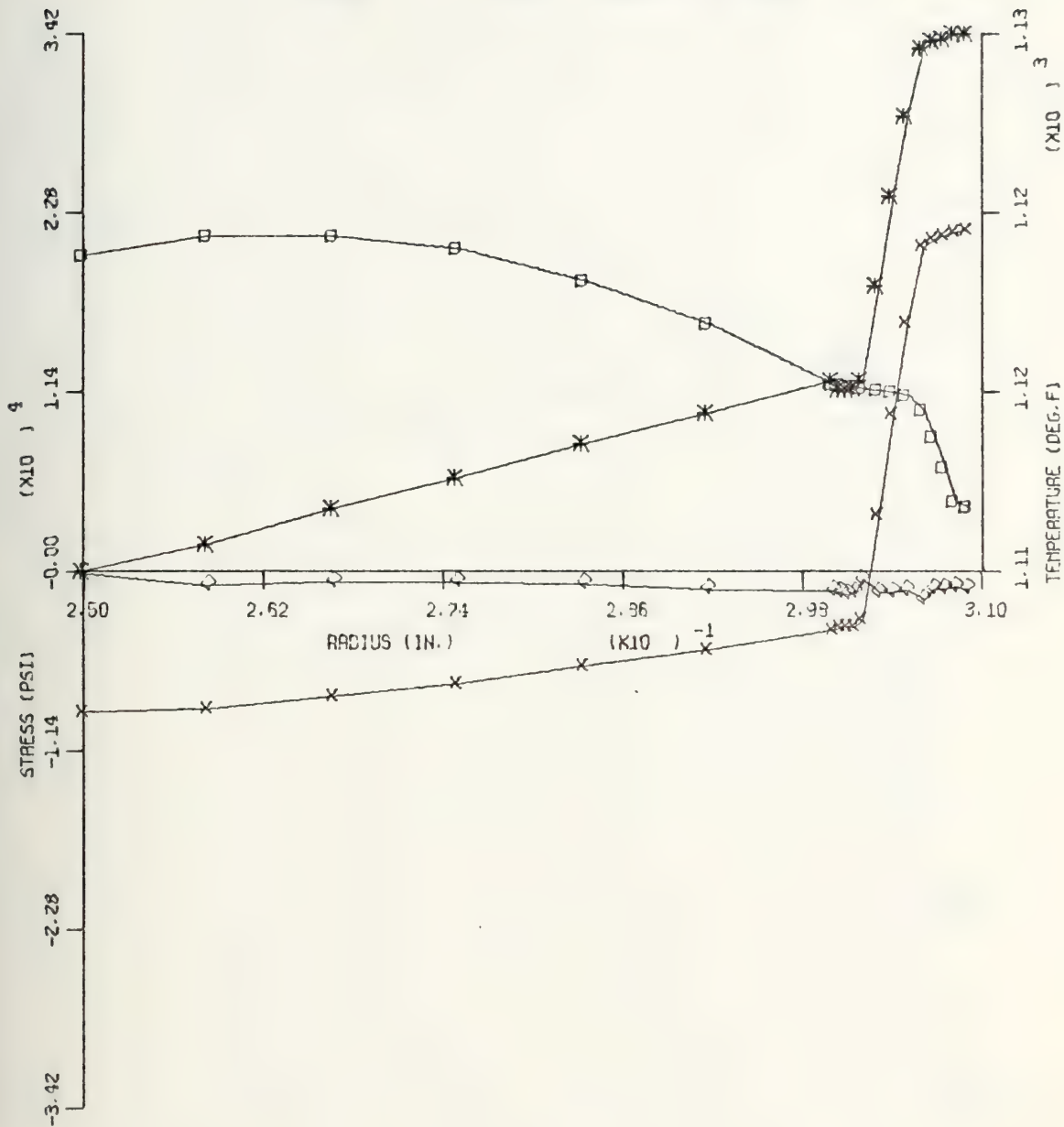
STRESS SCALE = 1.853×10^4 PSI/INCH

TEMPERATURE SCALE = 6.7 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 25(g)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 0.900 CONF.4-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

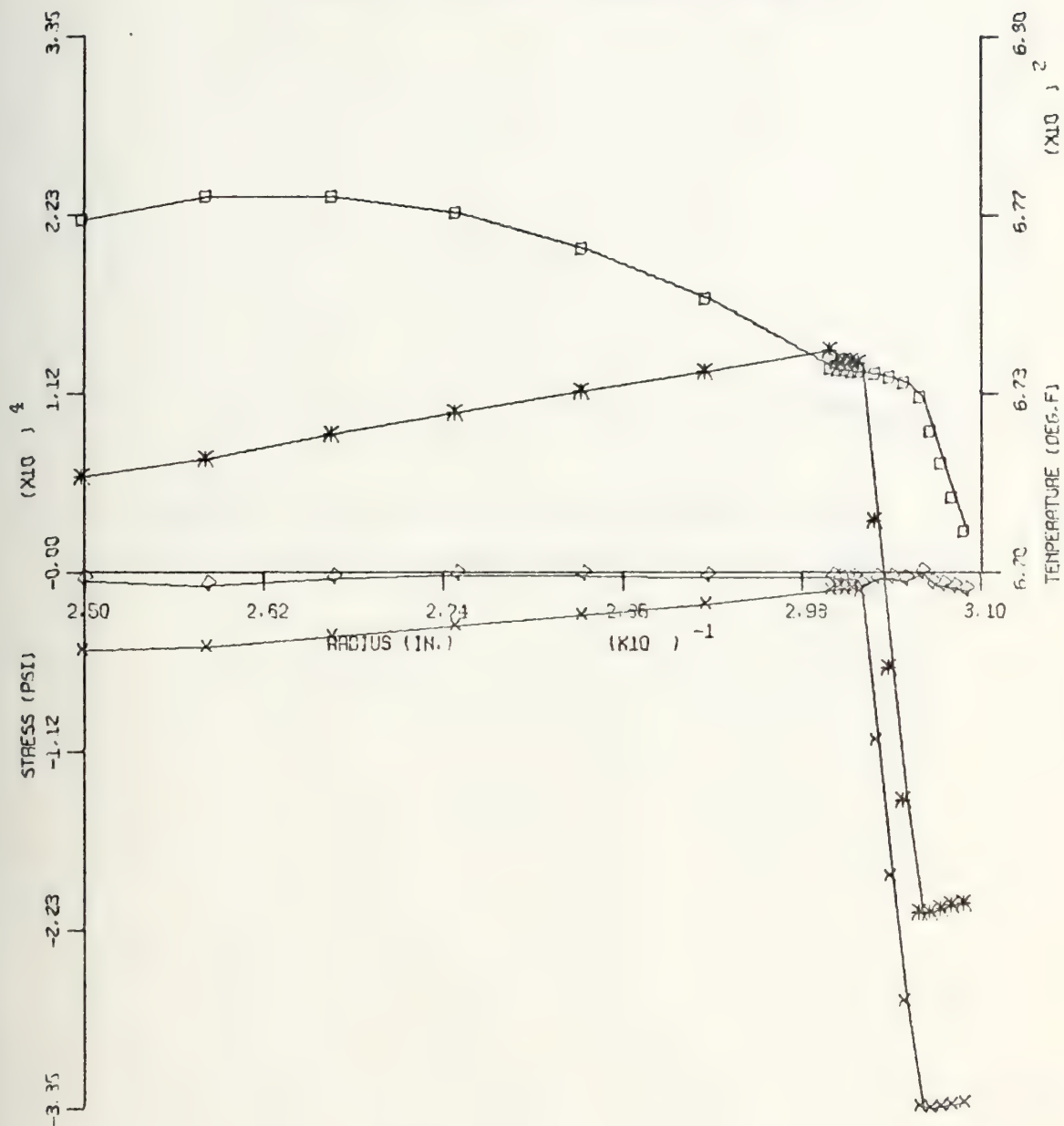
STRESS SCALE = 1.141×10^4 PSI/INCH

TEMPERATURE SCALE = 6.2 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 25(h)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.050 CONF.4-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

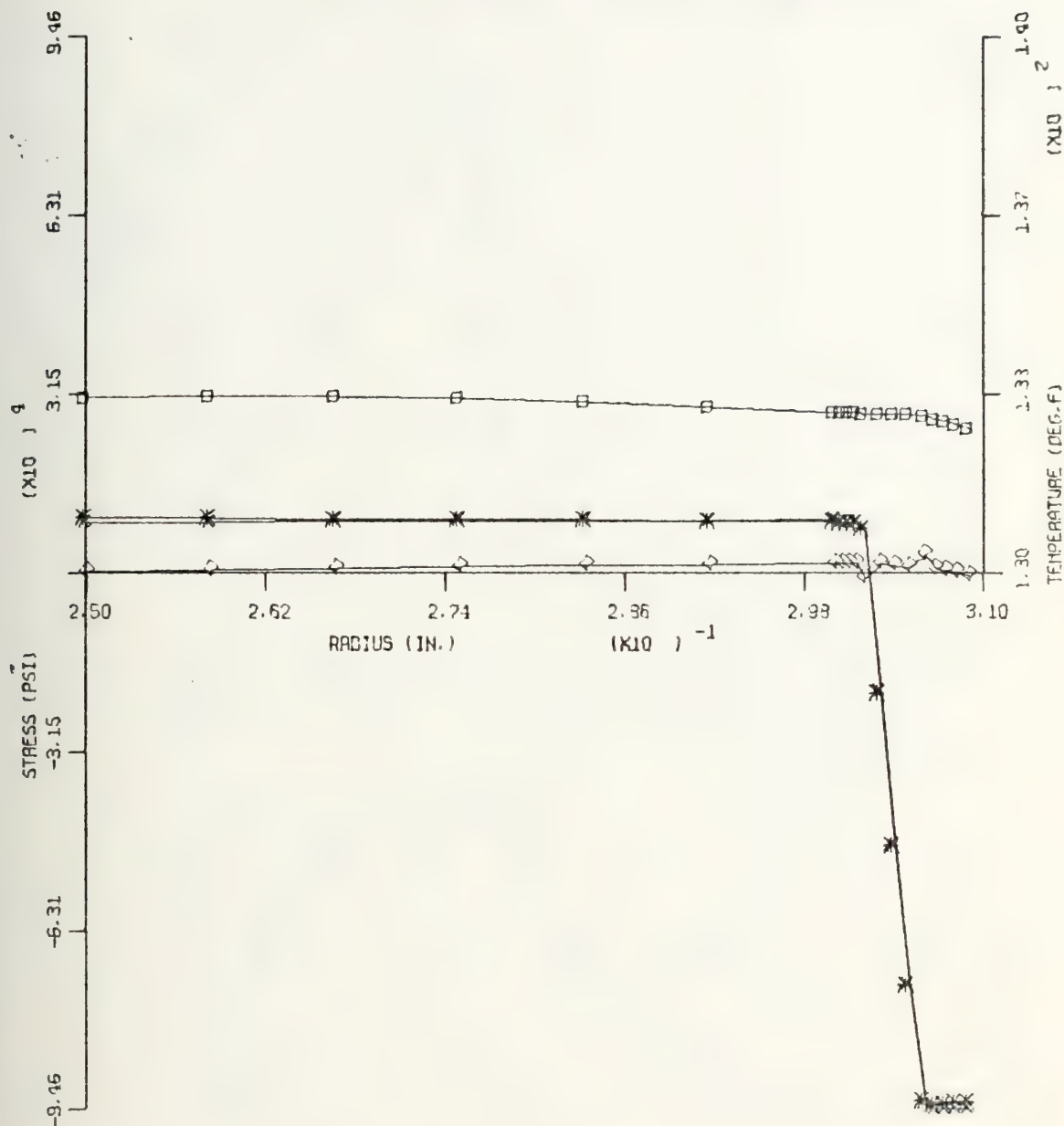
STRESS SCALE = 1.117×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 25(i)

STRESS AND TEMPERATURE
VERSUS RADIAL POSITION
TIME = 1.650 CONF.4-180



SIGMA R - DIAMOND SIGMA T - ASTERISK
SIGMA Z - CROSS TEMPERATURE - SQUARE

STRESS SCALE = 3.153×10^4 PSI/INCH

TEMPERATURE SCALE = 3.3 DEGREES/INCH

RADIAL POSITION SCALE = 0.01 INCHES/INCH

FIGURE 25(j)

MAXIMUM AND MINIMUM PRINCIPAL STRESSES
VERSUS TIME
CONFIGURATION NUMBER FOUR

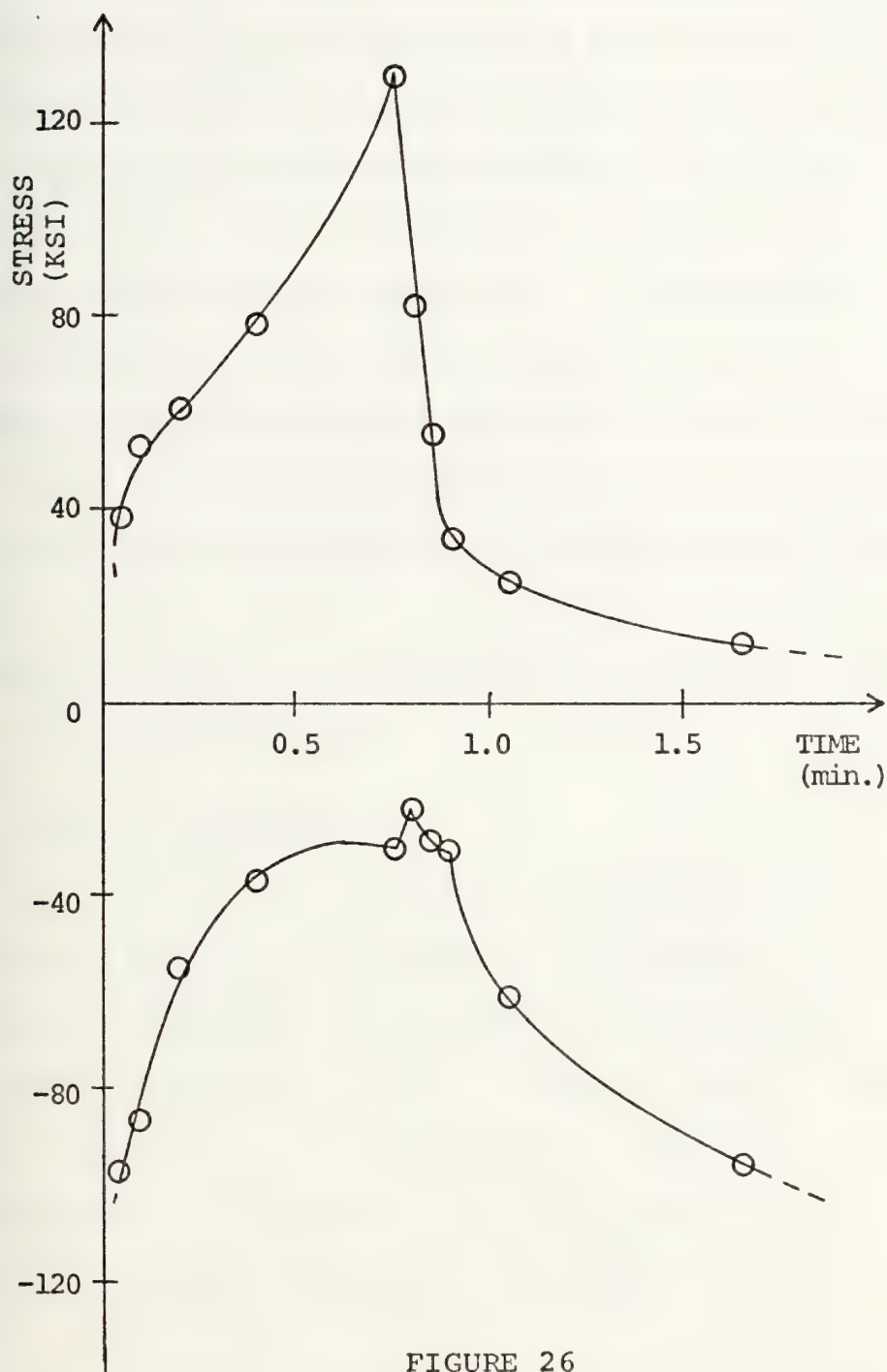


FIGURE 26

VI. CONCLUSIONS AND RECOMMENDATIONS

The thermal stress analysis of the various cylinder configurations showed that the maximum stress levels of the four coating arrangements were approximately equal. The maximum stress level was excessive enough to cause failure of the coatings and was dependent on the deposition temperature and the maximum temperature distribution. The abrupt change in the stress levels between the substratum and the outer coatings of configurations one and two was reduced by the use of the graded layers in configurations three and four.

The following recommendations are suggested for future development and analysis.

A. MATERIAL PROPERTIES

An extensive investigation of the temperature dependent properties should be completed. In addition to the various individual materials, the graded layer properties must be thoroughly understood before a complete design analysis can be completed. All creep data for these materials must be obtained. If possible, material properties for the compressive stress state should be obtained.

B. VARIATIONS TO THE PROBLEM

The following situations should be investigated.

1. Other geometries, such as wedges and turbine blades, would exhibit different temperature and stress distributions. Investigation of these geometries may show lower stress levels which would be less susceptible to failure. These investigations should be made.

2. The temperature distribution and maximum stress levels are directly related to the thickness of the zirconia coating. This coating depth should be varied to determine an optimal thickness for the minimization of the stress levels.

3. Since the deposition temperature affects the maximum tensile stresses, more investigation to determine the most appropriate temperature for coating application should be made. More information about the compressive state will also allow a better evaluation of the effect of the deposition temperature selection.

4. Although changes in the material property variation of the graded layer produced minor changes in the stress levels, the effect of the thickness of the graded layer should be investigated.

5. Since thermal load conditions are only part of the total load conditions on a turbine blade, an investigation of the effect of body forces on the stress levels in the blade should be completed. The program used in this analysis allows the specification of these transient body forces.

PART II. TRANSIENT THERMAL STRESS ANALYSIS

The classical equations governing thermal stress analysis are coupled to the heat conduction equations, however, in most studies of this topic including this one, the stress analysis and the heat conduction are separated. The analysis of the transient thermal stress condition is performed in two steps. In the first one, the unsteady heat equation is solved for the temperature distribution. Then, using this temperature distribution, the equations of elasticity can be solved to determine the stress distribution. Analytically this solution process is possible only in the simplest of problems. However, through the use of the finite element techniques, an approximate solution can be obtained for more realistic problems.

The computer program which was used as a basis to solve the heat conduction equation was written by Lew [7]. Several modifications were made to this program as described in Section I. The computer program used as a basis to solve the stress problem was written by Leonidas [8]. Several changes, as described in Section II, were also made to this program.

Since three dimensional finite element analysis generates such a voluminous amount of information, an additional post-processing program was written to condense the relevant information.

All programs were written using Fortran IV G Level language and solutions were obtained using the IBM 360/67 computer system.

I. TEMPERATURE DISTRIBUTION

The unsteady heat conduction equation, expressed in rectangular coordinates, is

$$\frac{\partial}{\partial x}(k_x \frac{\partial T}{\partial x}) + \frac{\partial}{\partial y}(k_y \frac{\partial T}{\partial y}) + \frac{\partial}{\partial z}(k_z \frac{\partial T}{\partial z}) + Q = \rho C \frac{\partial T}{\partial t} \quad (2)$$

This equation is subject to the following boundary conditions:

$$T = T_B \quad \text{on the boundary or}$$

$$k_x \frac{\partial T}{\partial x} l_x + k_y \frac{\partial T}{\partial y} l_y + k_z \frac{\partial T}{\partial z} l_z + q_s + h(T - T_f) = 0 \quad (3)$$

on the boundary

and a specified initial temperature distribution. The symbols used in these equations have the following definitions:

Q	- rate of internal energy generation per unit volume
ρ	- density
C	- specific heat
k_x, k_y, k_z	- thermal conductivity in the x, y, and z directions
l_x, l_y, l_z	- the direction cosines at a point on the boundary
q_s	- external heat flux
h	- surface heat transfer coefficient
T_f	- fluid temperature

As described in Ref. 9, the unsteady heat equation can be transformed into a discretized form by the finite element technique. Using the symbol [] for an $n \times n$ matrix, $\langle \rangle$ for a row vector, and { } for a column vector,

$$[H] \{T\} + [C] \dot{\{T\}} + \{F\} = 0 \quad (4)$$

where the temperature T of equation (2) has been expressed as the product of the nodal shape functions and the nodal temperature values.

$$T = \langle N \rangle \{T\} \quad (5)$$

The time derivative of the temperature $\frac{\partial T}{\partial t}$ has been expressed as

$$\frac{\partial T}{\partial t} = \dot{T} = \langle N \rangle \dot{\{T\}} \quad (6)$$

The matrices [H], [C], {F} are defined at the element level by

$$[H]_e = \int_{V_e} (k_x \left\{ \frac{\partial N}{\partial x} \right\} \left\langle \frac{\partial N}{\partial x} \right\rangle + k_y \left\{ \frac{\partial N}{\partial y} \right\} \left\langle \frac{\partial N}{\partial y} \right\rangle + k_z \left\{ \frac{\partial N}{\partial z} \right\} \left\langle \frac{\partial N}{\partial z} \right\rangle) dx dy dz \quad (7)$$

$$[C]_e = \int_{V_e} \{N\} \rho C \langle N \rangle dx dy dz \quad (8)$$

$$\{F\}_e = - \int_{V_e} Q\{N\} dx dy dz + \int_{S_e} q\{N\} dS_e + \left(\int_{S_e} \{N\} h \langle N \rangle dS_e \right) \{T_i\} \quad (9)$$

where

$$q = q_s - h T_f$$

The program developed by Lew [7] solves the discretized equation subject to the listed boundary conditions. In order to adapt the program to the thermal stress problem, several alterations were incorporated.

A. TIME INTEGRATION SCHEME

Reference 7 provides numerous time integration schemes to deal with the first time derivative of the temperature. In this study only the best integration schemes were retained. A five point backward difference formula was used to approximate the time derivative. For the i th time step, this approximation can be expressed as

$$\left(\frac{\partial T}{\partial t}\right)_i = \dot{T}_i = \frac{1}{12\Delta t} (3T_{i-4} - 16T_{i-3} + 36T_{i-2} - 48T_{i-1} + 25T_i) \quad (10)$$

where Δt is the specified time step. The truncation error for this expression is $+\frac{1}{5}(\Delta t)^4 \frac{\partial^5 T}{\partial t^5}$.

If equation (10) is written in vector form and substituted into equation (4), the final form of the discretized heat equation is obtained.

$$(12\Delta t[H] + 25[C]) \cdot \{T_i\} =$$

$$48\{T_{i-1}\} - 36\{T_{i-2}\} + 16\{T_{i-3}\} - 3\{T_{i-4}\} - 12\Delta t\{F\} \quad (11)$$

If $[H]$, $[C]$, $\{F\}$, Δt and the four preceding values of $\{T_i\}$ are known, the system of simultaneous equations is solved for the next set of $\{T_i\}$.

The initial four values for the temperature distribution is obtained by using the trapezoid method for a specified number of steps at a small Δt . After each n steps at Δt , the values of $\{T_i\}$ are saved. When three sets of $\{T_i\}$ plus the initial known distribution are available, the transient solution using a time step of $(n\Delta t)$ can proceed. For example, an initial step size of 0.001 taken for 150 steps results in a step size of 0.05 for the remainder of the transient problem.

Using the solution technique described above, the transient temperature distribution for an infinite hollow cylinder were determined. The problem characteristics are:

Inside diameter	= 0.5 inches
Outside diameter	= 0.6 inches
Initial temperature	= 932 deg. F
Internal and external heat transfer coefficients	= 25.0 BTU/hr-sq.ft.-deg.F
Internal Fluid temperature	= 896 deg. F
External Fluid temperature	= 1994 deg. F
Initial time step size	= 0.001 min.
Number of initial steps	= 150

Results of the approximate solution and the exact solution are given in Table 3.

B. NODAL MATERIAL PROPERTIES

The usual method of specifying properties to be constant throughout a finite element becomes troublesome when composite materials have continuously varying properties. Many separate elements would be required in order to adequately represent a structure which has material properties that are continuous functions of spatial coordinates.

In order to alleviate the problem, several routines in the basic program of Ref. 7 were altered to accommodate nodal properties in addition to element properties. A property P is expressed as $P = N_i P_i$, where P_i is a vector of nodal values and N_i are the same shape functions used in equation (5) for the temperature variable. The matrices $[H]$, $[C]$, and $\{F\}$ are formed by the same integration techniques as used in the basic program. However, the properties are no longer constant during the integration, and must be re-evaluated at each integration point.

As an example, consider steady state heat flow through a cylinder in which the thermal conductivity varies continuously along a radius. As proposed above, the development of a twenty nodal point element could represent exactly any linear or quadratic variation of material properties. Table 4 displays the results of such a problem as compared to the exact solution. The column heading "Element Property

TABLE 3

TRANSIENT TEMPERATURE OF THE INFINITE HOLLOW CYLINDER

Exact Solution (Deg. F)

Radius (in)	t = 0.25 (min)	0.5	1.0	3.0
0.25	1277.165	1409.796	1479.776	1491.715
0.26	1278.061	1410.916	1481.013	1492.977
0.27	1279.104	1412.062	1482.214	1494.182
0.28	1280.291	1413.236	1483.381	1495.348
0.29	1281.618	1414.438	1484.518	1496.473
0.30	1283.081	1415.669	1485.626	1497.561

Finite Element SolutionUsing 5-Point Finite Difference Method (Deg. F)

Radius (in)	t = 0.25 (min)	0.5	1.0	3.0
0.25	1277.220	1409.897	1479.879	1491.810
0.26	1278.116	1411.016	1481.116	1493.068
0.27	1279.160	1412.163	1482.316	1494.278
0.28	1280.347	1413.336	1483.483	1495.443
0.29	1281.674	1414.538	1484.612	1496.568
0.30	1283.138	1415.769	1485.726	1497.654

TABLE 4
STEADY HEAT FLOW THROUGH A HOLLOW CYLINDER

Inside Diameter	0.5 in.
Outside Diameter	0.6 in.
Inside Surface Temperature	500.0 °F
Outside Surface Temperature	1000.0 °F
Thermal Conductivity Variation in Outward Radial Direction:	
(a)	For linear function, 0.05 to 0.25 BTU/in-min-°F
(b)	For quadratic function, 0.2025 to 0.225625 (at mid-radius) to 0.25 BTU/in-min-°F

Linear Variation of Thermal Conductivity
(Two Radial Elements)

Radius (in.)	Exact Solution (deg. F)	Element Property Solution (deg. F)	Nodal Property Solution (deg. F)
0.25	500.0	500.0	500.0
0.2625	575.884	590.871	576.502
0.275	669.145	677.510	670.301
0.2875	794.697	842.256	796.982
0.3	1000.0	1000.0	1000.0

Quadratic Variation of Thermal Conductivity
(Two Radial Elements)

Radius (in.)	Exact Solution (deg. F)	Element Property Solution (deg. F)	Nodal Property Solution (deg. F)
0.25	500.0	500.0	500.0
0.2625	623.879	627.602	624.213
0.275	748.213	749.253	748.803
0.2875	873.441	877.342	873.747
0.3	1000.0	1000.0	1000.0

Solution" identifies the results obtained by specifying properties constant over an element. The column heading "Nodal Property Solution" identifies the results obtained by specifying the properties variable over an element, in other words nodal properties.

C. BOUNDARY CONDITIONS

The load vector represented by equation (9) contains two surface integrals which pertain to boundary conditions described by equation (3). The development of this load vector is shown in Ref. 9.

The usual technique for specification of the boundary conditions is to consider q and h as constants over the applicable element surface. In order to specify convection boundary conditions which vary over an element surface, the flux and heat transfer coefficient have been treated as nodal values in the same manner as nodal material properties. The heat generation term, Q , is also treated in this manner. Therefore, instead of the constant terms in equation (9), the usual shape functions N_i are used with the nodal values to form the variable terms $Q = N_i Q_i$, $q = N_i q_i$, and $h = N_i h_i$. The integration processes proceed as described in Ref. 7, with the addition of these terms.

The surface integrals are computed using a coordinate transformation to allow the use of Gauss Quadrature. This transformation maps the Cartesian coordinates (x,y,z) into

a local coordinate system (ξ, η, ζ) . As described by Hanson [10], the unit area vector in the (x, y, z) system is evaluated in the (ξ, η, ζ) system by

$$\hat{n} \, dS_e = \left(\frac{\partial \bar{\mathbf{r}}}{\partial \alpha} \times \frac{\partial \bar{\mathbf{r}}}{\partial \beta} \right) d\alpha \, d\beta, \quad (12)$$

where

$$\bar{\mathbf{r}} = x(\xi, \eta, \zeta) \hat{i} + y(\xi, \eta, \zeta) \hat{j} + z(\xi, \eta, \zeta) \hat{k} \quad (13)$$

specifies the surface and α, β represent ξ, η , or ζ .

The vector $\mathbf{V} = \frac{\partial \bar{\mathbf{r}}}{\partial \alpha} \times \frac{\partial \bar{\mathbf{r}}}{\partial \beta}$ is normal to the surface area $d\alpha \, d\beta$ and must be evaluated during the integration process. It is convenient to express equation (12) as

$$\hat{n} \, dS_e = |\bar{\mathbf{V}}| \left(\frac{\bar{\mathbf{V}}}{|\bar{\mathbf{V}}|} \right) d\alpha \, d\beta = |\bar{\mathbf{V}}| \hat{n} \, d\alpha \, d\beta, \quad (14)$$

where \hat{n} is a unit normal vector to surface area $d\alpha \, d\beta$.

Therefore, during the integration process, the factor

$$\begin{aligned} |\bar{\mathbf{V}}| = & \left[\left(\frac{\partial y}{\partial \alpha} \frac{\partial z}{\partial \beta} - \frac{\partial z}{\partial \alpha} \frac{\partial y}{\partial \beta} \right)^2 + \left(\frac{\partial x}{\partial \alpha} \frac{\partial z}{\partial \beta} - \frac{\partial z}{\partial \alpha} \frac{\partial x}{\partial \beta} \right)^2 \right. \\ & \left. + \left(\frac{\partial x}{\partial \alpha} \frac{\partial y}{\partial \beta} - \frac{\partial y}{\partial \alpha} \frac{\partial x}{\partial \beta} \right)^2 \right]^{1/2} \end{aligned} \quad (15)$$

must be evaluated. If the surface integral is performed over an area on which one of the variables x , y , or z is

constant, the quantity $|\bar{V}|$ becomes identical to the determinant of the Jacobian matrix for the surface transformation. For a general surface $r(x,y,z)$, the complete expressions for $|\bar{V}|$ must be used.

II. STRESS DISTRIBUTION

The finite element technique provides the required methods for the solution of the three dimensional elastic stress analysis problem. Reference 9 describes the derivations for the following equations. A structure can be divided into several elements, each formed by a specified number of nodes. If this structure is acted upon by various forces, the nodal displacements $\{r\}$ are determined by solving the following equation:

$$[K]\{r\} = \{R\} - \{F\}_B - \{F\}_P - \{F\}_{\epsilon_o} - \{F\}_{\sigma_o}, \quad (16)$$

where the matrices and vectors are described by

$$[K] = \int_V [B]^T [D] [B] dV \quad (17)$$

$$\{F\}_B = - \int_V [N]^T \{f_B\} dV \quad (18)$$

$$\{F\}_P = - \int_S [N]^T \{f_P\} dV \quad (19)$$

$$\{F\}_{\epsilon_o} = - \int_V [B]^T [D] \{\epsilon_o\} dV \quad (20)$$

$$\{F\}_{\sigma_o} = + \int_V [B]^T \{\sigma_o\} dV \quad (21)$$

and $\{R\}$ is the vector of concentrated nodal loads. The other symbols used in these equations are defined as:

$[N]$	-	vector of shape functions
$[B]$	-	matrix of shape function derivatives
$[D]$	-	elasticity matrix
$\{f_B\}$	-	vector of body forces
$\{f_p\}$	-	vector of surface forces
$\{\epsilon_0\}$	-	vector of initial strains
$\{\sigma_0\}$	-	vector of initial stresses

Following the solution of equation (16), the nodal displacements $\{r\}$ are used to evaluate the nodal strains.

$$\{\epsilon\} = [B] \{r\} \quad (22)$$

The final step in the analysis is the calculation of the nodal stresses.

$$\{\sigma\} = [D] (\{\epsilon\} - \{\epsilon_0\}) + \{\sigma_0\} \quad (23)$$

A computer program to make these calculations was written at the Naval Postgraduate School by Leonidas [8]. In that version of the program, the right hand side load vector was allowed to contain only concentrated loads $\{R\}$. In order to permit the presence of thermal stresses and body

forces, several changes were made to allow $\{\epsilon_0\}$ and $\{f_B\}$ to have non-zero values.

A. GENERATION OF LOAD VECTORS

Temperature differences always generate strains but sometimes stresses may still be zero, for example, when the temperature differences are constant throughout, ϵ in equation (23) would be equal to ϵ_0 giving zero stress for one material. The presence of a non-uniform temperature distribution in a solid body causes stresses and strains due to non-uniform expansion. In addition, a solid body composed of materials with differing coefficients of linear expansion, λ , will at times experience stresses and strains. This happens when the temperature distributions differ from that of the zero-stress condition.

The forces present in a body during a thermal stress state can be calculated by setting $\epsilon_0 = \lambda \Delta T$, where ΔT is the difference between the actual temperature distribution and the zero-stress temperature distribution. In order to form a consistent load vector, these forces, calculated by equation (20), are added to any other nodal forces present on the right hand side of equation (16).

The body forces, due to some force field, are converted into nodal forces by using equation (17). These are added to the right hand vector of equation (16) to form the total consistent load vector.

B. NODAL MATERIAL PROPERTIES

As discussed in Section IB, the use of nodal material properties is a more convenient method for composite body analysis. In the stress analysis operations, the elastic matrix and the coefficient of linear expansion enter into the calculations. The elastic matrix is composed of the elastic modulus and Poisson's ratio. For the different operations, these properties are evaluated by using $E = N_i E_i$, $\mu = N_i \mu_i$, and $\lambda = N_i \lambda_i$. During the various integration processes these property values are evaluated at each integration step.

The use of nodal properties requires careful consideration as to the effect the shape functions have on the properties within the element. If an element of one material is adjacent to an element of another material, the use of nodal properties can lead to erroneous results. Since the nodes common to the elements can have only one set of property values, one of the elements will not have the expected uniform properties. It would be better to use constant element properties or combine the two elements into one and use variable properties over that element. The amount of property variation between the two elements will also assist the analyst in determining the proper technique.

III. DESCRIPTION OF THE PROGRAMS

Three programs were used to complete the thermal stress analysis. As noted in Sections I and II, the programs of Lew and Leonidas [7,8] were used as a basis to develop the programs for the temperature distribution and elastic stress analysis. The third program was written to process the information generated in the stress analysis. The computer listings for these three programs are provided in Appendices F, G, and H.

A. TEMPERATURE PROGRAM

The main alterations to the temperature program [7] were: (1) the elimination of the second time derivative of the quasi-harmonic equation, (2) the addition of the time integration scheme described in Section IA, and (3) changes to subroutines FORMC, CUBE, and BCOND to allow the specification of nodal material properties and nodal boundary conditions. The version of the program listed in Appendix F allows 278 nodes and a bandwidth of 40. These specifications were varied to adjust the computed core requirements to the particular problem being solved. A problem with 278 nodes and a bandwidth of 40 required approximately 350,000 bytes of storage.

The assumptions made in the solution of a problem using this program are: (1) material properties are constant with time and independent of temperature, (2) directional material

properties must align with the global coordinate axes, and
(3) the boundary conditions are constant within a heat transfer phase, but may be altered at the end.

1. Subroutine TIMEHB

As discussed in Section IA, this subroutine performs a time integration using a five point backward difference method. During each heat transfer phase, boundary conditions are constant. Following applications of these boundary conditions to the appropriate matrices and vectors, the trapezoid integration method is used to generate three starting values. Along with the initial temperature distribution, these three starting values provide the necessary information to use the five point difference scheme. The output of temperature distributions is controlled in a manner similar to that used in the original program [7]. These output values are adjusted to the reference temperature supplied in the data deck. During a stress analysis problem, this reference temperature is the zero-stress temperature. Since these temperature distributions are used in a subsequent stress analysis, proper control is provided to punch cards or transfer the information to temporary storage devices. The program in its present form allows up to twenty separate temperature vectors to be stored for future use.

2. Subroutine FORMC, CUBE

Subroutine FORMC was modified to allow nodal properties to be used in the integration process for the matrices

[H] and [C]. The subroutine CUBE performs the cubic integration used in the formation of these matrices and therefore required slight modification.

3. Subroutine BCOND

Subroutine BCOND and its subsidiary routines were modified to accommodate nodal boundary conditions. Subroutine JACOB was altered to evaluate the factor given in equation (15). This factor was used in the surface integration as discussed in Section IC.

4. Data Preparation for the Temperature Program

The data required for a transient solution is comprised of eight groupings of cards. Careful detail must be followed to insure proper formats are used and the correct order of cards maintained. A data check can be performed without any major calculations. The data check normally requires 20-30 seconds, however, the full computer core requirement is needed.

Preparation of the data cards is explained in the initial comment cards of the program listing in Appendix F. A sample data deck for a 32 node, 2 element program is provided in Appendix A. Several aspects of the data preparation must be thoroughly understood.

a. Nodal Properties

The method of input for values which are common to several nodes was provided to reduce the number of input cards. Care must be taken to insure that every node has

properties assigned to it, and that the order of material property cards follow the order of ascending nodal numbers.

b. Output Storage

The number of times stored output is requested must match the number of loads used in the stress analysis program.

c. Steady State Solution

If a steady state problem is being analyzed and no convection or internal generation conditions exist, a blank card must be inserted in place of the heat transfer phase data.

d. Stacking of Problems

If this program is used in conjunction with the elastic stress analysis program, only one problem can be loaded at a time. Therefore, the final comments concerning stacking problems do not apply, and a "STOP" card must be used.

e. Boundary Condition Data

When preparing the boundary condition data, the number of cards with boundary property values (NODECD) is given on the first card of each boundary condition data set. Ensure that NODECD cards are used in that particular set.

5. Job Control Language

The necessary job control language for using the temperature program is listed in Appendix B. The two cards beginning with //GO.FT10F001 and //GO.FT11F001 are

necessary for the storage of the matrices [H] and [C] during the time integration process. They are not required for the steady state solution. The card beginning with //GO.FT07F001 is used to store temperature distributions on temporary files for use in the stress analysis program. If punched output is desired, this card should be replaced by

```
                //GO.FT07F001  DD SYSOUT=B
```

The card beginning with //GO.FT06F001 is necessary to allow large amounts of output from the computer printing devices. If less than 3000 lines of output is anticipated, this card can be removed. These job control cards are designed for use at the W.R. CHURCH COMPUTER CENTER, NAVAL POSTGRADUATE SCHOOL, Monterey, California. Use of the program at another installation will require appropriate modifications.

B. ELASTIC STRESS ANALYSIS PROGRAM

Several changes were required to add a transient thermal stress analysis capability to the stress analysis program [8]. These included: (1) Addition of a consistent load generation subroutine LODGEN, (2) Alteration of the subroutine SOLVE to allow multiple loads, (3) Changes to subroutines INPUT, SORT, STRESS to provide steps for various calculations pertaining to the temperature distributions, (4) The capability to use nodal properties requiring the addition of subroutine NDPROP and numerous alterations to the integration process for the computation of the "stiffness matrix", and (5) The addition of a subroutine PRNSTR to calculate the magnitude of the principal stresses.

The program uses external storage for all information. Therefore, numerous additions and changes were made to the various WRITE and READ statements. A thorough knowledge of the direct-access statements used in Fortran IV is essential to understand the program used in this analysis.

1. Subroutine LODGEN

This subroutine performs calculations used to form the consistent load vectors as described in Section IIA. A lengthy computing time is required for this subroutine due to the number of computations and the many direct access file operations. The number of Gauss points (NGPL) used can be specified independently of the number used for the formation of the "stiffness matrix". The consistent load vector forms the right hand side of equation (16) and must be of sufficient accuracy to insure valid final results. For the calculations made in Part I, four Gauss points were considered sufficient.

2. Subroutine SOLVE

The basic subroutine used to solve equation (15) is contained in Refs. 8 and 11. In order to solve multiple load vectors, changes were incorporated to make necessary computations to all load vectors simultaneously. As described in Refs. 8 and 11, calculations are performed on each block of values in the load vector until all nodes have been processed. The changes which were made performed these calculations to a specified block for all load vectors prior to advancing to the next block. This method was used

to minimize the number of computations and the direct access operations.

3. Subroutine NDPROP

This subroutine calculates the material properties at a node or Gauss point. If the elastic matrix is requested, the various properties are calculated for the position in question, and the matrix is formed. If the coefficient of thermal expansion is requested, it is calculated for the position specified and returned to the calling program. This subroutine uses a separate subroutine, SHAPE, to evaluate the shape function values used in the calculation of the property values.

4. Preparation of Input Data

The preparation of all data cards is explained in the initial comment cards of the program listing, Appendix G. Special care must be given to the details of the input cards.

a. Object Time Formats

The use of object time formats is made throughout the input subroutine. The use of object time formats eliminates the need to recall what format a certain data card is to follow. The object time format technique is covered in most books on the Fortran language.

b. Data Check

A data check can be made by specifying NGPS=0. This requires the full 250,000 bytes of storage, but is completed in less than 20 seconds.

c. External Storage of Load Conditions

If the load conditions of temperature distribution and body forces are prepared by a preprocessing program, these values are passed using files 33 and 34. File 35 for use with nodal displacements was added to the input routine for future development. When the values are passed from a preprocessing program, a time value must be provided at the beginning of each set of load conditions.

5. Job Control Language

Since the stress analysis program stores all information on external devices, the job control language is a very critical part of the program. A very thorough understanding of the specific requirements for a particular computer facility is needed to use this program. All examples and instructions presented here apply to the W.R. CHURCH COMPUTER CENTER AT THE NAVAL POSTGRADUATE SCHOOL, Monterey, California.

External storage operations used in the program are divided into two types. One type is the sequential file READ or WRITE statement. This method of external storage performs the operation on a specified record length in a sequential manner. The second type of operation is the direct access method. In this case, a specified record length is written into or retrieved from a given location within the file. In the first type of operation, no changes to the program language are necessary. The second

type of operation requires changes to the DEFINE FILE statements listed in the control program. These statements will vary according to the size of the problem being analyzed.

Each file used by either a sequential type operation or a direct access operation must be described in the job control language. For the case of the direct access operations, the DEFINE FILE statements and the job control language must agree both in record length and the number of records.

In order to provide a complete summary of all files, Table 5 was prepared. If a particular problem does not require a file, its job control language and any DEFINE FILE statements can be eliminated. Table 6 describes each DEFINE FILE statement and job control card. For each file, a formula is supplied which determines the record length and the number of records which that file will contain. The program listing, Appendix G, and the job control card listing, Appendix D, are examples for use with a thermal stress problem using 278 nodes, 27 elements, and 10 load vectors.

C. POST PROCESSING PROGRAM

The elastic stress analysis program provides a considerable amount of information. For a problem of 200 nodes with 10 load conditions, about 16,000 lines of output are generated. Interpretation of this amount of output is both tedious and time consuming. Usually only a few nodes are

TABLE 5
FILE DEFINITIONS

File Number	- Definition
7	- Storage of Total Stiffness Matrix Blocks
8	- Storage of Element Stiffness Matrices
9	- Storage of Element Connectivity
10	- Storage of Nodal Coordinate Vectors
11	- Storage of Nodal Properties
12	- Storage of Nodal Stress Vectors
13	- Storage of Nodal Strain Vectors
14	- Storage of Concentrated Load Vectors
15	- Storage of Boundary Condition Vectors
16	- Storage of Element Material Numbers
17	- Storage of Reaction Vectors
18	- Storage of Nodal Temperatures
19	- Storage of Nodal Coordinates by Element
20	- Storage of Load Vector Blocks
21	- Storage of Nodal Displacement Vectors
22	- Storage of Predefined Boundary Displacements
23	- Storage of Nodal Properties by Element
24	- Storage of Nodal Temperatures by Element
25	- Storage of Nodal Body Forces
26	- Storage of Nodal Displacements
27	- Storage of Nodal Body Forces by Element

File Number	- Definition
28	- Storage of Nodal Displacements by Element
29	- Storage of Nodal Force Vector
30	- Storage of Time Vectors
33	- Storage of Preprocessed Temperature Distributions
34	- Storage of Preprocessed Body Force Distributions
35	- Storage of Preprocessed Nodal Displacement Distributions

TABLE 6

DEFINE FILE Statements and Job Control Cards

General format for these cards:

DEFINE FILE XX(NRX,LRWX,U,IXX)

//GO.FTXXF001 DD UNIT=SYSDA,SPACE=(LRBX,(NRX,1)),DISP=(NEW,DELETE)

where XX,NRX,LRBX are defined in the table.

FILE	Number of Records	Record length in BYTES	Record length in WORDS
XX	NRX	LRBX	LRWX
7	MMxNNxNREC7	NCOUNTx8/NREC7	NCOUNTx2/NREC7
8	4xNEL	7200	1800
9	NEL	80	20
10	NDPT	24	6
11	NDPT	24	6
12	NDPTxNLOAD	56	14
13	NDPT	56	14
14	NCLD	32	8
15	NPBC	16	4
16	NEL	4	1
17	NPBC	32	8
18	NDPTxNLOAD	8	2
19	NEL	480	120
20	NNxNLOAD	NSx8	NSx2
21	NNxNS/NDF	24	6



XX	NRX	LRBX	LRWX
22	NPBC2	24	6
23	NEL	480	120
24	NLOADxNEL	160	40
25	NLOADxNDPT	24	6
26	NLOADxNDPT	24	6
27	NLOADxNEL	480	120
28	NLOADxNEL	480	120
29	NLOADxNDPT	24	6
30	NLOAD	8	2
33	NLOADxNDPT	8	2
34	NLOADxNDPT	24	6
35	NLOADxNDPT	24	6

Notes:

- (a) For a 20 nodal point brick, NREC7 = 4
- (b) For a 32 nodal point brick, NREC7 = 6
- (c) Variables MM, NN, NS, etc. are defined in Ref. 8

considered at one time and the remainder of the information is reserved for later use.

Through the use of job control language, the important information can be stored on external storage devices and processed by a separate program. A post-processing program was written to analyze the stored information.

1. Program Description

The program listed in Appendix G performs the following operations: (a) information stored on external devices is read for a specified number of nodes; (b) the x,y, and z components of all stress vectors are analyzed for the principal stress values, which are used to determine the maximum and minimum values for a given load condition; (c) Offline and Calcomp plots of stress versus coordinate position and temperature versus coordinate position can be obtained; (d) Offline and Calcomp plots can be obtained for the stress values in polar coordinates.

2. Input Data Preparation

The input data deck has been designed to consist of three or four cards. The preparation of these cards is explained in the initial comments of the program in Appendix H. When more than one problem is processed, care must be taken to limit the number of Calcomp plots requested. All Calcomp plots are performed twice to make the lines dark enough for reproduction by other means.

3. Job Control Language

Since the information is to be passed from the stress analysis program, a few changes to the job control language discussed in Section IIIB are necessary. Files 10, 12, 18, and 30 are changed to the following format:

```
//GO.FTXXF001 DD UNIT=XXXXX,SPACE=(LRBX,(NRX,1)),  
// DISP=(NEW,KEEP),DSN=YYYYY,VOLUME=SER=ZZZZZ
```

where XX, LRBX, NRX - defined in Section IIIB

XXXXX - a unit number

YYYYY - a file name, e.g. STRESS

ZZZZZ - a device number

The unit number and device number are obtained from the particular computer facility.

The job control language for the post-processing program must match with that passed by the elastic stress analysis program. Files 8, 9, 10, and 11 are used for the stress values, coordinate values, temperature values, and time values respectively. The job control cards for the post-processor have the same form as those in the elastic stress analysis program. The linkage between the two programs is formed by specifying the same unit number, device number, and file name for the information given by the stress program and received by the post-processor. For example, file 12 in the stress program would be

```
//GO.FT12F001 DD UNIT=2311,SPACE=(LRBX,(NRX,1)),  
// DISP=(NEW,KEEP),DSN=STRESS,VOLUME=SER=SYS001
```

and, therefore, File 8 in the post-processor would be


```
//GO.FT08F001 DD UNIT=2311,SPACE=(LRBX,(NRX,1)),  
// DISP=(OLD,KEEP),DSN=STRESS,VOLUME=SER=SYS001 .
```

The post processing program also contains DEFINE FILE statements which follow the guidelines in Section IIIB.

Further details on the procedures for job control language can be found in Ref. 12.

D. SYSTEM USAGE OF ALL PROGRAMS

The discussions thus far have pertained to the individual programs, however, the actual production usage of the programs was done in a multistep process. The usage of the programs as a system first requires the formation of the system, then the production run of a problem.

1. System Formation

Appendix D lists the job control language used to create the library of programs. Details of this procedure are provided in Ref. 12.

2. Production Runs

Appendix E lists the job control language for the multistep process of temperature and stress distribution determination. The major change in the methods is the transport of the temperature distribution from the first step to the second step. This is accomplished by the card beginning with //GO.FT07F001 for the first step job control language and the card beginning with //GO.FT33F001 of the second step job control language. These two cards connect the informatin generated and used by the two programs.

Reference 12 contains full details of the job control language used.

IV. RECOMMENDATIONS

As with any endeavor to construct a usable tool for engineering analysis, the job is never done. The following changes are recommended for future development:

A. PROGRAM FOR THE TEMPERATURE DISTRIBUTION

The present version of the temperature distribution is designed for an "in-core" solution technique, which restricts the problem size to within the computer core size. The stress program uses an "out of core" technique and is restricted only by the availability of external storage devices. In order to eliminate the core size restriction, the temperature program requires complete revision to permit the use of external storage devices.

B. INITIAL STRESS DISTRIBUTION

The load generator should be altered to allow a non-zero initial stress distribution.

C. JOB CONTROL LANGUAGE

The amount of job control language allowed at a particular computer facility is limited. For the facility used during this analysis, it appears that no more job control language can be used with this program. The various files used in this program must be combined to reduce the number of job control cards needed.

D. MATERIAL PROPERTIES

In order to more accurately solve the real engineering problems, the capability of temperature dependent properties should be added to both the temperature program and the stress program.

SAMPLE DATA DECK FOR TEMPERATURE PROGRAM

01234567890123456789012345678901234567890123456789

[illegible]

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APPENDIX B

JOB CONTROL CARD LISTING FOR TEMPERATURE PROGRAM

0123456789012345678901234567890123456789012345678901234567890123456789

```
// (JOBNAME CARD)
// EXEC FORTCLG,REGION.GD=350K
//FORT.SYSIN DD *
```

SOURCE CODE OF TEMPERATURE PROGRAM FOLLOWED BY /* CARD

```
//GO.FTCT7F001 DD UNIT=SYSDA,SPACE=(CYL,(2,1)),DSN=&TEMP,DISP=(NEW,PASS)
//GO.FT10F001 DD UNIT=SYSDA,SPACE=(CYL,(2,1)),DISP=(NEW,DELETE)
//GO.FT11F001 DD UNIT=SYSDA,SPACE=(CYL,(2,1)),DISP=(NEW,DELETE)
//GO.SYSIN DD *
```

DATA DECK FOLLOWED BY /* CARD



APPENDIX C

SAMPLE DATA DECK FOR STRESS PROGRAM

0123456789012345678901234567890123456789012345678901234567890123456789

START	TEST	PROBLEM	20	-1	0	26	0	32	0	-1	0	4	3	5
1	2	32	60											
		ELEMENT CONNECTIVITY												
		(1015,/,1115)												
	28	25	20	1	6	9	17	29	21					
	30	10	30	22	15	3	17	11	18	1				
	30	26	15	3	17	11	18	31	23					
	4	12	32	24	16	5	8	13	19	1				
		JOINT COORDINATES												
		(116,3F15.5,15)												
		1		0.0			0.0	0.0	0.0	0.60000				
		2		0.0			0.0	0.0	0.0	0.58750				
		3		0.0			0.0	0.0	0.0	0.57500				
		4		0.0			0.0	0.0	0.0	0.56250				
		5		1.50000			0.0	0.0	0.0	0.55000				
		6		1.50000			0.0	0.0	0.0	0.57500				
		7		1.50000			0.0	0.0	0.0	0.55000				
		8		3.00000			0.0	0.0	0.0	0.58750				
		9		3.00000			0.0	0.0	0.0	0.57500				
		10		3.00000			0.0	0.0	0.0	0.56250				
		11		3.00000			0.0	0.0	0.0	0.55000				
		12		0.0			0.0	0.21213	0.0	0.51213				
		13		0.0			0.0	0.19445	0.0	0.49445				
		14		0.0			0.0	0.17677	0.0	0.47677				
		15		0.0			0.0	0.21213	0.0	0.51213				
		16		0.0			0.0	0.19445	0.0	0.49445				
		17		0.0			0.0	0.17677	0.0	0.47677				
		18		0.0			0.0	0.30000	0.0	0.30000				
		19		0.0			0.0	0.28750	0.0	0.30000				
		20		0.0			0.0	0.27500	0.0	0.30000				
		21		0.0			0.0	0.26250	0.0	0.30000				
		22		0.0			0.0	0.25000	0.0	0.30000				
		23		1.50000			0.0	0.30000	0.0	0.30000				
		24		1.50000			0.0	0.27500	0.0	0.30000				
		25		1.50000			0.0	0.25000	0.0	0.30000				
		26		3.00000			0.0	0.30000	0.0	0.30000				
		27		3.00000			0.0	0.28750	0.0	0.30000				
		28		3.00000			0.0	0.27500	0.0	0.30000				
		29		3.00000			0.0	0.26250	0.0	0.30000				



APPENDIX D
JOB CONTROL CARD LISTING
FOR STRESS PROGRAM

```
// (JOBNAME CARD)
// EXEC FORTCLG,REGION.GO=250K
//FORT.SYSIN DD *

      SOURCE CODE OF STRESS PROGRAM FOLLOWED BY /* CARD

//GO.FT07F001 DD UNIT=SYSDA,SPACE=(7200,(0120,1)),DISP=(NEW,DELETE)
//GO.FT08F001 DD UNIT=SYSDA,SPACE=(7200,(0100,1)),DISP=(NEW,DELETE)
//GO.FT09F001 DD UNIT=SYSDA,SPACE=(0080,(0027,1)),DISP=(NEW,DELETE)
//GO.FT10F001 DD UNIT=SYSDA,SPACE=(0024,(0278,1)),DISP=(NEW,DELETE)
//GO.FT11F001 DD UNIT=SYSDA,SPACE=(0024,(0278,1)),DISP=(NEW,DELETE)
//GO.FT12F001 DD UNIT=SYSDA,SPACE=(0056,(2780,1)),DISP=(NEW,DELETE)
//GO.FT13F001 DD UNIT=SYSDA,SPACE=(0056,(0278,1)),DISP=(NEW,DELETE)
//GO.FT14F001 DD UNIT=SYSDA,SPACE=(0032,(0278,1)),DISP=(NEW,DELETE)
//GO.FT15F001 DD UNIT=SYSDA,SPACE=(0016,(0100,1)),DISP=(NEW,DELETE)
//GO.FT16F001 DD UNIT=SYSDA,SPACE=(0004,(0027,1)),DISP=(NEW,DELETE)
//GO.FT17F001 DD UNIT=SYSDA,SPACE=(0032,(0100,1)),DISP=(NEW,DELETE)
//GO.FT18F001 DD UNIT=SYSDA,SPACE=(0008,(2780,1)),DISP=(NEW,DELETE)
//GO.FT19F001 DD UNIT=SYSDA,SPACE=(0480,(0027,1)),DISP=(NEW,DELETE)
//GO.FT20F001 DD UNIT=SYSDA,SPACE=(0480,(0400,1)),DISP=(NEW,DELETE)
//GO.FT21F001 DD UNIT=SYSDA,SPACE=(0024,(0800,1)),DISP=(NEW,DELETE)
//GO.FT23F001 DD UNIT=SYSDA,SPACE=(0480,(0027,1)),DISP=(NEW,DELETE)
//GO.FT24F001 DD UNIT=SYSDA,SPACE=(0160,(0270,1)),DISP=(NEW,DELETE)
//GO.FT29F001 DD UNIT=SYSDA,SPACE=(0024,(2780,1)),DISP=(NEW,DELETE)
//GO.FT30F001 DD UNIT=SYSDA,SPACE=(0008,(0010,1)),DISP=(NEW,DELETE)
//GO.FT33F001 DD UNIT=SYSDA,DSN=&TEMP,DISP=(OLD,DELETE)
//GO.SYSIN DD *
```

DATA DECK FOLLOWED BY /* CARD

APPENDIX E
JOB CONTROL CARD LISTING
FOR FORMING PROGRAM LIBRARIES

FIRST FORM THE TEMPERATURE LIBRARY

```
// (JOBNAME CARD)
// EXEC FORTCL,PARM.LINK='NCAL,LET'
// FORT.SYSIN DD *

INSERT SOURCE CODE OF TEMPERATURE PROGRAM EXCEPT FOR CONTROL PROGRAM

INSERT /* CARD

// LINK.SYSLMOD DD DSN=SI312,TRITRNL(INPUT),DISP=(NEW,KEEP),
// UNIT=2321,VOLUME=SER=CELO03,LABEL=RETPD=120,
// SPACE=(CYL,(10,1))

INSERT /* CARD
```

NOW FORM THE STRESS PROGRAM LIBRARY

```
// (JOBNAME CARD)
// EXEC FORTCL,PARM.LINK='NCAL,LET'
// FORT.SYSIN DD *

INSERT SOURCE CODE OF THE STRESS PROGRAM EXCEPT FOR CONTROL PROGRAM

INSERT /* CARD

// LINK.SYSLMOD DD DSN=SI312,TRISOP(MAINE),DISP=(NEW,KEEP),
// UNIT=2321,VOLUME=SER=CELO03,LABEL=RETPD=120,
// SPACE=(CYL,(10,1))

INSERT /* CARD
```


JOB CONTROL CARD LISTING
FOR MULTISTEP PROCESS

```
// (JOBNAME CARD)
//STEP1 EXEC FORTCLG,REGION.GO=350K
//FORT.SYSIN DD *

      INSERT CONTROL PROGRAM FOR TEMPERATURE ROUTINE

/* CARD

//LINK.SYSLIB DD DISP=SHR,DSNAME=SYS1.FORTLIB
//              DD DISP=SHR,DSNAME=SYS1.MPSLIB
//              DD DISP=SHR,DSNAME=SL312.TRITRNL(INPUT),
//              VOLUME=SER=CEL003,UNIT=2321
//GO.FT06F001 DD SPACE=(CYL,(1,1)),SYSCOUT=0
//GO.FT07F001 DD UNIT=SYSDA,SPACE=(CYL,(2,1)),DSN=ETEMP,DISP=(NEW,PASS)
//GO.FT10F001 DD UNIT=SYSDA,SPACE=(CYL,(2,1)),DISP=(NEW,DELETE)
//GO.FT11F001 DD UNIT=SYSDA,SPACE=(CYL,(2,1)),DISP=(NEW,DELETE)
//GO.SYSIN DD *
```

INSERT TEMPERATURE DATA DECK FOLLOWED BY /* CARD

```
//STEP2 EXEC FORTCLG,REGION.GO=250K
//FORT.SYSIN DD *
```

INSERT CONTROL PROGRAM FOR STRESS ROUTINE

/* CARD

```
//LINK.SYSLIB DD DISP=SHR,DSNAME=SYS1.FORTLIB
//              DD DISP=SHR,DSNAME=SYS1.MPSLIB
//              DD DISP=SHR,DSNAME=SL312.TRISOP(MAINE),VOLUME=SER=CEL005,
//              UNIT=2321
//GO.FT06F001 DD SPACE=(CYL,(2,1)),SYSCOUT=0
//GO.FT07F001 DD UNIT=SYSDA,SPACE=(7200,(0150,1)),DISP=(NEW,DELETE)
//GO.FT08F001 DD UNIT=SYSDA,SPACE=(7200,(0120,1)),DISP=(NEW,DELETE)
//GO.FT09F001 DD UNIT=SYSDA,SPACE=(0080,(0027,1)),DISP=(NEW,DELETE)
//GO.FT10F001 DD UNIT=2311,SPACE=(0024,(0278,1)),DISP=(NEW,KEEP),
//              DSN=CUARD3,VOLUME=SER=SYS001
//GO.FT11F001 DD UNIT=SYSDA,SPACE=(0024,(0278,1)),DISP=(NEW,DELETE)
//GO.FT12F001 DD UNIT=2311,SPACE=(0056,(2780,1)),DISP=(NEW,KEEP),
//              DSN=STRESS3,VOLUME=SER=SYS001
//GO.FT13F001 DD UNIT=SYSDA,SPACE=(0056,(0278,1)),DISP=(NEW,DELETE)
//GO.FT14F001 DD UNIT=SYSDA,SPACE=(0032,(0278,1)),DISP=(NEW,DELETE)
//GO.FT15F001 DD UNIT=SYSDA,SPACE=(0016,(0120,1)),DISP=(NEW,DELETE)
//GO.FT16F001 DD UNIT=SYSDA,SPACE=(0004,(0027,1)),DISP=(NEW,DELETE)
```



```

//GO.FT17F001 DD UNIT=SYSDA,SPACE=(0032,(0120,1)),DISP=(NEW,DELETE)
//GO.FT18F001 DD UNIT=2311,SPACE=(0008,(2780,1)),DISP=(NEW,KEEP),
//DSN=TEMP3, VOLUME=SER=SYS001
//GO.FT19F001 DD UNIT=SYSDA,SPACE=(0480,(0027,1)),DISP=(NEW,DELETE)
//GO.FT20F001 DD UNIT=SYSDA,SPACE=(0480,(0400,1)),DISP=(NEW,DELETE)
//GO.FT21F001 DD UNIT=SYSDA,SPACE=(0024,(0800,1)),DISP=(NEW,DELETE)
//GO.FT22F001 DD UNIT=SYSDA,SPACE=(0480,(0027,1)),DISP=(NEW,DELETE)
//GO.FT23F001 DD UNIT=SYSDA,SPACE=(0160,(0270,1)),DISP=(NEW,DELETE)
//GO.FT24F001 DD UNIT=SYSDA,SPACE=(0024,(2780,1)),DISP=(NEW,DELETE)
//GO.FT25F001 DD UNIT=2311,SPACE=(0008,(0010,1)),DISP=(NEW,KEEP),
//GO.FT30F001 VOLUME=SER=SYS001
//DSN=TIME3, VOLUME=SER=SYS001
//GO.FT33F001 DD UNIT=SYSDA,DSN=&TEMP,DISP=(OLD,DELETE)
//GO.SYSIN DD *

```

INSERT DATA DECK FOR STRESS PROGRAM FOLLOWED BY /* CARD

CNTPOOL0

THIS PROGRAM SOLVES THE TRANSIENT HEAT EQUATION IN THREE DIMENSIONS. IN ADDITION THE STEADY STATE SOLUTION CAN BE OBTAINED DIRECTLY. THIS PROGRAM IS BASED ON A PROGRAM ORIGINALLY WRITTEN BY LCDR. LEW. NAVAL POSTGRADUATE SCHOOL, 1972. PRESENT VERSION WAS WRITTEN BY LT. BUBECK, NAVAL POSTGRADUATE SCHOOL, 1975.

07002CNIP0070

- (1) PROPERTIES ARE CONSTANT
- (2) STATIONARY BOUNDARIES AND ELEMENTS ARE REQUIRED.
- (3) ANISOTROPIC MATERIALS MUST ALIGN WITH AXES.

(1) TITLE CARD (10A8) - ANY TITLE IN COLUMNS 2-80: A NUMERAL "1" PLACED IN COLUMN 1 STARTS PRINTOUT ON A NEW PAGE.

```

NNODE.....NUMBER OF NODES...MAXIMUM OF 278, WITH A MAXIMUM
                BAND WIDTH OF 40
CNTP02240
CNTP02250
CNTP02260
CNTP02270
CNTP02280
CNTP02290
CNTP02300
CNTP02310
CNTP02320
CNTP02330
CNTP02340
CNTP02350
CNTP02360
CNTP02370
CNTP02380
CNTP02390
CNTP02400
CNTP02410
CNTP02420
CNTP02430
CNTP02440
CNTP02450
CNTP02460
CNTP02470
CNTP02480
CNTP02490
CNTP02500
CNTP02510
CNTP02520
CNTP02530
CNTP02540
CNTP02550
CNTP02560
CNTP02570
CNTP02580
CNTP02590
CNTP02600
CNTP02610
CNTP02620
CNTP02630
CNTP02640
CNTP02650
CNTP02660
CNTP02670
CNTP02680
CNTP02690
CNTP02700
CNTP02710
CNTP02720
CNTP02730
CNTP02740
CNTP02750
CNTP02760
CNTP02770
CNTP02780
CNTP02790
CNTP02800
CNTP02810
CNTP02820
CNTP02830
CNTP02840
CNTP02850
CNTP02860
CNTP02870
CNTP02880
CNTP02890
CNTP02900
CNTP02910
CNTP02920
CNTP02930
CNTP02940
CNTP02950
CNTP02960
CNTP02970
CNTP02980
CNTP02990
CNTP03000

```

```

NGP.....NUMBER OF GAUSS POINTS...1 - RECTANGULAR;
IF NGP = 0, A DATA CHECK...IS PERFORMED.
IGO.....ELEMENT TYPE...1 - LINEAR; 2 - QUADRATIC; 3 - CUBIC
NNBC.....NUMBER OF SPECIFIED NODAL VALUES...MAXIMUM OF 150
IPROB.....PROBLEM TYPE...1 - STEADY STATE; 2 - TRANSIENT
COORDINATE SYSTEM...1 - RECTANGULAR;

```

IPUN.....PUNCHED OUTPUT INDICATOR...0 - NO PUNCH; A POSITIVE
 INTEGER SPECIFIES THE NUMBER OF SETS OF VALUES TO
 BE PUNCHED OR STORED.
 NTSC.....NUMBER OF HEAT TRANSFER PHASES...EACH PHASE HAS

CNTP0420
CNTP0430
CNTP0440
CNTP0450
CNTP0460
CNTP0470
CNTP0480
CNTP0490
CNTP0500
CNTP0510
CNTP0520
CNTP0530
CNTP0540
CNTP0550
CNTP0560
CNTP0570
CNTP0580
CNTP0590
CNTP0600
CNTP0610
CNTP0620
CNTP0630
CNTP0640
CNTP0650
CNTP0660
CNTP0670
CNTP0680
CNTP0690
CNTP0700
CNTP0710
CNTP0720
CNTP0730
CNTP0740
CNTP0750
CNTP0760
CNTP0770
CNTP0780
CNTP0790
CNTP0800
CNTP0810
CNTP0820
CNTP0830
CNTP0840
CNTP0850
CNTP0860
CNTP0870
CNTP0880
CNTP0890

PHIREF... A NEW SET OF BOUNDARY CONDITIONS.
REFERENCE VALUE FOR PHI.

- (3) MATERIAL PROPERTIES - IF ELEMENTAL PROPERTIES, NMAT CARDS;
IF NODAL PROPERTIES, NUMBER OF CARDS
IS VARIABLE.
FOR ELEMENTAL PROPERTIES (15,4F15.0)
ONE INTEGER FOR MATERIAL NUMBER, FOUR FLOATING POINT
NUMBERS FOR KX,KY,KZ,AND RHO*C.
FOR NODAL PROPERTIES (2I3,4F18.0)
CASE I: IF ALL NODES(CONSECUTIVE) BETWEEN NODE "N" AND
NODE "M" ARE OF SAME MATERIAL, THE TWO INTEGERS ARE "N" AND
"M" RESPECTIVELY, AND THE FOUR FLOATING POINT NUMBERS ARE
PROPERTIES.
CASE II: IF A NODE "N" HAS UNIQUE PROPERTIES, BOTH
INTEGERS ARE "N" AND PROPERTIES FOLLOW.

- (4) NODE CARDS; TWO SETS OF NNODE CARDS PER PROBLEM
(DATA ASSUMED ZERO IF NOT SPECIFIED)

SET I:
COLS 1-10 NODE NUMBER
COLS 11-30 X COORDINATE OR RADIUS
COLS 31-50 Y COORDINATE OR ANGLE IN DEGREES RELATIVE TO
X AXIS (+ CCW)
COLS 51-70 Z COORDINATE OR ANGLE IN DEGREES RELATIVE TO
Z AXIS (+ CW)

SET II: (NOT REQUIRED FOR THE STEADY STATE PROBLEM)
CASE I: IF ALL NODES(CONSECUTIVE) BETWEEN NODE "N" AND
NODE "M" ARE AT SAME INITIAL TEMPERATURE, THE TWO INTEGERS
ARE "N" AND "M" RESPECTIVELY, AND THE FLOATING POINT
NUMBER IS THE TEMPERATURE.
CASE II: IF A NODE "N" HAS A UNIQUE INITIAL TEMPERATURE,
BOTH INTEGERS ARE "N" AND THE TEMPERATURE FOLLOWS.

- (5) NODE BOUNDARY VALUES; NNBC CARDS PER PROBLEM
INPUT IN ORDER OF ASCENDING NODE NUMBERS
(OMIT IF DATA NOT REQUIRED)
COLS 1-10 JOINT NUMBER
COLS 11-50 SPECIFIED NODAL VALUE

- (6) CONNECTIVITY DATA, NEL CARDS PER PROBLEM (2I5,32I3)
COLS 1-5 ELEMENT NUMBER
COLS 6-10 MATERIAL IDENTIFICATION NUMBER
COLS 11-107 CONNECTIVITY, CCW FROM UPPER RIGHT
LOOKING INTO Y-Z PLANE FROM (+) X AXIS

CC

NODES 1-23 ON CARD I (215,2313)
 NODES 24-32 ON CARD II (913)

(7) TIMES FOR PUNCHED OR STORED OUTPUT. UP TO 20 SEPARATE
 TIMES MAY BE REQUESTED IN A FORMAT(10F8.0,/,10F8.0). VALUES
 ARE PUNCHED OR STORED DEPENDING ON THE JOB CONTROL LANGUAGE
 USING AN OUTPUT FORMAT OF (15,F20.0).

HEAT TRANSFER PHASE DATA

FOR EACH PHASE, THE REMAINING DATA MUST BE SUPPLIED. IF STEADY
 STATE PROBLEM ONLY ONE SET IS USED.

TRANSIENT PROBLEM DATA (SUPPLY ONE BLANK CARD FOR STEADY STATE
 PROBLEM WHICH HAS NELBC = 0); OTHERWISE (315,3F10.0)
 NUMBER OF INITIAL STEPS(ISTP)
 NUMBER OF STEPS BETWEEN PRINTINGS(IPRT)
 NUMBER OF BOUNDARY CONDITIONS(NELBC), MAXIMUM OF 48
 BEGIN TIME FOR THIS PHASE(TIM)
 INITIAL STEPSIZE FOR THIS PHASE(DT)
 END TIME FOR THIS PHASE(ENDT)

BOUNDARY CONDITION DATA: NELBC SETS PER PHASE
 EACH SET PERTAINS TO A PARTICULAR CONDITION ON ONE ELEMENT
 SURFACE. A SET CONTAINS A FIRST CARD PLUS A VARIABLE NUMBER
 OF CARDS.

FIRST CARD: (415), ELEMENT NUMBER, BOUNDARY CONDITION TYPE(KBT),
 SURFACE APPLICABLE(KBS), NUMBER OF CARDS SPECIFYING PROPERTY
 VALUE(NODECD); WHERE

BOUNDARY CONDITION TYPE(KBT)
 1 = VOLUME INTEGRAL OF LOADING
 2 = SURFACE INTEGRAL OF LOADING
 3 = SURFACE LOSS COEFFICIENT
 SURFACE APPLICABLE (KBS)
 0 = VOLUME INTEGRAL (USE WITH KBT = 1 ONLY)
 1 = SURFACE INTEGRAL OVER XI = +/- 1 FACE
 2 = SURFACE INTEGRAL OVER ETA = +/- 1 FACE
 3 = SURFACE INTEGRAL OVER ZETA = +/- 1 FACE
 *** KBS = 1, 2 OR 3 MUST BE SIGNED TO INDICATE ***
 *** PLUS (+) OR MINUS (-) FACE ***

NODECD CARDS (VARIABLE), (215,1G20.8)
 IF ALL NODES (CONSECUTIVE NUMBERING) ON PARTICULAR ELEMENT
 SURFACE BETWEEN NODE "N" AND NODE "M" HAVE SAME BOUNDARY
 CONDITION PROPERTY VALUE, THE TWO INTEGERS ARE "N" AND "M",
 AND PROPERTY VALUE FOLLOWS.

CNTP0900
 CNTP0910
 CNTP0920
 CNTP0930
 CNTP0940
 CNTP0950
 CNTP0960
 CNTP0970
 CNTP0980
 CNTP0990
 CNTP1000
 CNTP1010
 CNTP1020
 CNTP1030
 CNTP1040
 CNTP1050
 CNTP1060
 CNTP1070
 CNTP1080
 CNTP1090
 CNTP1100
 CNTP1110
 CNTP1120
 CNTP1130
 CNTP1140
 CNTP1150
 CNTP1160
 CNTP1170
 CNTP1180
 CNTP1190
 CNTP1200
 CNTP1210
 CNTP1220
 CNTP1230
 CNTP1240
 CNTP1250
 CNTP1260
 CNTP1270
 CNTP1280
 CNTP1290
 CNTP1300
 CNTP1310
 CNTP1320
 CNTP1330
 CNTP1340
 CNTP1350
 CNTP1360
 CNTP1370

INP00160

WRITE (6,360) TITLE

READ PROBLEM CHARACTERISTICS

READ (5,370) NNODE, NEL, NMAT, NGP, IGO, NNBC, IPROB, KC, IPUN, NTSC, PHIREF
 WRITE (6,380) NNODE, NEL, NMAT, NGP, NNBC, IPROB, PHIREF
 NPCL = 12*(IGO-1)+8
 WRITE (6,390) IGO, NPCL
 IF (IPUN.GE.1) WRITE (6,400)

ZERO STORAGE AREAS

10 NPI = NPEL+1
 DO 10 J=1, NPI
 DC 10 I=1, NEL
 NCON(I, J) = 0
 DO 20 I=1, NNODE
 F(I) = 0.000
 PHI(I) = 0.000
 PHIDGT(I) = 0.000
 PHWRK(I) = 0.000
 DO 20 J=1, 3
 CCARD(I, J) = 0.000
 DO 30 J=1, 4
 NNODE
 DO 30 I=1, NNODE
 PPROP(I, J) = 0.000
 DO 40 I=1, 48
 DO 40 J=1, 3
 KELB(I, J) = 0
 DO 50 J=1, 3
 DO 50 I=1, NNODE
 DO 50 I, J) = 0.000

READ MATERIAL PROPERTIES

IF (NMAT.LT.0) GO TO 80
 IF (NMAT.EQ.0) GO TO 130
 DO 60 I=1, NMAT
 READ (5,410) ID, (PROP(ID, J), J=1, 4)
 WRITE (6,420)
 DO 70 I=1, NMAT
 WRITE (6,430) I, (PROP(I, J), J=1, 4)
 DO 70 GO TO 130
 INNODE = 1
 IF (INNODE.EQ.NNODE) GO TO 110
 READ (5,440) INNODE, INUDE, PROPI, PROP2, PROP3, PROP4
 DO 100 I=1, INNODE, INUDE
 PROP(I, 1) = PROPI

INP00550
INP00560
INP00570
INP00580
INP00590
INP00600
INP00610

INP00620
INP00630
INP00640
INP00650
INP00660
INP00670
INP00680
INP00690
INP00700
INP00710
INP00720
INP00730
INP00740
INP00750
INP00760
INP00770
INP00780
INP00790
INP00800
INP00801

INP00810
INP00820
INP00830
INP00840
INP00850
INP00860
INP00870
INP00880
INP00890
INP00900
INP00910
INP00920
INP00930
INP00940

```

      PROP(I,2) = PROP2
      PROP(I,3) = PROP3
      PROP(I,4) = PROP4
100  GC TO 90
110  WRITE (6,450)
      DO 120 I=1,NNODE
120  WRITE (6,460) I, (PROP(I,J), J=1,4)
C
      READ JOINT COORDINATES, INITIAL NODAL VALUES, ANY FIXED BOUNDARY
      VALUES.  CONVERT IF NECESSARY
C
130  DO 140 I=1,NNODE
140  READ (5,470) IJT, (COORD(IJT,J), J=1,3)
      IF (IPRUB.LE.1) GO TO 170
      I = I
150  IF (I.EQ.NNODE) GO TO 170
      READ (5,480) IJT1, IJT2, PHIVAL
      DO 160 I=IJT1, IJT2
160  PHI(I) = PHIVAL
      GC TO 150
170  IF (NNEC.LE.0) GO TO 200
      DO 180 I=1,NNBC
180  READ (5,490) KNODE(I), PHIWRK(KNODE(I))
      WRITE (6,500)
      DO 190 I=1,NNBC
190  ID = KNODE(I)
      WRITE (6,510) ID, PHIWRK(ID)
200  WRITE (6,520)
      DO 210 I=1,NNODE
210  WRITE (6,530) I, (COORD(I,J), J=1,3), PHI(I)
      IF (KC.GT.1) CALL MARK(COORD, KC, NNODE)
C
      READ CONNECTIVITY, COMPUTE HALF BAND WIDTH
C
220  DO 220 I=1, NEL
      READ (5,540) IEL, (NCON(IEL,J), J=1,NP1)
      WRITE (6,550)
230  DO 230 I=1, NEL
      WRITE (6,560) I, (NCON(I,J), J=1,NP1)
      NBAND = 0
      DO 250 I=1, NEL
250  DO 240 J=2, NP1
      JK = J+1
      DO 240 K=JK, NP1
240  NBAND = MAX0(NBAND, IABS(NCON(I,J)-NCON(I,K)))
250  CONTINUE
      NBAND = NBAND+1
      WRITE (6,570) NBAND

```


UUU

READ TIMES FOR PUNCHED OR STORED OUTPUT

INP00950
INP00960
INP00970
INP00980
INP00990

```
IF (IPUN.EQ.0) GO TO 260
IF (IPROB.EQ.1) GO TO 260
IF READ (5,580) {PUNCH(I),I=1,IPUN)
IF WRITE (6,590) {PUNCH(I),I=1,IPUN)
IF (NGP.NE.0) RETURN
```

260

READ PHASE DATA AND BOUNDARY CONDITIONS

uuu

INP01000
INP01010
INP01020
INP01030
INP01040
INP01050
INP01060
INP01070
INP01080
INP01090
INP01100
INP01110
INP01120
INP01130
INP01140
INP01150
INP01160
INP01170
INP01180
INP01190
INP01200
INP01210
INP01220
INP01230
INP01240
INP01250
INP01260
INP01270
INP01280
INP01290
INP01300
INP01310
INP01320
INP01330
INP01340
INP01350
INP01360

```

270 IHTC = 1
    READ (5,600) ISTP,IPRT,NELBC,TIM,DT,ENDT
    WRITE (6,610)
    WRITE (6,620) ISTP,IPRT,NELBC,TIM,DT,ENDT
    IF (NELBC.EQ.0) GO TO 350
    WRITE (6,650)
    DO 340 I=1,NELBC
        READ (5,640) (KELB(I,J),J=1,3),NODECD
        IELTYP = KELB(I,1)
        NBCTYP = KELB(I,2)
        GO TO (280,300,320), NBCTYP
    DO 290 II=1,NODECD
        READ (5,630) IJT1,IJT2,PROPB
        DC 290 IJT=IJT1,IJT2
        PROB(IJT,1) = PROPB
    WRITE (6,660) IEL,IJT,NBCTYP,KELB(I,3),PROB(IJT,1)
    GO TO 340
280 DO 310 II=1,NODECD
        READ (5,630) IJT1,IJT2,PROPB
        DC 310 IJT=IJT1,IJT2
        PROB(IJT,2) = PROPB
    WRITE (6,660) IEL,IJT,NBCTYP,KELB(I,3),PROB(IJT,2)
    GO TO 340
320 DO 330 II=1,NODECD
        READ (5,630) IJT1,IJT2,PROPB
        DO 330 IJT=IJT1,IJT2
        PROB(IJT,3) = PROPB
    WRITE (6,660) IEL,IJT,NBCTYP,KELB(I,3),PROB(IJT,3)
    CCNTINUE
340 IF (NGP.NE.0) RETURN
350 IF (IHTC.EQ.NTSC) RETURN
    IHTC = IHTC+1
    GO TO 270
360 FORMAT (10A8)
370 FORMAT (10I5,F2C.0)
380 FORMAT (///,11X, : NUMBER OF NODES = ',I5,///,11X, :
    ITS = ',I5,///,11X, : NUMBER OF MATERIALS = ',I5,///,11X, :

```

INPO1330
INPO1340
INPO1350
INPO1360





SUBROUTINE MERGE (IMTRX)

THIS SUBROUTINE FORMS "H" AND "C" MATRICES

```

IMPLICIT REAL*8(A-H,O-Z)
COMMON /MTRX/  CORD(32,3), ENN(32,32), DCOL(32,3), COL(32), AJ(3,3), AJI(3,3),
1(3,3), ENW(32,32), FE(32), PROP(278,4), CUARD(278,3), F(278), PHI(278), PMER000040
2HIDDT(278), PHIRK(278), PHIDDT(278), BGH(278,40), BGC(278,40), PRGB(278,40), PRGB(278,40),
3(3,3), PUNCH(20)
COMMON /IMTRX/  NCON(48,33), KELB(48,3), KNODE(278)
COMMON /FLP/  DTJ, AKX, AKY, AKZ, FACT, PHIRKF, APZRO, ABIGN, TIM, DI, ENDT
COMMON /INT/  IGO, NBAND, NNODE, NMAT, NEL, NPCL, NGP, NELBC, NNBC, IPROB, KC
1, NTSC, LPUN, IPR1, ISTR

```

ZERO MASTER AND ELEMENT MATRICES

```

IF (IMTRX.EQ.2) GO TO 20
DO 10 J=1,NBAND
DO 10 I=1,NNODE
10 BGH(I,J) = 0.0D0
GO TO 40
DO 20 J=1,NBAND
DO 30 I=1,NNODE
20 BGC(I,J) = 0.0DQ
DO 40 I=1,NEL
DO 50 JJ=1,3
DO 50 II=1,NPEL
50 CURD(II,JJ) = 0.0D0
DO 60 II=1,NPEL
DO 60 JJ=1,NPEL
60 ENW(JJ,II) = 0.0D0
DO 70 J=1,NPEL
70 INDX = NCON(I,J+1)
DO 70 K=1,3
70 CORE(J,K) = COARD(INDX,K)
IF (IMTRX.EQ.2) GO TO 90

```

FORM ELEMENTAL "H" MATRIX

```

      IF (NMAJ.LT.0) GO TO 80
      IMAT = NCON(I,1)
      AKX = PRCP(IMAT,1)
      AKY = PRCP(IMAT,2)
      AKZ = PRCP(IMAT,3)
      CALL CUBE (NPEL,NGP,1,I,NMAT)
80    GO TO 120

```

FCRM ELEMENTAL "C" MATRIX


```
C
90 IF (NMAT.LT.O) GO TO 100
   IMAT = NCON(I,I)
   FACT = PROP(IMAT,4)
100 CALL CUBE (NPCL,NGP,2,I,NMAT)
    DO 110 I=1,NPEL
      ICX = NCON(I,I+1)
      DO 110 JJ=1,NPEL
        IDY = NCON(I,JJ+1)-ICX+1
        IF (IDY.LT.I) GO TO 110
        BGC(IDX,IDY) = BGC(IDX,IDY)+ENW(II,JJ)
110 GC CONTINUE
    GC TO 140
C
C
C
ASSEMBLE APPROPRIATE MASTER MATRIX
C
120 DO 130 II=1,NPEL
     IDX = NCON(II,II+1)
     DO 130 JJ=1,NPEL
       IDY = NCON(II,JJ+1)-IDX+1
       IF (IDY.LT.II) GO TO 130
       BGH(IDX,IDY) = BGH(IDX,IDY)+ENW(II,JJ)
130 CONTINUE
140 RETURN
END
C
C
C
SUBROUTINE STEADY
C
C
C THIS SUBROUTINE SOLVES THE STEADY STATE PROBLEM
C
C
C IMPLICIT REAL*8(A-H,O-Z)
COMMON /MTRX/ CORD(32,3), ENN(32,32), DCOL(32,3), COL(32), AJ(3,3), AJSTY000020
1(3,3), ENW(32,32), FE(32), PHIRK(278), PHI(278), PHI(278), PROB(278) STY000030
2HIDOT(278), PHIRK(278), PHIDD(278), BGH(278,40), BGC(278,40), STY000040
38,3), PUNCH(20) STY000050
COMMON /IMTRX/ NCON(48,33), KELB(48,3), KNODE(278) STY000060
COMMON /FLP/ DTJ,AKX,AKY,AKZ,FACT,PHIREF,APZRO,ABIGN,TIM,DT,ENDT STY000070
COMMON /INT/ IGO,NBANO,NNODE,NMAT,NEL,NPEL,NGP,NELBC,NNBC,I,PROB,KC STY000080
1,NTSC,IPUN,IPRT,ISTP STY000090
DO 110 I=1,NNODE STY000100
PHI(I) = PHIRK(I) STY000110
PHIRK(I) = F(I) STY000120
PHIRK(I) = -F(I) STY000130
110 F(I) = -F(I) STY000140
C
C
C
MER000380
MER000390
MER000400
MER000410
MER000420
MER000430
MER000440
MER000450
MER000460
MER000470
MER000480
MER000490
C
MER000500
MER000510
MER000520
MER000530
MER000540
MER000550
MER000560
MER000570
MER000580
MER000590
C
*****
STY000010
C
C
C
STY000020
STY000030
STY000040
STY000050
STY000060
STY000070
STY000080
STY000090
STY000100
STY000110
STY000120
STY000130
STY000140
```


C

C FIND APPROXIMATE VALUE OF ZERO

```

CALL BNSN (BGH,NNODE,APZRO,ABIGN)
WRITE (6,200) APZRO,ABIGN
IF (NNBC.LE.0) GO TO 150

```

C C

C APPLY ANY KNOWN NODAL VALUES AND SOLVE EQUATION

```

NEND = NBAND-1
DO 140 I=1,NNBC
  ID = KNODE(I)
  NCHK = NNODE-ID
  DO 130 J=1,NEND
    IROW = ID-J
    ICOL = J+1
    INDX = ID+J
    IF (IROW.LE.0) GO TO 120
    F(IROW) = F(IROW)-BGH(IROW,ICOL)*PHI(ID)
    IF (J.GT.NCHK) GO TO 130
    F(INDX) = F(INDX)-BGH(ID,ICOL)*PHI(ID)
  120 CONTINUE
  130 F(ID) = F(ID)-BGH(ID,1)*PHI(ID)
  BGH(ID,1) = ABIGN
  CONTINUE
  140 CALL LDLT (BGH,NNODE,NBAND,APZRO)
  150 CALL SLV (BGH,F,NNODE,NBAND)
  IF (NNBC.LE.0) GO TO 170
  DO 160 I=1,NNBC
    ID = KNODE(I)
    PHIWRK(ID) = PHIWRK(ID)+ABIGN*F(ID)
  160 F(ID) = PHI(ID)

```

C C

C FORM TEMPERATURE VALUES AND LOAD VALUES

```

170 WRITE (6,210)
DO 180 I=1,NNODE
  PHI(I) = F(I)+PHIREF
180 WRITE (6,220) I,(COORD(I,J),J=1,3),PHI(I),PHIWRK(I)
  TOT = 0.000
DO 190 I=1,NNODE
  TOT = TOT+PHIWRK(I)
190 WRITE (6,230) TOT
  WHEN = 1.0D20
  IF (IPUN.NE.0) CALL PUNC(PHI,WHEN)
  RETURN
200 FORMAT ('0',10X,'THE APPROXIMATE VALUE OF ZERO = ',1PG24.16,'/',23X
1,' ',ABIGN = ',C24.16,/')
210 FORMAT ('1',1,' ',NODE',7X,'X COORDINATE',8X,'Y COORDINATE',8X,'Z COOR

```

STY000150
 STY000160
 STY000170

STY000180
 STY000190
 STY000200
 STY000210
 STY000220
 STY000230
 STY000240
 STY000250
 STY000260
 STY000270
 STY000280
 STY000290
 STY000300
 STY000310
 STY000320
 STY000330
 STY000340
 STY000350
 STY000360
 STY000370
 STY000380
 STY000390
 STY000400

STY000410
 STY000420
 STY000430
 STY000440
 STY000450
 STY000460
 STY000470
 STY000480
 STY000490
 STY000491
 STY000500
 STY000510
 STY000520
 STY000530

TME00310
TME00320
TME00330
TME00340

TME00350
TME00360
TME00370
TME00380

TME00390
TME00400
TME00410
TME00420

TME00430
TME00440
TME00450
TME00460
TME00470
TME00480
TME00490
TME00500
TME00510
TME00520
TME00530
TME00540
TME00550
TME00560
TME00570
TME00580
TME00590
TME00600
TME00610
TME00620
TME00630
TME00640
TME00650
TME00660
TME00670
TME00680
TME00690

```
20 BGC(I,J) = AHJJ+CIJ
   CALL BNSN (BGC,NNODE,APZRO,ABIGN)
   WRITE (6,360) APZRO,ABIGN
   IF (NNBC.LE.0) GO TO 40
```

C
C
C
APPLY ANY KNOWN NODAL TEMPERATURES

```
DO 30 I=1,NNBC
  ID = KNODE(I)
  PHI(ID) = PHIWRK(ID)
30 BGC(ID,1) = ABIGN
```

C
C
C
DECOMPOSE COEFFICIENT MATRIX FOR STARTING VALUES

```
40 CALL LDLT (BGC,NNODE,NBAND,APZRO)
   TIME(1) = TIM
   DO 50 I=1,NNODE
50 OUT(I,1) = PHI(I)+PHIREF
```

C
C
C
SOLVE FOR STARTING VALUES, STORE THOSE REQUIRED

```
DO 60 I=1,NNODE
60 PHISAV(I,1) = PHI(I)
   ISTEP3 = ISTEP/3
   ISTEP2 = 2*ISTEP3
   DO 170 IT=1,ISTEP
   CALL MULT (BGC,PHI,PHIDOT,NNODE,NBAND)
   DO 70 I=1,NNODE
70 PHIDOT(I) = -PHIDOT(I)-F(I)
   IF (NNBC.LE.0) GO TO 90
   DC 80 I=1,NNBC
   ID = KNODE(I)
   PHIDOT(ID) = 0.000
```

```
80 CALL SLV (BGC,PHIDOT,NNODE,NBAND)
90 DO 100 I=1,NNODE
100 PHI(I) = PHI(I)+PHIDOT(I)
   IF (IT.NE.ISTEP3) GO TO 130
   DO 110 I=1,NNODE
110 PHISAV(I,2) = PHI(I)
   IXX = IXX+1
   IF (IPRT.NE.IXX) GO TO 130
   IXX = 0
```

```
120 TIME(IDP) = DT*IT+TIM
   DO 120 I=1,NNODE
120 OUT(I,IDP) = PHI(I)+PHIREF
130 IF (IT.NE.ISTEP2) GO TO 160
   DC 140 I=1,NNODE
```



```

140 PHISAV(I,3) = PHI(I)
    IXX = IXX+1
    IF (IPRT.NE.IXX) GO TO 160
    IXX = 0
    IDP = IDP+1
    TIME(IDP) = DT*IT+TIM
    DO 150 I=1,NNODE
    OUT(I,IDP) = PHI(I)+PHIREF
150 OUT(I,IDP) = PHI(I)+PHIREF
160 WHEN = TIM+DT*IT
    IF (IPUN.NE.0) CALL PUNC(PHI,WHEN)
170 CONTINUE
    DO 180 I=1,NNODE
    PHISAV(I,4) = PHI(I)
180 IXX = IXX+1
    IF (IPRT.NE.IXX) GO TO 200
    IXX = 0
    IDP = IDP+1
    TIME(IDP) = DT*IT+TIM
    DO 190 I=1,NNODE
190 OUT(I,IDP) = PHI(I)+PHIREF

```

REBUILD LEFT HAND SIDE FOR FIVE POINT FINITE DIFFERENCE METHOD
FIND ABIGN,APZRO

```

200 REWIND 10
    READ (10) BGC
    DT = DT*DFLOAT(ISTP3)
    ITIM = (ENDT-TIM)/DT
    NTP = (ITIM/IPRT)+1
    DO 210 J=1,NNODE
    CIJ = BGC(I,J)*25.000/DT
    AHIJ = BGH(I,J)*12.000
    BGH(I,J) = CIJ+AHIJ
210 CALL BNSN (BGH,NNODE,APZRO,ABIGN)
    IF (NNBC.EQ.0) GO TO 230

```

APPLY ANY KNOWN NODAL VALUES

```

    DO 220 I=1,NNBC
    IC = KNODE(I)
220 BGH(IC,1) = ABIGN
230 CALL LDLT (BGH,NNODE,NBAND,APZRO)

```

COMMENCE SOLUTION, PRINTING AND PUNCHING OUTPUT AS REQUIRED

```

    DO 340 IT=4,ITIM
    DO 240 I=1,NNODE

```

TME00700
TME00710
TME00720
TME00730
TME00740
TME00750
TME00760
TME00770
TME00780
TME00781
TME00790
TME00800
TME00810
TME00820
TME00830
TME00840
TME00850
TME00860
TME00870
TME00880

TME00890
TME00900
TME00910
TME00920
TME00930
TME00940
TME00950
TME00960
TME00970
TME00980
TME00990
TME01000

TME01010
TME01020
TME01030
TME01040

TME01050
TME01060



```

240 PHI(I) = (48.0D0*PHISAV(I,4)-36.0D0*PHISAV(I,3)+16.0D0*PHISAV(I,2)
1-3.0D0*PHISAV(I,1))/DT
CALL MULT (BGC,PHI,PHIDOT,NNODE,NBAND)
DO 250 I=1,NNODE
250 PHI(I) = PHIDOT(I)-12.0D0*F(I)
IF (NNBC.EC.O) GO TO 270
DO 260 I=1,NNBC
260 PHI(ID) = ABIGN*PHIWRK(ID)
270 CALL SLV (BGH,PHI,NNODE,NBAND)
DO 280 I=1,NNODE
PHISAV(I,1) = PHISAV(I,2)
PHISAV(I,2) = PHISAV(I,3)
PHISAV(I,3) = PHISAV(I,4)
PHISAV(I,4) = PHI(I)
280 IXX = IXX+1
IF (IXX.NE.IPRT) GO TO 330
IXX = 0
IDP = IDP+1
TIME(IDP) = DT*IT+TIM
DO 290 I=1,NNODE
290 OUT(I,IDP) = PHI(I)+PHIREF
IF (NTP.LT.JE) JE = NTP
IF (IDP.LT.JE) GO TO 330
NL1 = 1
NL2 = NLINE
WRITE (6,370) DT, (TIME(J), J=1,JE)
300 WRITE (6,380)
WRITE (6,390)
DO 310 I=NL1,NL2
310 WRITE (6,400) I, (COORD(I,J), J=1,3), (OUT(I,K), K=1,JE)
IF (NL2.EQ.NNODE) GO TO 320
NL1 = NL2+1
NL2 = NL2+NLINE
IF (NL2.GT.NNODE) NL2=NNODE
WRITE (6,410)
GO TO 300
320 NTP = NTP+JE
IDP = 0
330 WHEN = TIM+DT*IT
IF (IPUN.NE.O) CALL PUNC(PHI,WHEN)
340 CCNTINUE
350 CCNTINUE
RETURN
360 FCFORMAT (///,11X,'THE APPROXIMATE VALUE(FOR STARTING VALUES) OF ZER
10 = ' ,1PG24.16,///,23X,' ABIGN = ' ,G24.16,///)
370 FORMAT(11,1,38X,'DISTRIBUTION OF TEMPERATURE VERSUS TIME/SPATIAL PTIME01530
10SITION',/,42X,'FINITE DIFFERENCE METHOD WITH TRAPEZOIDAL STARTER',TIME01540

```






```

IDGP = NGP-1
DO 40 I=1,NGP
  XI = GP(I,IDGP)
  WX = WF(I,IDGP)
  DO 40 J=1,NGP
    ETA = GP(J,IDGP)
    WY = WF(J,IDGP)
  DO 40 K=1,NGP
    ZETA = GP(K,IDGP)
    WZ = WF(K,IDGP)
  WFACT = WX*WY*WZ
  IF (ITYPE.EQ.1) GO TO 10
  CALL FORMC (XI,ETA,ZETA,NPEL,IEL,NMAT)
  GO TO 20
10 CALL FORMH (XI,ETA,ZETA,NPEL,IEL,NMAT)
20 DO 30 L=1,NPEL
  DO 30 M=1,NPEL
    ENW(L,M) = ENW(L,M)+WFACT*ENN(L,M)
40 CONTINUE
DO 50 I=1,NPEL
  DO 50 J=1,NPEL
    ENW(I,J) = 5.0-I*(ENW(I,J)+ENW(J,I))
50 ENW(J,I) = ENW(I,J)
RETURN

```

CC

FOR USE WITH BOUNDARY CONDITIONS

```

ENTRY CUBEB(NPEL,NGP,KBT,KBS,IEL)
NGPT = NGP
NGP = 5
IDGP = NGP-1
KBSABS = IABS(KBS)
GO TO (60,100,100), KBT
60 DO 90 I=1,NGP
  XI = GP(I,IDGP)
  WX = WF(I,IDGP)
  DO 90 J=1,NGP
    ETA = GP(J,IDGP)
    WY = WF(J,IDGP)
  DO 90 K=1,NGP
    ZETA = GP(K,IDGP)
    WZ = WF(K,IDGP)
  WFACT = WX*WY*WZ
  CALL FORMV (XI,ETA,ZETA,NPEL)
  CONST = 0.000
DO 70 L=1,NPEL
  ID = NCGN(IEL,L+1)
70 CONST = CONST+PROB(ID,1)*COL(L)

```

CUB000250
CUB000260
CUB000270
CUB000280
CUB000290
CUB000300
CUB000310
CUB000320
CUB000330
CUB000340
CUB000350
CUB000360
CUB000370
CUB000380
CUB000390
CUB000400
CUB000410
CUB000420
CUB000430
CUB000440
CUB000450
CUB000460
CUB000470
CUB000480

CUB000490
CUB000500
CUB000510
CUB000520
CUB000530
CUB000540
CUB000550
CUB000560
CUB000570
CUB000580
CUB000590
CUB000600
CUB000610
CUB000620
CUB000630
CUB000640
CUB000650
CUB000660
CUB000670
CUB000680
CUB000690


```

CONST = CONST*DTJ*WFACT
DO 80 L=1,NPEL
80 FE(L) = FE(L)-COL(L)*CONST
90 CONTINUE
100 GO TO 220
C3 = DFLQAT(KBS/KBSABS)
DO 200 I=1,NGP
C1 = GP(I,IDGP)
W1 = WF(I,IDGP)
DO 200 J=1,NGP
C2 = GP(J,IDGP)
W2 = WF(J,IDGP)
WFACT = W1*W2
GO TO (110,120,130), KBSABS
110 XI = C3
ETA = C1
ZETA = C2
GO TO 140
120 XI = C1
ETA = C3
ZETA = C2
GO TO 140
130 XI = C1
ETA = C2
ZETA = C3
140 IF (KBT.EQ.3) GO TO 170
CALL FORMSQ (XI,ETA,ZETA,NPEL,KBSABS)
CONST = 0.000
DO 150 L=1,NPEL
ID = NCON(IEL,L+1)
CONST = CONST+PROB(ID,2)*COL(L)
CONST = CONST*DTJ*WFACT
DO 160 L=1,NPEL
FE(L) = FE(L)+CONST*COL(L)
GO TO 200
170 CALL FORMSA (XI,ETA,ZETA,NPEL,KBSABS)
CONST = 0.000
DO 180 L=1,NPEL
ID = NCON(IEL,L+1)
CONST = CONST+PROB(ID,3)*COL(L)
CONST = CONST*WFACT*DTJ
DO 190 L=1,NPEL
DO 190 M=1,NPEL
ENW(L,M) = ENW(L,M)+ENN(L,M)*CONST
200 CONTINUE
IF (KBT.NE.3) GO TO 220
DO 210 I=1,NPEL
DO 210 J=1,NPEL

```

```

CUB00700
CUB00710
CUB00720
CUB00730
CUB00740
CUB00750
CUB00760
CUB00770
CUB00780
CUB00790
CUB00800
CUB00810
CUB00820
CUB00830
CUB00840
CUB00850
CUB00860
CUB00870
CUB00880
CUB00890
CUB00900
CUB00910
CUB00920
CUB00930
CUB00940
CUB00950
CUB00960
CUB00970
CUB00980
CUB00990
CUB01000
CUB01010
CUB01020
CUB01030
CUB01040
CUB01050
CUB01060
CUB01070
CUB01080
CUB01090
CUB01100
CUB01110
CUB01120
CUB01130
CUB01140
CUB01150
CUB01160
CUB01170

```





```

140 DCOL(I,3) = AJI(3,1)*DNX+AJI(3,2)*DNY+AJI(3,3)*DNZ
    IF (NMAT.GT.0) GO TO 160
    AKX = 0.000
    DO 150 I=1,NPEL
        ID = NCON(IEL,I+1)
        AKX = AKX+PROP(ID,1)*COL(I)
    150 DO 170 I=1,NPEL
        BX = DCOL(I,1)*DTJ
    160 DO 170 J=1,NPEL
        ENN(I,J) = BX*AKX*DCOL(J,1)
    170 IF (NMAT.GT.0) GO TO 190
        AKY = 0.000
    180 DO 180 I=1,NPEL
        ID = NCON(IEL,I+1)
        AKY = AKY+PROP(ID,2)*COL(I)
    190 DO 200 I=1,NPEL
        BY = DCOL(I,2)*DTJ
    200 DO 200 J=1,NPEL
        ENN(I,J) = BY*AKY*DCOL(J,2)+ENN(I,J)
    IF (NMAT.GT.0) GO TO 220
        AKZ = 0.000
    210 DO 210 I=1,NPEL
        ID = NCON(IEL,I+1)
        AKZ = AKZ+PROP(ID,3)*COL(I)
    220 DO 230 I=1,NPEL
        BZ = DCOL(I,3)*DTJ
    230 DO 230 J=1,NPEL
        ENN(I,J) = BZ*AKZ*DCOL(J,3)+ENN(I,J)
        RETURN

```

C
C
C

FOR USE WITH B.C. CALCULATIONS

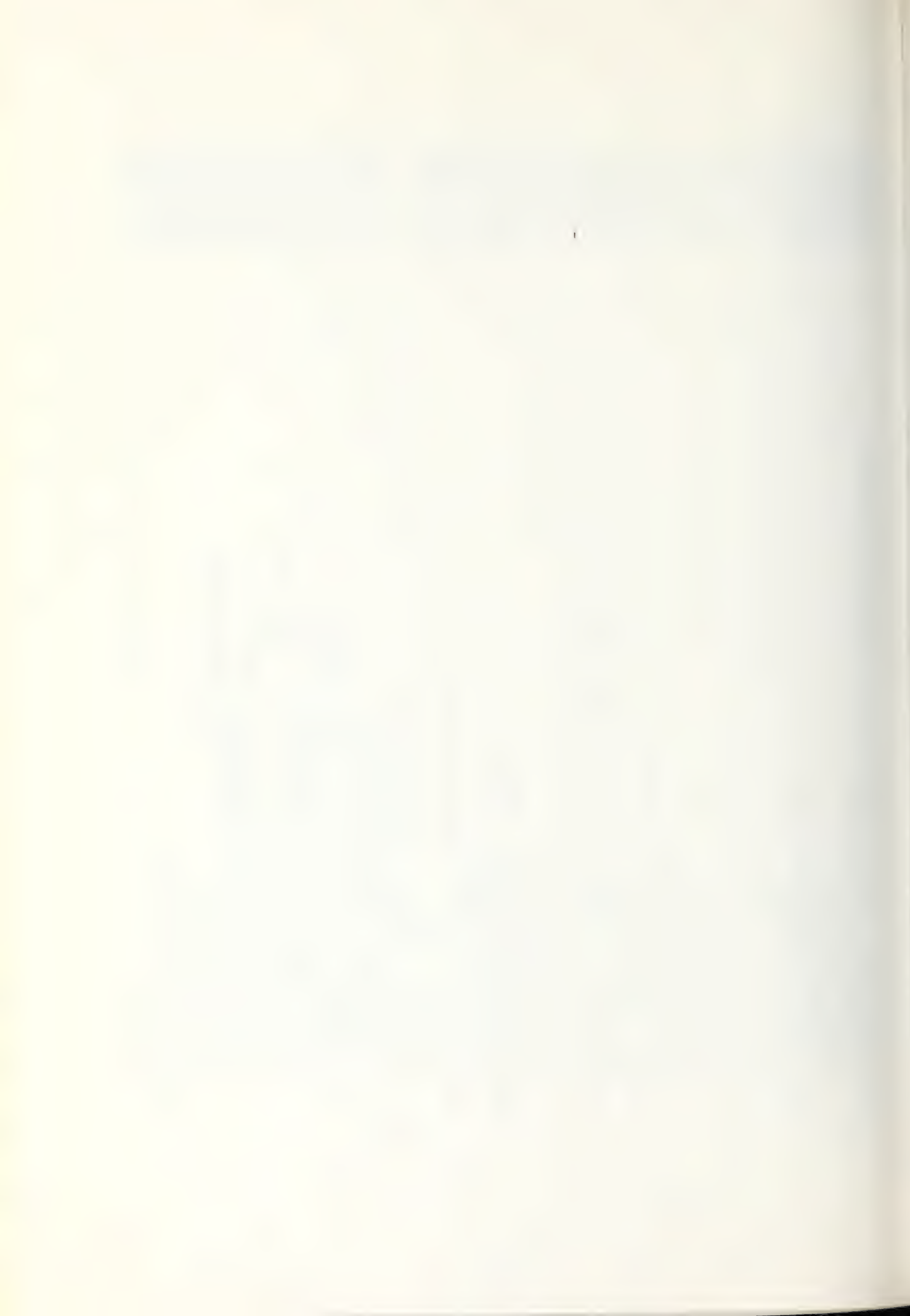
```

ENTRY FORMV(XI,ETA,ZETA,NPEL)
CALL SHAPE (XI,ETA,ZETA,NPEL)
CALL JACOB (NPEL,AJ,DCOL,CORD,AJI,DTJ)
RETURN
ENTRY FORMSQ(XI,ETA,ZETA,NPEL,KBSABS)
CALL SHAPE (XI,ETA,ZETA,NPEL)
CALL JACOB (NPEL,AJ,DCOL,CORD,DTJ,KBSABS)
RETURN
ENTRY FORMSA(XI,ETA,ZETA,NPEL,KBSABS)
CALL SHAPE (XI,ETA,ZETA,NPEL)
CALL JACOB (NPEL,AJ,DCOL,CORD,DTJ,KBSABS)
DO 240 I=1,NPEL
    A = COL(I)
    DO 240 J=1,NPEL
        ENN(I,J) = A*COL(J)
    RETURN
240

```

FMCO0310
FMCO0320
FMCO0330
FMCO0340
FMCO0350
FMCO0360
FMCO0370
FMCO0380
FMCO0390
FMCO0400
FMCO0410
FMCO0420
FMCO0430
FMCO0440
FMCO0450
FMCO0460
FMCO0470
FMCO0480
FMCO0490
FMCO0500
FMCO0510
FMCO0520
FMCO0530
FMCO0540
FMCO0550
FMCO0560
FMCO0570
FMCO0580
FMCO0590

FMCO0600
FMCO0610
FMCO0620
FMCO0630
FMCO0640
FMCO0650
FMCO0660
FMCO0670
FMCO0680
FMCO0690
FMCO0700
FMCO0710
FMCO0720
FMCO0730
FMCO0740
FMCO0750



SHP000380
 SHP000390
 SHP000400
 SHP000410
 SHP000420
 SHP000430
 SHP000440
 SHP000450
 SHP000460
 SHP000470
 SHP000480
 SHP000490

SHP000500
 SHP000510
 SHP000520
 SHP000530
 SHP000540
 SHP000550
 SHP000560
 SHP000570
 SHP000580
 SHP000590
 SHP000600
 SHP000610
 SHP000620
 SHP000630

SHP000640
 SHP000650
 SHP000660
 SHP000670
 SHP000680
 SHP000690
 SHP000700
 SHP000710
 SHP000720
 SHP000730
 SHP000740
 SHP000750
 SHP000760
 SHP000770
 SHP000780

```

110 ICOL = 0
    DO 120 IZ=1,2
      DO 120 IX=1,4
        ICOL = ICOL+1
        XI = CORDX(IX)
        YI = CORDY(IX)
        ZI = CORDZ(IZ)
        CUL(ICOL) = FL(XI,ETA,ZETA,XI,YI,ZI)
        DCUL(ICOL,1) = DFL(ETA,ZETA,XI,YI,ZI)
        DCOL(ICOL,2) = DFL(ZETA,XI,YI,ZI,XI)
        DCOL(ICOL,3) = DFL(XI,ETA,ZI,XI,YI)
      GO TO 190
120

```

C
 C
 C
 C
 QUADRATIC FUNCTIONS
 CCRNER NODES

```

130 DO 140 IZ=1,2
      II = I2*(IZ-1)+1
      IT = II+6
      IX = 0
      DO 140 ICOL=II,IT,2
        IX = IX+1
        YI = CORDY(IX)
        XI = CORDX(IX)
        ZI = CORDZ(IZ)
        CUL(ICOL) = FQC(XI,ETA,ZETA,XI,YI,ZI)
        DCUL(ICOL,1) = DFQC(XI,ETA,ZETA,XI,YI,ZI)
        DCOL(ICOL,2) = DFQC(ETA,ZETA,XI,YI,ZI,XI)
        DCOL(ICOL,3) = DFQC(ZETA,XI,ETA,ZI,XI,YI)
      IC9 = 8
140

```

C
 C
 C
 MID-SIDE NODES

```

DO 150 IZ=1,2
  IT = I2*(IZ-1)+2
  DO 150 IIC2=II,IT,4
    IC4 = IC2+2
    IC9 = IC9+1
    IX = IC9-8
    XI = CORDX(IN)
    YI = CORDY(IN)
    ZI = CORDZ(IN)
    CUL(IC2) = FQM(XI,ETA,ZETA,ZI,YI)
    CUL(IC4) = FQM(ETA,ZETA,XI,YI,-ZI)
    CUL(IC9) = FQM(ZETA,XI,ETA,XI,YI)
    DCOL(IC2,1) = DFQMX(XI,ETA,ZETA,ZI,YI)
    DCOL(IC2,2) = DFQMY(XI,ZETA,ZI,YI)

```


SHPO00790
SHPO00800
SHPO00810
SHPO00820
SHPO00830
SHPO00840
SHPO00850
SHPO00860

SHPO00870
SHPO00880
SHPO00890
SHPO00900
SHPO00910
SHPO00920
SHPO00930
SHPO00940
SHPO00950
SHPO00960
SHPO00970
SHPO00980
SHPO00990
SHPO01000

SHPO1010
SHPO1020
SHPO1030
SHPO1040
SHPO1050
SHPO1060
SHPO1070
SHPO1080
SHPO1090
SHPO1100
SHPO1110
SHPO1120
SHPO1130
SHPO1140
SHPO1150
SHPO1160
SHPO1170
SHPO1180
SHPO1190

DCOL(IC2,3) = DFQMY(XI,ETA,ZETA,XI,ZI,YI)
DCOL(IC4,1) = DFQMY(ETA,ZETA,-ZI,YI)
DCOL(IC4,2) = DFQMX(ETA,XI,ZETA,-ZI,YI)
DCOL(IC4,3) = DFQMY(ETA,XI,YI,-ZI)
DCOL(IC9,1) = DFQMY(ZETA,ETA,XI,YI)
DCOL(IC9,2) = DFQMY(ZETA,XI,YI,XI)
DCOL(IC9,3) = DFQMX(ZETA,XI,ETA,XI,YI)
150 GU TO 190

C
C
C
C

CUBIC FUNCTIONS
CLNR NUDES

160 DO 170 IZ=1,2
II = 20*(IZ-1)+1
IT = II+9
IX = 0
DO 170 ICOL=II,IT,3
IX+1
IX = CORUX(IX)
YI = CORDY(IX)
ZI = CORDZ(IZ)
CCL(ICOL) = FCC(XI,ETA,ZETA,XI,YI,ZI)
DCOL(ICOL,1) = DFCC(XI,ETA,ZETA,XI,YI,ZI)
DCOL(ICOL,2) = DFCC(ETA,ZETA,XI,YI,ZI,XI)
DCOL(ICOL,3) = DFCC(ZETA,XI,ETA,ZI,XI,YI)
170 I13 = 12

C
C
C

MID-SIDE NUDES

DO 180 IZ=1,2
ID = 9-2*IZ
IX = 0
DO 180 II=1,2
IS = 20*II-19+IZ
IT = 20*II-10-IZ
DO 180 ICOL=IS,IT,ID
I13 = I13+1
I5 = ICOL+3
IX = IX+1
YI = CORDY(IX)
ZI = CORDZ(IX)
THIRD = TRD(IZ)
COL(ICOL) = FCM(XI,ETA,ZETA,THIRD,ZI,YI)
COL(I5) = FCM(ETA,ZETA,XI,THIRD,YI,-ZI)
CCL(I13) = FCM(ZETA,XI,ETA,THIRD,XI,YI)
DCOL(ICOL,1) = DFQMX(XI,ETA,ZETA,THIRD,ZI,YI)
DCOL(ICOL,2) = DFQMY(XI,ZETA,THIRD,ZI,YI)



```

180 DCOL(I COL,3) = DFCMY(XI,ETA,THIRD,Y1,Z1)
190 DCOL(I5,1) = DFCMY(ETA,ZETA,XI,THIRD,-Z1,Y1)
200 DCOL(I5,2) = DFCMX(ETA,XI,THIRD,Y1,-Z1)
210 DCOL(I13,1) = DFCMY(ETA,ETA,THIRD,X1,Y1)
220 DCOL(I13,2) = DFCMY(ZETA,XI,THIRD,Y1,X1)
230 DCOL(I13,3) = DFCMX(ZETA,XI,ETA,THIRD,X1,Y1)
240 RETURN
250 END

```

081
081

SUBROUTINE JACOB (NPEL,AJ,DCOL,CORD,AJI,DTJ)
JC800010

THIS SUBROUTINE WILL FORM THE JACOBIAN AND ITS DETERMINANT

```

IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION AJ(3,3), AJI(3,3), CORD(32,3), DCOL(32,3)

```

FORM JACOBIAN

```
DO 110 I=1,3  
DO 110 J=1,3  
DO 110 K=1,NPEL  
AJ(I,J)=AJ(I,J)+DCOL(K,I)*CORD(K,J)  
110 AJ(I,J)  
JC800040  
JC800050  
JC800060  
JC800070  
JC800080
```

CCMPUTE DETERMINANT

[illegible]

FOR USE WITH B.C. CALCULATIONS

ENTRY JACOB(NPEL,AJ,D COL,CORD,DTJ,KS)
DO 120 I=1,3



C

C THIS SUBROUTINE FORMS C=A*B

```

      IMPLICIT REAL*8(A-H,O-Z)
      DIMENSION A(278,40), B(278), C(278)
      DO 120 I=1,N
      IS1 = I-M-1
      C(I) = B(I)*A(I,1)
      DO 120 J=2,M
      IROW = IS1+J
      IF (IROW.LE.0) GO TO 110
      ICOL = M-J+2
      C(I) = C(I)+A(IROW,ICOL)*B(IROW)
      110 IR = I+J-1
      IF (IR.GT.N) GO TO 120
      C(I) = C(I)+A(I,J)*B(IR)
      CCNTINUE
      120 RETURN
      END

```

C

```

C *****
C *****
C *****

```

SUBROUTINE LDLT (A,N,M,ASN)

LDI00010

THIS SUBROUTINE DECOMPOSES THE COEFFICIENT MATRIX, CODED BY GILLES
CANTIN, NAVAL POSTGRADUATE SCHOOL, MAY, 1972

C

C

C

```

      IMPLICIT REAL*8(A-H,O-Z)
      DIMENSION A(278,40)
      AVSN = ASN*ASN
      NM = N-1
      DO 130 I=1,NM
      DIAG = A(I,1)
      DO 110 J=2,M
      IF (DABS(A(I,J)).LE.AVSN) A(I,J)=0.0D0
      A(I,J) = A(I,J)/DIAG
      CCNTINUE
      110 DO 130 J=2,M
      L = I+J-1
      IF (L.GT.N) GO TO 130
      IF (DABS(A(I,J)).LE.AVSN) GO TO 130
      AA = A(I,J)*DIAG
      DO 120 K=J,M
      ML = 1+K-J
      A(L,ML) = A(L,ML)-AA*A(I,K)
      120 CCNTINUE
      130 RETURN

```

C

```

MUL00020
MUL00030
MUL00040
MUL00050
MUL00060
MUL00070
MUL00080
MUL00090
MUL00100
MUL00110
MUL00120
MUL00130
MUL00140
MUL00150
MUL00160
MUL00170

```

```

LDI00020
LDI00030
LDI00040
LDI00050
LDI00060
LDI00070
LDI00080
LDI00090
LDI00100
LDI00110
LDI00120
LDI00130
LDI00140
LDI00150
LDI00160
LDI00170
LDI00180
LDI00190
LDI00200
LDI00210

```


C

THIS ENTRY POINT PERFORMS FORWARD SUBSTITUTION

ENTRY SLV(Z,B,N,M)
DIMENSION Z(278,40), B(278)

NM = N-1

DO 140 I=1,NM

BI = B(I)

DO 140 J=2,M

L = I+J-1

IF (L.GT.N) GO TO 140

B(L) = B(L)-Z(I,J)*BI

140 CONTINUE

DO 150 I=1,N

150 B(I) = B(I)/Z(I,1)

DO 160 L=2,N

IR = N-L+1

BIRP = B(IR+1)

DO 160 J=2,M

IRO = IR-J+2

IF (IRO.LE.0) GO TO 160

B(IRO) = B(IRO)-Z(IRO,J)*BIRP

160 CONTINUE

RETURN

END

C

SUBROUTINE MARK (C,K,M)

MRK00010

THIS SUBROUTINE CONVERTS CYLINDRICAL OR SPHERICAL COORDINATES TO
RECTANGULAR COORDINATES.

IMPLICIT REAL*8(A-H,O-Z)

DIMENSION C(278,40)

PI = 3.141592653589793D0

IF (K.NE.2) GO TO 120

DO 110 I=1,M

ANG = C(I,2)*PI/1.8D2

R = C(I,1)

C(I,1) = R*DCOS(ANG)

C(I,2) = R*DSIN(ANG)

WRITE (6,160)

GO TO 140

DO 130 I=1,M

ANG2 = C(I,3)*PI/1.8D2

ANGX = C(I,2)*PI/1.8D2

R = C(I,1)

110

120

MRK000020
MRK000030
MRK000040
MRK000050
MRK000060
MRK000070
MRK000080
MRK000090
MRK000100
MRK000110
MRK000120
MRK000130
MRK000140
MRK000150
MRK000160


```

      SIN = DSIN(ANGZ)
      C(I,1) = R*SIN*DCOS(ANGX)
      C(I,2) = R*SIN*DSIN(ANGX)
      C(I,3) = R*DCOS(ANGZ)
130 WRITE (6,I70)
140 DO I=1,M
150 WRITE (6,I80) J,(C(J,L),L=1,3)
      RETURN
160 FORMAT ('D',5X,' COORDINATES GIVEN IN CYLINDRICAL SYSTEM OF REFERENCE NUMBER',I,5X,', NODE NUMBER',I,5X,', FOLLOW',///,6X,' RECTANGULAR COORDINATES FOLLOW',///,5X,', NODE NUMBER',I,5X,', FOLLOW',///,5X,', Z COORDINATE',///)
170 FORMAT ('O',5X,' COORDINATES GIVEN IN SPHERICAL SYSTEM OF REFERENCE NUMBER',I,5X,', NODE NUMBER',I,5X,', FOLLOW',///,6X,' RECTANGULAR COORDINATES FOLLOW',///,5X,', NODE NUMBER',I,5X,', FOLLOW',///,5X,', Y COORDINATE',///,5X,', Z COORDINATE',///)
180 FORMAT (10X,I5,5X,I'P3G24.16')
      END

```

[illegible]

PUN00010

THIS SUBROUTINE PUNCHES REQUESTED OUTPUT

SUBROUTINE PUNC(A, WHEN)

```

1 IMPLICIT REAL*8(A-H,O-Z)
2 COMMON /MTX/ CORD(32,3), ENN(32,32), DCOL(32,3), COL(32), AJ(3,3), AJIPUN00030
3 1(3,3), ENW(32,32), FE(32), PROPP(278,4), COARD(278,3), FI(278), PHI(278), PPUN00040
4 2HIDOT(278), PHIMWK(278), PHIDDT(278), BGH(278,40), BGC(278,40), PROB(278,40), PROB(278,40), PPUN00050
5 38,3), PUNCH(20)
6 COMMON /FLP/ DTJ, AKX, AKY, AKZ, FACT, PHIREF, APZRO, ABIGN, TIM, DT, ENDT, PPUN00060
7 COMMON /INT/ IGO, NBAND, NNODE, NMAT, NEL, NPCL, NGP, NELBC, NNBC, IPROB, KC, PPUN00070
8 1, NTSC, IPUN, IPRT, ISTP
9 DIMENSION A(278)
10 DATA IFLAG/1/
11 IF (IPROB.EQ.1) GO TO 120
12 TEST = PUNCH(IFLAG)-WHEN
13 IF (DABS(TEST).LE.1.0D-06) GO TO 110
14 RETURN
15 WRITE (7,140) WHEN
16 DO 130 I=1, NNODE
17 AP = A(I)+PHIREF
18 WRITE (7,150) I, AP
19 IFLAG = IFLAG+1
20 RETURN
21 FORMAT (F10.5)
22 FORMAT (I5,F9.3)
23 END

```


APPENDIX G

COMPUTER LISTING FOR STRESS PROGRAM

```

*****
THIS PROGRAM PERFORMS A STRESS AND STRAIN ANALYSIS OF THREE
DIMENSIONAL OBJECTS USING NUMERICALLY INTEGRATED ISOPARAMETRIC
FINITE ELEMENTS. THIS PARTICULAR VERSION USES TWENTY (20) NODAL
POINT ELEMENTS ALLOWING TRIQUADRATIC VARIATION OF THE DISPLACEMENT
FUNCTIONS THROUGHOUT THE ELEMENTS.
PROFESSOR GILLES CANTIN, DECEMBER 1970, REVISED IN 1971 BY
LCDR. LEONIDAS, NAVAL POSTGRADUATE SCHOOL, UPDATED IN 1975 BY
LT. BUBECK, NAVAL POSTGRADUATE SCHOOL, TO INCLUDE TRANSIENT THERMAL
STRESSES.
*****
INPUT CARDS NEEDED
*****
INITIALIZATION CARD (1A8,15) WITH THE WORD START IN THE FIRST 8
COLUMNS, PLUS AN INTEGER SPECIFYING THE NUMBER OF SETS OF LOAD
CONDITIONS( MAX OF 10)
*****
PROBLEM IDENTIFICATION CARD (10A8) , ANY TITLE CAN BE USED
*****
PARAMETER CARD (16I15), ONE CARD PER PROBLEM
*****
COL. VARIABLE TOTAL NUMBER OF ELEMENTS
1-5 NEL TOTAL NUMBER OF NODAL POINTS
6-10 NDPT TOTAL SIZE USED IN THE EQUATION SOLVER
11-15 NS BLOCK SIZE OF NODES PER ELEMENT
16-20 NPEL NUMBER OF MATERIALS (MAX.=10)
21-25 NMAT IF NODAL PROPERTIES, NMAT=-1
26-30 NCLD TOTAL NUMBER OF NODES WITH CONCENTRATED LOADS
31-35 NPBC TOTAL NUMBER OF NODES WITH BOUNDARY CONDITIONS
36-40 NPBG2 TOTAL NUMBER OF NODES WITH PREDEFINED DISPLACEMENTS
41-45 NLINE NUMBER OF LINES WITH OUTPUT PAGE DESIRED FOR THE
TABLE OF JOINT STRESSES AND STRAINS
46-50 KEL IF THIS IS 0 OR BLANK ELEMENT STRESSES AND
51-55 NTL0 THERMAL FORCE PRESENT
56-60 NBLD BODY FORCE PRESENT
61-65 NSLD INITIAL FORCE PRESENT
66-70 NKEC7 INITIAL OF RECORDS ON UNIT SEVEN(4/6 FOR QUADS/CUBICS)
71-75 NGPS NUMBER OF GAUSS POINTS FOR STIFFNESS INTEGRATION
76-80 NGPL IF NGPS=0, A DATA CHECK IS PERFORMED.
NUMBER OF GAUSS POINTS FOR LOAD GENERATION
STRAINS WILL NOT BE PRINTED, IF THIS IS 1 A PRINTING
RESULTS. THIS PRINTING CAN BE VOLUMINOUS.
*****

```

CC


```

CNIPO420
CNIPO430
CNIPO440
CNIPO450
CNIPO460
CNIPO470
CNIPO480
CNIPO490
CNIPO500
CNIPO510
CNIPO520
CNIPO530
CNIPO540
CNIPO550
CNIPO560
CNIPO570
CNIPO580
CNIPO590
CNIPO600
CNIPO610
CNIPO620
CNIPO630
CNIPO640
CNIPO650
CNIPO660
CNIPO670
CNIPO680
CNIPO690
CNIPO700
CNIPO710
CNIPO720
CNIPO730
CNIPO740
CNIPO750
CNIPO760
CNIPO770
CNIPO780
CNIPO790
CNIPO800
CNIPO810
CNIPO820
CNIPO830
CNIPO840
CNIPO850
CNIPO860
CNIPO870
CNIPO880
CNIPO890

* * * * * N-O-T-A B-E-N-E
* * * * *
FOR THE REST OF THE INPUT CARDS, OBJECT TIME FORMATS ARE USED
THROUGHOUT, UNFAMILIAR USERS SHOULD LEARN ABOUT THIS FEATURE FROM
ANY GOOD STANDARD BOOK ON FORTRAN 4
* * * * *
***ELEMENT CARD DECK
  TITLE CARD (10A8) ANY TITLE IN COLUMNS 2-80
  FORMAT CARD (10A8) FORMAT SELECTED FOR THE CONNECTIVITY CARDS
                      THE SAME FORMAT WILL BE USED FOR THE ECHO
                      CHECK. (SAME FOR ALL SUBSEQUENT DECKS)
                      THE FORMAT MUST ACCOMMODATE 20 OR 21
                      INTEGERS AS DESCRIBED NEXT
    CONNECTIVITY CARDS (NEL) PER DECK, CONTAINING THE STANDARD
    SEQUENCE OF TWENTY NODES PER ELEMENT FOLLOWED BY A MATERIAL
    IDENTIFICATION NUMBER IF NEEDED
***JOINT CARD DECK
  TITLE CARD (10A8) ANY TITLE IN COLUMNS 2-80
  FORMAT CARD (10A8) FORMAT SELECTED FOR THE JOINT CARDS
                      (1 INTEGER, 3 FL. PT, 1 INTEGER)
    COORDINATE CARDS (NDPT) PER DECK CONTAINING FIRST AN INTEGER
    IDENTIFICATION NUMBER FOR THE JOINT FOLLOWED BY EITHER THREE
    CARTESIAN COORDINATES (X,Y,Z) OR THREE CYLINDRICAL COORDINATES
    (R,PHI,Z) FOLLOWED BY AN INTEGER DIFFERENT FROM ZERO IF
    CYLINDRICAL COORDINATES ARE USED
***MATERIAL PROPERTY TABLE
  IF NUDAL MATERIAL PROPERTIES(NMAT=-1) ARE USED REFER TO (2)
  (1) TITLE CARD (10A8) ANY TITLE IN COLUMNS 2-80
      FORMAT CARD (10A8) FORMAT SELECTED FOR THE MATERIAL PROPERTIES
                      (1 INTEGER, 3 FL. POINT NUMBERS)
    MATERIAL PROPERTY (NMAT) CARDS, MUST BE LESS THAN 10
    CONTAINING AN INTEGER IDENTIFICATION NUMBER FOR THE MATERIAL
C

```



FOLLOWED BY THE VALUES OF YOUNG'S MODULUS, POISSON'S RATIO
AND COEFFICIENT OF THERMAL EXPANSION

(2) TITLE CARD (10A8) ANY TITLE IN COLUMNS 2-80

FORMAT CARD (10A8) FORMAT SELECTED FOR NODAL MATERIAL
PROPERTIES(2 INTEGERS, 3 FL. POINT NUMBERS)

EACH NODE MUST HAVE PROPERTIES ASSIGNED TO IT. IN ORDER TO
REDUCE THE NUMBER OF INPUT CARDS, IF ALL NODES(CONSECUTIVE)
BETWEEN NODE "N" AND NODE "M" ARE THE SAME, THE TWO INTEGERS
SPECIFIED ARE "N" AND "M". IF NODE "N" IS UNIQUE IN ITS
PROPERTIES, THE BOTH INTEGERS ARE "N". THE FLOATING POINT
NUMBERS ARE YOUNG'S MODULUS, POISSON RATIO, AND THE COEFFICIENT
OF THERMAL EXPANSION.

***BOUNDARY CONDITION DECK

TITLE CARD (10A8) ANY TITLE IN COLUMNS 2-80

FORMAT CARD (10A8) FORMAT SELECTED FOR THE BOUNDARY CARDS
(4 INTEGERS)

BOUNDARY CARD (NPBC) PER DECK CONTAINING FIRST AN INTEGER
IDENTIFICATION FOR THE NODE FOLLOWED BY A 0 FOR A FREE NODE
FOLLOWED BY 1 FOR A ZERO DISPLACEMENT CONSTRAINT, FOLLOWED
BY A 2 FOR A PREDEFINED DISPLACEMENT. CARDS MUST BE
SEQUENCED IN ASCENDING ORDER OF NODE NUMBER

***PREDEFINED DISPLACEMENT DECK (OMIT THIS DECK IF NPBC2 IS ZERO)

TITLE CARD (10A8) ANY TITLE IN COLUMNS 2-80

FORMAT CARD (10A8) FORMAT SELECTED FOR THE DISPLACEMENT CARDS
(1 INTEGER, 3 FL. POINT NUMBERS)

SPECIFIED DISPLACEMENTS (NPBC2) PER DECK CONTAINING AN INTEGER
IDENTIFICATION NUMBER FOR THE NODE FOLLOWED BY THE SPECIFIED
DISPLACEMENT COMPONENTS IN THE GLOBAL COORDINATE SYSTEM (X,Y,Z)

***CONCENTRATED LOAD CARD DECK(OMIT IF NCLED=0)

TITLE CARD (10A8) ANY TITLE IN COLUMNS 2-80

FORMAT CARD (10A8) FORMAT SELECTED FOR THE LOAD CARDS
(1 INTEGER, 3 FL. POINT NUMBERS)

LOAD CARDS (NCLED) CARDS PER DECK, CONTAINING AN INTEGER

CNTP0900
CNTP0910
CNTP0920
CNTP0930
CNTP0940
CNTP0950
CNTP0960
CNTP0970
CNTP0980
CNTP0990
CNTP1000
CNTP1010
CNTP1020
CNTP1030
CNTP1040
CNTP1050
CNTP1060
CNTP1070
CNTP1080
CNTP1090
CNTP1100
CNTP1110
CNTP1120
CNTP1130
CNTP1140
CNTP1150
CNTP1160
CNTP1170
CNTP1180
CNTP1190
CNTP1200
CNTP1210
CNTP1220
CNTP1230
CNTP1240
CNTP1250
CNTP1260
CNTP1270
CNTP1280
CNTP1290
CNTP1300
CNTP1310
CNTP1320
CNTP1330
CNTP1340
CNTP1350
CNTP1360
CNTP1370

IDENTIFICATION FOLLOWED BY THE VALUES OF THE COMPONENTS OF THE
JOINT LOAD IN THE GLOBAL SYSTEM OF REFERENCE (X,Y,Z)
LOAD CARDS MUST BE IN ASCENDING ORDER OF THE JOINT NUMBER

***NLOAD SETS OF LOAD CONDITIONS ARE READ AS DESCRIBED.

***THERMAL LOAD CONDITIONS(OMIT IF NTLD=0)

TITLE CARD(10A8) ANY TITLE INCOLUMNS 2-80

FORMAT CARD(10A8) FORMAT SELECTED FOR NODAL TEMPERATURES
(1 INTEGER, 1 FL. POINT NUMBER)

SPECIFY NODAL NUMBER AND TEMPERATURE, SEQUENCED IN ORDER OF
ASCENDING NODAL NUMBER; NNODE CARDS REQUIRED.

IF NTLD=-1, TEMPERATURES ARE READ FROM FILE 33.

***BODY FORCE LOAD CONDITIONS(OMIT IF NBLD=0)

TITLE CARD(10A8) ANY TITLE IN COLUMNS 2-80

FORMAT CARD(10A8) FORMAT SELECTED FOR BODY FORCES
(1 INTEGER, 3 FL. POINT NUMBERS)

SPECIFY NODAL NUMBER AND X,Y,Z COMPONENTS OF BODY FORCE,
SEQUENCED IN ORDER OF ASCENDING NODAL NUMBERS, NNODE CARDS.

IF NBLD=-1, BODY FORCES ARE READ FROM FILE 34.

***INITIAL STRETCH LOAD CONDITIONS(OMIT IF NSLD=0)

TITLE CARD(10A8) ANY TITLE IN COLUMNS 2-80

FORMAT CARD(10A8) FORMAT SELECTED FOR INITIAL STRETCH CONDITION
(1 INTEGER, 3 FL. POINT NUMBERS)

SPECIFY NODAL NUMBER AND X,Y,Z COMPONENTS OF INITIAL STRETCH
SEQUENCED IN ORDER OF ASCENDING NODAL NUMBER, NNODE CARDS

IF NSLD=-1, STRETCH CONDITIONS ARE READ FROM FILE 35.

IF ANOTHER PROBLEM FOLLOWS PLACE COMPLETE DECK WITH INITIAL
"START" CARD. IF END OF RUN PLACE A CARD WITH "STOP" IN FIRST
8 COLUMNS.



CC

INPUT ALL DATA

```

CALL SETIME
CLOCK = ITIME(0)*0.01
CALL INPUT
CALL GETIME (IET)
CPUTM = IET*0.000026
WRITE (6,440)
WRITE (6,430)
CLOCK = ITIME(0)*0.01-CLOCK
WRITE (6,520) CLOCK,CPUTM
IF (NGPS.EQ.0) RETURN

```

CCC

FORM ELEMENT NODAL ARRAYS BY CALLING SORT

```

CALL SETIME
CLOCK = ITIME(0)*0.01
CALL SORT
CALL GETIME (IET)
CPUTM = IET*0.000026
WRITE (6,450)
CLOCK = ITIME(0)*0.01-CLOCK
WRITE (6,520) CLOCK,CPUTM

```

CCC

FORM STIFFNESS MATRICES

```

CALL SETIME
CLOCK = ITIME(0)*0.01
CALL ESTF
CALL GETIME (IET)
CPUTM = IET*0.000026
WRITE (6,460)
CLOCK = ITIME(0)*0.01-CLOCK
WRITE (6,520) CLOCK,CPUTM

```

CCC

FORM TOTAL STIFFNESS MATRIX IN BLOCKS

```

CALL SETIME
CLOCK = ITIME(0)*0.01
CALL MERGE
CALL GETIME (IET)
CPUTM = IET*0.000026
WRITE (6,470)
CLOCK = ITIME(0)*0.01-CLOCK
WRITE (6,520) CLOCK,CPUTM
IF (IABS(NTLD)+IABS(NBLD)+IABS(NSLD).EQ.0) GO TO 10

```

CC

FORM LOAD VECTOR BY LOAD GENERATION

SMA00180
SMA00190
SMA00200
SMA00210
SMA00220
SMA00230
SMA00240
SMA00250
SMA00260
SMA00270

SMA00280
SMA00290
SMA00300
SMA00310
SMA00320
SMA00330
SMA00340
SMA00350

SMA00360
SMA00370
SMA00380
SMA00390
SMA00400
SMA00410
SMA00420
SMA00430

SMA00440
SMA00450
SMA00460
SMA00470
SMA00480
SMA00490
SMA00500
SMA00510
SMA00520



C

```

CALL SETIME
CLOCK = ITIME(0)*0.01
CALL LODGEN (IET)
CALL GETIME (IET)
CPUTM = IET*0.000026
WRITE (6,440)
WRITE (6,480)
CLOCK = ITIME(0)*0.01-CLOCK
WRITE (6,520) CLOCK,CPUTM

```

C C

```

FORM TOTAL LOAD VECTORS

```

10

```

CALL SETIME
CLOCK = ITIME(0)*0.01
CALL DLOAD
CALL GETIME (IET)
CPUTM = IET*0.000026
WRITE (6,490)
CLOCK = ITIME(0)*0.01-CLOCK
WRITE (6,520) CLOCK,CPUTM
IF (NPBC2.NE.0) GO TO 20

```

C C

```

APPLY ANY BOUNDARY CONDITIONS

```

20

```

CALL SETIME
CLOCK = ITIME(0)*0.01
CALL BCOND
CALL GETIME (IET)
CPUTM = IET*0.000026
WRITE (6,500)
CLOCK = ITIME(0)*0.01-CLOCK
WRITE (6,520) CLOCK,CPUTM
GO TO 30

```

20

```

CALL SETIME
CLOCK = ITIME(0)*0.01
CALL BCOND2
CALL GETIME (IET)
CPUTM = IET*0.000026
WRITE (6,510)
CLOCK = ITIME(0)*0.01-CLOCK
WRITE (6,520) CLOCK,CPUTM

```

C C

```

SOLVE SYSTEM OF EQUATIONS FOR DISPLACEMENTS

```

30

```

CALL SETIME
CLOCK = ITIME(0)*0.01
CALL SOLVE

```

SMA00530
SMA00540
SMA00550
SMA00560
SMA00570
SMA00580
SMA00590
SMA00600
SMA00610

SMA00620
SMA00630
SMA00640
SMA00650
SMA00660
SMA00670
SMA00680
SMA00690
SMA00700

SMA00710
SMA00720
SMA00730
SMA00740
SMA00750
SMA00760
SMA00770
SMA00780
SMA00790
SMA00800
SMA00810
SMA00820
SMA00830
SMA00840
SMA00850
SMA00860
SMA00870

SMA00880
SMA00890
SMA00900



```
CALL GETIME (IET)
CCPUTM = IET*0.000026
WRITE (6,440)
WRITE (6,430)
CLOCK = ITIME (0)*0.000026
IF (TLD.EQ.-1) REMIN=
IF (NILD.NE.0) REMIN=
```

LOOP ON DIFFERENT LOADS TO COMPUTE ALL STRESSES AND STRAINS

```

DC 370 IL=1,NLOAD
IF (NTLD.EQ.-1) READ (30) TIME
WRITE (6,590) IL
IF (NTLD.EQ.-1) WRITE (6,380) TIME
IF (NTLD.EQ.0) GO TO 60
NL1 = 1
NL2 = NLINE
WRITE (6,390)
DO 50 I=NL1,NL2
NTRK(I,1) = NTRK(I,1)
CALL RDISKI(NTRK)
WRITE (6,600) I,THNP
IF (NL2.EQ.NDPT) GO TO 60
NL1 = NL2+1
NL2 = NL2+NLINE
IF (NL2.GT.NDPT) NL2=NDPT
WRITE (6,620)
GO TO 40
IF (NBLD.EQ.0) GO TO 90
WRITE (6,620)
NL1 = 1
NL2 = NLINE
WRITE (6,400)
DO 80 I=NL1,NL2
NTRK(I,1) = NTRK(I,1)
CALL RDISKI(NTRK)
WRITE (6,610) I,BYNP
IF (NL2.EQ.NDPT) GO TO 90
NL1 = NL2+1
NL2 = NL2+NLINE
IF (NL2.GT.NDPT) NL2=NDPT
WRITE (6,620)
GO TO 70
IF (IABS(NTLD)+IABS(NBLD)+IABS(NSL
TLOAD1 = 0.000
TLOAD2 = 0.000
TLOAD3 = 0.000

```



```

100 NL1 = NLINE
    NL2 = (6,410) IL
    WRITE (6,410) IL
110 DO I=1,NL1,NL2
    NTK = NTRK(IL,I)
    CALL RDISLFF(NTK)
    WRITE (6,610) I,FLOAD
    TLOAD1 = TLOAD1+FLOAD(1)
    TLOAD2 = TLOAD2+FLOAD(2)
    TLOAD3 = TLOAD3+FLOAD(3)
    IF (NL2.EQ.NDPT) GO TO 120
    NL1 = NL2+1
    NL2 = NL2+NLINE
    IF (NL2.GT.NDPT) NL2=NDPT
    GO TO 100
120 IF (NCLD.EQ.0) GO TO 140
    REWIND 14
    DO I=1,NCLD
    READ (14) CLOAD
    TLOAD1 = TLOAD1+CLOAD(1)
    TLOAD2 = TLOAD2+CLOAD(2)
    TLOAD3 = TLOAD3+CLOAD(3)
130 TLOAD1,TLOAD2,TLOAD3
140 WRITE (6,420) TLOAD1,TLOAD2,TLOAD3

```

uuu

FORM NOCAL DISPLACEMENTS AND REACTIONS

```

150 CALL SETIME
    CLOCK = ITIME(0)*0.01
    CALL DISP (IL)
    CALL GETIME (IET)
    CPUTM = IET*0.000026
    WRITE (6,440)
    WRITE (6,540)
    CLOCK = ITIME(0)*0.01-CLOCK
    WRITE (6,520) CLOCK,CPUTM
    CALL SETIME
    CLOCK = ITIME(0)*0.01
    REWIND 9
    REWIND 19
DO 160 I=1,7
    SSNN1(I) = 0.000
    SSS1(I) = 0.000
DO 170 I=1,NDPT
    IS = (IL-1)*NDPT+I
    CALL WDISKS (IS)
    CALL WDISKN (I)
    IF (NMAT.EQ.1) GO TO 200
    IF (NMAT.EQ.-1) GO TO 220

```



```

NMAT1 = 0
REWIND 16
DO 190 I=1, NEL
  READ (19) COREL
  READ (16) NCON21
  IF (NTLD.NE.O) READ (24) TEEL
  IF (NMAT1.EQ.NCON21) GO TO 180
NMAT1 = NCON21
ALPHA = ELDAT(NCON21,3)
CALL ELPROP (NMAT1)
CALL STRESS (I,IL)
180 CCNTINUE
190 GO TO 240
200 ALPHA = ELDAT(1,3)
DO 210 I=1, NEL
  IF (NTLD.NE.O) READ (24) TEEL
  READ (19) COREL
  CALL STRESS (I,IL)
210 GO TO 240
220 REWIND 23
DO 230 I=1, NEL
  READ (19) COREL
  READ (23) ELDATE
  IF (NTLD.NE.O) READ (24) TEEL
  CALL STRESS (I,IL)
230 NPAGE = NDPT/NLINE
240 NL1 = NPAGE*NLINE
NL1 = NL1+1
NLAST = NDPT-NL
IF (NPAGE.EQ.O) GO TO 270
DO 260 I=1, NPAGE
  WRITE (6,580)
  I1 = (I-1)*NLINE+1
  I2 = I1+NLINE-1
  DO 260 J=I1, I2
    JS = (I1-1)*NDPT+J
  CALL RDISKS (JS)
  DO 250 K=1, 6
    SSSS1(K) = SSSS1(K)/SSSS1(7)
250 SSSS1(K) = SSSS1(K)/SSSS1(7)
  CALL PRNSTRT (SSSS1,S1,S2,S3)
260 WRITE (6,570) J, SSSS1,S1,S2,S3
  IF (NLAST.EQ.O) GO TO 300
270 WRITE (6,580)
  DO 290 I=NL1,NDPT
    IS = (I-1)*NDPT+1
  CALL RDISKS (IS)
  DO 280 K=1, 6
    SSSS1(K) = SSSS1(K)/SSSS1(7)
280 SSSS1(K) = SSSS1(K)/SSSS1(7)

```

SMA01810
 SMA01820
 SMA01830
 SMA01840
 SMA01850
 SMA01860
 SMA01870
 SMA01880
 SMA01890
 SMA01900
 SMA01910
 SMA01920
 SMA01930
 SMA01940
 SMA01950
 SMA01960
 SMA01970
 SMA01980
 SMA01990
 SMA02000
 SMA02010
 SMA02020
 SMA02030
 SMA02040
 SMA02050
 SMA02060
 SMA02070
 SMA02080
 SMA02090
 SMA02100
 SMA02110
 SMA02120
 SMA02130
 SMA02140
 SMA02150
 SMA02160
 SMA02170
 SMA02180
 SMA02190
 SMA02200
 SMA02210
 SMA02220
 SMA02230
 SMA02240
 SMA02250
 SMA02260
 SMA02270
 SMA02280




```

290 CALL PRNSIR (SSSS1,S1,S2,S3)
300 WRITE (6,570) I,SSSS,S1,S2,S3
CONTINUE
IF (NPAGE.EQ.0) GO TO 330
DO 320 I=1,NPAGE
WRITE (6,560)
I1 = (I-1)*NLINE+1
I2 = I1+NLINE-1
DO 320 J=I1,I2
CALL RDISKN (J)
DO 310 K=1,6
SSNN1(K) = SSNN1(K)/SSNN1(7)
310 SSNN1(K) = SSNN1(K)/SSNN1(7)
320 WRITE (6,570) J,SSNN
IF (NLAST.EQ.0) GO TO 360
330 WRITE (6,560)
DO 350 I=NLI,NDPT
CALL RDISKN (I)
DO 340 K=1,6
SSNN1(K) = SSNN1(K)/SSNN1(7)
340 SSNN1(K) = SSNN1(K)/SSNN1(7)
350 WRITE (6,570) I,SSNN
360 CONTINUE
CALL GETIME (IET)
CPUTM = IET*0.000026
WRITE (6,440)
WRITE (6,550)
CLOCK = ITIME(0)*0.01-CLOCK
WRITE (6,520) CLOCK,CPUTM
370 CONTINUE
RETURN
380 FORMAT ('0',14X,' TIME OF LOAD VECTOR = ',F10.5)
390 FORMAT ('0',14X,' NODAL TEMPERATURES',//,14X,'NODE',6X,'TEMPERATUR
1E,/)
400 FORMAT ('0',14X,' NODAL BODY FORCES',//,14X,'NODE',6X,'BODY FORCE')
410 FORMAT ('1',25X,' CONSISTENT LOAD VECTOR NO.',15,/,',',25X,'X',25
1X,'Y',25X,'Z',/)
420 FORMAT ('0',TOTAL LOADS ARE ',IP3D25.16)
430 FORMAT (5X,'***** INPUT *****')
440 FORMAT (//)
450 FORMAT (5X,'***** SORT *****')
460 FORMAT (5X,'***** FSTF *****')
470 FORMAT (5X,'***** MERGE *****')
480 FORMAT (5X,'***** LODGEN *****')
490 FORMAT (5X,'***** DLOAD *****')
500 FORMAT (5X,'***** BCOND *****')
510 FORMAT (5X,'***** BCOND2 *****')
520 FORMAT (5X,7H TIME =,1F7.2,8H SECONDS,6H (CPU =,1F7.2,9H SECONDS)//)
530 FORMAT (5X,'***** SOLVE *****')

```




```

540 FORMAT (5X, '***** DISP *****')
550 FORMAT (5X, '***** STRESS *****')
560 FORMAT (1H1, '//, ' A-V-E-R-A-G-E S-T-R-A-I-N-S AT THE JOINTS ',
1 //, 2X, 'JOINT',
2 9X, 'SNX', 12X, 'SNZ', 12X, 'SNXY', 11X, 'SNYZ', 11X, 'SNZX')
570 FORMAT (2X, 14, 2X, 6(1PD15.6), 1X, 3(1PD11.3))
580 FORMAT (1H1, '//, ' A-V-E-R-A-G-E S-T-R-E-S-S-E-S AT THE JOINTS ',
1 //, 2X, 'JOINT',
2 7X, 'SSX', 12X, 'SSZ', 12X, 'SSXY', 11X, 'SSYZ', 11X, 'SSZX', 10X,
3 51, 9X, 'S3', '//)
590 FORMAT (1, 120(*,)), //, 45X, 'SOLUTION FOR LCAD VECTOR NO.', 15, '//,
1 120(*,))
600 FORMAT (1, 116, 8X, F10.5)
610 FORMAT (1, 116, 1P3D25.16)
620 FORMAT (1, 1)
END

```

C *****
C *****
C *****
C *****

INP00010

SUBROUTINE INPUT

THIS SUBROUTINE READS ALL DATA

```

IMPLICIT REAL*8(A-H,O-Z)
COMMON /NB1/ NEL,NDPT,NPEL,NDF,NEQ,NN,MM,NS,NCOUNT,NST,NSTF,NCLD,N
1PBC,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NTLD,NBLD,NSL
20
COMMON /NB2/ NCON(20),NBC(4)
COMMON /GAUSS/ NGPS,NGPL
COMMON /B1/ ELDAT(10,3),CLOAD(4),BCON(3)
COMMON /B12/ ELDATN(3),BYNP(3),THNP,FLOAD(3)
COMMON /B2/ ELAST(6,6),COORD(3),COREL(20,3),UJNT(3),SSSS1(7),SSNN1
1(7),ALPHA
COMMON /CNTRL/ NLOAD
DIMENSION TITLE(10), FMT(10), CLOAD1(3)
DATA STOP/,STOP
EQUIVALENCE (CLOAD(1),CLOAD1(1)), (TITLE(1),COREL(1,1)), (FMT(1),C
1COREL(1,2))
1NTRK(1,J)) = (I-1)*NDPT+J
PI = 3.14159265359
REWIND 9
REWIND 14
REWIND 15
READ (5,290) CHKWD,NLOAD
IF (NLOAD.EQ.0) NLOAD=1
IF (CHKWD.EQ.STOP) STOP
READ (5,300) TITLE

```



```

WRITE (6,300) TITLE
READ GENERAL CHARACTERISTICS
READ (5,320) NEL,NDPT,NS,NPEL,NMAT,NCLD,NPBC,NPBC2,NLINE,KEL,NTLD,
1 NBLD,NSLD,NREC7,NGPS,NGPL
WRITE (6,330) NEL,NDPT,NMAT,NS,NPBC,NPBC2,NCLD,NTLD,NBLD,NSLD,NGPS
1,NGPL,NLOAD
NDF = 3
NDFP = 4
NEQ = NDPT*NDF
NCOUNT = NS*NS
NST = NPEL*NDF
NSTF = NST*NST
NSB = 6*NST
NN = (NEQ*NS-1)/NS
READ (5,300) TITLE
READ (5,300) FMT
WRITE (6,310) TITLE
WRITE (6,300)
NBAND = 0

```

INP00260

```

READ CONNECTIVITY

```

```

IF (IABS(NMAT).EQ.1) GO TO 20
REWIND 16
DO 10 I=1,NEL
  READ (5,FMT) NCON,NCON21
  WRITE (6,FMT) NCON,NCON21
  WRITE (9) NCON
  NPELM = NCON-1
  DO 10 J=1,NPELM
    JP = J+1
    DO 10 K=JP,NPEL
      NBAND = MAX0(NBAND,IABS(NCON(J)-NCON(K)))
    GO TO 40
  DO 30 I=1,NEL
    READ (5,FMT) NCON
    WRITE (6,FMT) NCON
    WRITE (9) NCON

```

INP00450
INP00460
INP00470
INP00480
INP00490
INP00500
INP00510
INP00520
INP00530
INP00540
INP00550
INP00560
INP00570
INP00580
INP00590
INP00600
INP00610

```

CCOMPUTE HALF BANDWIDTH

```

```

NPELM = NPEL-1
DO 50 J=1,NPELM
  JP = J+1

```

INP00620
INP00630
INP00640



INP00650
INP00660
INP00670
INP00680
INP00690
INP00700

INP00710
INP00720

INP00730
INP00740
INP00750
INP00760
INP00770
INP00780
INP00790
INP00800
INP00810
INP00820
INP00830
INP00840
INP00850
INP00860
INP00870

INP00880
INP00890
INP00900
INP00910
INP00920
INP00930
INP00940
INP00950
INP00960
INP00970
INP00980
INP00990
INP01000
INP01010
INP01020
INP01030

```

30 DO 30 K=JP,NPEL
40 NBAND = MAXO(NBAND,IABS(NCON(J)-NCON(K)))
  NBAND = (NBAND+1)*NDF
  MM = (NBAND+2*(NS-1))/NS
  IF (NN.LT.MM) MM = NN
  WRITE (6,340) NEQ,NBAND,NN,MM,NCOUNT,NS*F,NREC7

```

CALL 'DISK' TO SET UP VARIABLES

```

CALL DISK
WRITE (6,310)

```

READ COORDINATES AND CONVERT IF NECESSARY

```

READ (5,300) TITLE
WRITE (6,300) TITLE
READ (6,310)
WRITE (5,300) FMT
DO 60 I=1,NDPT
  READ (5,FMT) IEL,COORD,KND
  IF (KND.EQ.0) GO TO 50
  PHI = PI*COORD(2)/180.000
  X = COORD(1)*DCOS(PHI)
  Y = COORD(1)*DSIN(PHI)
  COORD(1) = X
  COORD(2) = Y
  WRITE (6,FMT) IEL,COORD,KND
60 CALL WDISK(I)
  WRITE (6,310)

```

READ MATERIAL PROPERTIES

```

READ (5,300) TITLE
WRITE (6,300) TITLE
READ (6,310)
WRITE (5,300) FMT
IF (NMAT.EQ.-1) GO TO 80
DO 70 I=1,NMAT
  READ (5,FMT) N,ELDAT(N,J),J=1,3)
  WRITE (6,FMT) N,(ELDAT(N,J),J=1,3)
70 WRITE (6,310)
  GO TO 110
80 NNODE2 = 1
  IF (NNODE2.EQ.NDPT) GO TO 110
90 READ (5,FMT) NODE1,NODE2
  DO 100 I=NODE1,NODE2
    ELCATN(1) = YM
    ELDATN(2) = PR

```




```

ELDATN(3) = ALPHA
WRITE (6,370) I,(ELDATN(J),J=1,3)
100 CALL WDISKP (1)
GC TO 90

```

CC

READ ANY BOUNDARY CONDITIONS

```

110 READ (5,300) TITLE
WRITE (6,310)
WRITE (6,300) TITLE
WRITE (6,310)
READ (5,300) FMT
READ (5,FMT) NBC
WRITE (6,FMT) NBC
NBCCL = NBC(1)
NBC(1) = NDF*NBC(1)-NDF
WRITE (15) NBC
NPBCL = NPBCL-1
DO 120 I=2,NPBCL
READ (5,FMT) NBC
WRITE (6,FMT) NBC
NBC(1) = NDF*NBC(1)-NDF
120 WRITE (15) NBC
READ (5,FMT) NBC
WRITE (6,FMT) NBC
NBCCL = NBC(1)
NBC(1) = NDF*NBC(1)-NDF
WRITE (15) NBC
IF (NPBCL.EQ.0) GO TO 140
READ (5,300) TITLE
WRITE (6,310)
WRITE (6,300) TITLE
READ (5,300) FMT
DO 130 I=1,NPBCL2
READ (5,FMT) NDBC,BCON
WRITE (6,FMT) NDBC,BCON
130 WRITE (22) BCON
140 IF (NCLD.EQ.0) GO TO 160

```

CC

READ ANY CONCENTRATED LOADS

```

READ (5,300) TITLE
WRITE (6,310)
WRITE (6,300) TITLE
WRITE (6,310)
READ (5,300) FMT
DO 150 I=1,NCLD

```

INP01040
INP01050
INP01060
INP01070

INP01080
INP01090
INP01100
INP01110
INP01120
INP01130
INP01140
INP01150
INP01160
INP01170
INP01180
INP01190
INP01200
INP01210
INP01220
INP01230
INP01240
INP01250
INP01260
INP01270
INP01280
INP01290
INP01300
INP01310
INP01320
INP01330
INP01340
INP01350
INP01360
INP01370
INP01380
INP01390

INP01400
INP01410
INP01420
INP01430
INP01440
INP01450



INP01460
INP01470
INP01480
INP01490
INP01500

INP01510
INP01520
INP01530

INP01540
INP01550
INP01560
INP01570
INP01580
INP01590
INP01600
INP01610
INP01620
INP01630
INP01640
INP01650
INP01660
INP01670
INP01680
INP01690
INP01700
INP01710

INP01720
INP01730
INP01740
INP01750
INP01760
INP01770
INP01780
INP01790
INP01800
INP01810
INP01820
INP01830
INP01840

```

READ (5,FMT) NCL,CLOAD1
WRITE (6,FMT) NCL,CLOAD1
ANCL = NCL
CLOAD(NDFF) = ANCL
150 WRITE (14) CLOAD
CC
CC
    LOOP ON VARIOUS LOAD CONDITIONS
160 DO 280 IL=1,NLOAD
    IF (NTLD.EQ.0) GO TO 200
    IF (NTLD.EQ.-1) GO TO 180
CC
CC
    READ TEMPERATURE DISTRIBUTIONS
    READ (5,300) TITLE
    WRITE (6,310) TITLE
    WRITE (6,300) TITLE
    READ (5,300) FMT
    DO 170 I=1,NDPT
    READ (5,FMT) IJT,THNP
    WRITE (6,FMT) IJT,THNP
    NTK = NTRK(IL,I)
    CALL WDISKT (NTK)
170 GO TO 200
180 IF (NGPS.EQ.0) GO TO 200
    READ (33,350) TIME
    WRITE (30) TIME
    DO 190 I=1,NDPT
    READ (33,360) IJT,THNP
    NTK = NTRK(IL,I)
    CALL WDISKT (NTK)
190
CC
CC
    READ ANY BODY FORCES
200 IF (NBLD.EQ.0) GO TO 240
    IF (NBLD.EQ.-1) GO TO 220
    READ (5,300) TITLE
    WRITE (6,310) TITLE
    WRITE (6,300) TITLE
    READ (5,300) FMT
    DO 210 I=1,NDPT
    READ (5,FMT) IJT,BYNP
    WRITE (6,FMT) IJT,BYNP
    CALL WDISKT (1)
210 GO TO 240
220 IF (NGPS.EQ.0) GO TO 240

```



```

READ (34,350) TIME
WRITE (30) TIME
DO 230 I=1,NDPT
  READ (34,370) IJT, BYNP
  NTK = NTRK(IL, I)
  CALL WDISBF (NTK)
230

```

READ ANY DISPLACEMENTS

240	IF (NSLD:EQ:0) GO TO 280	INP01910
	IF (NSLD:EQ:-1) GO TO 260	INP01920
	LEAD (5,300) TITLE	INP01930
	WRITE (6,310)	INP01940
	WRITE (6,300) TITLE	INP01950

```

250
CALL WDISSE (I)
WRITE (5, FMT) IJJ, STNP
DO 250 I=1, NDUPT
READ (5, 300) FMT
END

```

250 CALL WDISF (I)
GO TO 280

260 IF (NGPS.EV.0) GO TO 280

```

270 CALL WDISSE (NIN, I)
      NTK = NTRK(I, I)
      DUA = (35,370)
      WRITE (30,1) NDT
      READ (35,350) TIME
      TIME = TIME + 1
      STNP = STNP + 1
      IF (TIME .EQ. 360) THEN
        WRITE (30,2)
        CALL WDISSE (NIN, I)
        DUA = (35,370)
        NTK = NTRK(I, I)
        TIME = 0
      END IF
    END DO
  END DO
  CALL WDISSE (NIN, I)
  DUA = (35,370)
  NTK = NTRK(I, I)
  TIME = 0
  STNP = 0
  IF (TIME .EQ. 360) THEN
    WRITE (30,2)
    CALL WDISSE (NIN, I)
    DUA = (35,370)
    NTK = NTRK(I, I)
    TIME = 0
  END IF
END DO
CALL WDISSE (NIN, I)
DUA = (35,370)
NTK = NTRK(I, I)
TIME = 0
STNP = 0
IF (TIME .EQ. 360) THEN
  WRITE (30,2)
  CALL WDISSE (NIN, I)
  DUA = (35,370)
  NTK = NTRK(I, I)
  TIME = 0
END IF

```

270 CALL WDISSF (NTRK)
280 CCNT INUE

```

290 FORMAT (1A8,I5)
300 FORMAT (10A8)
310 FORMAT (//)
320 FORMAT (16I5)
330 FORMAT (//,5X,

```

[illegible]






```

188888888888,0.5555555555555555,1.000,1.000,0.3478548451374539,0.652LDG00280
21451548625461,0.6521451548625461,0.3478548451374539,1.000,0.236926LDG00290
38850561891,0.47862867049933665,0.568888888888889,0.47862867049933665,LDG00300
40.2369268850561891/NGPM = NGPL-1LDG00310
NTRK(I,J) = (I-1)*NDPT+JLDG00320
NGPM = NGPL-1LDG00330
DO 110 I=1,NGPLDGM = NGP-1LDG00340
DO 110 J=1,NGPLDGM = NGP-1LDG00350
DO 110 K=1,NGPLDGM = NGP-1LDG00360
DO 110 L=1,NGPLDGM = NGP-1LDG00370
DO 110 M=1,NGPLDGM = NGP-1LDG00380
DO 110 N=1,NGPLDGM = NGP-1LDG00390
DO 110 O=1,NGPLDGM = NGP-1LDG00400
DO 110 P=1,NGPLDGM = NGP-1LDG00410
DO 110 Q=1,NGPLDGM = NGP-1LDG00420
DO 110 R=1,NGPLDGM = NGP-1LDG00430
DO 110 S=1,NGPLDGM = NGP-1LDG00440
DO 110 T=1,NGPLDGM = NGP-1LDG00450
DO 110 U=1,NGPLDGM = NGP-1LDG00460
DO 110 V=1,NGPLDGM = NGP-1LDG00470
DO 110 W=1,NGPLDGM = NGP-1LDG00480
DO 110 X=1,NGPLDGM = NGP-1LDG00490
DO 110 Y=1,NGPLDGM = NGP-1LDG00500
DO 110 Z=1,NGPLDGM = NGP-1LDG00510
DO 110 AA=1,NGPLDGM = NGP-1LDG00520
DO 110 AB=1,NGPLDGM = NGP-1LDG00530
DO 110 AC=1,NGPLDGM = NGP-1LDG00540
DO 110 AD=1,NGPLDGM = NGP-1LDG00550
DO 110 AE=1,NGPLDGM = NGP-1LDG00560
DO 110 AF=1,NGPLDGM = NGP-1LDG00570
DO 110 AG=1,NGPLDGM = NGP-1LDG00580
DO 110 AH=1,NGPLDGM = NGP-1LDG00590
DO 110 AI=1,NGPLDGM = NGP-1LDG00600
DO 110 AJ=1,NGPLDGM = NGP-1LDG00610
DO 110 AK=1,NGPLDGM = NGP-1LDG00620
DO 110 AL=1,NGPLDGM = NGP-1LDG00630
DO 110 AM=1,NGPLDGM = NGP-1LDG00640
DO 110 AN=1,NGPLDGM = NGP-1LDG00650
DO 110 AO=1,NGPLDGM = NGP-1LDG00660
DO 110 AP=1,NGPLDGM = NGP-1LDG00670
DO 110 AQ=1,NGPLDGM = NGP-1LDG00680
DO 110 AR=1,NGPLDGM = NGP-1LDG00690
DO 110 AS=1,NGPLDGM = NGP-1LDG00700
DO 110 AT=1,NGPLDGM = NGP-1LDG00710
DO 110 AU=1,NGPLDGM = NGP-1LDG00720
DO 110 AV=1,NGPLDGM = NGP-1LDG00730
DO 110 AW=1,NGPLDGM = NGP-1LDG00740
DO 110 AX=1,NGPLDGM = NGP-1LDG00750

```

110 AIA(I,J,K) = AIA(I,J,K)*AI(J,NGPM)*AI(K,NGPM)

IF (NTLD.NE.O) REWIND 24

IF (NBLO.NE.O) REWIND 27

IF (NSLD.NE.O) REWIND 28

DO 340 IL=1,NLOAD

REWIND 9

REWIND 19

IF (NMAT.EQ.-1) REWIND 16

IF (NMAT.EQ.-1) REWIND 23

DO 120 I=1,3

FLOAD(I) = 0.000

DO 130 I=1,NDPT

NTRK = 1

NTRK = NTRK(IL,NTRACK)

CALL WDISLF (NTRK)

DO 310 I=1,NEL

DO 140 J=1,NST

CEUD(J) = 0.000

READ (9) NCDN

READ (19) COREL

IF (NTLD.NE.O) READ (24) TEEL

IF (NBLO.NE.O) READ (27) BYEP

IF (NSLD.NE.O) READ (28) STEP

IF (NMAT.EQ.-1) GO TO 150

ALPHA = ELDT(1,3)

IF (NMAT.EQ.+1) GO TO 160

READ (16) NCON21

ALPHA = ELDT(NCON21,3)

CALL ELPRDP (NCON21)

GO TO 160

150 READ (23) ELDATE

160 X = XI(J,NGPM)

DO 280 K=1,NGP

Y = XI(K,NGPM)

DO 280 L=1,NGP

Z = XI(L,NGPM)

FACT = AIA(J,K,L)



CC

FORM LOADS DUE TO TEMPERATURE

```

IF (NTLD.EQ.0) GO TO 200
IF (NMAT.EQ.-1) CALL NDPROP(X,Y,Z,2)
CALL FORMK (X,Y,Z,DTJ,2)
DO 170 II=1,NST
DO 170 JJ=1,5
  BTD(II,JJ) = 0.000
DO 170 KK=1,6
  BTD(II,JJ) = BTD(II,JJ)+B(KK,II)*ELAST(KK,JJ)
170 CALL SHAPE (X,Y,Z,COL)
  CFT(1) = 0.000
DO 180 II=1,NPEL
  CFT(II) = CFT(1)+COL(II)*TEEL(II)
180 CFT(1) = CFT(1)*ALPHA*DTJ
DO 190 II=1,NST
  CELOD(II) = CELOD(II)+(BTD(II,1)+BTD(II,2)+BTD(II,3))*CFT(1)*FACT
190

```

CC

FORM LOADS DUE TO BODY FORCES

```

200 IF (NBLD.EQ.0) GO TO 230
CALL SHAPE (X,Y,Z,COL)
CALL FORMK (X,Y,Z,DTJ,1)
CFT(1) = 0.000
CFT(2) = 0.000
CFT(3) = 0.000
DO 210 II=1,NPEL
  CFT(II) = CFT(1)+COL(II)*BYEP(II,JJ)
210 CONTINUE
DO 220 II=1,NPEL
  DO 220 KK=1,NDF
    IK = 3*II+KK-3
    CELOD(IK) = CELOD(IK)+COL(II)*CFT(KK)*DTJ*FACT
220

```

CC

FORM LOADS DUE TO DISPLACEMENT

```

230 IF (NSLD.EQ.0) GO TO 260
CALL FORMK (X,Y,Z,DTJ,2)
IF (NMAT.EQ.-1) CALL NDPROP(X,Y,Z,0)
DO 240 II=1,NST
DO 240 JJ=1,6
  BTD(II,JJ) = 0.000
DO 240 KK=1,6
  BTD(II,JJ) = BTD(II,JJ)+B(KK,II)*ELAST(KK,JJ)
240

```



```

250 AK1(I1,JJ) = 0.0D0
    DU 260 I1=1,NST
    DO 260 JJ=1,NST
    DO 260 KK=1,6
260 AK1(I1,JJ) = AK1(I1,JJ)
    DU 270 I1=1,NST
    DO 270 JJ=1,NPEL
    DO 270 KK=1,NDF
    IJ = 3*JJ+KK-3
270 CELOD(IJ) = CELOD(IJ)
280 CONTINUE
    DO 300 I1=1,NPEL
    NTRACK = NCON(I1)
    NTK = NTRK(I1,NTRACK)
    CALL RDISLF (NTK)
    DO 290 JJ=1,NDF
    IJ = 3*I1+JJ-3
290 FLOAD(JJ) = CELOD(IJ)
300 CALL WDISLF (NTK)
310 CONTINUE

```

```

ZRC = 0.0D0
REWIND (15) NBS
READ (15) (NBS(I), I=1,N)
DO 330 NTK = NISL,NBS
CALL (1,N,N,E,J=1,N)
IF (NBS(NTK) .NE. NBS(NTK))
DO 320 J=1,N
IF (NBS(NTK) .NE. NBS(NTK))
CONTINUE
320

```

```

CALL WDISLF (NTK)      GO TO 330
IF (I.EQ.NDPT) GO TO 330
READ (15) NBC
NBCI = (NBC(I)+NDF)/NDF
CONTINUE
RETURN
END
330
340

```

○



C
C
C
C

SUBROUTINE BCOND

THIS SUBROUTINE APPLIES ANY BOUNDARY CONDITIONS TO APPROPRIATE
BLOCKS OF THE TOTAL STIFFNESS MATRIX

BCD000010

IMPLICIT REAL*8(A-H,O-Z)

BCD000020

COMMON /NB1/ NEL,NDPT,NPEL,NDF,NEQ,NN,MM,NS,NCOUNT,NST,NSTF,NCLD,NBCL,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NTLD,NBLD,NSL
1PBC,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NTLD,NBLD,NSL
BCD000030
BCD000040
BCD000050
BCD000060
BCD000070
BCD000080
BCD000090
BCD000100
BCD000110
BCD000120
BCD000130
BCD000140
BCD000150
BCD000160
BCD000170
BCD000180
BCD000190
BCD000200
BCD000210
BCD000220
BCD000230
BCD000240
BCD000250
BCD000260
BCD000270
BCD000280
BCD000290
BCD000300
BCD000310
BCD000320
BCD000330
BCD000340
BCD000350
BCD000360
BCD000370
BCD000380
BCD000390
BCD000400
BCD000410
BCD000420
BCD000430

COMMON /NB2/ NCON(20),NBC(4)
COMMON /B1/ ELDAT(10,3),CLOAD(4),BCON(3)
COMMON /B3/ AK1(60,60),AK2(60,60),AK3(60,60),RB1(60),RB2(60),RB3(60)

10 COMMON /CNTRL/ NLOAD
DIMENSION AKW1(3600), AKW2(3600)
EQUIVALENCE (AKW1(1),AK1(1,1)), (AKW2(1),AK2(1,1))
NTRK(I,J) = (I-1)*(MM)+J
NTRK(I,J) = (I-1)*NN+J
ABIGN = 1.0D20
REWIND 15
READ (15) NBC
NBC1 = NBC(1)+NDF
INDEX = 1
DO 50 I=1,NN
IH = I*NS
IF (I-1)*NS+1 GO TO 50
NTRK(I,1) = NTRK(I,1)
CALL RDDISK (NTK,AKW1)
IBL = NBC(1)+1-NS*(I-1)
IBH = IBL+NDF-1
IC = 1
DO 20 K=IBL,IBH
IC = IC+1
IF (NBC(IC).NE.1) GO TO 20
AK1(K,K) = AK1(K,K)+ABIGN
CONTINUE
IF (INDEX.EQ.NPBC) GO TO 30
READ (15) NBC
NBC1 = NBC(1)+NDF
INDEX = INDEX+1
IF (NBC1.LE.IH) GO TO 10
GO TO 40
CALL WRDISK (NTK,AKW1)
RETURN
CALL WRDISK (NTK,AKW1)
CONTINUE

20
30
40
50



RETURN

ENTRY 'BCOND2' IS FOR ANY PREDEFINED DISPLACEMENTS

ENTRY BCOND2
DIMENSION BCON1(60,2)
EQUIVALENCE (BCON1(1,1),AK3(1,1))

ABIGN = 1.0D20
DC 310 IL=1, NLOAD

I1 = 1
I5 = 0

MMF = NN-MM+1
IFR = (NBCL*NDF+NS-1)/NS

IR1 = 1
IFR (IFR, LE, MM) GO TO 60

IR1 = IFR-MM+1
DO 290 I=IR1, NN

IFR1 = IFR-I+1
IF (IFR1.GT.MM) GO TO 290

REWIND 15
REWIND 22

NTK1 = NTKR(IL, I)
CALL RDSKRI (NTK1)

DO 70 L=1, I1
READ (15) NBC

NBC1 = NBC(1)+NDF
NBC2 = NBC1/NDF

IF (15.EQ.0) GO TO 90
DO 80 L=1, I5

READ (22) BCON
IFR1 = IFR

J3 = 2
IF (IFR.NE.I) GO TO 200

I4 = 0
I# = I*NS

NTK1 = NTKR(I, I)
CALL RDISK (NTK1, AKW1)

I61 = 0
I6 = 0

DO 150 K=1, NDF
K1 = K+1

M = NBC(K1)+1
I1 = NBC(1)-(I-1)*NS+K

GO TO (150, 140, 110), M
IF (I6.NE.0) GO TO 120

READ (22) BCON
I5 = I5+1

I6 = I

60

70

80
90

100

110

BCD000440

BCD000450
BCD000460
BCD000470
BCD000480
BCD000490
BCD000500
BCD000510
BCD000520
BCD000530
BCD000540
BCD000550
BCD000560
BCD000570
BCD000580
BCD000590
BCD000600
BCD000610
BCD000620
BCD000630
BCD000640
BCD000650
BCD000660
BCD000670
BCD000680
BCD000690
BCD000700
BCD000710
BCD000720
BCD000730
BCD000740
BCD000750
BCD000760
BCD000770
BCD000780
BCD000790
BCD000800
BCD000810
BCD000820
BCD000830
BCD000840
BCD000850
BCD000860
BCD000870
BCD000880



```

120 I61 = 1
130 DO 130 L=1,NS
130 R81(L) = RB1(L) - AK1(L,11)*BCON(K)
140 I4 = I4+1
150 BCON1(I4,1) = I1
BCON1(I4,2) = BCON(K1)
AK1(I1,I1) = AK1(I1,I1)+ABIGN
150 CONTINUE
J3 = MM+1
MML = I/MMF-(I-MMF)/MMF
MM1 = MM-(I-MMF)*MML
J4 = MM1
IF (NBC2.EQ.NBCH) GO TO 160
READ (15) NBC
NBC1 = NBC(I)+NDF
NBC2 = NBC1/NDF
I1 = I+1
IF (NBC1.LE.IH) GO TO 100
J1 = (NBC1+NS-1)/NS
J2 = I+1
J3 = J1-I+1
160 IFR = J1
IF (I61.EQ.0) GO TO 190
IH = J1*NS
IF (J1.EQ.J2) GO TO 190
J4 = J3-1
IF (J4.GT.MM) J4 = MM
IF (I.EQ.NN) GO TO 300
DO 180 J=2,J4
NTK2 = NTRK(I,J)
CALL RDDISK (NTK2,AKW2)
NTKR2 = J+I-1
NTKR2 = NTKK(I1,NTKR2)
CALL RDSKR2 (NTK)
DO 170 LI=1,I4
NTK = BCON1(LI,1)
170 DO 170 L2=1,NS
R82(L2) = RB2(L2)-AK2(IR,L2)*BCON1(L1,2)
180 CALL WDSKR2 (NTK)
190 CALL WKDISK (NTK1,AKW1)
200 IF (J3.GT.MM) GO TO 280
MML = I/MMF-(I-MMF)/MMF
MM1 = MM-(I-MMF)*MML
DC 270 J=J3,MML
IH = (J+I-1)*NS
NTK2 = NTRK(I,J)
CALL RDDISK (NTK2,AKW2)
IHL = (J+I-2)*NS+1

```

```

BCD000890
BCD000900
BCD000910
BCD000920
BCD000930
BCD000940
BCD000950
BCD000960
BCD000970
BCD000980
BCD000990
BCD001000
BCD001010
BCD001020
BCD001030
BCD001040
BCD001050
BCD001060
BCD001070
BCD001080
BCD001090
BCD001100
BCD001110
BCD001120
BCD001130
BCD001140
BCD001150
BCD001160
BCD001170
BCD001180
BCD001190
BCD001200
BCD001210
BCD001220
BCD001230
BCD001240
BCD001250
BCD001260
BCD001270
BCD001280
BCD001290
BCD001300
BCD001310
BCD001320
BCD001330
BCD001340
BCD001350
BCD001360

```



```

210 IPD = (IH+I-NBC1)*(IHL-NBC1)
    IF (IPD.GT.O) GO TO 250
    I6 = 0
    DO 240 K=1,NDF
        KI = K+1
        IF (NBC(KI).NE.2) GO TO 240
        IF (I6.NE.O) GO TO 220
        READ (22) BCON
        I6 = 1
        IF = NBC(1)+K-(I+J-2)*NS
        DO 230 L=1,NS
            RB1(L) = RB1(L)-AK2(L,I)*BCON(K)
        CCNTINUE
        IF (NBC2.EQ.NBCH) GO TO 250
        READ (15) NBC
        NBC1 = NBC(1)+NDF
        NBC2 = NBC1/NDF
        IF (NBC1.GT.IH) GO TO 250
        GO TO 210
220 IFR6 = IFR1*I61
    IF (IFR6.NE.I) GO TO 270
    NTKR2 = J+I-1
    NTK = NTKR(IL,NTKR2)
    CALL RDSKR2 (NTK)
    DO 260 LI=1,I4
        IR = BCON(LI,1)
    DO 260 L2=1,NS
        RB2(L2) = RB2(L2)-AK2(IR,L2)*BCON1(L1,2)
    CALL WDSKR2 (NTK)
    CCNTINUE
    CALL WDSKR1 (NTKR1)
270 CCNTINUE
280 CALL WDSKR1 (NTKR1)
290 CALL WROISK (NTK1,AKW1)
    RETURN
300 CALL WDSKR1 (NTKR1)
    RETURN
310 CALL WDSKR1 (NTKR1)
    RETURN
    END

```

[illegible]

SLV00010

SUBROUTINE SOLVE

THIS SUBROUTINE IS THE BLOCK SOLVER PATTERNED AFTER A ROUTINE
ORIGINALLY WRITTEN BY PROF. J.G. CANTIN AT THE U.S. NAVAL
POSTGRADUATE SCHOOL

U U U U U U U U U U U U



```

IMPLICIT REAL*8(A-H,O-Z)
COMMON /NB1/ NEL,NDPT,NPEL,NDF,NEQ,NN,MM,NS,NCOUNT,NST,NSTF,NCLD,NSL
1PBC,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NILD,NBLD,NSL
2D
CCMMCN /B3/ AT1(60,60),AT2(60,60),AT3(60,60),RT1(60),RT2(60),RT3(60)
10,
COMMON /CNTRL/ NLOAD
DIMENSION AK1(3600), AK2(3600), AK3(3600), RB1(60), RB2(60), RB3(60)
10,
EQUIVALENCE (AK1(1),AT1(1,1)), (AK2(1),AT2(1,1)), (AK3(1),AT3(1,1))
1), (RB1(1),RT1(1)), (RB2(1),RT2(1)), (RB3(1),RT3(1))
NTKK(I,J) = (I-1)*(MM)+J
NTRK(I,J) = (I-1)*NN+J

```

CC

FCW REDUCTION

```

KMM = MM
N = 0
10 NTK = N+1
CALL RDDISK (NTK,AK2)
CALL SYMINV (AK2,NS,RB1,RB2,IFLG)
IF (IFLG.EQ.1) WRITE (6,150) N
IF (IFLG.EQ.1) STOP
IF (IFLG.EQ.2) WRITE (6,160) N
NTR = N
DO 20 IL=1,NLOAD
NTK = NTRK(IL,N)
CALL RDDISK (NTK)
CALL MULT (AK2,RB1,RB2,NS,NS,1)
CALL WDSKR2 (NTK)
CCNTINUE
20 IF (N.EQ.NN) GO TO 100
IF (N.GT.(NN-MM+1)) KMM=KMM-1
DO 40 K=2,KMM
NTK = NTRK(N,K)
CALL RDDISK (NTK,AK1)
CALL MULT (AK2,AK1,AK3,NS,NS,NS)
CALL WRDISK (NTK,AK3)
DO 30 I=1,NS
II = (I-1)*NS
DO 30 J=1,NS
IL = I+J
IR = I+(J-1)*NS
30 AK3(IL) = AK1(IR)
NTK = NTRK(NN,K)
CALL WRDISK (NTK,AK3)
40 CCNTINUE

```

SLV000020
SLV000030
SLV000040
SLV000050
SLV000060
SLV000070
SLV000080
SLV000090
SLV000100
SLV000110
SLV000120
SLV000130
SLV000140

SLV000150
SLV000160
SLV000170
SLV000180
SLV000190
SLV000200
SLV000210
SLV000220
SLV000230
SLV000240
SLV000250
SLV000260
SLV000270
SLV000280
SLV000290
SLV000300
SLV000310
SLV000320
SLV000330
SLV000340
SLV000350
SLV000360
SLV000370
SLV000380
SLV000390
SLV000400
SLV000410
SLV000420
SLV000430
SLV000440
SLV000450
SLV000460



SLV00470
SLV00480
SLV00490
SLV00500
SLV00510
SLV00520
SLV00530
SLV00540
SLV00550
SLV00560
SLV00570
SLV00580
SLV00590
SLV00600
SLV00610
SLV00620
SLV00630
SLV00640
SLV00650
SLV00660
SLV00670
SLV00680
SLV00690
SLV00700
SLV00710
SLV00720
SLV00730
SLV00740
SLV00750

```

DO 90 L=2,KMM
I = N+L-1
IF (I.GT.NN) GO TO 90
J = 0
NTK = NTRK(NN,L)
CALL RDDISK (NTK,AK2)
DO 60 K=L,KMM
J = J+1
NTK = NTRK(N,K)
CALL RDDISK (NTK,AK1)
CALL MULT (AK2,AK1,AK3,NS,NS,NS)
NTK = NTRK(I,J)
CALL RDDISK (NTK,AK1)
DO 50 I1=1,NCOUNT
50 AK1(I1) = AK1(I1)-AK3(I1)
CALL WRDISK (NTK,AK1)
60 CONTINUE
DO 80 IL=1,NLOAD
NTK = NTRK(IL,N)
CALL RUSKR2 (NTK)
CALL MULT (AK2,RB2,RB3,NS,NS,I)
NTR = I
NTK = NTRK(IL,NTR)
CALL RUSKR1 (NTK)
DO 70 I1=1,NS
70 RB1(I1) = RB1(I1)-RB3(I1)
80 CALL WRDISK (NTK)
90 CONTINUE
GO TO 10

```

C
C

BACK SUBSTITUTION

```

100 N = N-1
IF (N.EQ.0) GO TO 140
NT1 = N
DO 130 IL=1,NLOAD
NTK = NTRK(IL,NT1)
CALL RUSKR3 (NTK)
DO 120 K=2,KMM
L = N+K-1
IF (L.GT.NN) GO TO 120
NTK = NTRK(N,K)
CALL RDDISK (NTK,AK1)
NTR = L
NTK1 = NTRK(IL,NTR)
CALL RUSKR1 (NTK1)
CALL MULT (AK1,RB1,RB2,NS,NS,I)
DO 110 K1=1,NS

```

SLV00760
SLV00770
SLV00780
SLV00790
SLV00800
SLV00810
SLV00820
SLV00830
SLV00840
SLV00850
SLV00860
SLV00870
SLV00880
SLV00890
SLV00900
SLV00910



```

1110 RB3(K1) = RB3(K1)-RB2(K1)
1120 CCNTINUE
      NTK = NTKR(IL,NT1)
      CALL WDSKR3 (NTK)
1130 CCNTINUE
      KMM = KMM+1
      IF (KMM.GT.MM) KMM = MM
      GO TO 1100
1140 RETURN
1150 FORMAT ('0',10X,'***** BLOCK NO.',I3,' IS SINGULAR. EXECUTION STOP
1160 1S *****')
      1CN FORMAT ('0',10X,'***** BLOCK NO.',I3,' IS NEARLY SINGULAR. EXECUTI
      1CN CCNTINUES *****')
      END

```

SUBROUTINE DISP (ILL) DSP00010

THIS SUBROUTINE FORMS NODAL DISPLACEMENTS AND REACTIONS AT
BOUNDARY CONDITION NODES

```

10      IMPLICIT REAL*8(A-H,O-Z)
11      COMMON /NB1/ NEL,NDPT,NPEL,NDF,NEQ,NN,MM,NS,NCOUNT,NST,NSIF,NCLD,
12      1PBC,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NTLD,NBLD,NSL
13      2D      COMMON /NB2/ NCON(20),NBC(4)
14      10      COMMON /B3/ AK1(60,60),AK2(60,60),AK3(60,60),RB1(60),RB2(60),RB3(60)
15      10      COMMON /CNTRL/ NLOAD
16      10      DIMENSION REACT(4), REACT1(3)
17      10      EQUIVALENCE (REACT(1),RB2(1)), (REACT(2),REACT1(1))
18      10      NTKR(I,J) = (I-1)*NN+J
19      10      ABIGN = 1.0D20
20      10      ZRO = 0.0D0
21      10      REWIND 15
22      10      READ 15 NBC
23      10      NBC1 = NBC(1)+NDF
24      10      INDEX = 1
25      10      NPBL = NS/NDF
26      10      JLL = NDPT-(NDPT/NPBL)*NPBL
27      10      IF (JLL.EQ.0) JLL = NPBL
28      10      NLI = 0
29      10      WRITE (6,250)
30      10      DO 220 I=1,NN
31      10      NTK = NTKR(ILL,I)

```



```

CALL RDSKRI (NTK)
CALL WDISKB (I)
IM1 = I-1
ICTI = IM1*NPBL
IH = I*NS
IL = IM1*NS+1
I11CHECK = (NBC1-IH)*(IL-NBC1)
110 IF (I11CHECK.LT.0) GO TO 160
DO 120 J=1,NDEF
120 REACT(J) = ZRO
I1 = NBC(1)-NS*(I-1)+1
I2 = I1+1
I3 = I2+1
IF (NBC(2).EQ.0) GO TO 130
REACT(2) = RB1(I1)*ABIGN
RB1(I1) = ZRO
130 IF (NBC(3).EQ.0) GO TO 140
REACT(3) = RB1(I2)*ABIGN
RB1(I2) = ZRO
140 IF (NBC(4).EQ.0) GO TO 150
REACT(4) = RB1(I3)*ABIGN
RB1(I3) = ZRO
150 ANBC = NBC(1)/NDF+1
REACT(1) = ANBC
WRITE (17) REACT
IF (INDEX.EQ.NPBC) GO TO 160
READ (15) NBC
NBC1 = NBC(1)+NDF
INDEX = INDEX+1
GO TO 110

```

C
C
C

PRINT DISPLACEMENTS

```

160 NL2 = NL1+NPBL
JL = NPBL
IF (I.EQ.NN) JL = JLL
IF (NL2.GT.NLINE) GO TO 180
DO 170 M=1,JL
170 ICT = ICTI+M
K1 = (M-1)*NDF+1
K2 = K1+2
170 WRITE (6,270) ICT,(RB1(N1),N1=K1,K2)
NL1 = NL2
GO TO 220
180 J2 = NLINE-NL1
IF (J2.LE.0) GO TO 200
DO 190 N=1,J2
190 ICT = ICTI+N

```

DSP00270
DSP00280
DSP00290
DSP00300
DSP00310
DSP00320
DSP00330
DSP00340
DSP00350
DSP00360
DSP00370
DSP00380
DSP00390
DSP00400
DSP00410
DSP00420
DSP00430
DSP00440
DSP00450
DSP00460
DSP00470
DSP00480
DSP00490
DSP00500
DSP00510
DSP00520
DSP00530
DSP00540
DSP00550
DSP00560

DSP00570
DSP00580
DSP00590
DSP00600
DSP00610
DSP00620
DSP00630
DSP00640
DSP00650
DSP00660
DSP00670
DSP00680
DSP00690
DSP00700
DSP00710



DSP00720
 DSP00730
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 DSP00750
 DSP00760
 DSP00770
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 DSP00790
 DSP00800
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DSP00850
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 DSP00950
 DSP00960
 DSP00970
 DSP00980
 DSP00990
 DSP01000
 DSP01010
 DSP01020
 DSP01030
 DSP01040
 DSP01050
 DSP01060
 DSP01070
 DSP01080
 DSP01090
 DSP01100
 DSP01110
 DSP01120

```

K1 = (N-1)*NDF+1
K2 = K1+2
190 WRITE (6,270) ICT,(RBI(N2),N2=K1,K2)
200 NLI = NPBL-J2
    IF (JL.LE.0) GO TO 220
    WRITE (6,250)
    J1 = J2+1
    DO 210 M1=J1,JL
      ICT = ICT+M1
    K1 = (M1-1)*NDF+1
    K2 = K1+2
210 WRITE (6,270) ICT,(RBI(N3),N3=K1,K2)
220 CONTINUE

PRINT REACTIONS
CHECK EQUILIBRIUM

SUMX = ZRO
SUMY = ZRO
SUMZ = ZRO
I2 = ZRO
REWIND 17
I1 = 1
I2 = I2+NLIN
I2 = (NPBC.LE.I2) I2 = NPBC
WRITE (6,260) I2
DO 240 L3=I1,I2
  READ (17) REACT
  ICT = REACT(1)
  WRITE (6,270) ICT,REACT1
  SUMX = SUMX+REACT(2)
  SUMY = SUMY+REACT(3)
  SUMZ = SUMZ+REACT(4)
CONTINUE
I1 = I2+1
IF (I2.LT.NPBC) GO TO 230
WRITE (6,280) SUMX,SUMY,SUMZ
RETURN

250 1 FFORMAT (1H1,/,5X,' D-I-S-P-L-A-C-E-M-E-N-T-S ',
260 1 FFORMAT (1H1,/,5X,' N-PT,12X,3H X,23X,2H Y,23X,2H Z,/)
270 1 FFORMAT (1H1,/,5X,' R-E-A-C-T-I-O-N-S ',
280 1 FFORMAT (1H1,/,5X,' N-PT,12X,3H X,23X,2H Y,23X,2H Z,/)
    FFORMAT (5X,15,3(1PD25.16)),
    FFORMAT (///,' EQUILIBRIUM CHECK ',/,10X,3(1PD25.16))
  END

```

C*****
 C*****
 C*****





PST00440
PST00450
PST00460
PST00470
PST00480
PST00490
PST00500

S2 = FACT*DCOS(PHI3+2.094395103D0)
S3 = FACT*DCOS(PHI3+4.18879206D0)
S1 = S1-PHI1/3.0D0
S2 = S2-PHI1/3.0D0
S3 = S3-PHI1/3.0D0
RETURN
END

C*****
C*****
C*****
C*****

SUBROUTINE ELPRUP (I)
ELP00010

THIS SUBROUTINE FORMS THE ELASTIC MATRIX FOR ELEMENTS

IMPLICIT REAL*8(A-H,O-Z)
COMMON /B1/ ELDAT(10,3),CLOAD(4),BCON(3)
COMMON /B2/ ELAST(6,6),COORD(3),COREL(20,3),SSSS1(7),UJNT(3),SSSS1(7),SSNN1
1(7),ALPHA
DO 110 L=1,6
DO 110 J=1,6
ELAST(L,J)=0.0D0
E = ELDAT(I,1)
PR = ELDAT(I,2)
EK = E/((1.0D0+PR)*(1.0D0-2.0D0*PR))
ELAST(L,J)=ER*(1.0D0-PR)
ELAST(L,J)=ER*PR
ELAST(1,1)=EK
ELAST(1,2)=EK
ELAST(2,1)=EK
ELAST(2,2)=EK
ELAST(3,1)=EK
ELAST(3,2)=EK
ELAST(4,3)=EK
ELAST(4,4)=ER*(1.0D0-2.0D0*PR)/2.0D0
ELAST(5,5)=EK
ELAST(5,6)=EK
ELAST(6,6)=EK
RETURN
END

C*****
C*****
C*****
C*****

SUBROUTINE FORMK (X,Y,Z,DTJ,INDIC)
FMK00010

THIS SUBROUTINE CALCULATES SHAPE FUNCTION DERIVATIVE VALUES,

C





MIDSIDE NODES

FMK00440
FMK00450
FMK00460
FMK00470
FMK00480
FMK00490
FMK00500
FMK00510
FMK00520
FMK00530
FMK00540
FMK00550
FMK00560
FMK00570
FMK00580
FMK00590
FMK00600
FMK00610
FMK00620
FMK00630
FMK00640
FMK00650

```

130 K=1,2
11=(K-1)*I2+4
11=11+4
130 L=1,1,1,1,1,4
X1=CORDG(L,1)
Y1=CORDG(L,2)
Z1=CORDG(L,3)
W1(1,L)=D4(Y,Z,X1,Z1)
W1(2,L)=D4(Y,Z,X1,Z1,X1)
W1(3,L)=D4(Y,X,Z1,X1)
140 L=9,12
X1=CORDG(L,1)
Y1=CORDG(L,2)
Z1=CORDG(L,3)
W1(1,L)=D4(Z,Y,X1,Y1)
W1(2,L)=D4(Z,X,Y1,X1)
W1(3,L)=D2(Z,X,Y,X1,Y1)
150 I=1,3
DO 150 J=1,3
AJ(I,J)=0.000
DO 150 K=1,NPEL
AJ(I,J)=AJ(I,J)+W1(I,K)

```

FERRE JACOBIAN

```

DTJ = AJ(1,1)*AJ(2,2)*AJ(3,3)+AJ(1,2)*AJ(2,3)*AJ(3,1)+AJ(1,3)*AJ(2,1)*AJ(3,2)
1)*AJ(3,2)-AJ(3,1)*AJ(2,2)*AJ(1,3)-AJ(2,3)*AJ(1,1)-AJ(1,2)*AJ(3,3)
2)*AJ(2,1)*AJ(1,2)
IF (INDIC.EQ.1) RETURN
AJIN(1,1) = (AJ(2,2)*AJ(3,3)-AJ(3,2)*AJ(2,3))/DTJ
AJIN(2,1) = -(AJ(2,1)*AJ(3,3)-AJ(3,1)*AJ(2,2))/DTJ
AJIN(3,1) = (AJ(2,1)*AJ(3,2)-AJ(3,1)*AJ(2,3))/DTJ
AJIN(1,2) = -(AJ(1,2)*AJ(3,3)-AJ(3,2)*AJ(1,3))/DTJ
AJIN(2,2) = (AJ(1,1)*AJ(3,3)-AJ(3,1)*AJ(1,3))/DTJ
AJIN(3,2) = -(AJ(1,1)*AJ(3,2)-AJ(3,1)*AJ(1,2))/DTJ
AJIN(1,3) = (AJ(1,2)*AJ(2,3)-AJ(2,1)*AJ(1,3))/DTJ
AJIN(2,3) = -(AJ(1,1)*AJ(2,3)-AJ(2,1)*AJ(1,3))/DTJ
AJIN(3,3) = (AJ(1,1)*AJ(2,2)-AJ(2,1)*AJ(1,2))/DTJ
CC 160 I=1,5
DO 160 J=1,NPEL
DNX(I,J) = 0.0D0
DO 160 K=1,3
DNX(I,J) = DNX(I,J)+AJIN(I,K)*W1(K,J)
DO 170 I=1,6
DO 170 J=1,NST
BI(J,I) = 0.0D0

```





```

1PBC,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NTLD,NBLD,NSL
2D COMMON /B2/ ELAST(6,6),COORD(3),COREL(20,3),UJNT(3),SSSS1(7),SSNN1
1(7),ALPHA
CCMMGN /B3/ AK1(60,60),AK2(60,60),AK3(60,60),RB1(60),RB2(60),RB3(60)
10, DIMENSION STK(3600), AK(3600), B(360), ELDATN(20,3)
EQUIVALENCE (AK1(1,1),STK(1)), (AK2(1,1),AK(1)), (AK3(1,1),B(1)),
1(RB2(1),ELDATN(1))
REWIND 19
IF (NMAT.EQ.-1) GO TO 150
IF (NMAT.EQ.1) GO TO 130

```

CC

FOR USE WITH MUKI THAN ONE ELEMENT PROPERTY

```

REWIND 16
N2 = -1
DO 120 I=1,NEL
READ (19) COREL
READ (16) NCON21
N = NCON21
IF (N.EQ.N2) GO TO 110
CALL ELPROP (N)
N2 = N
110 CALL CUB
CALL WDISK1 (1)
120 CONTINUE
RETURN

```

CC

CNE ELEMENT PROPERTY

```

130 CALL ELPROP (1)
DO 140 I=1,NEL
READ (19) COREL
CALL CUB
CALL WDISK1 (1)
140 CONTINUE
RETURN

```

CC

NODAL PROPERTIES

```

150 REWIND 23
DO 160 I=1,NEL
READ (19) COREL
READ (23) ELDATN
CALL CUB
CALL WDISK1 (1)
160 CONTINUE

```



RETURN
END

SUBROUTINE MERGE

THIS SUBROUTINE FORMS THE TOTAL STIFFNESS MATRIX

```

1 IMPLICIT REAL*8(A-H,O-Z)
2 COMMON /NB1/ NEL,NDPT,NPEL,NDF,NEQ,NN,MM,NS,NCOUNT,NST,NSTF,NCLD,
3 IPEC,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NTLD,NBLD,NSL
4
5 DO
6   CGMMCN /NB2/ NCUN(20),NBC(4)
7   CGMMCN /B3/ AK1(60,60),AK2(60,60),AK3(60,60),RB1(60),RB2(60),RB3(60)
8
9   DIMENSION A(60,60), S(60,60), B(3600), T(3600), LM(20)
10  EQUIVALENCE (AK2(1,1),A(1,1),B(1)), (AK1(1,1),S(1,1),T(1)), (LM(1)
11  ,RB1(1))
12  NTRK(1,J) = (I-1)*(MM)+J
13  ZRO = 0.000
14  DO 190 I=1,NN
15  DO 190 J=1,MM
16  DO 110 I1=1,NS
17  DO 110 J1=1,NS
18  A(I1,J1) = ZRO
19  REWIND 9
20  DO 180 K=1,NEL
21  READ (9) NCUN
22  DO 120 L=1,NPEL
23  LM(L) = NDF*NCON(L)-NDF
24  LMIN = 5000
25  LMAX = 0
26  DO 130 I2=1,NPEL
27  LMIN = MINO(LMIN,LM(I2))
28  LMAX = MAXO(LMAX,LM(I2))
29  LMIN = LMIN+1
30  LMAX = LMAX+1
31  NSI = NS*I
32  NSIM = NSI-NS
33  IF (LMIN.GT.NSI) GO TO 180
34  IF (LMAX.LE.NSIM) GO TO 180
35  CALL RDISK1(K)
36  DO 170 I1=1,NPEL
37  DO 160 JJ=1,NPEL
38  DO 150 KK=1,NDF
39  I11 = LM(I1)+KK-NS*(I-1)

```



```

IF ((III.GT.NS).OR.(III.LT.I)) GO TO 150
KKK = NDF*II-NDI+KK
DU 140 LL=1, NDF
JJJ = LM(JJ)+LL-NS*(J+I-2)
IF ((JJJ.GT.NS).OR.(JJJ.LT.I)) GO TO 140
LLL = NDF*JJ-NDI+LL
A(III,JJJ) = A(III,JJJ)+S(KKK,LLL)
CCCONTINUE
140 CCCONTINUE
1150 CCCONTINUE
1160 CCCONTINUE
1170 CCCONTINUE
1180 CCCONTINUE

```

WRITE COMPLETED BLOCK ON DISK

```

190      NTK = NTRK(I,J)
        CALL WRDISK (NTK,B)
        CCNT INUE
        IF (NN*NS.EQ.NEQ) GO TO 210
        NTK = NTRK(NN,I)
        CALL RDISK (NTK,B)
        DO 200 I=1,NS
        IF (DABS(A(I,I)).LT.1.0D-14) A(I,I)=1.0D0
200      CCNT INUE
        CALL WRDISK (NTK,B)
210      CCNT INUE
        RETURN
      END

```

○

SUBROUTINE STRESS (I,ILL)

THIS SUBROUTINE COMPUTES NODAL STRESSES

```

IMPLICIT REAL*8(A-H,O-Z)
COMMON /NB1/ NEL,NUPT,NPEL,NDF,NEQ,NN,MM,NS,NCOUNT,NST,NSIF,NCLD,NS
1PBC,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NTLD,NBLD,NSL
2D
COMMON /NB2/ NCON(20),NBC(4)
COMMON /B1/ ELDAT(10,3),CLOAD(4),BCON(3)
COMMON /B2/ ELAST(6,6),COORD(3),COREL(20,3),UJNT(3),SSSS1(7),SSNN1
1(7),ALPHA
COMMON /B3/ AK1(60,60),AK2(60,60),AK3(60,60),RB1(60),RB2(60),RB3(6
10)
DIMENSION SNELM(20,6),UEL(60),SSELM(20,6),SSNN(6),B(6,60),SSS
1S(6),CORD6(20,3),LM(20),TEEL(20),ELDATE(20,3)
STR000020
STR000030
STR000040
STR000050
STR000060
STR000070
STR000080
STR000090
STR00100
STR00110
STR00120
STR00130

```

315





CC

FORMS STRESS AND STRAIN VECTORS

```

NTK = NTRK(ILL,JJ)
CALL RDISKS (NTK)
CALL RDISKN (JJ)
SSSI(7) = SSSI(7)+1.000
SSNN(7) = SSNN(7)+1.000
DO 180 K=1,6
  SNELM(J,K) = SSNN(K)
  SSELN(J,K) = SSSI(K)
  SSNN(K) = SSNN(K)+SSNN(K)
  SSNN(K) = SSNN(K)
180 CCNTINUE
CALL WDISKS (NTK)
CALL WDISKN (JJ)
190 CCNTINUE
IF (KEL.EQ.0) GO TO 210

```

CC

PRINT ELEMENT NODAL VALUES IF REQUIRED

```

DO 200 K=1,NPEL
  WRITE (6,230) LM(K),(SNELM(K,L),L=1,6)
  WRITE (6,230) LM(K),(SSELN(K,M),M=1,6)
200 WRITE (6,240)
210 CCNTINUE
  RETURN
220 FORMAT (1H1,/, ' S-T-R-A-I-N-S / S-T-R-E-S-S-E-S FOR ELEMENT ',I4,
1 /,2X,JOINT,6X,
2 ,SNX/SSX,8X,'SNZ/SSZ',8X,'SNXY/SSXY',6X,'SNYZ/SSYZ',
3 ,6X,'SNZX/SSZX',/)
230 FCNRMAT (2X,I4,4X,6(1PD15.6))
240 FORMAT (1H )
END

```

CC

SUBROUTINE CUB

THIS SUBROUTINE PERFORMS CUBIC INTEGRATION

```

IMPLICIT REAL*8(A-H,O-Z)
COMMON /NB1/ NEL,NOPT,NPEL,NDF,NEQ,NN,MM,NS,NCOUNT,N,NSTF,NCLD,NPBC
1C,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NILD,NBLD,NSLD
COMMON /B3/ AK1(60,60),AK2(60,60),AK3(60,60),RB1(60),RB2(60),RB3(60)
10)
COMMON /GAUSS/ NGPS,NGPL

```

CUB00010

CUB000020
CUB000030
CUB000040
CUB000050
CUB000060
CUB000070





C

THIS SUBROUTINE COMPUTES SHAPE FUNCTION VALUES

```

      IMPLICIT REAL*8(A-H,O-Z)
      DIMENSION CORDX(4), CORDY(4), CORDZ(4), COL(1)
      DATA CORDX/1.0D0,-1.0D0,-1.0D0,-1.0D0,1.0D0/
      DATA CORDY/1.0D0,1.0D0,-1.0D0,-1.0D0,-1.0D0/
      DATA CORDZ/1.0D0,-1.0D0,1.0D0,-1.0D0,-1.0D0/
      FQC(XI,ETA,ZETA,XI,YI,ZI)=(1.0D0+XI*XI)*(1.0D0+XI*XI)*
      1*(1.0D0+ETA*YI)*(1.0D0+ETA*YI)*(1.0D0+ZETA*Z1
      1)/4.0D0
      DO 110 IZ=1,2
      II = IZ*(IZ-1)+1
      IT = II+6
      IX = 0
      DO 110 ICOL=II,IT,2
      IX = IX+1
      YI = CORDY(IX)
      XI = CORDX(IX)
      ZI = CORDZ(IZ)
      COL(ICOL) = FQC(XI,ETA,ZETA,XI,YI,ZI)
      IC9 = 8
      DO 120 IZ=1,2
      II = IZ*(IZ-1)+2
      IT = II+4
      DO 120 IC2=II,IT,4
      IC4 = IC2+2
      IC9 = IC9+1
      IN = IC9-8
      XI = CORDX(IN)
      YI = CORDY(IN)
      ZI = CORDZ(IN)
      COL(IC2) = FQM(XI,ETA,ZETA,ZI,YI)
      COL(IC4) = FQM(ETA,ZETA,XI,YI,-ZI)
      COL(IC9) = FQM(ZETA,XI,ETA,XI,YI)
      120 RETURN
      END

```

C

SUBROUTINE MULT (A,B,C,NRA,NCA,NCB)

THIS SUBROUTINE FORMS C=A*B

```

      IMPLICIT REAL*8(A-H,O-Z)
      DIMENSION A(1), B(1), C(1)
      ZRO = 0.0D0

```

C

MUL000020
MUL000030
MUL000040

MUL00050
MUL00060
MUL00070
MUL00080
MUL00090
MUL00100
MUL00110
MUL00120
MUL00130
MUL00140
MUL00150

```

DO 110 I=1,NRA
DO 110 J=1,NCB
IC = I+(J-I)*NRA
C(IC) = ZRO
DO 110 K=1,NCA
IA = I+(K-I)*NRA
IB = K+(J-I)*NCA
C(IC) = C(IC)+A(IA)*B(IB)
110 CCONTINUE
      RETURN
      END

```

C *****
C *****
C *****
C *****

SUBROUTINE DISK

THIS SUBROUTINE CONTROLS ALL READ AND WRITE OPERATIONS TO THE
VARIOUS FILES USED IN THE EXTERNAL STORAGE.

```

      IMPLICIT REAL*8(A-H,O-Z)
      COMMON /NB1/NEL,NDPI,NPEL,NDF,NEQ,NN,MM,NS,NCOUNT,NST,NSTF,NCLD,
1NPPBC,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NTLD,NBLD,
2NSLD
      COMMON /B12/ ELDATN(3),BYNP(3),STNP(3),THNP,FLOAD(3)
      COMMON /B2/ ELAST(6,6),COORD(3),COREL(20,3),UJNT(3),SSSS1(7),
1SSNN1(7),ALPHA
      COMMON /B3/ AK1(60,60),AK2(60,60),AK3(60,60),RB1(60),RB2(60),
1RB3(60)
      COMMON /IFILE/ I7,I8,I10,I11,I12,I13,I18,I20,I21,I25,I26,I29
      DIMENSION STK1(900),STK2(900),STK3(900),STK4(900),A(1)
      EQUIVALENCE (STK1(1),AK1(1,1)),(STK2(1),AK1(1,16)),
1(STK3(1),AK1(1,31)),(STK4(1),AK1(1,46))
      NCCOUNT=NS*NS
      LREC7=NCOUNT/NREC7
      NS1=NS/NDF
      RETURN
      ENTRY WRDISK(NTRACK,A)
      I7=(NTRACK-1)*NREC7+1
      N1=1
      DO 10 L=1,NREC7
      N2=L*LREC7
      WRITE(7,I7)
      N1=N2+1
      RETURN
      ENTRY RDDISK(NTRACK,A)
      I7=(NTRACK-1)*NREC7+1
      N1=1

```

DSK00020
DSK00030
DSK00040
DSK00050
DSK00060
DSK00070
DSK00080
DSK00090
DSK00100
DSK00110
DSK00120
DSK00130
DSK00140
DSK00150
DSK00160
DSK00170
DSK00180
DSK00190
DSK00200
DSK00210
DSK00220
DSK00230
DSK00240
DSK00250
DSK00260
DSK00270
DSK00280
DSK00290



```

20  DO 20 L=1,NREC7
    N2=L*LRREC7
    READ(7,17,ERR=500)
    NI=N2+1
    RETURN
    ENTRY WDISK1(NTRACK)
    I8=(NTRACK-1)*4+1
    WRITE(8,18) STK1
    WRITE(8,18) STK2
    WRITE(8,18) STK3
    WRITE(8,18) STK4
    RETURN
    ENTRY ROISK1(NTRACK,A)
    I8=(NTRACK-1)*4+1
    READ(8,18,ERR=500) STK1
    READ(8,18,ERR=500) STK2
    READ(8,18,ERR=500) STK3
    READ(8,18,ERR=500) STK4
    RETURN
    ENTRY WDISK(NTRACK)
    WRITE(10,NTRACK) COORD
    RETURN
    ENTRY RDISK(NTRACK)
    READ(10,NTRACK,ERR=500) COORD
    RETURN
    ENTRY WDISKP(NTRACK)
    WRITE(11,NTRACK) ELDATN
    RETURN
    ENTRY RDISKP(NTRACK)
    READ(11,NTRACK,ERR=500) ELDATN
    RETURN
    ENTRY RDISKS(NTRACK)
    READ(12,NTRACK,ERR=500) SSSS1
    RETURN
    ENTRY WDISKS(NTRACK)
    WRITE(12,NTRACK) SSSS1
    RETURN
    ENTRY RDISKN(NTRACK)
    READ(13,NTRACK,ERR=500) SSNN1
    RETURN
    ENTRY WDISKN(NTRACK)
    WRITE(13,NTRACK) SSNN1
    RETURN
    ENTRY WDISKT(NTRACK)
    WRITE(18,NTRACK) THNP
    RETURN
    ENTRY ROISKT(NTRACK)
    READ(18,NTRACK,ERR=500) THNP

```

```

DSK00300
DSK00310
DSK00320
DSK00330
DSK00340
DSK00350
DSK00360
DSK00370
DSK00380
DSK00390
DSK00400
DSK00410
DSK00420
DSK00430
DSK00440
DSK00450
DSK00460
DSK00470
DSK00480
DSK00490
DSK00500
DSK00510
DSK00520
DSK00530
DSK00540
DSK00550
DSK00560
DSK00570
DSK00580
DSK00590
DSK00600
DSK00610
DSK00620
DSK00630
DSK00640
DSK00650
DSK00660
DSK00670
DSK00680
DSK00690
DSK00700
DSK00710
DSK00720
DSK00730
DSK00740
DSK00750
DSK00760
DSK00770

```



```

RETURN RDSKR1(NTRACK)          RB1
ENRTRY READ(20,NTRACK,ERR=500)
RETURN
ENRTRY RDSKR2(NTRACK)          RB2
ENRTRY READ(20,NTRACK,ERR=500)
RETURN
ENRTRY RDSKR3(NTRACK)          RB3
ENRTRY READ(20,NTRACK,ERR=500)
RETURN
ENRTRY WDSKR1(NTRACK)          RB1
ENRTRY WRITE(20,NTRACK)
RETURN
ENRTRY WDSKR2(NTRACK)          RB2
ENRTRY WRITE(20,NTRACK)
RETURN
ENRTRY WDSKR3(NTRACK)          RB3
ENRTRY WRITE(20,NTRACK)
ENRTRY WDISKB(NTRACK)
I2L1=(NTRACK-1)*NS1+1
J1=1
J2=NDF
DO 600 I=1,NS1
WRITE(21,I2L1) (RB1(J),J=J1,J2)
J1=J2+1
J2=J2+NDF
CONTINUE
RETURN
ENRTRY RDISKB(NTRACK)          UJNT
ENRTRY READ (21,NTRACK,ERR=500)
RETURN
ENRTRY WDISBF(NTRACK)
ENRTRY WRITE(25,NTRACK) BYNP
RETURN
ENRTRY RDISBF(NTRACK)
ENRTRY READ(25,NTRACK,ERR=500) BYNP
RETURN
ENRTRY WDISSF(NTRACK)
ENRTRY WRITE(26,NTRACK) STNP
RETURN
ENRTRY RDISSF(NTRACK)
ENRTRY READ(26,NTRACK,ERR=500) STNP
RETURN
ENRTRY WDISLF(NTRACK)
ENRTRY WRITE(29,NTRACK) FLOAD
RETURN
ENRTRY RDISLF(NTRACK)

```

DSK00780
 DSK00790
 DSK00800
 DSK00810
 DSK00820
 DSK00830
 DSK00840
 DSK00850
 DSK00860
 DSK00870
 DSK00880
 DSK00890
 DSK00900
 DSK00910
 DSK00920
 DSK00930
 DSK00940
 DSK00950
 DSK00960
 DSK00970
 DSK00980
 DSK00990
 DSK01000
 DSK01010
 DSK01020
 DSK01030
 DSK01040
 DSK01050
 DSK01060
 DSK01070
 DSK01080
 DSK01090
 DSK01100
 DSK01110
 DSK01120
 DSK01130
 DSK01140
 DSK01150
 DSK01160
 DSK01170
 DSK01180
 DSK01190
 DSK01200
 DSK01210
 DSK01220
 DSK01230
 DSK01240
 DSK01250

[illegible]




```

1130 ELAST(5,5) = ELAST(4,4)
      ELAST(6,6) = ELAST(4,4)
      IF (IFLAG.EQ.0) RETURN
      ALPHA = 0.000
      DO I40 I=1,NPEL
        ALPHA = ALPHA+CCL(I)*ELI
      RETURN
      END

```

SMV00010

SUERROUTINE SYMINV (A,N,B,C,IFLG)

THIS SUBROUTINE INVERTS A SYMMETRIC MATRIX

```

SMV000020
SMV000030
SMV000040
SMV000050
SMV000060
SMV000070
SMV000080
SMV000090
SMV000100
SMV000110
SMV000120
SMV000130
SMV000140
SMV000150
SMV000160
SMV000170
SMV000180
SMV000190
SMV000200
SMV000210
SMV000220
SMV000230
SMV000240
SMV000250
SMV000260
SMV000270
SMV000280
SMV000290
SMV000300
SMV000310
SMV000320
SMV000330

IMPLICIT REAL*8(A-H,O-Z)
DIMENSION A(1), B(1), C(1)
IFLG = 0
ZRO = 0.000
ABIGN = 1.0D15
TRACE = ZRO
DO 110 I=1,N
  IAD = (I-1)*N+I
  IF (A(IAD)-GE.ABIGN) GO TO 110
  TRACE = TRACE+DABS(A(IAD))
CCNTINUE
APZRO = TRACE*1.0D-12
IF (APZRO.GT.1.0D-20) APZRO=1.0D-20
LF = N
DO 120 I=2,LP
  J=I,LP
  ICOL = J+LP*(I-2)
  IROW = I+{(J-I)*LP-1}
  IF (A(ICOL).EQ.A(IROW)) GO TO 120
  A(ICOL) = 0.500*(A(ICOL)+A(IROW))
  A(IROW) = A(ICOL)
CCNTINUE
DO 140 I=1,N
  B(I) = ZRO
  II = (I-1)*N
  DO 130 J=1,N
    JJ = II+J
    IF (A(JJ)-GE.ABIGN) GO TO 130
    B(I) = B(I)+DABS(A(JJ))
    CCNTINUE
  CONTINUE
  ANR = ZRO
110
120
130
140
```




```

150 DC 150 I=1,N
    ANR = DMAXI(ANR,B(I))
    DO 220 I=1,N
    NR = (I-1)*N
    DO 160 J=1,N
    K = NR+J
    160 B(J) = A(K)
    D = B(I)
    IF (D.EQ.ZRO) GO TO 270
    IF (DABS(D).LT.APZRO) IFLG=2
    DO 170 J=1,N
    C(J) = -B(J)/D
    170 L = I
    DC 200 J=1,N
    M = L
    DO 190 K=J,N
    DB = DABS(B(J))
    IF (DB.LE.1.00-40) GO TO 180
    DC = DABS(C(K))
    IF (DC.LE.1.00-40) GO TO 180
    IF ((DB.LE.APZRO).AND.(DC.LE.APZRO)) GO TO 180
    A(L) = A(L)+B(J)*C(K)
    180 CCNTINUE
    A(M) = A(L)
    M = M+N
    190 L = L+1
    200 L = L+J
    C(I) = -1.00G/D
    M = I
    DO 210 J=1,N
    K = NR+J
    A(K) = C(J)
    A(M) = C(J)
    M = M+N
    210 CCNTINUE
    NS = N*N
    DO 230 J=1,NS
    A(J) = -A(J)
    230 DO 250 I=1,N
    B(I) = ZRO
    II = (I-1)*N
    DO 240 J=1,N
    JJ = II+J
    IF (A(JJ).EQ.1.000) GO TO 240
    B(I) = B(I)+DABS(A(JJ))
    240 CCNTINUE
    250 AINK = ZRO

```

SMV00340
SMV00350
SMV00360
SMV00370
SMV00380
SMV00390
SMV00400
SMV00410
SMV00420
SMV00430
SMV00440
SMV00450
SMV00460
SMV00470
SMV00480
SMV00490
SMV00500
SMV00510
SMV00520
SMV00530
SMV00540
SMV00550
SMV00560
SMV00570
SMV00580
SMV00590
SMV00600
SMV00610
SMV00620
SMV00630
SMV00640
SMV00650
SMV00660
SMV00670
SMV00680
SMV00690
SMV00700
SMV00710
SMV00720
SMV00730
SMV00740
SMV00750
SMV00760
SMV00770
SMV00780
SMV00790
SMV00800
SMV00810



SMV000820
SMV000830
SMV000840
SMV000850
SMV000860
SMV000870
SMV000880
SMV000890
SMV000900
SMV000910

```

DO 260 I=1,N
  AINR = DMAX1(AINR,B(I))
  IF (AINR.LT.1.0D-15) AINR=1.0D0/ANR
  CNBR = ANR*AINR
  WRITE (6,280) CNBR,ANR,AINR
  RETURN
270 IF LG = 1
  RETURN
280 FORMAT (5X,'CONDITION NUMBER ',5X,1PD25.16,5X,2(1PD25.16))
END

```

C*****
C*****
C*****

DL000010

SUBROUTINE DLOAD

THIS SUBROUTINE FORMS TOTAL LOAD VECTORS

```

IMPLICIT REAL*8(A-H,O-Z)
COMMON /NB1/ NEL,NDPT,NPEL,NDF,NEQ,NN,MM,NS,NCOUNT,NST,NSTF,NCLD,NCLD,NCLD,NSL
1PBC,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NTLD,NBLD,NSL
2D
COMMON /B1/ ELDAT(10,3),CLOAD(4),BCON(3)
COMMON /B2/ ELDATN(3),BYNP(3),STNP(3),IHNP,FLOAD(3)
COMMON /B3/ AK1(60,60),AK2(60,60),AK3(60,60),RB1(60),RB2(60),RB3(60)
10)
COMMON /CNTRL/ NLOAD
DIMENSION A(60,60), S(60,60), R1(60), R2(60)
EQUIVALENCE (A(1,1),AK1(1,1)), (S(1,1),AK2(1,1)), (R1(1),RB1(1)),
1(R2(1),RB2(1))
NTRK(1,J) = (I-1)*NN+J
NTRK(I,J) = (I-1)*NS3+J
NTRK(I,J) = (I-1)*NDPT+J
NS3 = NS/3
ZRO = 0.0D0
DO 130 IL=1,NLOAD
  IF (NCLD.NE.0) REWIND 14
  IF (NCLD.EQ.0) GO TO 10
  READ (14) CLOAD
  INDEX = 1
  INCH = CLOAD(NDFP)*NDF
10 DO 90 I=1,NN
  DO 20 J=1,NS
  RI(J) = ZRO
  IF (IABS(NTLD)+IABS(NBLD)+IABS(NSLD).EQ.0) GO TO 50
  DO 40 J=1,NS3
  NTK1 = NTRK(I,J)
  IF (NTK1.GT.NDPT) GO TO 50

```

DL000020
DL000030
DL000040
DL000050
DL000060
DL000070
DL000080
DL000090
DL000100
DL000110
DL000120
DL000130
DL000140
DL000150
DL000160
DL000170
DL000180
DL000190
DL000200
DL000210
DL000220
DL000230
DL000240
DL000250
DL000260
DL000270
DL000280
DL000290
DL000300
DL000310



```

NTK = NTRK(IL,NTK1)
CALL RDISLF (NTK)
DO 30 K=1,NDF
  IR = (J-1)*NDF+K
  RI(IR) = FLOAD(K)
  CCNTINUE
50 IF (NCLD.EQ.0) GO TO 80
  IH = I*NS
  IF (INCH.GT.IH) GO TO 80
  IRL = INCH-(I-1)*NS-NDF
  DO 70 K=1,NDF
    IR = IRL+K
    RI(IR) = CLOAD(K)
    IF (INDEX.EQ.NCLD.AND.NLOAD.EQ.0) GO TO 100
    IF (INDEX.EQ.NCLD) GO TO 80
    READ (14) CLOAD
    INDEX = INDEX+1
    INCH = CLOAD(NDFP)*NDF
    GO TO 60
80 NTK = NTKR(IL,I)
  CALL WDSKRI (NTK)
  CCNTINUE
90 GO TO 130
100 NTK = NTKR(IL,I)
  CALL WDSKRI (NTK)
  I1 = I+1
  IF (I1.GT.NN) RETURN
  DO 120 K=1,NN
    DO 110 L=1,NS
      RI(L) = ZRO
110 NTK = NTKR(IL,K)
  CALL WDSKRI (NTK)
  CCNTINUE
120 CCNTINUE
130 RETURN
END

```

SUBROUTINE SORT

FOR ALL LOAD CONDITIONS THIS SUBROUTINE SORTS NODAL VALUES INTO
ELEMENT STORAGE

```

IMPLICIT REAL*8(A-H,O-Z)
COMMON /NB1/ NEL,NDPT,NPEL,NDF,NEQ,NN,MM,NS,NCOUNT,NST,NSIF,NCLD,NSRT000030
1PBC,NPBC2,NBCL,NBCH,NMAT,NCON21,NDFP,NLINE,NREC7,KEL,NILD,NBLD,NSLSRT000040

```

SRT000020

SRT000030

NSLSRT000040




```

20 COMMON /NB2/ NCON(20),NBC(4)
COMMON /B2/ ELAST(6,6),COORD(3),COREL(20,3),UJNT(3),SSSS1(7),SSNN1
1(7),ALPHA
COMMON /B12/ ELDATN(3),BYNP(3),STNP(3),THNP,FLOAD(3)
COMMON /CNTRL/ NLOAD
DIMENSION ELDATE(20,3), BYEP(20,3), STEP(20,3), TEEL(20)
EQUIVALENCE (COREL(1,1),ELDATE(1,1),BYEP(1,1),STEP(1,1),TEEL(1,1))
NTRK(I,J) = (I-1)*NDPT+J
REWIND 9
REWIND 19
REWIND 23
REWIND 24
REWIND 27
REWIND 28

```

SRT000050
 SRT000060
 SRT000070
 SRT000080
 SRT000090
 SRT000100
 SRT000110
 SRT000120
 SRT000130
 SRT000140
 SRT000150
 SRT000160
 SRT000170
 SRT000180
 SRT000190

C
 C
 C
 SORT COORDINATES

```

DO 130 I=1,NEL
READ (9) NCON
DO 110 J=1,NPEL
NTRACK = NCON(J)
CALL RDISKP (NTRACK)
DO 110 K=1,NDF
COREL(J,K) = COORD(K)
CONTINUE
110 WRITE (19) COREL

```

SRT000200
 SRT000210
 SRT000220
 SRT000230
 SRT000240
 SRT000250
 SRT000260
 SRT000270
 SRT000280

C
 C
 C
 SCRT NODAL PROPERTIES

```

IF (NMAT.GT.0) GO TO 130
DO 120 J=1,NPEL
NTRACK = NCON(J)
CALL RDISKP (NTRACK)
DO 120 K=1,NDF
ELDATE(J,K) = ELDATN(K)
CONTINUE
120 WRITE (23) ELDATE
130 CONTINUE
IF (IABS(NTLO)+IABS(NBLD)+IABS(NSLD).EQ.0) RETURN
DO 200 IL=1,NLOAD
REWIND 9
DO 190 I=1,NEL
READ (9) NCON
IF (NBLD.EQ.0) GO TO 150

```

SRT000290
 SRT000300
 SRT000310
 SRT000320
 SRT000330
 SRT000340
 SRT000350
 SRT000360
 SRT000370
 SRT000380
 SRT000390
 SRT000400
 SRT000410
 SRT000420
 SRT000430

C
 C
 C
 SCRT BODY FORCES



```

DO 140 J=1,NPEL
NTRACK = NCON(J)
NTRK = NTRK(I1,NTRACK)
CALL RDISBF (NTRK)
DO 140 K=1,NDF
BYEP(J,K) = BYNP(K)
CONTINUE
140 WRITE (27) BYEP
150 IF (NSLD.EQ.0) GO TO 170
C
C      SURT INITIAL DISPLACEMENTS
C
DO 160 J=1,NPEL
NTRACK = NCON(J)
NTRK = NTRK(I1,NTRACK)
CALL RDISSF (NTRK)
DO 160 K=1,NDF
STEP(J,K) = SYNPF(K)
CONTINUE
160 WRITE (28) STEP
170 IF (NTLD.EQ.0) GO TO 190
C
C      SORT NODAL TEMPERATURES
C
DO 180 J=1,NPEL
NTRACK = NCON(J)
NTRK = NTRK(I1,NTRACK)
CALL RDISKT (NTRK)
TEEL(J) = THNP
CONTINUE
180 WRITE (24) TEEL
190 CONTINUE
200 RETURN
END

```

SRT000440
 SRT000450
 SRT000460
 SRT000470
 SRT000480
 SRT000490
 SRT000500
 SRT000510
 SRT000520

SRT000530
 SRT000540
 SRT000550
 SRT000560
 SRT000570
 SRT000580
 SRT000590
 SRT000600
 SRT000610

SRT000620
 SRT000630
 SRT000640
 SRT000650
 SRT000660
 SRT000670
 SRT000680
 SRT000690
 SRT000700
 SRT000710
 SRT000720



APPENDIX H

COMPUTER LISTING OF POSTPROCESSING PROGRAM

```

THIS PROGRAM PROCESSES INFORMATION FROM THE STRESS ANALYSIS.
PRESENTATION OF RESULTS IS PROVIDED IN A CONCISE FORMAT AND
PLCTS OF INFORMATION ARE PROVIDED. LT. R. B. BUBECK, NAVAL
PCSTGRADUATE SCHOOL, 1975

      POST PROCESSING PROGRAM
      THIS PROGRAM PROCESSES INFORMATION FROM THE STRESS ANALYSIS.
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INPUT DATA
TITLE CARD(344)
THE TITLE APPEARS ON THE RESULT PRESENTATION AND THE PLOTS.
PROBLEM PARAMETERS: SIX INTEGERS, ONE FLOATING POINT (6I4,F10.0)
NLOAD1...NUMBER OF FIRST LOAD TO BE PROCESSED
NLOAD2...NUMBER OF LAST LOAD TO BE PROCESSED
NPT...NUMBER OF POINTS TO BE PROCESSED
NCORD...COORDINATE TO BE USED FOR PLOTS(1 - X,2 - Y,3 - Z)
NDPT...TOTAL NUMBER OF POINTS IN THE MESH
NPLUT...PLOT INDICATOR 0 - PRINTER PLOT, 1 - CALCOMP PLOT
      PROVIDED.
TREF...ANY REFERENCE TEMPERATURE FOR ZERO STRESS CONDITION

      NODAL POINT INFORMATION(20I4)
      NCCN - THE NODAL NUMBERS OF POINTS TO BE PROCESSED.

REAL*8 SSSS1(7),COORD(3),THNP
DIMENSION NCON(20),RNG(4)
DIMENSION SS(7),X(20),Y(20),Z(20),SX(20),SY(20),SZ(20)
EQUIVALENCE (SX(1),SSSS(1,1)),(SY(1),SSSS(1,2)),(SZ(1),SSSS(1,3)),
1(X(1),COORD(1,1)),(Y(1),COORD(1,2)),(Z(1),COORD(1,3))
COMMON /MTRX/ SSSS(20,6),COORD(20,3),TEMP(20),TITLE(3)
DATA STOP//STOP//
DEFINE FILE 8(2800, 14,U,18),
1 9( 278, 6,U,19),
2 10(2800, 2,U,110)
1 READ(5,1000) TITLE
IF(TITLE(1).EQ.STOP) STOP
WRITE(6,1800) TITLE
WRITE(6,1900) NLOAD1,NLOAD2,NPT,NCORD,NDPT,NPLUT,TREF
READ(5,1100) NCON(1),I=1,NPT)
WRITE(6,1101) NLOAD1,NLOAD2,NPT,NCORD,NDPT,NPLUT,(NCON(I),I=1,NPT)

```

MAIN0010
 MAIN0020
 MAIN0030
 MAIN0040
 MAIN0041
 MAIN0050
 MAIN0060
 MAIN0070
 MAIN0080
 MAIN0090
 MAIN0100
 MAIN0110
 MAIN0111
 MAIN0120
 MAIN0130
 MAIN0140
 MAIN0150
 MAIN0160
 MAIN0161
 MAIN0170
 MAIN0180
 MAIN0190
 MAIN0200
 MAIN0210
 MAIN0220
 MAIN0221
 MAIN0230
 MAIN0240
 MAIN0250
 MAIN0260
 MAIN0270
 MAIN0280
 MAIN0290
 MAIN0300
 MAIN0310
 MAIN0320
 MAIN0330
 MAIN0340
 MAIN0350
 MAIN0360
 MAIN0370
 MAIN0380

CCCCCCCCCCCCCCCCCCCCCCCCCCCC



MAIN0390
 MAIN0400
 MAIN0410
 MAIN0420
 MAIN0430
 MAIN0440
 MAIN0450
 MAIN0460
 MAIN0470
 MAIN0480
 MAIN0490
 MAIN0500
 MAIN0510
 MAIN0520
 MAIN0530
 MAIN0540
 MAIN0550
 MAIN0560
 MAIN0570
 MAIN0580
 MAIN0590
 MAIN0600
 MAIN0610
 MAIN0620
 MAIN0630
 MAIN0640
 MAIN0650
 MAIN0660
 MAIN0670
 MAIN0680
 MAIN0690
 MAIN0700
 MAIN0710
 MAIN0720
 MAIN0730
 MAIN0740
 MAIN0750
 MAIN0760
 MAIN0770
 MAIN0780
 MAIN0790
 MAIN0800
 MAIN0810
 MAIN0820
 MAIN0830
 MAIN0840
 MAIN0850
 MAIN0860

```

DO 900 IL=NLOAD1,NLOAD2
SMAX=-1.0E20
SMIN=+1.0E20
TMAX=-1.0E20
TMIN=+1.0E20
XMAX=-1.0E20
XMIN=+1.0E20
REWIND 11
DO 10 I=1, IL
  READ(11) TIME
DO 50 I=1,NPT
  NODE=NCON(I)
  READ(9,NODE) COORD
DC 50 J=1,3
  COORD(I,J)=DABS(COORD(J))
DO 100 I=1,NPT
  NODE=NCON(I)
  NTK=(IL-1)*NDPT+NODE
  READ(8,NTK) SSSS1
  READ(10,NTK) THNP
DC 110 J=1,6
  SSSS(I,J)=SSSS1(J)/SSSS1(7)
  TEMP(I)=THNP+TREF
  IF(NPLOT.NE.0) GO TO 100
  SMAX=AMAX1(SMAX,SSSS(I,1),SSSS(I,2),SSSS(I,3))
  SMIN=AMIN1(SMIN,SSSS(I,1),SSSS(I,2),SSSS(I,3))
  TMAX=AMAX1(TMAX,TEMP(I))
  TMIN=AMIN1(TMIN,TEMP(I))
  XMAX=AMAX1(XMAX,COORD(I,NCORD))
  XMIN=AMIN1(XMIN,COORD(I,NCORD))
CONTINUE
WRITE(6,1400) TIME
DO 200 I=1,NPT
  DO 205 J=1,7
    SSS(J)=SSSS(I,J)
    CALL PRNSTR(SSS,S1,S2,S3)
    WRITE(6,1205) COORD(I,1),SSSS(I,4),S1
    WRITE(6,1300) NCON(I),COORD(I,2),TEMP(I),SSSS(I,5),S2
    WRITE(6,1200) COORD(I,3),SSSS(I,6),S3
    IF(NCORD.LE.3) GO TO 300
    CALL RADIAL(NPT,NCORD,NPLOT,NCON,TIME)
    GO TO 860
  IF(NPLOT.NE.0) GO TO 850
  RNG(1)=XMAX
  RNG(2)=XMIN
  RNG(3)=SMAX
  RNG(4)=SMIN
  GO TO (600,700,800),NCORD
  
```




```

600 WRITE(6,1801)
    CALL UTPLLOT(X,SX,NPT,RNG,1,1)
    CALL UTPLLOT(X,SY,NPT,RNG,1,2)
    CALL UTPLLOT(X,SZ,NPT,RNG,1,3)
    WRITE(6,1802)
    RNG(3)=TMAX
    RNG(4)=TMIN
    CALL UTPLLOT(X,TEMP,NPT,RNG,1,0)
    GO TO 850
700 WRITE(6,1803)
    CALL UTPLLOT(Y,SX,NPT,RNG,1,1)
    CALL UTPLLOT(Y,SY,NPT,RNG,1,2)
    CALL UTPLLOT(Y,SZ,NPT,RNG,1,3)
    WRITE(6,1804)
    RNG(3)=TMAX
    RNG(4)=TMIN
    CALL UTPLLOT(Y,TEMP,NPT,RNG,1,0)
    GO TO 850
800 WRITE(6,1805)
    CALL UTPLLOT(Z,SX,NPT,RNG,1,1)
    CALL UTPLLOT(Z,SY,NPT,RNG,1,2)
    CALL UTPLLOT(Z,SZ,NPT,RNG,1,3)
    WRITE(6,1806)
    RNG(3)=TMAX
    RNG(4)=TMIN
    CALL UTPLLOT(Z,TEMP,NPT,RNG,1,0)
    IF(NPLOT.NE.0) CALL GRAPH(NPT,NCORD,TIME)
850 P1MAX=-1.0E20
860 P1MIN=+1.0D20
    DO 875 I=1,NDPT
        NTK=(IL-1)*NDPT+I
        READ(8,NK) SSS1
    DO 880 J=1,7
        SSS(J)=SSS1(J)/SSSS1(7)
    CALL PRNSFR(SSS,S1,S2,S3)
    P2MAX=AMAX1(P1MAX,S1,S2,S3)
    P2MIN=AMIN1(P1MIN,S1,S2,S3)
    IF(P1MAX.NE.P2MAX) IMAX=I
    IF(P1MIN.NE.P2MIN) IMIN=I
    P1MAX=P2MAX
    P1MIN=P2MIN
875 CONTINUE
900 WRITE(6,1900) TIME,P1MAX,IMAX,P1MIN,IMIN
    CONTINUE
    GO TO 1
1000 FORMAT(3A4)
1100 FORMAT(20I4)
1110 FORMAT(6I4,F10.0)

```

```

MAIN0870
MAIN0880
MAIN0890
MAIN0900
MAIN0910
MAIN0920
MAIN0930
MAIN0940
MAIN0950
MAIN0960
MAIN0970
MAIN0980
MAIN0990
MAIN1000
MAIN1010
MAIN1020
MAIN1030
MAIN1040
MAIN1050
MAIN1060
MAIN1070
MAIN1080
MAIN1090
MAIN1100
MAIN1110
MAIN1120
MAIN1130
MAIN1140
MAIN1150
MAIN1160
MAIN1170
MAIN1180
MAIN1190
MAIN1200
MAIN1210
MAIN1220
MAIN1230
MAIN1240
MAIN1250
MAIN1260
MAIN1270
MAIN1280
MAIN1290
MAIN1300
MAIN1310
MAIN1320
MAIN1330
MAIN1331

```




```

11101 FORMAT('0',14X,'NUMBER OF LOADS TO BE PROCESSED =',2I5,'//',15X,'NUMBER OF POINTS TO BE PROCESSED =',15,'//',15X,'COORDINATE TO BE USED =',15,'//',15X,'TOTAL NUMBER OF NODES IN MESH =',15,'//',15X,'VALUE OF NPLT =',15,'//',15X,'NODAL NUMBERS TO BE PROCESSED',//,20X,
42015,///)
1200 FCORMAT('0',15X,IP1G18.6,18X,IP3G18.6)
1205 FCORMAT('0',15X,IP1G18.6,18X,IP3G18.6)
1300 FCORMAT('1',10X,15,IP5G18.6)
1400 FCORMAT('1',15X,15,IP5G18.6)
13X,NODE,5X,COORDINATE,8X,TEMPERATURE AND STRESS VALUES, TIME =,F10.5,/,/,1MAINI1420
21X,PRINCIPAL,/,25X,X/Y/Z,31X,X/Y/Z,12X,XY/YZ/ZX,09X,STRESS,14X,TAU,1MAINI1430
35,/)
1500 FCORMAT('1',25X,3A4,/)
1700 FCORMAT('1',5X,IP3G20.6)
1701 FCORMAT('1',5X,IP5G20.6)
1800 FCORMAT('1')
1801 FCORMAT('1',53X,STRESS VERSUS X COORDINATE,/,44X,SIGMA X - "
1,SIGMA Y - "
1802 FCORMAT('1',51X,TEMPERATURE VERSUS X COORDINATE,/,/,/)
1803 FCORMAT('1',53X,STRESS VERSUS Y COORDINATE,/,44X,SIGMA X - "
1,SIGMA Y - "
1804 FCORMAT('1',51X,TEMPERATURE VERSUS Y COORDINATE,/,/,/)
1805 FCORMAT('1',53X,STRESS VERSUS Z COORDINATE,/,44X,SIGMA X - "
1,SIGMA Y - "
1806 FCORMAT('1',51X,TEMPERATURE VERSUS Z COORDINATE,/,/,/)
1900 FCORMAT('0',/,/,/,25X,AT TIME =,F10.5,PRINCIPAL STRESS ARE :,1MAINI1580
1/,25X,MAXIMUM PRINCIPAL STRESS =,IP1G20.6,AT NODE,15,/,25X,MAINI1590
2,MINIMUM PRINCIPAL STRESS =,IP1G20.6,AT NODE,15)
END

```

[illegible]

```

SUBROUTINE PRNSTR(STR,S1,S2,S3)
CCOMPLEX*8 AC,BC,CC
DIMENSION STR(1)
DATA CC/(0.000,1.732050808D0)/,CR/0.3333333333333D0/
SX=STR(1)
SY=STR(2)
SZ=STR(3)
TXY=STR(4)
TYZ=STR(5)
TXZ=STR(6)
PHI=-((SX+SY+SZ)
PHI1=PHI
THEIA=SX*SY*SZ-(TXY*TXZ+TXZ*TXZ+TXZ*TXZ)
PSI=-((SX*SY*SZ+2.0*TYZ*TXZ*TXZ+SY*TXZ*TXZ+SZ*TXZ*TXZ))
A=THEIA-PHI*PHI/3.000

```



```

B=2.000*PHI*PHI*PHI/27.000-PHI*THETA/3.000+PSI
AB=B*B/4.000+A*A*A/27.000
IF(AB.LT.0.000) GO TO 100
AB=SQRT(AB)
BPAB=-B/2.000+AB
BMAB=-B/2.000-AB
IF(BPAB.LT.0.000) GO TO 10
AC=BPAB**CK
GO TO 20
10 AC=-((-BPAB)**CR)
20 IF(BMAB.LT.0.000) GO TO 30
30 BC=BMAB**CK
40 GO TO 40
30 BC=-((-BMAB)**CR)
40 S1=REAL(AC+BC)
S2=REAL(-(AC+BC)/2.000+CC*(A-B)/2.000)
S3=REAL(-(AC+BC)/2.000-CC*(A-B)/2.000)
S1=S1-PHI/3.000
S2=S2-PHI/3.000
S3=S3-PHI/3.000
RETURN
100 AB=(-B/2.000)/SQRT(-A*A/27.0)
PHI=ARCOS(AB)
FACT=2.000*SQRT(-A/3.0)
PHI3=PHI/3.000
S1=FACT*COS(PHI3)
S2=FACT*COS(PHI3+2.094395103)
S3=FACT*COS(PHI3+4.18879206)
S1=S1-PHI/3.000
S2=S2-PHI/3.000
S3=S3-PHI/3.000
RETURN
END

```

```

C*****
C*****
C*****

```

```

SUBROUTINE GRAPH(NPT, NCORD, TIME)
REAL*4 BCD(4)/ZCA000000,Z5C000000,ZCC000000,ZCB000000/,
1SP(+)/0.04,0.08,0.04,0.05/
COMMON /MTRX/ SSSS(20,6),COORD(20,3),TEMP(20),TITLE(3)
DIMENSION TITLEX(4),TITLYL(2),TITLYR(3),BCDI(1),
1NCCN(20),SX(20),SY(20),SZ(20),SXYZ(60),
2X(20),Y(20),Z(20)
EQUIVALENCE (SX(1),SSSS(1,1)),(SY(1),SSSS(1,2)),(SZ(1),SSSS(1,3)),
1(X(1),COORD(1,1)),(Y(1),COORD(1,2)),(Z(1),CORD(1,3))
DATA TITLEX/,'SPAT',,'IAL',,'POSTI',,'TION',,'/,'TITLYL/,'STRE',,'SS',,'/,
1TITLYR/,'TEMP',,'ERAT',,'URE',,'/,'

```

```

GRPH0010
GRPH0020
GRPH0030
GRPH0040
GRPH0050
GRPH0060
GRPH0070
GRPH0080
GRPH0090
GRPH0100
GRPH0110

```



```

C
C
INITIALIZE PLOTS
CALL PLOTS
SCALE AXES
IF(NCORD.EQ.1) CALL SCAL(X,NPT,5.0,XMIN,DX)
IF(NCORD.EQ.2) CALL SCAL(Y,NPT,5.0,XMIN,DX)
IF(NCORD.EQ.3) CALL SCAL(Z,NPT,5.0,XMIN,DX)
NPT2=2*NPT
NPT3=3*NPT
DO 50 I=1,NPT
  XYZ(I)=SX(I)
  XYZ(I+NPT)=SY(I)
  XYZ(I+NPT2)=SZ(I)
  CALL SCAL(XYZ,NPT3,6.0,SMIN,DS)
  CALL SCAL(TEMP,NPT,3.0,TMIN,DT)
  DRAW AXES
DO 75 I=1,NPT
  SX(I)=XYZ(I)
  SY(I)=XYZ(I+NPT)
  SZ(I)=XYZ(I+NPT2)
  XS=0.0
  YS=0.0
  IF(XMIN.LT.0.0) XS=ABS(XMIN)/DX
  IF(SMIN.LT.0.0) YS=ABS(SMIN)/DS
  CALL AXIS(XS,0.0,TITLEL,+6.6,0.90,0.0,SMIN,DS)
  CALL AXIS(XS,0.0,TITLEL,+6.6,0.90,0.0,SMIN,DS)
  CALL AXIS(0.0,YS,TITLEX,-16.5,0.0,0.0,XMIN,DX)
  CALL AXIS(0.0,YS,TITLEX,-16.5,0.0,0.0,XMIN,DX)
  XSP=XSP+5.0
  CALL AXIS(XSP,YS,TITLEYR,-11.3,0.90,0.0,TMIN,DT)
  CALL AXIS(XSP,YS,TITLEYR,-11.3,0.90,0.0,TMIN,DT)
DO 80 I=1,NPT
  TEMP(I)=TEMP(I)+YS
PLCT VALUES
GU TO (100,200,300),NCORD
CALL LINE(X,SX,NPT,1,1)
CALL LINE(X,SX,NPT,1,1)
CALL LINE(X,SY,NPT,1,1)
CALL LINE(X,SY,NPT,1,1)
CALL LINE(X,SZ,NPT,1,1)
CALL LINE(X,SZ,NPT,1,1)
CALL LINE(X,TEMP,NPT,1,1)
CALL LINE(X,TEMP,NPT,1,1)
CALL SYMBOL(.75,-.5,0.14,'SPATIAL POSITION - X COORDINATE',0.0,31)
CALL SYMBOL(.75,-.5,0.14,'SPATIAL POSITION - X COORDINATE',0.0,31)
GU TO 400
CALL LINE(Y,SX,NPT,1,1)
CALL LINE(Y,SX,NPT,1,1)
CALL LINE(Y,SY,NPT,1,1)
CALL LINE(Y,SY,NPT,1,1)

```

```

GRPH0120
GRPH0130
GRPH0140
GRPH0150
GRPH0160
GRPH0170
GRPH0180
GRPH0190
GRPH0200
GRPH0210
GRPH0220
GRPH0230
GRPH0240
GRPH0250
GRPH0260
GRPH0270
GRPH0280
GRPH0290
GRPH0300
GRPH0310
GRPH0320
GRPH0330
GRPH0340
GRPH0350
GRPH0360
GRPH0370
GRPH0380
GRPH0390
GRPH0400
GRPH0410
GRPH0420
GRPH0430
GRPH0440
GRPH0450
GRPH0460
GRPH0470
GRPH0480
GRPH0490
GRPH0500
GRPH0510
GRPH0520
GRPH0530
GRPH0540
GRPH0550
GRPH0560
GRPH0570
GRPH0580
GRPH0590

```




```

CALL LINE(Y,SY,NPT,1,1)
CALL LINE(Y,SZ,NPT,1,1)
CALL LINE(Y,TEMP,NPT,1,1)
CALL LINE(Y,TEMP,NPT,1,1)
CALL SYMBOL(.75,-.5,0.14,'SPATIAL POSITION - Y COORDINATE',0.0,31)
CALL SYMBOL(.75,-.5,0.14,'SPATIAL POSITION - Y COORDINATE',0.0,31)
GC TO 400
CALL LINE(Z,SX,NPT,1,1)
CALL LINE(Z,SX,NPT,1,1)
CALL LINE(Z,SY,NPT,1,1)
CALL LINE(Z,SY,NPT,1,1)
CALL LINE(Z,SZ,NPT,1,1)
CALL LINE(Z,SZ,NPT,1,1)
CALL LINE(Z,TEMP,NPT,1,1)
CALL LINE(Z,TEMP,NPT,1,1)
CALL SYMBOL(.75,-.5,0.14,'SPATIAL POSITION - Z COORDINATE',0.0,31)
CALL SYMBOL(.75,-.5,0.14,'SPATIAL POSITION - Z COORDINATE',0.0,31)
DO 450 I=1,3
BCDI(1)=BCD(I)
DO 450 J=1,NPT
XN=COORD(J,NCORD)-SP(I)
YX=SSS(J,I)-SP(I)
CALL SYMBOL(XN,YN,0.14,BCDI,0.0,1)
CALL SYMBOL(XN,YN,0.14,BCDI,0.0,1)
BCDI(1)=BCD(4)
DO 475 I=1,NPT
YN=COORD(I,NCORD)-SP(4)
XN=TEMP(I)-SP(4)
CALL SYMBOL(XN,YN,0.14,BCDI,0.0,1)
CALL SYMBOL(XN,YN,0.14,BCDI,0.0,1)
CALL SYMBOL(.5,-0.75,0.14,'SIGMA X - DIAMOND',0.0,17)
CALL SYMBOL(.5,-0.75,0.14,'SIGMA X - DIAMOND',0.0,17)
CALL SYMBOL(2.75,-.75,0.14,'SIGMA Y - ASTERISK',0.0,18)
CALL SYMBOL(2.75,-.75,0.14,'SIGMA Y - ASTERISK',0.0,18)
CALL SYMBOL(2.75,-.75,0.14,'SIGMA Z - CROSS',0.0,15)
CALL SYMBOL(0.5,-1.0,0.14,'SIGMA Z - CROSS',0.0,15)
CALL SYMBOL(0.5,-1.0,0.14,'SIGMA Z - CROSS',0.0,15)
CALL SYMBOL(2.5,-1.0,0.14,'TEMPERATURE - SQUARE',0.0,20)
CALL SYMBOL(2.5,-1.0,0.14,'TEMPERATURE - SQUARE',0.0,20)
CALL SYMBOL(0.5,-1.25,0.07,'STRESS SCALE = PSI/INCH',0.0,10)
CALL SYMBOL(0.5,-1.25,0.07,'STRESS SCALE = PSI/INCH',0.0,10)
CALL ANN(DS,DSS,KEX)
PEX=KEX
CALL NUMBER(1.375,-1.25,0.07,DSS,0.0,3)
CALL NUMBER(1.375,-1.25,0.07,DSS,0.0,3)
CALL NUMBER(1.880,-1.18,0.07,PEX,0.0,-1)

```






```

IMIN=INT(TMIN)
TMIN=10.0*IMIN
YMIN=TMIN
IF(YMAX.GT.0.0.AND.YMIN.GT.0.0) GO TO 14
IF(ABS(YMAX).GT.ABS(YMIN)) YMIN=SIGN(YMAX,YMIN)
IF(ABS(YMIN).GT.ABS(YMAX)) YMAX=SIGN(YMIN,YMAX)
DY=(YMAX-YMIN)/S
GO TO 17
14 TMAX=YMAX*100.0
15 TMAX=INT(TMAX)+1
TMAX=FLOAT(TMAX)/100.0
TMIN=YMIN*100.0
TMIN=INT(TMIN)
TMIN=FLOAT(TMIN)/100.0
DY=(TMAX-TMIN)/S
SCALE DATA
17 SLOPE=1.0/DY
DO 20 I=1,N
20 Y(I)=SLOPE*(Y(1)-YMIN)
RETURN
END

```

```

C *****
C *****
C *****
C *****

```

```

SUBROUTINE RADIAL(NPT,NCORD,NPLOT,NCON,TIME)
REAL*4 BCD(4)/ZCA000000,Z5C000000,ZCC000000,ZCB000000/,
1 SP(4)/0.04,0.06,0.04,0.05/
DIMENSION RNG(4),NCON(20),XYZ(60),BCD1(1)
COMMON /MTRX/ SSSS(20,6),COORD(20,3),TEMP(20),TITLE(3)
RAD=57.295779
SMAX=-1.0E20
SMIN=+1.0E20
TMAX=-1.0E20
TMIN=+1.0E20
XMAX=-1.0E20
XMIN=+1.0E20
WRITE(6,1000) TIME

```

```

C C C
      SET UP COORDINATE INDICATORS
NCORD1=NCORD/10
NCORD2=NCORD-NCORD1*10
NCCRD3=6-NCORD1-NCORD2
IF(NCCRD1*NCCRD2.NE.2) GO TO 20
NCCRD4=4
NCCRD5=5
NCCRD6=6

```



```

20 IF(NCORD1*NCORD2.NE.3) GO TO 30
NCCRD4=6
NCCRD5=4
NCCRD6=5
30 IF(NCORD1*NCORD2.NE.6) GO TO 1
NCCRD4=5
NCCRD5=6
NCCRD6=4
1 DO 10 I=1,NPT
C
C
C      CONVERT COORDINATES
X=COORD(I,NCORD1)
Y=COORD(I,NCORD2)
R=SQRT(X*X+Y*Y)
THETA=ARCOS(X/R)
COORD(I,NCORD1)=R
COORD(I,NCORD2)=THETA*RAD
C
C
C      CONVERT STRESSES
SX=SSSS(I,NCORD1)
SY=SSSS(I,NCORD2)
SZ=SSSS(I,NCORD3)
TXZ=SSSS(I,NCORD4)
TYZ=SSSS(I,NCORD5)
TXZ=SSSS(I,NCORD6)
S=SIN(THETA)
SC=cos(THETA)
S2=S*S
C2=C*C
CS2=2.0*C*S
SSSS(I,1)=SX*C2+SY*S2+TXZ*CS2
SSSS(I,2)=SX*S2+SY*C2-TXZ*CS2
SSSS(I,3)=SZ
SSSS(I,4)=(SY-SX)*C*S+TXZ*(C2-S2)
SSSS(I,5)=TXZ*C-TXZ*S
SSSS(I,6)=TXZ*C-TXZ*S
WRITE(6,1002) COORD(I,NCORD1),SSSS(I,1),SSSS(I,4)
WRITE(6,1001) NCON(I),COORD(I),COORD(I,NCORD2),TEMP(I),SSSS(I,6)
WRITE(6,1003) COORD(I,NCORD3),SSSS(I,3),SSSS(I,6)
SMAX=AMAX1(SMAX,SSSS(I,1),SSSS(I,2),SSSS(I,3))
SMIN=AMIN1(SMIN,SSSS(I,1),SSSS(I,2),SSSS(I,3))
XMAX=AMAX1(R,XMAX)
XMIN=AMIN1(XMIN,R)
TMAX=AMAX1(TMAX,TEMP(I))
TMIN=AMIN1(TMIN,TEMP(I))
10 IF(NPLJ1.NE.0) GO TO 100
RADL0240
RADL0250
RADL0260
RADL0270
RADL0280
RADL0290
RADL0300
RADL0310
RADL0320
RADL0330
RADL0340
RADL0350
RADL0360
RADL0370
RADL0380
RADL0390
RADL0400
RADL0410
RADL0420
RADL0430
RADL0440
RADL0450
RADL0460
RADL0470
RADL0480
RADL0490
RADL0500
RADL0510
RADL0520
RADL0530
RADL0540
RADL0550
RADL0560
RADL0570
RADL0580
RADL0590
RADL0600
RADL0610
RADL0620
RADL0630
RADL0640
RADL0650
RADL0660
RADL0670
RADL0680
RADL0690
RADL0700
RADL0710

```


RADL0720
RADL0730
RADL0740
RADL0750
RADL0760
RADL0770
RADL0780
RADL0790
RADL0800
RADL0810
RADL0820
RADL0830
RADL0831
RADL0840
RADL0850
RADL0860
RADL0870
RADL0880
RADL0890
RADL0900
RADL0910
RADL0920
RADL0930
RADL0940
RADL0950
RADL0960
RADL0970
RADL0980
RADL0990
RADL1000
RADL1010
RADL1020
RADL1030
RADL1040
RADL1050
RADL1060
RADL1070
RADL1080
RADL1090
RADL1100
RADL1110
RADL1120
RADL1130
RADL1140
RADL1150
RADL1160
RADL1170
RADL1180

```

RNG(1)=XMAX
RNG(2)=XMIN
RNG(3)=SMAX
RNG(4)=SMIN
WRITE(6,2000)
DC 50 I=1,3
CALL UTPLUT(COORD(1,NCORD1),SSSS(1,I),NPT,RNG,1,1,I)
50 CONTINUE
WRITE(6,2001)
RNG(3)=TMAX
RNG(4)=TMIN
CALL UTPLUT(COORD(1,NCORD1),TEMP,NPT,RNG,1,0)
100 RETURN
NPT2=NPT*2
NPT3=NPT*3
CALL PLCTS
CALL PLOT(1.0,0.0,-3)

C
C
C SCALE AXES
CALL SCAL(COORD(1,NCORD1),NPT,5.0,XMIN,DX)
DO 150 I=1,NPT
  XYZ(I)=SSSS(I,1)
  XYZ(I+NPT)=SSSS(I,2)
  XYZ(I+NPT2)=SSSS(I,3)
150 CALL SCAL(XYZ,NPT3,6.0,SMIN,DS)
DO 175 I=1,NPT
  SSSS(I,1)=XYZ(I)
  SSSS(I,2)=XYZ(I+NPT)
  SSSS(I,3)=XYZ(I+NPT2)
175 CALL SCAL(TEMP,NPT,3.0,TMIN,DT)
XS=0.0
YS=0.0
IF(XMIN.LT.0.0) XS=ABS(XMIN)/DX
IF(SMIN.LT.0.0) YS=ABS(SMIN)/DS
CALL AXIS(XS,0.0,STRESS(PSI),+12,6.0,90.0,SMIN,DS)
CALL AXIS(XS,0.0,STRESS(PSI),+12,6.0,90.0,SMIN,DS)
CALL AXIS(YS,0.0,RADIUS(IN.),-12,5.0,0.0,XMIN,DX)
CALL AXIS(YS,0.0,RADIUS(IN.),-12,5.0,0.0,XMIN,DX)
XSP=XS+5.0
CALL AXIS(XSP,YS,TEMPERATURE(DEG.F),-19,3.0,90.0,TMIN,DT)
CALL AXIS(XSP,YS,TEMPERATURE(DEG.F),-19,3.0,90.0,TMIN,DT)
DO 160 I=1,NPT
  TEMP(I)=TEMP(I)+YS
160 CALL LINE(COORD(1,NCORD1),SSSS(1,1),NPT,1,1)
CALL LINE(COORD(1,NCORD1),SSSS(1,1),NPT,1,1)
CALL LINE(COORD(1,NCORD1),SSSS(1,2),NPT,1,1)
CALL LINE(COORD(1,NCORD1),SSSS(1,2),NPT,1,1)

```




```

CALL LINE(COORD(1,NCORD1),SSSS(1,3),NPT,1,1)
CALL LINE(COORD(1,NCORD1),SSSS(1,3),NPT,1,1)
CALL LINE(COORD(1,NCORD1),TEMP,NPT,1,1)
CALL LINE(COORD(1,NCORD1),TEMP,NPT,1,1)
DO 200 I=1,3
  BCDI(1)=BCD(I)
DO 200 J=1,NPT
  XN=COORD(J,NCORD1)-SP(I)
  YN=SSSS(J,1)-SP(I)
  CALL SYMBOL(XN,YN,0.14,BCDI,0.0,1)
  CALL SYMBOL(XN,YN,0.14,BCDI,0.0,1)
  BCDI(1)=BCD(4)
DO 250 I=1,NPT
  XN=COORD(I,NCORD1)-SP(4)
  YN=TEMP(I)-SP(4)
  CALL SYMBOL(XN,YN,0.14,BCDI,0.0,1)
  CALL SYMBOL(XN,YN,0.14,BCDI,0.0,1)
  CALL SYMBOL(0.5,-0.75,0.14,SIGMA R - DIAMOND SIGMA T - ASTERISK,
1 0.0,37)
  CALL SYMBOL(0.5,-0.75,0.14,SIGMA R - DIAMOND SIGMA T - ASTERISK,
1 0.0,37)
  CALL SYMBOL(0.5,-1.0,0.14,SIGMA Z - CROSS TEMPERATURE - SQUARE,
1 0.0,37)
  CALL SYMBOL(0.5,-1.0,0.14,SIGMA Z - CROSS TEMPERATURE - SQUARE,
1 0.0,37)
  CALL SYMBOL(0.5,-1.25,0.07,STRESS SCALE = X 10 PSI/INCH,OR
1 0.0,35)
  CALL SYMBOL(0.5,-1.25,0.07,STRESS SCALE = X 10 PSI/INCH,OR
1 0.0,35)
  CALL ANN(DS,DSS,KEX)
  PEX=KEX
  CALL NUMBER(1.375,-1.25,0.07,DSS,0.0,3)
  CALL NUMBER(1.375,-1.25,0.07,DSS,0.0,3)
  CALL NUMBER(1.880,-1.18,0.07,PEX,0.0,-1)
  CALL NUMBER(1.880,-1.18,0.07,PEX,0.0,-1)
  CALL SYMBOL(0.5,-1.5,0.07,TEMPERATURE SCALE =
1 0.0,38)
  CALL SYMBOL(0.5,-1.5,0.07,TEMPERATURE SCALE =
1 0.0,38)
  CALL NUMBER(1.6875,-1.5,0.07,DT,0.0,1)
  CALL NUMBER(1.6875,-1.5,0.07,DT,0.0,1)
  CALL SYMBOL(0.5,-1.75,0.07,RADIAL POSITION SCALE =
1 CH,0.0,40)
  CALL SYMBOL(0.5,-1.75,0.07,RADIAL POSITION SCALE =
1 CH,0.0,40)
  CALL NUMBER(1.9375,-1.75,0.07,DX,0.0,2)
  CALL NUMBER(1.9375,-1.75,0.07,DX,0.0,2)
  CALL SYMBOL(1.125,6.75,0.14,STRESS AND TEMPERATURE,0.0,22)
RADL1190
RADL1200
RADL1210
RADL1220
RADL1230
RADL1240
RADL1250
RADL1260
RADL1270
RADL1280
RADL1290
RADL1300
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RADL1480
RADL1490
RADL1500
RADL1510
RADL1520
RADL1530
RADL1540
RADL1550
RADL1560
RADL1570
RADL1580
RADL1590
RADL1600
RADL1610
RADL1620
RADL1630
RADL1640
RADL1650
RADL1660

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